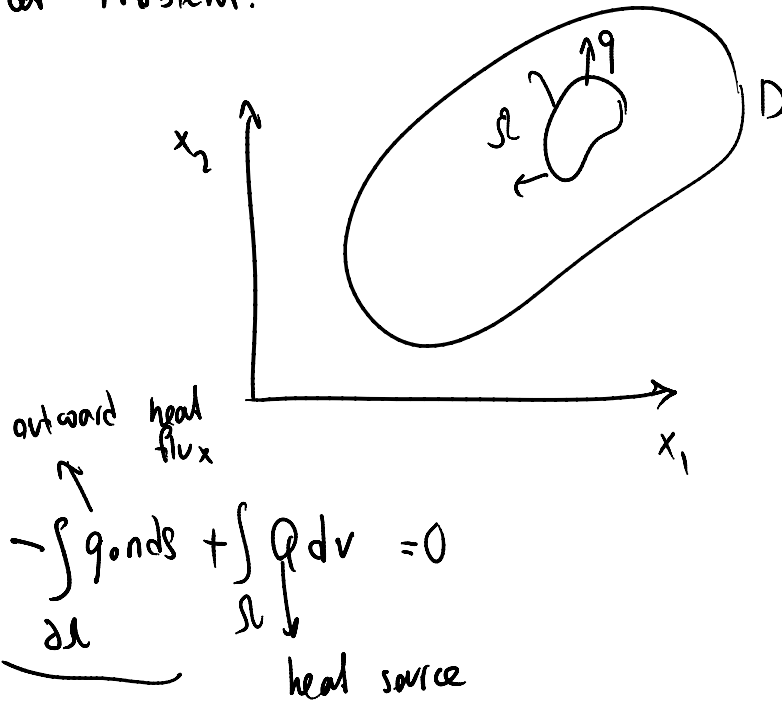


Heat Problem:



$$\int_{\partial\Omega} q \cdot n \, ds - \int_{\Omega} Q \, dv = 0$$

Balance law

$$\int_{\partial\Omega} q \cdot n \, ds = \int_{\partial\Omega} q_i n_i \, ds = \int_{\Omega} (\nabla \cdot q)_i \, dv$$

Divergence theorem

$$\int_{\Omega} (\nabla \cdot q - Q) \, dv = 0$$

localization (Ω is arbitrary)
 \implies

$$\boxed{\nabla \cdot q - Q = 0}$$

Strong form

$$[q] \cdot n = 0$$

jump condition

static problem
we don't deal
... ..

$$\nabla \cdot \mathbf{q} - Q = 0$$

units

we don't deal with physical jump

$$\nabla \cdot \mathbf{q} - Q = 0$$

strong form x_2

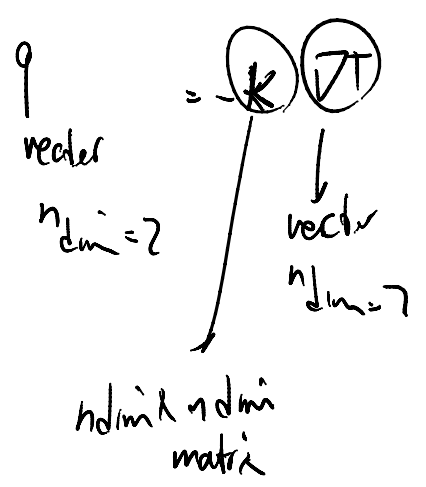
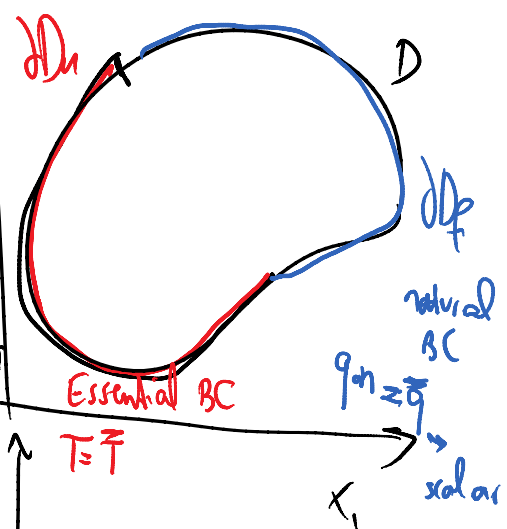
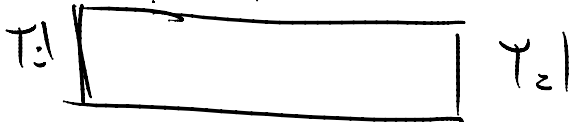
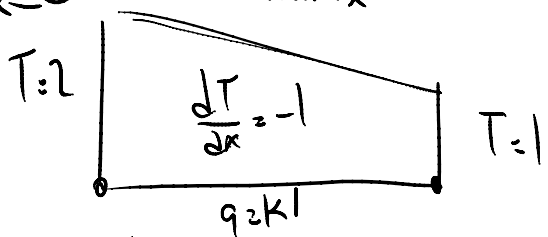
1 material properties k

