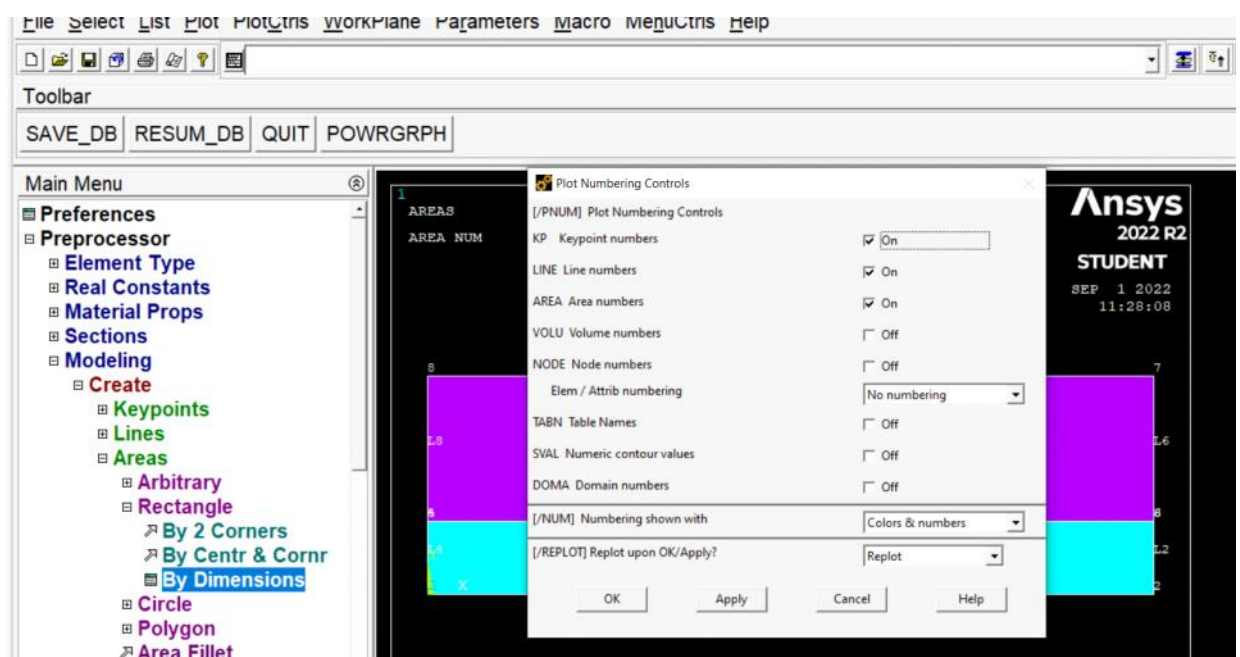
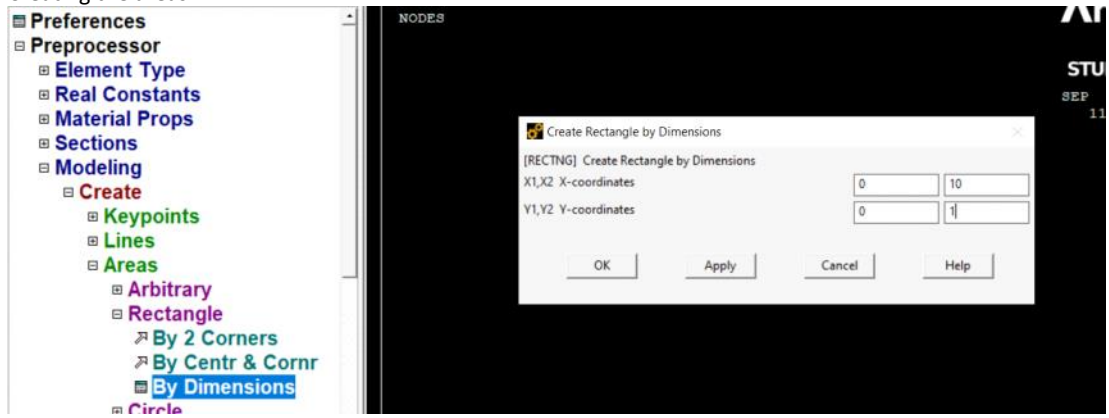
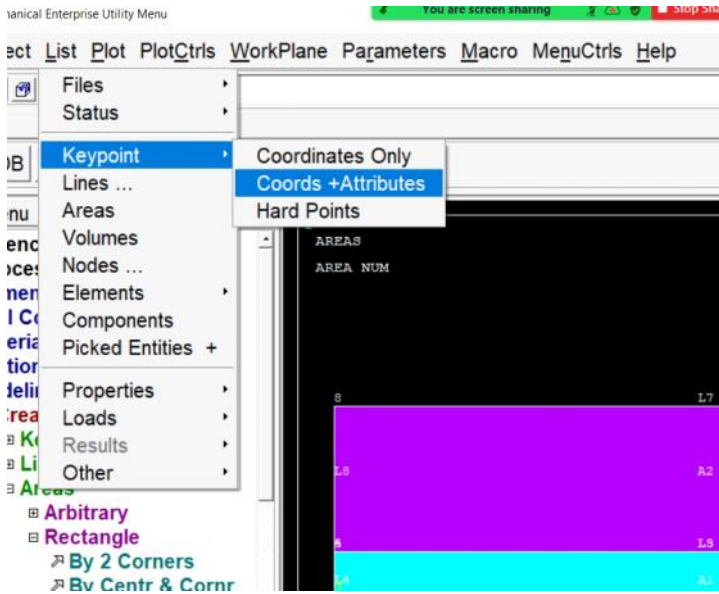
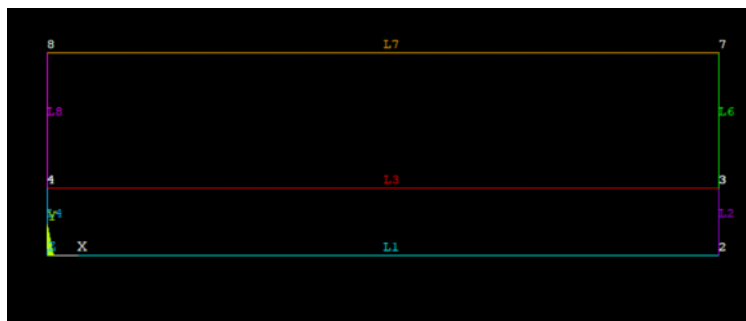
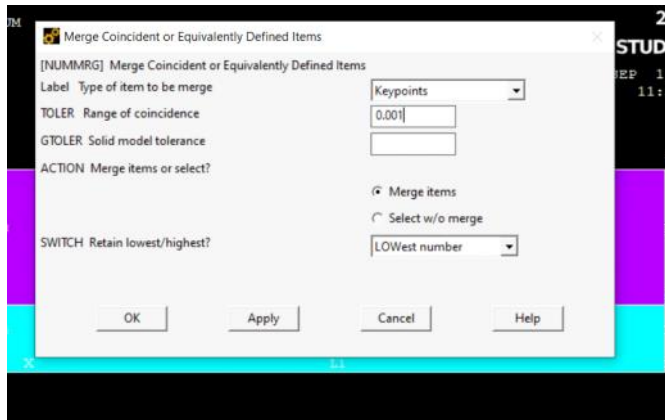


