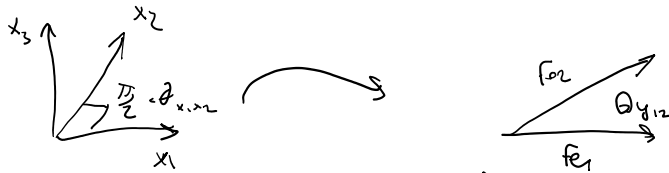


Relation of off-diagonal values of E and shear strain:



from last time

$$\sin \gamma_{ij} = \frac{e_i \cdot C_{ij}}{\sqrt{e_i \cdot C_{ii}} \sqrt{e_j \cdot C_{jj}}} \quad i \neq j$$

$$C = I + 2 \left(\frac{H+H^T}{2} \right) + H^T H = C + 2E + O(\epsilon^2), \quad \epsilon = |H|$$

$$e_i \cdot C_{ij} = e_i \cdot (I + 2E + O(\epsilon^2)) \cdot e_j = \delta_{ij} + 2E_{ij} + O(\epsilon^2)$$

for numerator $i \neq j$

$$\text{denominator } e_i \cdot C_{ii} = \delta_{ii} + 2E_{ii} + O(\epsilon^2) = 1 + O(\epsilon)$$

no summation on i

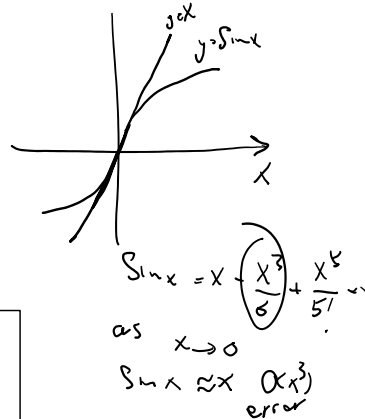
denominator

$$\sin \gamma_{ij} = \frac{2E_{ij} + O(\epsilon^2)}{\sqrt{1+O(\epsilon)} \sqrt{1+O(\epsilon)}} = \frac{2E_{ij} + O(\epsilon^2)}{(1+O(\epsilon))(1+O(\epsilon))}$$

$$\sqrt{1+x} = 1 + \frac{1}{2}x + O(x^2)$$

$$\sin \gamma_{ij} = 2E_{ij} + O(\epsilon^2)$$

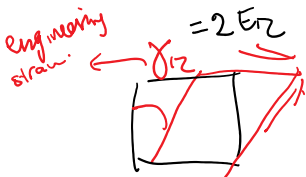
small number



$$\gamma_{ij} = 2E_{ij} + O(\epsilon^2)$$

For infinitesimal deformation theory

$$\text{tensorial strains } E_{ij} = \frac{\gamma_{ij}}{2} \quad \text{half of angle change}$$





$E = \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{12} & E_{22} & E_{23} \\ E_{13} & E_{23} & E_{33} \end{bmatrix}$

tensorial (half of engineering) shear strains between e_i & e_j ($i \neq j$)

normal strains along e_1, e_2, e_3

sym

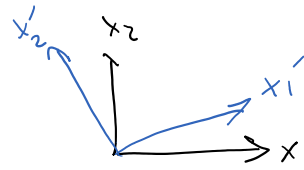
H
 $= \frac{\nabla u + \nabla u^T}{2}$

u is displacement field: tensor $\rightarrow \nabla u$ is a tensor

$\rightarrow E = \frac{\nabla u + \nabla u^T}{2}$ is a tensor & follows

coordinate transformation rules

$$E'_{ij} = Q_{im} Q_{jn} E_{mn}$$



will be Mohr-circle for sym 2nd order tensors discussed next week.

likewise $w = \frac{\nabla u - \nabla u^T}{2}$ is a rotator which is the infinitesimal theory counterpart to R

Vöigt notation:

E is a sym. 2nd order tensor

2D

$$E = \begin{bmatrix} E_{11} & E_{12} \\ E_{12} & E_{22} \end{bmatrix}$$

independent values

3D

$$E = \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{12} & E_{22} & E_{23} \\ E_{13} & E_{23} & E_{33} \end{bmatrix}$$

$$\gamma = \begin{bmatrix} E_{11} \\ E_{22} \\ 2E_{12} \end{bmatrix}$$

eng strains

$$\gamma = \begin{bmatrix} E_{11} \\ E_{12} \\ E_{33} \\ 2E_{12} \\ 2E_{13} \\ 2E_{23} \end{bmatrix}$$

the order here can change

1-array obviously not a vector

For finite deformation theory where H is no small, we need to use $F = RU$ or $F = VR$

