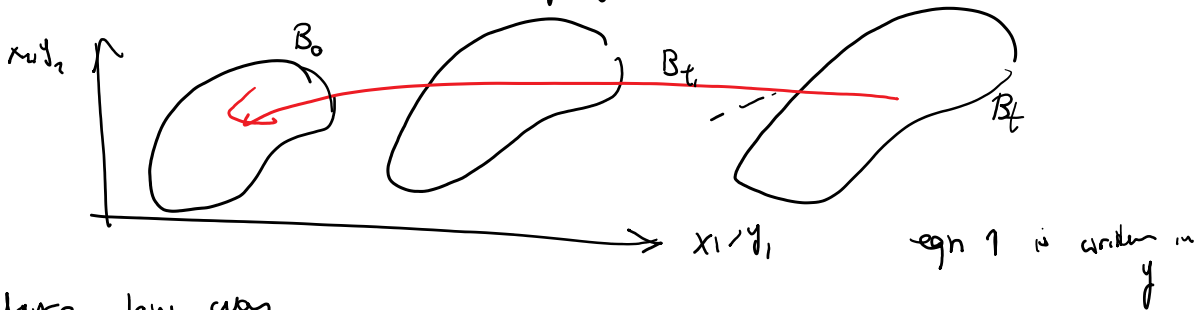


Balance of linear momentum / equation of motion (EOM)

$$\frac{dP}{dt} - \rho b + \operatorname{div} (\underbrace{\rho \otimes v}_{\text{ten density}} - \bar{T}) = 0 \quad (1)$$

$\frac{dP}{dt} = \sum F$ — are some the forces in the current configuration
 — are follow a blob of Mass (P_0)

Goal: Express (1) in Lagrangian framework.



Balance law was

$$\frac{DP}{Dt} = \underbrace{\frac{D}{Dt} \int_{B_0} p \, dV_y}_① = \underbrace{\int_{B_t} T_{ny} \, dA_y}_② + \underbrace{\int_{B_t} \rho b \, dV_y}_③$$

$$\frac{D}{Dt} \int_{B_t} p v \, dV_y = \int v \underbrace{\left(\frac{D}{Dt} \int_{B_0} p \, dV_x \right)}$$

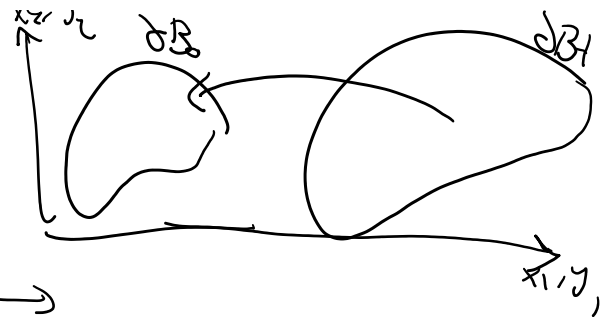
$$= \int_{B_0} \frac{D}{Dt} \left[\underbrace{v(x,t) \rho_0(x)}_{\rho_0(x,t) = \rho_0(x) v(x,t)} \right] dV_x$$

side note: reduced transport
 $\frac{D}{Dt} \int_{B_t} \underbrace{\rho p}_{dm} \, dV_y = \int \frac{Dg}{Dt} \underbrace{\rho_0 dx}_{dm}$

linear momentum density in reference configuration

$$\int T_{ny} \, dA_y$$

(2)

$$\int_{\partial B_t} T \cdot n_y dA_y$$


$$d\vec{A}_y = \mathcal{J} F^{-t} d\vec{A}_x$$

we showed this relation in kinematics section

$$\int_{\partial B_t} T d\vec{A}_y = \int_{\partial B_0} T (\mathcal{J} F^{-t}) d\vec{A}_x$$

