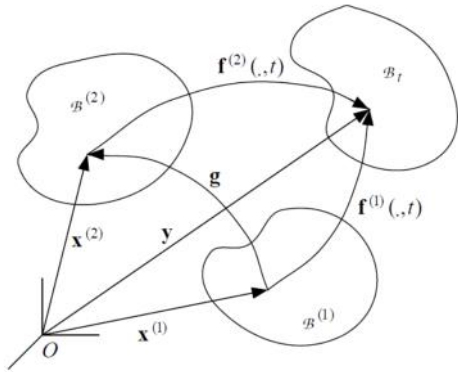


From last time



$$G^0(F \nabla g, g^{-1}(x^{(2)})) = G^2(F^{(2)}, x^{(2)})$$

$$\int_0(x^{(2)}) = \int_0(x^{(1)}) \det \nabla g$$

We restrict ourselves to maps for which density does not change from state 1 to 2

work with g's for which

$$H := \nabla g \text{ is unimodal}$$

$$\det H = 1$$

basically density in states ① & ② will be same identical

if for stress constitutive equation states ① & ② result in the same function expression we say  $H = \nabla g$  belong to stress const. eqn. symmetry group:

$$G(F) = G(FH) \quad \nabla g$$

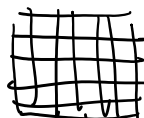
if H is sym group  $G^0 = G^2 = G$

det H = 1,  $G(FH, x) = G(F, x)$   
 H belongs to symmetry group of stress func: G

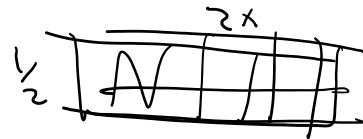
Example:

$$H = \begin{bmatrix} 2 & 0 \\ 0 & \frac{1}{2} \end{bmatrix}$$

cube



G



G

same const eqn  $G \searrow G$

inviscid elastic fluid all unimodal  $H$  belong to  $G$  sym. group

$$T = G(F) = \bar{G}(\underbrace{\det F}_J) = \bar{G}\left(\frac{p}{p_0}\right) = \bar{G}(p)$$

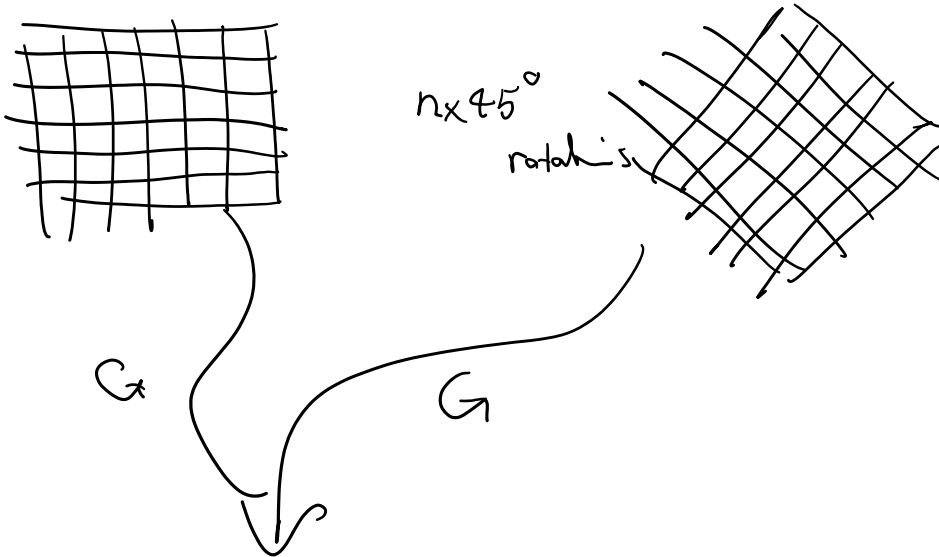
objectivity  
a func of  $\theta \dots$

we can show it will be like

$$T = -p(p, \theta) I$$

pressure temperature

Other example:



$$H = Rot(45^\circ)$$

$$G(F) = G(FH)$$

transverse isotropic



isotropic in xy plane

$$H = Rot_z(\theta) \text{ belongs to sym group}$$

HN orthotropic  
isotropic

Rot( $n \times 90$ )  $\in$  sym. group  
Rot( $\theta$ )  $0 < \theta \leq 360$   $2D$