Creating the areas





We need to merge the keypoints and after that the connecting lines will also merge



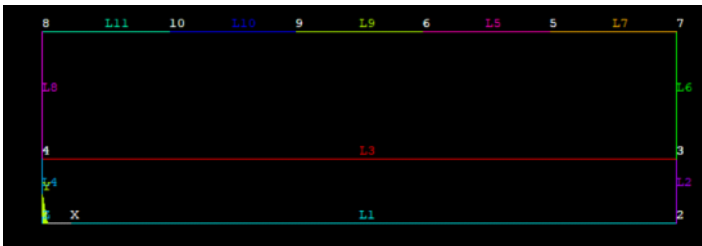
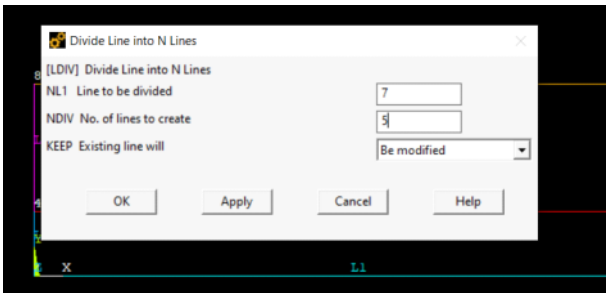
LIST ALL SELECTED KEYPOINTS. DSYN= 0

NO.	X,Y,Z LOCATION	KESIZE	NODE	ELEM	MAT	REAL	TYP
1	0.00 0.00 0.00	0.00	0	0	0	0	0
2	10.0 0.00 0.00	0.00	0	0	0	0	0
3	10.0 1.00 0.00	0.00	0	0	0	0	0
4	0.00 1.00 0.00	0.00	0	0	0	0	0
7	10.0 3.00 0.00	0.00	0	0	0	0	0
8	0.00 3.00 0.00	0.00	0	0	0	0	0