$\mathcal{J} F^{-t}$ is a scalar
 $\mathcal{J} T F^{-t}$

$$= \int_{\partial B_0} (\mathcal{J} T F^{-t}) \cdot n_x dA_x$$

$n_x dA_x = d\vec{A}_x$

$$\mathcal{S}(x,t) = \mathcal{J}(x,t) T(y(x,t),t) F^{-t}(x,t)$$

Piola Kirchhoff I stress tensor Cauchy stress
 PK-I

$$\int_{\partial B_t} T d\vec{A}_y = \int_{\partial B_0} \mathcal{S} d\vec{A}_x$$

(3)

$$\int_{B_t} \rho b dV_y = \int_{B_t} b (\rho dV_y) = \int_{B_t} b \rho_0 dV_x$$

Summary:

Balance of linear momentum

$$\frac{D}{Dt} \int_{B_t} \rho dV_y \quad \text{(1)} = \int_{\partial B_t} T n_y dA_y \quad \text{(2)} + \int_{B_t} \rho b dV_y \quad \text{(3)}$$

Spatial (Eulerian) not

$$\frac{D}{Dt} \int_{B_0} \rho_0 dV_x = \int_{\partial B} \rho n_x dA_x + \int_{B_0} \rho_0 b dV_x$$

(Eulerian) rep.
 Referential (Lagrangian) rep.

$$\rho_0 = \rho_0 \vec{V} \quad , \quad \rho = \rho_0 \mathbf{J} \mathbf{F}^{-T} \quad \text{PK-I}$$

Obtaining the strong form of bal. of lin. mom. in reference configuration

$$\frac{D}{Dt} \int_{B_0} \rho_0 dV_x = \int_{\partial B_0} \rho n_x dA_x + \int_{B_0} \rho_0 b dV_x$$

$$\int_{B_0} \left(\frac{D}{Dt} \rho_0 \right) dV_x = \int_{B_0} \text{Div} \rho \mathbf{S} dV_x + \int_{B_0} \rho_0 b dV_x$$

\downarrow
 \mathbf{e}_0

$$\forall B_0 : \int_{B_0} \left[\frac{D}{Dt} \rho_0 - \text{Div} \rho \mathbf{S} - \rho_0 b \right] dV_x = 0$$

→ Localized: $\frac{D \rho_0}{Dt} - \text{Div} \rho \mathbf{S} - \rho_0 b = 0$ (accelerati)

$$\frac{D \rho_0}{Dt} = \frac{D \rho_0(x) V(x,t)}{Dt} = \rho_0(x) \frac{D V(x,t)}{Dt} = \rho_0(x) \underbrace{a(x,t)}_{\ddot{U}(x,t)}$$

EOM in Lagrangian form is

$$\frac{D}{Dt} \int_{\mathcal{B}_0} \rho b = \int_{\mathcal{B}_0} \rho b$$

$$\int_{\mathcal{B}_0} \frac{D}{Dt} \rho = \rho a$$

$$Div \mathcal{S} = \rho_0 b$$

$$\begin{pmatrix} S_{11,1} + S_{12,2} + S_{13,3} \\ S_{21,1} + S_{22,2} + S_{23,3} \\ S_{31,1} + S_{32,2} + S_{33,3} \end{pmatrix}$$

$$\rho_0 \ddot{u}_1 - (S_{11,1} + S_{12,2} + S_{13,3}) = \rho_0 b_1$$

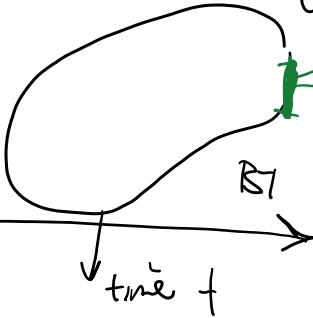
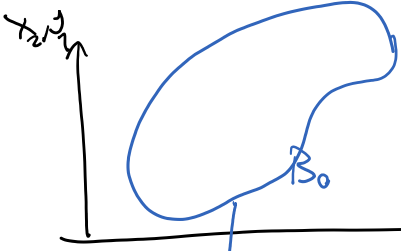
$$\rho_0 \ddot{u}_2 - (S_{21,1} + S_{22,2} + S_{23,3}) = \rho_0 b_2$$