Voigt stiffness

$$C = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ & C_{22} & C_{23} \\ & & C_{33} \end{bmatrix}_{3 \times 5}$$

$C_{11} = C_{22}$       $C_{12} = C_{23} = C_{13} = 0$   
 $C_{33} = \frac{C_{11} - C_{12}}{2}$

$\rightarrow$  elastic

isotropic material

$C_{ijkl} = \lambda \delta_{ij} \delta_{kl} + \mu (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk})$   
  
 Lamé's parameter

Solid Mechanics, isotropic material  
is characterized by 2 parameters ( $E, \nu$ ), ( $\lambda, \mu$ ), ...

**Definition 112 (Noll, 1958)** Given an elastic body and a reference configuration that corresponds to the region  $\overset{0}{B}$ , the material symmetry group at the material point identified by  $x$  in the reference configuration is the set

$$Msg_x = \{H \in Unim \mathcal{V}^+ : G(FH, x) = G(F, x) \forall F \in Lin \mathcal{V}^+\}$$

Again, it should be emphasized that the material symmetry group is characterized by tensors  $H$  that correspond to the gradients at  $x$  of deformations — not the deformations themselves. This is because the mass density and the elastic response function in the second reference configuration depend only on the gradient of the connecting deformation. Also, note that  $H \in Msg_x$  is not a tensor field, but rather the value of a tensor field at  $x$ .

The following theorem presents a property of all orthogonal elements of  $Msg_x$  that derives from the Principle of Material Frame-Indifference.

-----  
 What about H's that correspond to rotations, can we simply symmetry relation?

Theorem 174 is:

Let  $Q \in \underbrace{Orth^+ D}_{rotations}$  then  $Q \in Msg_x$  iff

$$\forall F \in \text{Lin}^+ \quad G(Q F Q^t) = Q G(F) Q^t \quad (2)$$

$Q \in \text{Msg}$  objectivity  
 ①  $G(F \overset{R}{Q}) = G(F)$   
 ②  $G(Q F) = Q G(F) Q^t$

use  $F' = Q F Q^t$  in ①

$$G((Q F Q^t) Q) = G(Q F Q^t)$$

$$G(Q F (Q^t Q)) = \quad =$$

$$G(Q F) = G(Q F Q^t)$$

We use ② to get

$$Q G(F) Q^t = G(Q F Q^t)$$

What is elasticity 4th order tensor and what's Voigt notation?

(A) Cauchy stress  $T = G(F) = F \bar{G}(C) F^t$   
 objectivity

"objective constitutive eqns for finite elasticity"  $\det H$  is not necessarily small  
 $H = F^{-1}$   
 want to show in infinitesimal theory

(B) PK-I  $S = C E$   
 material is stress free at initial state  
 4th order elasticity tensor  
 infinitesimal strain  $E = \frac{1}{2}(U + U^T)$   $\det H = \ll 1$

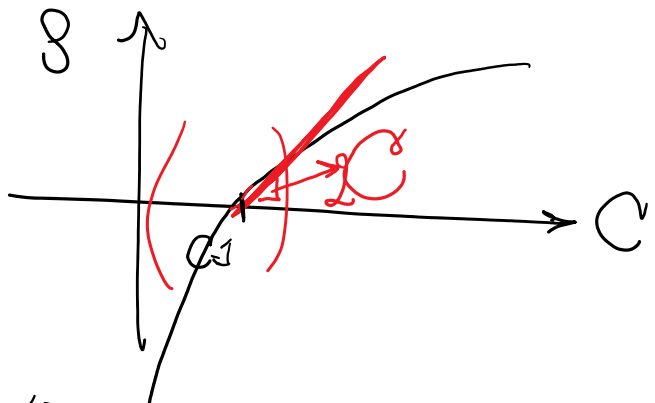
Proof

$$T = F \bar{G}(C) F^t$$

$$S = J^{-1} F^{-t} T = J^{-1} (F \bar{G}(C) F^t) F^{-t} \rightarrow \boxed{S = J^{-1} F \bar{G}(C)}$$

$\sigma(F, \psi) \Gamma \Gamma \rightarrow \boxed{\mathcal{D} = \delta + \bar{G}(C)}$

$C = F^t F$  at time zero  $y=x$   $F = \mathbb{I}_{2,2} = I$  so  $\mathcal{D} = I$