constitutive relation

vector $\leftarrow \mathbf{q} = -k \nabla T \rightarrow$ vector

compare matrix with $\leftarrow \mathbf{q} = C \cdot \varepsilon \rightarrow$ matrix

∇T similar to $\varepsilon = \nabla u + \nabla u^T$



$$k = k \mathbf{I} \quad \text{isotropic conductivity}$$

② weighted Residual

$$R_i = \nabla \cdot \mathbf{q} - Q$$

$$R_f = \bar{q} - \mathbf{q} \cdot \mathbf{n} \quad \text{on } \partial D_f \quad \text{Natural B}$$

$$R_u = \bar{T} - T \quad \text{on } \partial D_u$$

↳ typically satisfied strongly

Find $T \in V^{WR} = \left\{ u \mid \forall x \in \partial D_u, u(x) = \bar{T}(x) \right\}$ strong satisfaction of Essential BC

$$\ni \forall w$$

$$\int_D \omega R_i + \int_{\partial D_f} \omega R_f = 0$$

$$\int_D \omega (\nabla \cdot q - Q) dv + \int_{\partial D_f} \omega (\bar{q} - n \cdot q) = 0$$

Divergence theorem

$$\int_D f_{,i} dv = \int_{\partial D} f_i n_i dS$$

~~$$\int_D \omega q_{,i} dv$$~~

$$AB' = (AB)' - A'B$$

$$\omega q_{,i} = \underbrace{(\omega q_i)_{,i}} - \omega_{,i} q_i$$

∂D

$$\int_D \omega (\nabla \cdot \bar{q} - Q) dv + \int_{\partial D_f} \omega (\bar{q} - q \cdot n) ds = 0$$

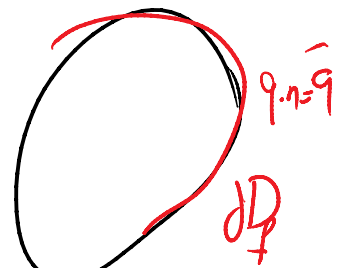
$$\int_D \omega q_{,i,i} dv = \int_D \left\{ (\omega q_i)_{,i} - \omega_{,i} q_i \right\} dv$$

$$= \int_D (\omega q_i)_{,i} dv - \int_D \omega_{,i} q_i dv$$

$$\int_{\partial D} \omega q_i \underline{n_i} ds - \int_D \omega_{,i} q_i dv$$

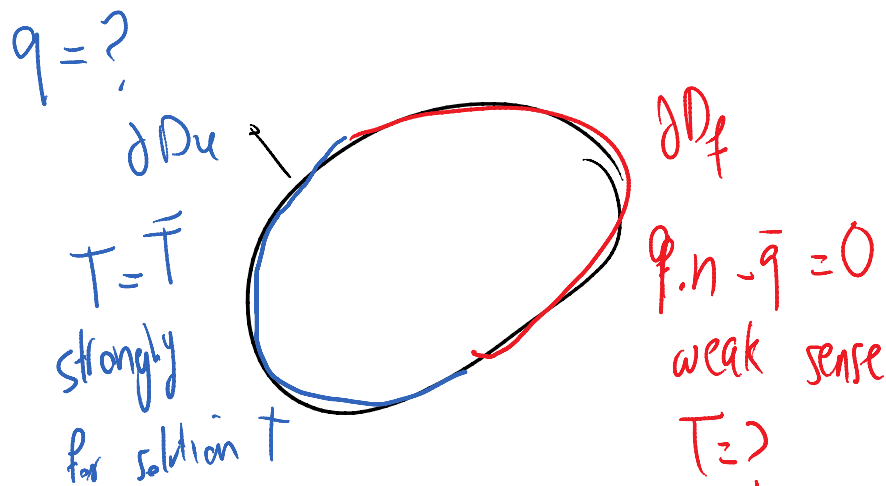
$$\int_D \omega (\nabla \cdot \bar{q} - Q) dv + \int_{\partial D_f} \omega (\bar{q} - q \cdot n) ds$$