$$\rho_0 \ddot{u}_3 - (S_{31,1} + S_{32,2} + S_{33,3}) = \rho_0 b_3$$

Interpretation of \mathcal{S} :

$$\frac{DP}{Dt} = F_{surface} + F_{body}$$

$$F_{surface} = \int T n_y dA_y$$



$$dA_y = n_y dA_y$$

$$T n_y = T n_y \quad (dF_y = T dA_y)$$

$$F_{surface} = \int \mathcal{S} n_x dA_x$$

$$d\vec{F} = \mathcal{S} d\vec{A}$$

Cauchy stress
 — maps y -surface-differential to
 y -force
 = y -normal vector to
 y -traction

PK-I
 maps x -surface differential to
 y -force
 = x -normal vector
 to y -force

$$\mathcal{S} = \mathcal{J} T F^{-t}$$

$$\mathcal{S}^t = - \mathcal{I} T^t$$

balance of angular momentum

$$\mathcal{S}^t = \mathcal{J} F^{-1} T^t = \mathcal{J} F^{-1} T^t \neq \mathcal{S}$$

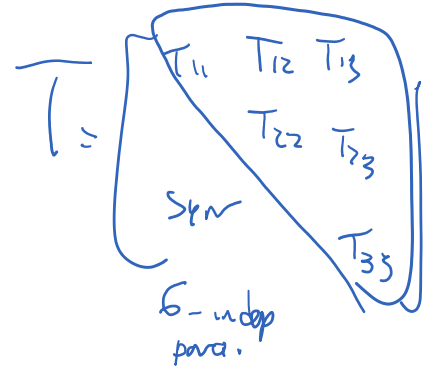
balance of angular momentum

$$\rightarrow T_{ij} = T_{ji} \quad (T = T^t)$$

T_{is} symmetric

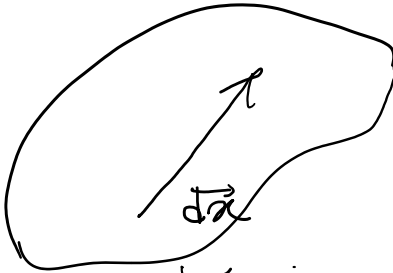
\mathcal{S} is not symmetric ;)

$$\mathcal{S} = \begin{pmatrix} S_{11} & S_{12} & S_{13} \\ S_{21} & S_{22} & S_{23} \\ S_{31} & S_{32} & S_{33} \end{pmatrix} \quad \begin{matrix} 9\text{-indep.} \\ \text{values} \end{matrix}$$

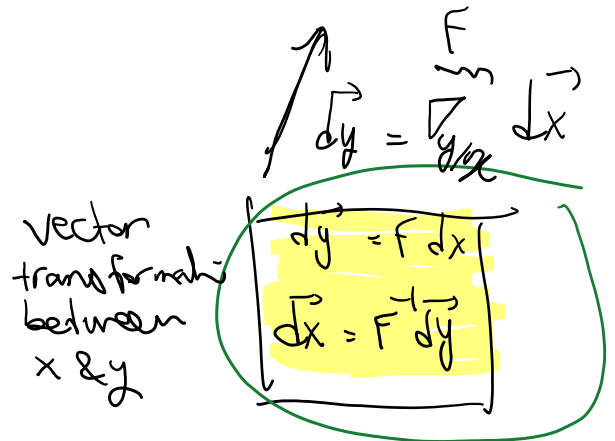


Idea: How about we map surface differential in x to FORCE **DIFFERENTIAL** in x ?