Dividing the top line to 5 segments so we can apply the load on the first

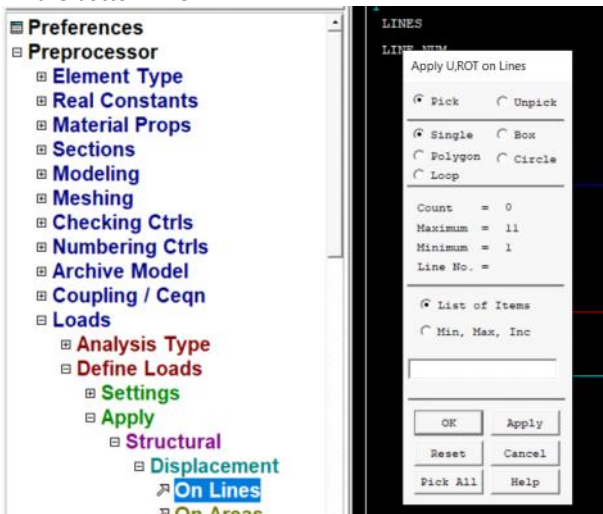
created segment from the left

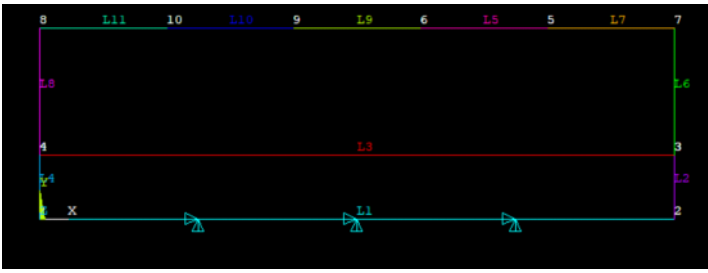
- ▣ Extrude
- ▣ Extend Line
- ▣ Booleans
  - ▣ Intersect
  - ▣ Add
  - ▣ Subtract
  - ▣ Divide
    - ▣ Volume by Area
    - ▣ Volu by WrkPlane
    - ▣ Area by Volume
    - ▣ Area by Area
    - ▣ Area by Line
    - ▣ Area by WrkPlane
    - ▣ Line by Volume
    - ▣ Line by Area
    - ▣ Line by Line
    - ▣ Line by WrkPlane
    - ▣ Line into 2 Ln's
    - ▣ Line into N Ln's



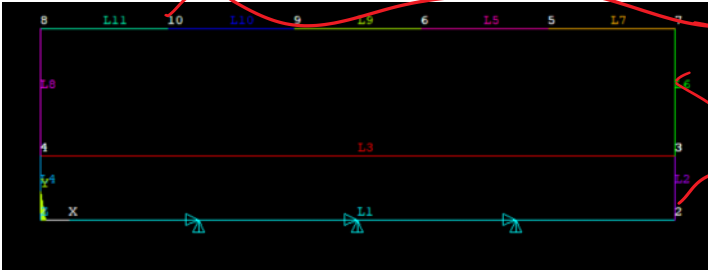
Boundary conditions

- We don't need to specify that the left edge is the axis of symmetry
- Fix the bottom line:



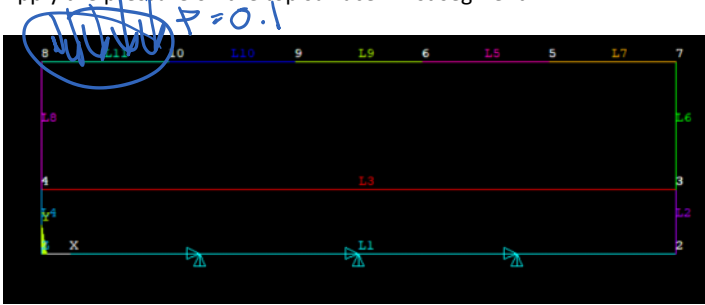


After fixing all dofs on the bottom surface

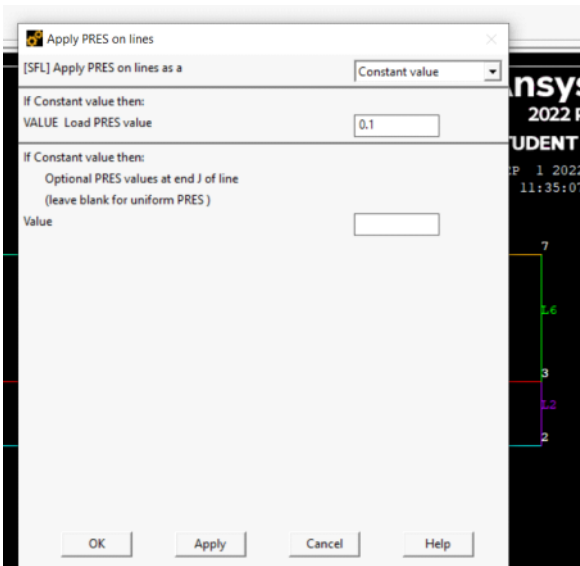


traction free  
no need to define BC here as this is FEM's default

Apply the pressure on the top surface - first segment

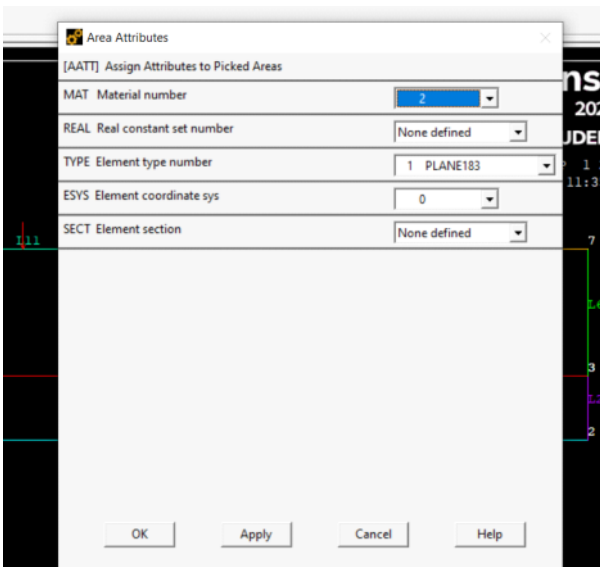
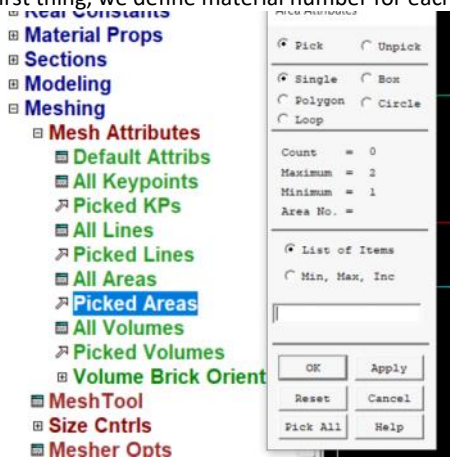


- ▣ Loads
  - ▣ Analysis Type
  - ▣ Define Loads
    - ▣ Settings
    - ▣ Apply
      - ▣ Structural
        - ▣ Displacement
        - ▣ Force/Moment
        - ▣ Pressure
          - ▣ On Lines



Meshing

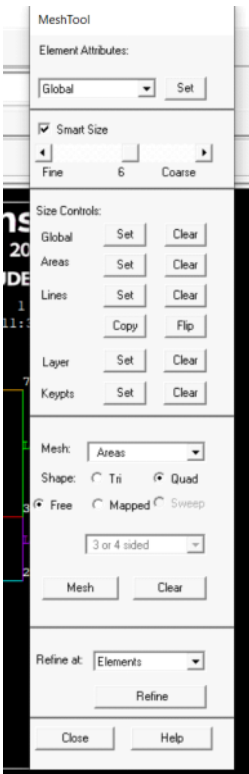
First thing, we define material number for each area



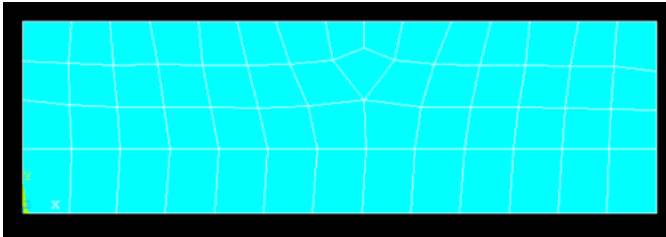
List the areas

LIST ALL SELECTED AREAS.