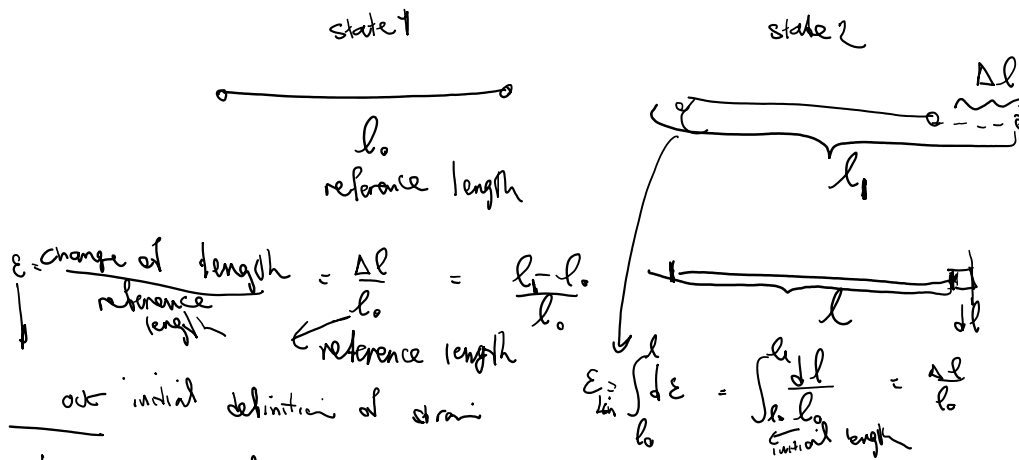
- U and V encompass the physical deformation of material (stretching and change of angle)
- R is a rotation

$U \leftrightarrow C \leftrightarrow G = 0.5(C - 1)$
 $V \leftrightarrow B$

--- a constitutive equation takes U and computes stress (and then incorporate the rotation part)

U is not a matrix of normal & shear strains
 the question is what "strain" measure best is related to stress

Background on Logarithmic strain



Logarithmic definition

$$E_{\text{Log}} = \int_{l_0}^{l_1} d\epsilon = \int_{l_0}^{l_1} \frac{\text{change of length}}{\text{current length}} = \ln\left(\frac{l_1}{l_0}\right)$$

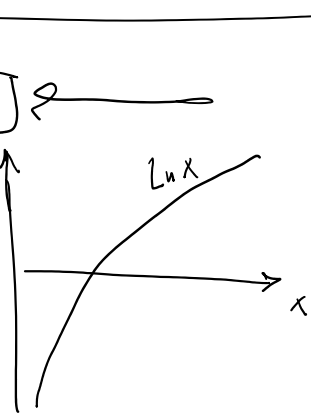
We push this all the way to $l_1 = 0^+$

$$E_{\text{Lin}} = \frac{l_1 - l_0}{l_0} = \frac{0 - l_0}{l_0} = -1$$

$$E_{\text{Log}} = \ln\left(\frac{l_1}{l_0}\right) = \ln(0^+) = -\infty$$

simple const. eqn

stress $\sigma = E \epsilon$
 (E constant)



To get the correct stress, we have two choices:

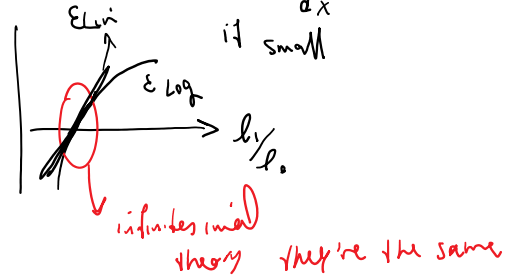
1. A simple linear const equation like above may be acceptable if we use logarithmic strain

2. Use linear strain BUT have more complex constitutive equation that gives -infinity stress for -1 linear strain.

In infinitesimal theory there is no distinction between these and in fact as mentioned before E is perfect.

related to here

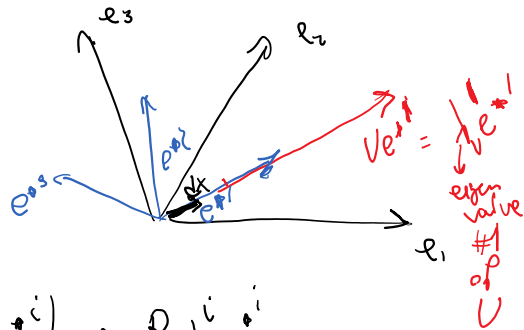
$$\begin{aligned} \epsilon_{\text{Log}} &= \ln\left(\frac{l_1}{l_0}\right) = \ln\left(\frac{l_0 + \Delta l}{l_0}\right) = \ln\left(1 + \frac{\Delta l}{l_0}\right) \\ &= \ln(1 + \epsilon_{\text{Lin}}) \\ &= \epsilon_{\text{Lin}}^2 + O(\epsilon_{\text{Lin}}^2) \end{aligned}$$



Background -> discussion of suitable strains for finite deformation theory

e^{*i} principal axes of C (& U)