traction t or force differential dF is a vector

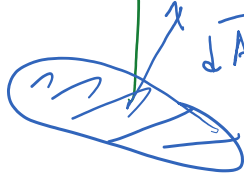


vector in x coordinate

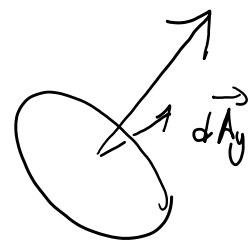


what is dF_x

$$d\vec{F}_x = F^{-1} d\vec{F}_y$$



$$d\vec{F}_y \checkmark \rightarrow d\vec{F}_x$$



$$\left. \begin{matrix} d\vec{F}_y \\ d\vec{F}_x \end{matrix} \right\} \begin{matrix} = \mathcal{S} \\ \leftarrow \end{matrix} \left. \begin{matrix} dA_x \\ dA_y \end{matrix} \right\} \rightarrow \left. \begin{matrix} d\vec{F}_x \\ d\vec{F}_y \end{matrix} \right\} = F^{-1} \mathcal{S} \left. \begin{matrix} dA_x \\ dA_y \end{matrix} \right\}$$

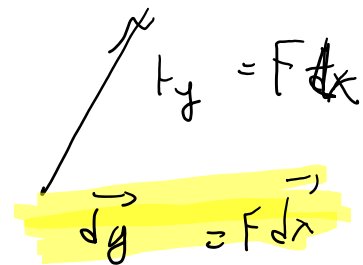
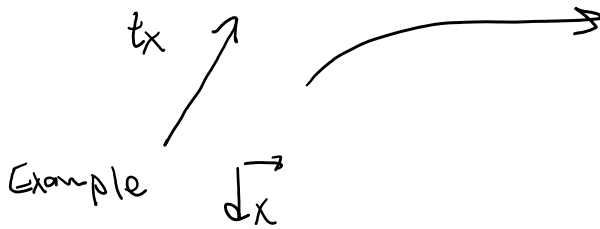
$$\left. \begin{aligned} - & \text{ } - & \text{ } - & \text{ } d^m x \\ d\vec{F}_x &= F^{-1} d\vec{F}_0 \end{aligned} \right\} \rightarrow \begin{aligned} dF_x &= t \cdot \delta \cdot dA_x \\ d\vec{F}_x &= \underbrace{(JF^{-1} T F^t)}_P d\vec{A}_x \end{aligned} \quad (PK-II)$$

PK-II maps $dA_x \rightarrow dF_x$
 or $\vec{n}_x \rightarrow \vec{F}_x$

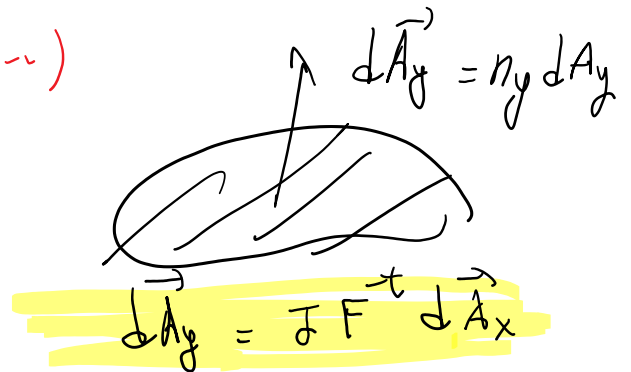
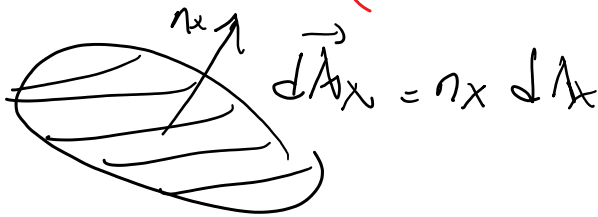
$$P = F^{-1} \delta = J F^{-1} T F^t$$

$$P^t = J (F^{-1})^t T^t (F^{-1})^t = J F^{-1} T F^t = P$$

vectors:
 forces, traction, line segment ($d\vec{y}$), ...
 referential



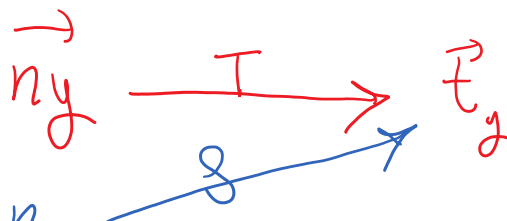
Covectors (surfaces...)