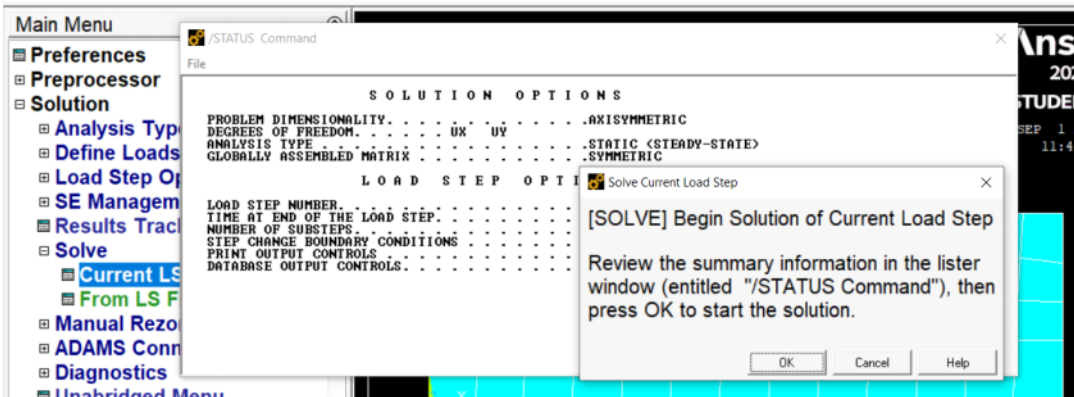
NUMBER	LOOP	LINES	AREA	ELEM SIZE	#NODES	#ELEM	MAT	REAL	TYP	ESYS	SECN
1	1	2 3 4	N/A	0.000	0 0	1	0	1	0	0	0
2	1	3 6 7 5	N/A	0.000	0 0	2	0	1	0	0	0
	9	10 11 8									



Pick all areas:

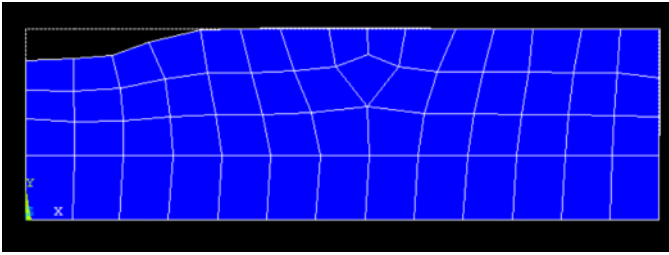


Solution:



Postprocess:





Contour plots

Choose 1st principal stress from the list of nodal contour plot:

Contour Nodal Solution Data

Item to be contoured

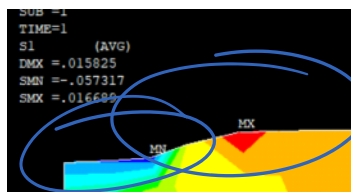
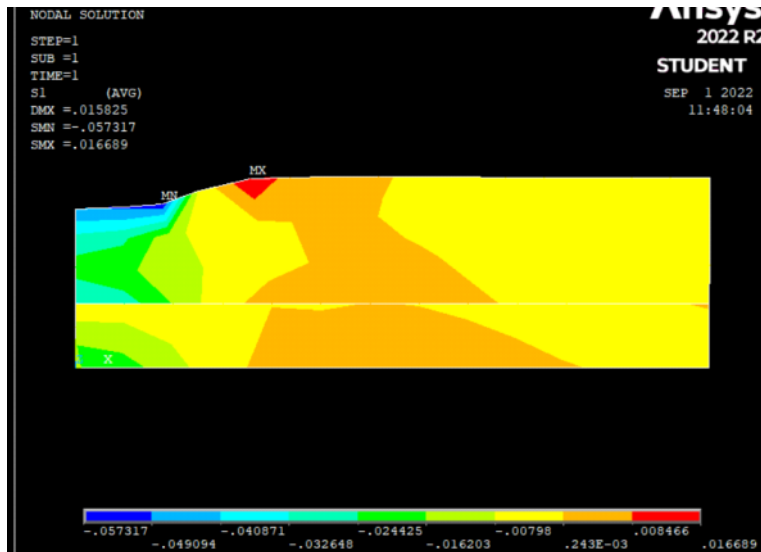
- X-Component of stress
- Y-Component of stress
- Z-Component of stress
- XY Shear stress
- YZ Shear stress
- XZ Shear stress
- 1st Principal stress**
- 2nd Principal stress
- 3rd Principal stress
- Stress intensity
- von Mises stress
- Plastic equivalent stress