$$\int_D (-\omega_{,i} q_i - \omega Q) dv + \int_{\partial D} \omega q_i n_i ds + \int_{\partial D_f} \omega (\bar{q} - q_i n_i) ds = 0$$



$$\begin{aligned}
 & + \int_{\partial D_f} \omega q_i n_i dS + \int_{\partial D_u} \omega q_i n_i dS + \int_{\partial D_f} \omega \bar{q} dS \\
 & - \int_{\partial D_f} \omega q_i n_i dS = 0
 \end{aligned}$$

$$\int_{\Omega} (\underbrace{\omega q_i}_{\partial D_f} - \omega q) dV + \int_{\partial D_u} \omega q_i n_i dS + \int_{\partial D_f} \omega \bar{q} dS = 0$$



we choose weight function to be zero @ ∂D_u

$q = -k \nabla T$ constitutive relation

Find $T \in V = \left\{ u \in C^1(\bar{\Omega}) \mid u|_{\partial D_u} = \bar{T}(x) \right\}$

$$\Rightarrow \forall w \in V_0 = \left\{ u \in C^1 \mid \forall x \in D_w \quad u(x) = \underline{0} \right\} \quad (\star)$$

in weak statement

$$\int_D + \nabla w \cdot k \nabla T = \int_D \omega Q - \int_{\partial D_f} \omega \bar{q}$$

weight functions satisfies homogeneous essential BC

3) Path to FE formulation

a) Discretization

$$T \rightarrow T^h = \sum_{i=1}^n a_i \phi_i(x) + \phi_p$$

discretized T

$a_1 \dots a_n$ the weight function

$$T^h \in V$$

$$\phi_i \in V_0$$

$$\phi_p \in V$$

$$w_i \in V_0$$

b) Galerkin method

$$V = \left\{ u \in H^1(D) \mid \forall x \in \partial D_u \quad u(x) = \bar{T}(x) \right\}$$

$$V_0 = \left\{ u \in H^1(D) \mid \forall x \in \partial D_u \quad u(x) = 0 \right\}$$

$$\text{For } i=1, \dots, n \quad \left. \begin{array}{l} \phi_i \in V_0 \\ \phi_p \in V \end{array} \right\}$$

Find a_1, \dots, a_n such that

$$T^h = \sum_{i=1}^n a_i \phi_i + \phi_p$$

satisfies

$$\int_D \nabla w \cdot k \nabla T^h = \int_D w Q - \int_{\partial D_f} w \hat{q} \, ds$$

in FE we use N (shape function) instead

n

c n

n

n

In it we use N_i (shape function) instead of ϕ (trial function)

6x6 system

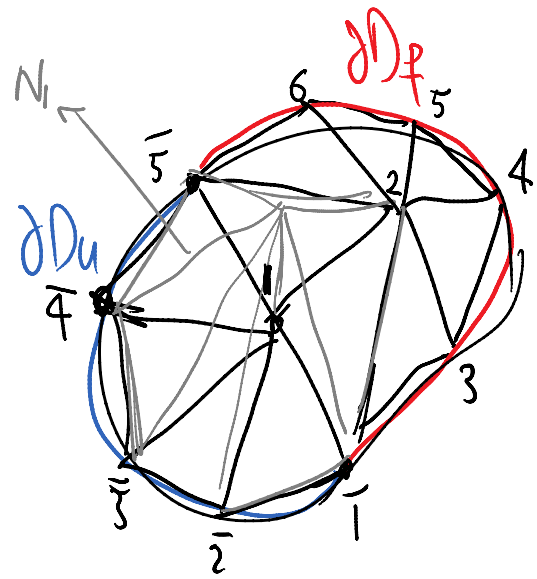
Shape functions

take the value of 1

at one **d.o.f** and zero elsewhere.
(Not node)