Recall

$$C = F^t F = (I + H)^t (I + H)$$

$$I + 2H + \frac{H^t + H}{2} + H^t H$$

$$\boxed{\mathcal{D} - I = 2E + O(\epsilon^2)}$$

$$= O(\epsilon)$$

in (1) I want to expand  $\bar{G}(C)$

$$\bar{G}(C) = \bar{G}(\underbrace{C - I}_{\substack{\text{small number for} \\ \text{infinitesimal} \\ \text{elasticity} \\ \text{=} \\ O(\epsilon)}} + I) = \bar{G}(I) + \left. \frac{\partial \bar{G}}{\partial C} \right|_{C=I} (C - I) + \text{HOT}$$

Taylor's expansion

like writing

$$g(1+\epsilon) = g(1) + \epsilon g' + O(\epsilon^2)$$

Why  $\bar{G}(C=I) = 0$  (why when  $F=I, T=0$ )

↓ assumption that at initial state stress is zero (stress free initial configuration)

$$\bar{G}(C) = \left. \frac{\partial \bar{G}}{\partial C} \right|_{C=I} (C - I) + O(\epsilon^2)$$

4th order elasticity tensor is defined as

$$C = 2 \frac{\partial^2 \bar{G}(C)}{\partial C^2}$$

4th order elasticity stiffness

Green strain

$$C_{ijkl} = 2 \left( \frac{\partial^2 \bar{G}_{ij}(C)}{\partial C_{kl}} \right)$$

$$\langle \dots \rangle_{ijkl} = 2 \left( \frac{0}{\delta C_{k\ell}} \right)$$

$$\bar{G}(C) = \frac{1}{2} C (C - I) + O(\epsilon^2)$$

$$(C - I) = 2E + O(\epsilon^2) \quad \rightarrow$$

$$\bar{G}(C) = \frac{1}{2} C (2E + O(\epsilon^2)) + O(\epsilon^2)$$

$$\bar{G}(C) = C E + O(\epsilon^2) \quad (2)$$

eqn (1)

$$S = JF \bar{G}(C)$$

$$S = JF (C E + O(\epsilon^2))$$

$$[I + \text{trace}(E) + O(\epsilon^2)] [I + H] (C E + O(\epsilon^2))$$

$\xrightarrow{\text{lower order terms}}$ 
 $\xrightarrow{O(\epsilon)}$

$$S = I C E + \text{HOT}$$

$$\underline{S} = \underline{C} \underline{E}$$

$\underline{S}$  ← PKT  
 $\underline{C}$  ← 4th or 6th elasticity tensor  
 $\underline{E}$  ←  $\frac{u+u^T}{2}$

infinitesimal elasticity

$$S_{ij} = C_{ijkl} E_{lk} = C_{ijkl} E_{kl}$$

$\underline{P}_u, \underline{T}_y$  are all the same in infinitesimal elasticity

$$\underline{S} = \delta F \bar{Q}(C)$$

General finite elasticity

$$\underline{S} = \underline{C} \underline{E}$$

3D  $\underline{S}_{3 \times 3}$   $\underline{E}_{3 \times 3}$   $\underline{C}_{3 \times 3 \times 3 \times 3}$

$$\underline{S} = \begin{pmatrix} S_{11} & S_{12} & S_{13} \\ S_{12} & S_{22} & S_{23} \\ S_{13} & S_{23} & S_{33} \end{pmatrix}$$

6 independent stresses

$$\underline{E} = \begin{pmatrix} E_{11} & E_{12} & E_{13} \\ E_{12} & E_{22} & E_{23} \\ E_{13} & E_{23} & E_{33} \end{pmatrix}$$

36 are independent

Voigt notation

$$s = \begin{pmatrix} S_{11} \\ S_{22} \\ S_{33} \\ S_{12} \\ S_{23} \\ S_{31} \end{pmatrix}$$

sym strain  $\rightarrow \gamma$

$$\begin{pmatrix} E_{11} \\ E_{22} \\ E_{33} \\ 2 E_{12} \\ 2 E_{23} \\ 2 E_{31} \end{pmatrix}$$

$$s = \underline{\tilde{C}} \gamma$$

6x6 Voigt stiffness

6x6 Voigt stiffness

all 36 components can be independent for general elastic material

$C_{ijkl}$   
3x3x3x3

we always have the following "minor" symmetries

$$\left[ \begin{array}{l} C_{ijkl} \\ C_{ijlk} \end{array} \right] = \left[ \begin{array}{l} C_{jilk} \\ C_{jlik} \end{array} \right]$$

2 line proofs in TAM 551

that's why we can use Voigt notation.