Undisplaced shape key: Deformed shape only

Scale Factor: Auto Calculated 31.595089072

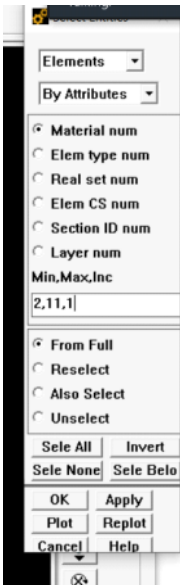
Additional Options

OK Apply Cancel Help

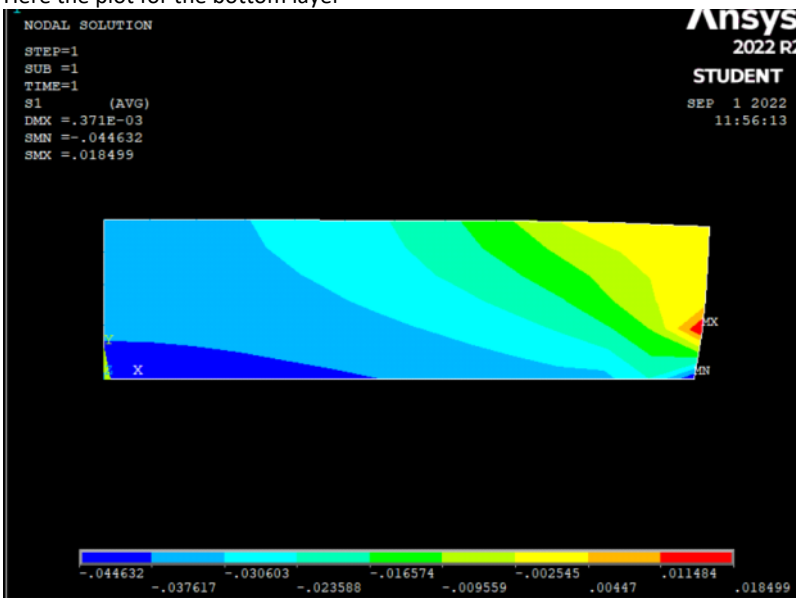


Min and max sigma<sub>1</sub> for the whole domain

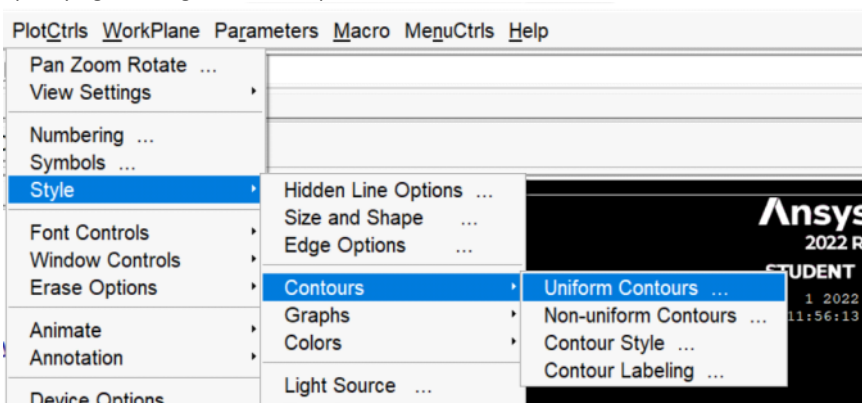
Plotting the results for certain number of layers  
 Select -> entities ->