$dy = F dx$
if $dx = \epsilon e^{*i}$
small number



$$dy = R U dx = R(U \epsilon e^{*i}) = \epsilon R(U e^{*i}) = \epsilon R \lambda^i e^{*i} = (\epsilon \lambda^i) R e^{*i}$$

current length

$$|dy| = \epsilon \lambda^i$$

initial length

$$|dx| = \epsilon$$

Linear strain for fiber along e^{*i}

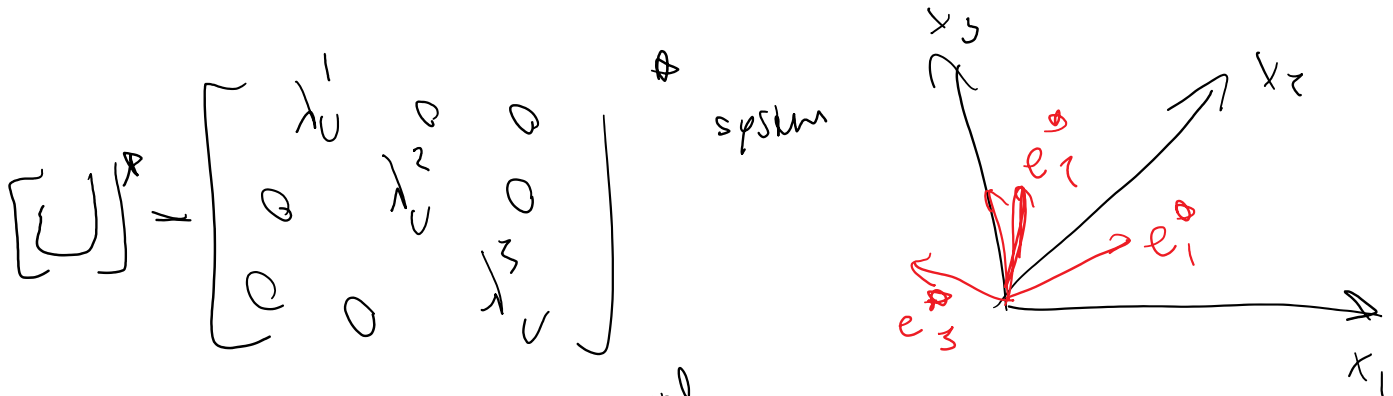
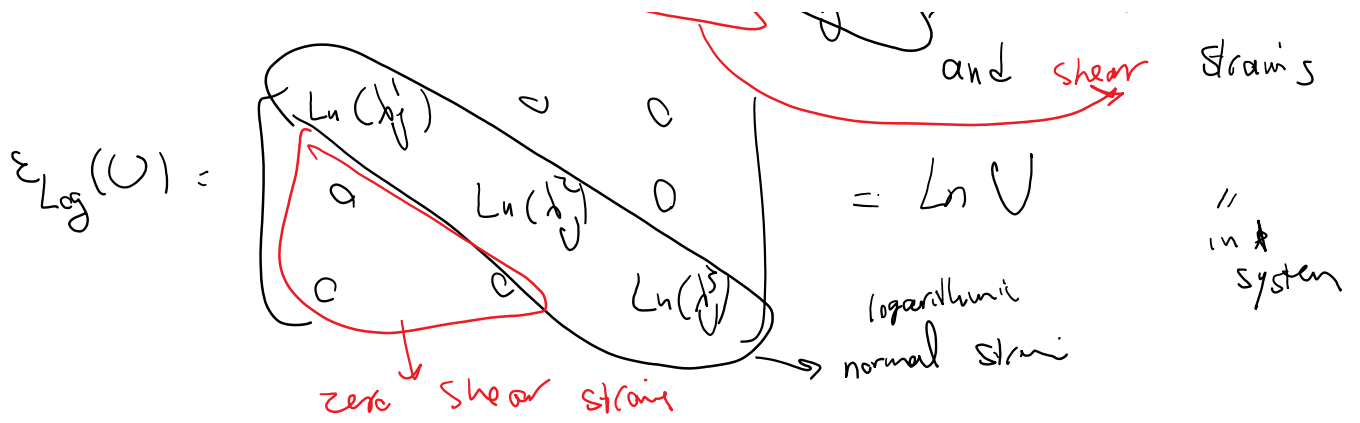
$$\frac{|dy| - |dx|}{|dx|} = \frac{\epsilon \lambda^i - \epsilon}{\epsilon} = \lambda^i - 1$$

Log strain

$$= \ln\left(\frac{|dy|}{|dx|}\right) = \ln(\lambda^i)$$

in $*$ system $\epsilon_{\text{Lin}}(U) = \begin{bmatrix} \lambda^1 - 1 & 0 & 0 \\ 0 & \lambda^2 - 1 & 0 \\ 0 & 0 & \lambda^3 - 1 \end{bmatrix} = U - \mathbb{1}$ expressed in A

these are linear normal and shear strains



Recall

$A v_i = \lambda_i v_i \rightarrow$ *eigenvektor*

$A U = U \Lambda$

$A = U \Lambda U^{-1}$

$f(A) = U f(\Lambda) U^{-1}$

for a diagonalizable matrix

if A is sym. $\rightarrow U = Q^t \quad f(U) = Q^T f(\Lambda) Q$