Hyperelastic material:

It's an elastic material that whose internal energy density depends on F

General elastic hyperelastic material

$$\begin{array}{l} T = G(F) \\ e = \check{e}(F) \end{array} \xrightarrow{\text{objectivity}} \begin{array}{l} T = F \bar{G}(C) F^t \\ e = \bar{e}(C) \end{array}$$

internal energy density

known  
what is  $\bar{G}$  for hyperelastic material?

$$\bar{G}(C) = \frac{2 p_0}{\sqrt{\det C}} \frac{\partial \bar{e}(C)}{\partial C}$$

Very short proof:

I'll give you a derivation that shows

arbitrary



I'll give you a derivation that shows

$$\underbrace{2\rho}_\text{sym} F \frac{\partial \bar{e}}{\partial C} F^t : D = T : \underbrace{D}_\text{sym}$$

arbitrary  
sym.  
tensor

$$L = \text{grad } v \quad L_{ij} = \frac{\partial v_i}{\partial y_j}$$

$$D = \frac{L + L^T}{2} \quad \text{stretching tensor}$$

$$W = \frac{L - L^T}{2} \quad \text{spin tensor}$$

(recall  $F = R \cdot U$   
(stretch tensor))

$$T = 2\rho F \frac{\partial \bar{e}}{\partial C} F^t$$

$$= F \left( 2 \frac{\rho}{J} \frac{\partial \bar{e}}{\partial C} \right) F^t = F \bar{G}(C) F^t$$

|| from objectivity  
T

$$\bar{G}(C) = \frac{2\rho_0}{\det F} \frac{\partial \bar{e}}{\partial C}$$

$$\bar{G}(C) = \frac{2\rho_0}{\sqrt{\det C}} \frac{\partial \bar{e}}{\partial C} \quad \textcircled{1}$$

$\bar{e} \rightarrow \bar{C}$   
energy  
function

$\bar{C}$   
stress  
constitutive eqn

I can't derive  $C_{ijkl}$  for hyperelastic materials

$$\bar{G}(C=I) = \frac{2\rho_0}{\sqrt{1}} \frac{\partial \bar{e}}{\partial C}(C=I) = 0$$

stress free  
initial  
condn

$$\rightarrow \left[ \frac{\partial \bar{e}}{\partial C}(C=I) = 0 \right] \textcircled{2}$$

Recall  $C = 2 \frac{\partial \bar{G}}{\partial C} \Big|_{C=I} \rightarrow$

$$C_{ijkl} = 2 \frac{\partial \bar{G}_{ij}}{\partial C_{kl}} \Big|_{C=I} \rightarrow$$

$$\bar{G}_{ij} = \frac{2\rho_0}{\sqrt{\det C}} \frac{\partial \bar{e}}{\partial C_{ij}}$$

$$C_{ijkl} = 4\rho_0 \frac{\partial}{\partial C_{kl}} \left( \frac{\partial \bar{e}}{\partial C_{ij}} \Big|_{C=I} \right) + \frac{4\rho_0}{\sqrt{\det C}} \frac{\partial^2 \bar{e}}{\partial C_{kl} \partial C_{ij}} \Big|_{C=I}$$

= 0 because of 2

$$C_{ijkl} = 4\rho_0 \frac{\partial^2 \bar{e}}{\partial C_{kl} \partial C_{ij}}$$

major symmetry =  $4\rho_0 \frac{\partial^2 \bar{e}}{\partial C_{ij} \partial C_{kl}} = C_{klij}$

Only for hyperelastic

$$\begin{bmatrix} S_{11} \\ S_{22} \\ S_{33} \\ S_{12} \\ S_{23} \\ S_{31} \end{bmatrix} = \underset{\substack{\sim \\ 6 \times 6 \\ \text{sym}}}{C} \begin{bmatrix} E_{11} \\ E_{22} \\ E_{33} \\ 2E_{12} \\ 2E_{23} \\ 2E_{31} \end{bmatrix}$$

sym with 21 components