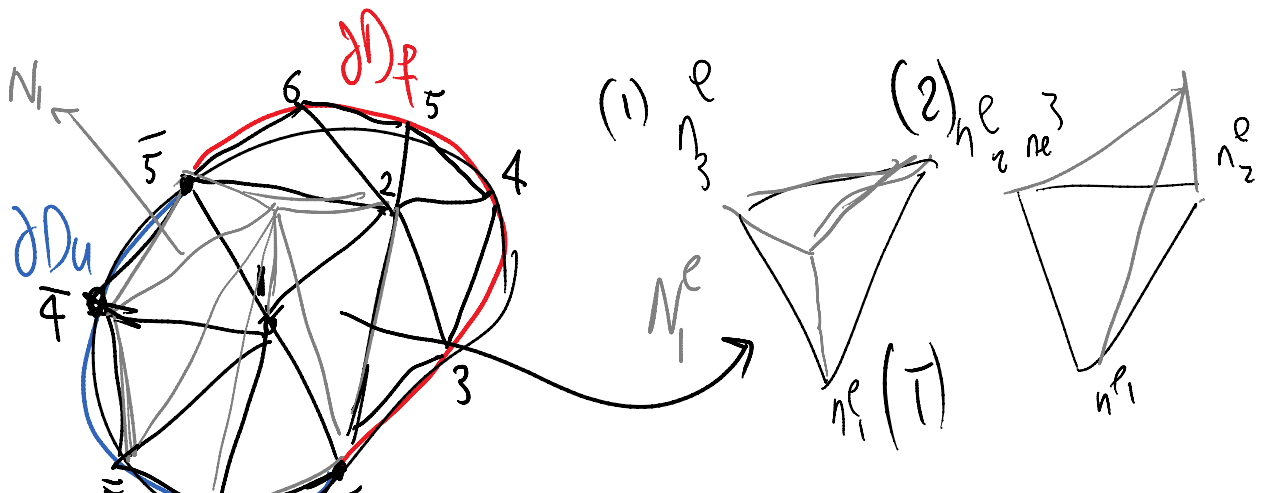
For this problem there is 1 dof per node

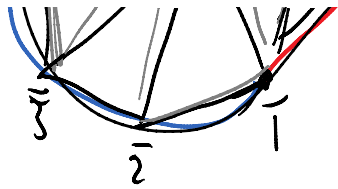
So it happens that shape functions take the value of 1 at 1 node and zero elsewhere



Globally

local @ element level





11/11

n

Formulas for element stiffness & forces:

$$f_r^e = \int N^T q$$

$$f_r = \int_{\partial \Omega} N^T \bar{q} ds$$

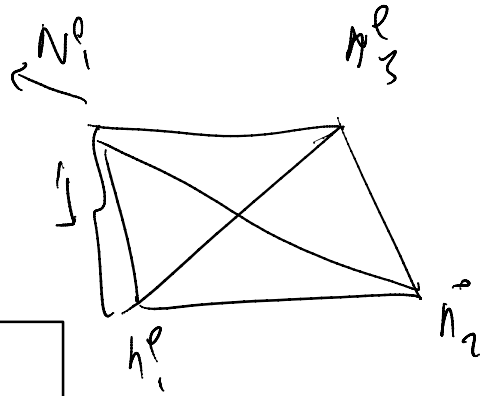
$$f_D = ka$$

$$k = k^e + k^e + k^e + k^e$$

$$\int_D \nabla^T w k \nabla T^h = \int_D w Q - \int_{\partial \Omega} w \bar{q} ds$$

$$N = [N_1 \ N_2 \ N_3]$$

$$T^h = N^e a^e$$



$$k^e = \int_e (N^T)^T k \nabla N dv$$

3×3 3×1

$$k^e = \int_e B^T D B dv$$

$$B = \nabla N \quad D = k$$

$$N = [N_1 \ N_2 \ N_3] \Rightarrow B = \nabla N = \begin{bmatrix} N_{1,x} & N_{2,x} & N_{3,x} \\ N_{1,y} & N_{2,y} & N_{3,y} \end{bmatrix}$$

$$\left[\begin{array}{c|c} r & N_{1,1} \dots N_{1,4} \end{array} \right] \left[\begin{array}{c|c} r & 0 \dots 0 \end{array} \right]$$

class. (1/2) 2/2

$$K^e = \int \begin{bmatrix} N_{1,x} & N_{1,y} \\ N_{2,x} & N_{2,y} \\ N_{3,x} & N_{3,y} \end{bmatrix} k \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} N_{1,x} & N_{2,x} & N_{3,x} \\ N_{1,y} & N_{2,y} & N_{3,y} \end{bmatrix} dV$$

.

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