Summary of stresses
 stress tensor maps

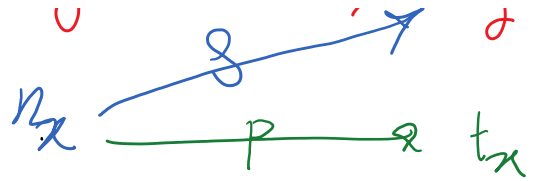
normal vector to traction

Cauchy T
 PK-T S



PK-I

S



PK-II

P

Lagrangian EOM $\rho_0 a - \text{Div} \cdot S = \rho_0 b$
 or $\rho_0 \ddot{u} - \text{Div}(FP) = \rho_0 b$

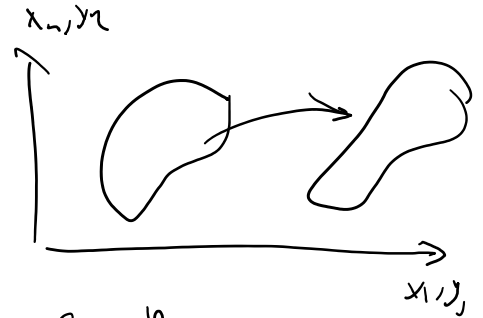
$S = J T F^{-t}$
 $P = J F^{-1} T F^{-t}$

P is better because it's sym.

$P = \begin{pmatrix} P_{11} & P_{12} & P_{13} \\ \text{sym} & P_{22} & P_{23} \\ & & P_{33} \end{pmatrix}$

Balance of energy (hand-out will be given for Thermodynamics laws I, II)
 Thermodynamics law I

E total energy for $\partial \Omega \subset B_t$



$$\frac{DE}{Dt} = \frac{D}{Dt} \int_{B_t} e \, dV_y = \int_{B_t} \dot{e} \, dV_y - \int_{\partial B_t} f_e^p \cdot dA_y$$

volumetric energy density

(Scalar)

$$e = \underbrace{\frac{1}{2} \rho \mathbf{v} \cdot \mathbf{v}}_{\text{kinetic energy density}} + \underbrace{U(F, \dot{F})}_{\text{strain-based energy density}}$$

Mechanical

+ internal energy from temperature

$+ C_v T$

↓ volumetric capacity

thermal

$$+ \frac{1}{2} (\mathbf{E} \cdot \mathbf{D} + \mathbf{H} \cdot \mathbf{B})$$

thermo

$$+ \frac{1}{2} (\mathbf{E} \cdot \mathbf{D} + \mathbf{H} \cdot \mathbf{B})$$

electric field flux magnetic field flux

$$\rho_{ed} = \mathbf{j} \cdot \mathbf{v} + Q$$

$$+ \underbrace{\mathbf{E} \cdot \mathbf{J}}_{\text{Joule's heating}} \text{ electric current density}$$

$$\rho_{\epsilon} = -\sigma \cdot \mathbf{v} + q \text{ outward heat flux} + \underbrace{\mathbf{E} \times \mathbf{H}}_{\text{Poynting vector}}$$

Simplified case: Ignore Mechanical & EM contributions

$$\left. \begin{array}{l} \rho = C_V T \\ \rho_{ed} = Q \\ \rho_{\epsilon} = q \end{array} \right\} \rightarrow$$

$$\frac{\partial C_V T}{\partial t} + \text{div } q = Q$$

no advection

if assume $q = -k \nabla T$ Fourier law

→ parabolic PDE