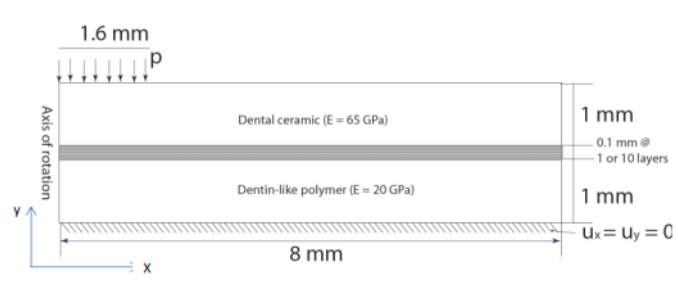
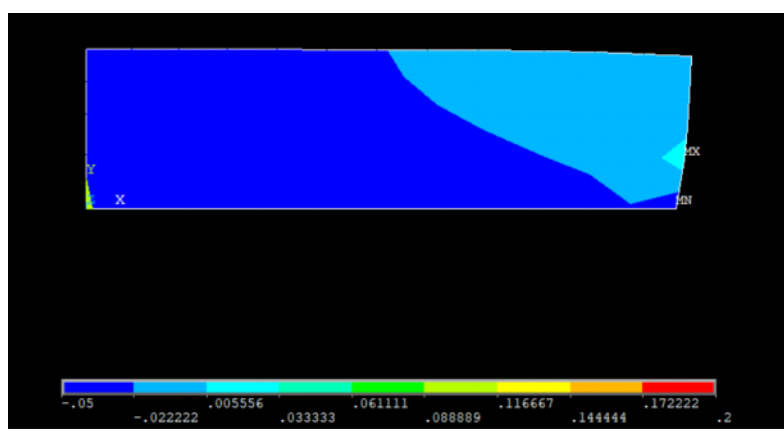
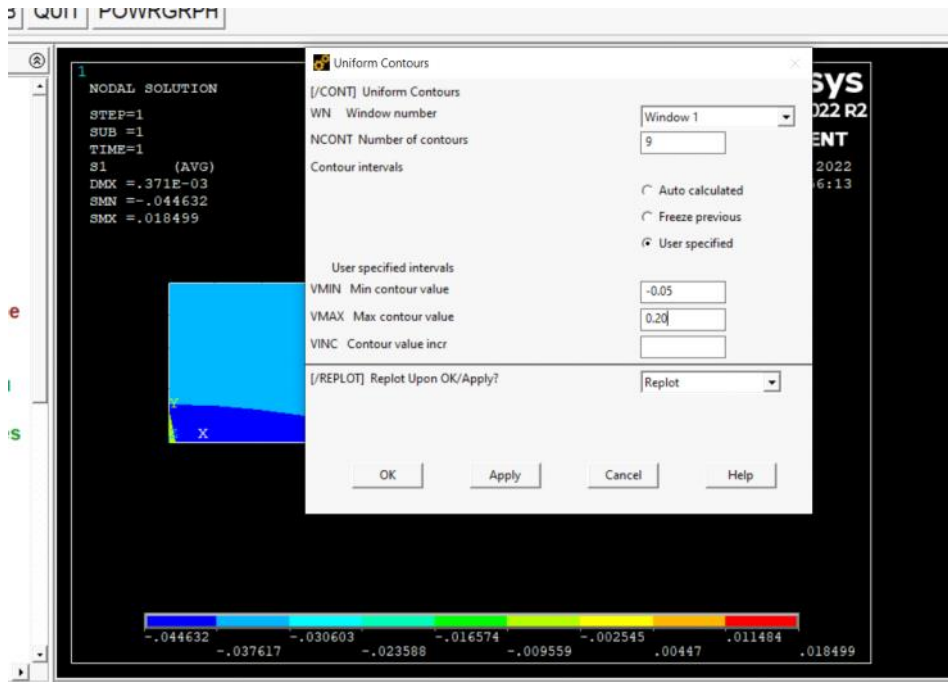
Here the plot for the bottom layer



Specifying the range of contour plot:





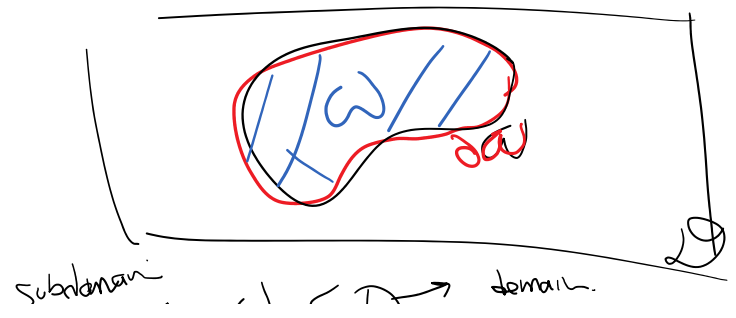


*I → make sure total length is 0.1 mm*

### FEM Formulation:

① Balance laws

$$\sum_{\omega} F = 0$$



$$\int_{\partial \Omega} \sigma \cdot n \, dS = 0$$

Subdomain  $\omega \subseteq D$  domain  $\Omega$

$$\int_{\partial \omega} \sigma \cdot n \, dS + \int_{\omega} p b = 0$$

$\sigma \cdot n$  stress tensor  
 $p b$  body force  
 $\omega$  arbitrary  
 $\partial \omega$  boundary of  $\omega$

②  $\longrightarrow$  we will derive strong form = (Partial) Differential Equation (PDE)

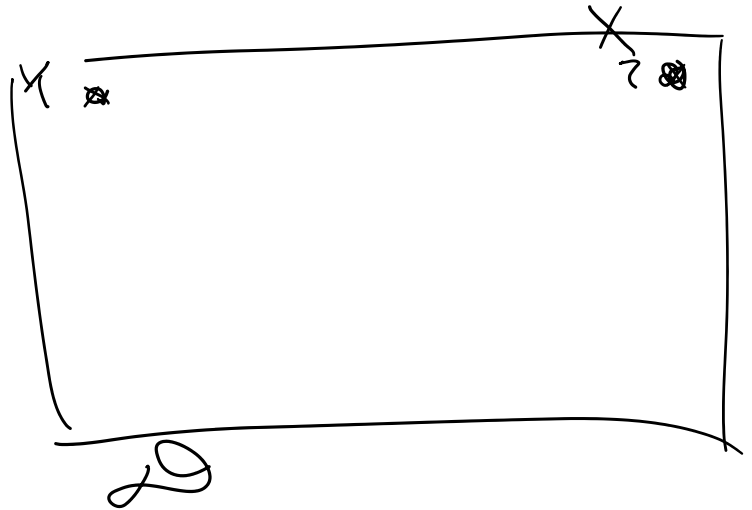
$$\nabla \cdot \sigma + p b = 0 \quad \forall x \in \Omega$$

divergence theorem

③

$$R = \nabla \cdot \sigma + p b$$

residual "error"



Weighted Residual Statement (WRS)

weight function  $\rightarrow$  arbitrary

$$\int_{\Omega} w (\nabla \cdot \sigma + p b) = 0$$

fixed  $w$



$\nabla W$   
 ↓  
 weight  
 functions

fixed domain

$R = \text{Residual}$

(4)

$$u = \sum_{i=1}^N a_i \phi_i(x)$$

shape functions known → we choose this

$\phi_1 = x$   
 $\phi_2 = x^2$   
 $\phi_3 = x^3$

unknowns

Discretization

∞ unknowns → N unknowns ( $a_1, \dots, a_n$ )

→ N equations

we'll satisfy (\*) for  $w_1, w_2, \dots, w_n$   
 we'll choose these functions:

$$\forall i \in \{1, \dots, N\} \int_D w_i (\nabla \cdot \sigma + f) = 0$$

1D  
 example  
 $w_1 = x$   
 $w_2 = x^2$   
 $w_3 = x^3$   
 ⋮

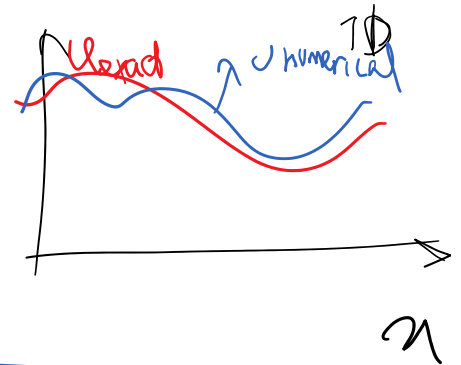
Find  $a_i$ 's

$$u = \sum a_i \phi_i(x)$$

fully known

is obtained

$u \rightarrow \Sigma \rightarrow \sigma \dots$



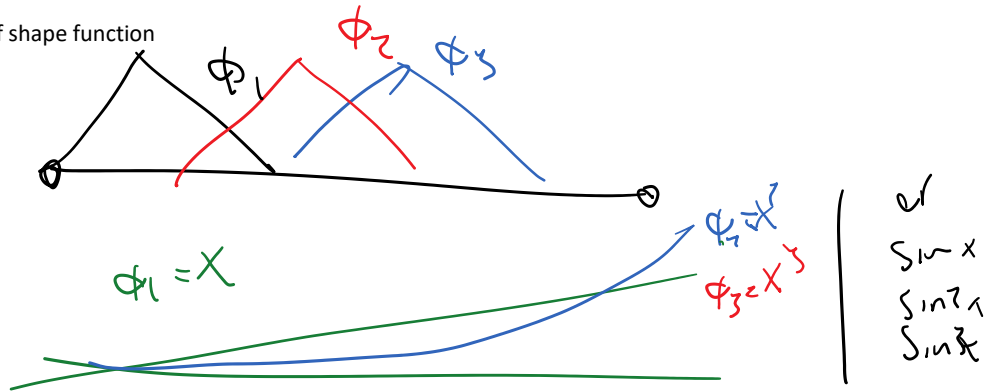
Different methods have different forms of shape function

$\phi_1 \quad \phi_2 \quad \phi_3$

Different methods have different forms of shape function

FEM

Spectral method

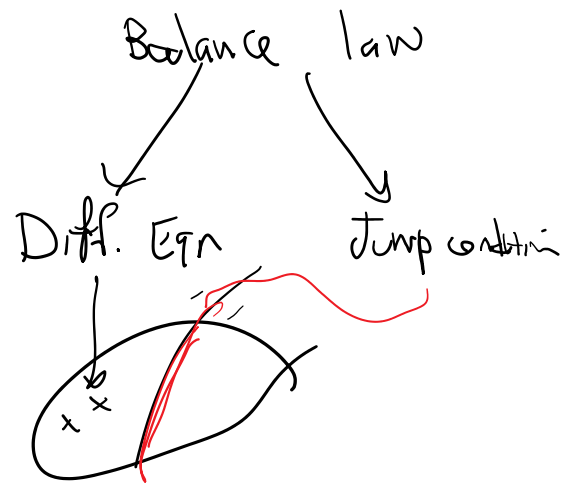


Discontinuous Galerkin -> Different basis functions

FEM formulation in detail

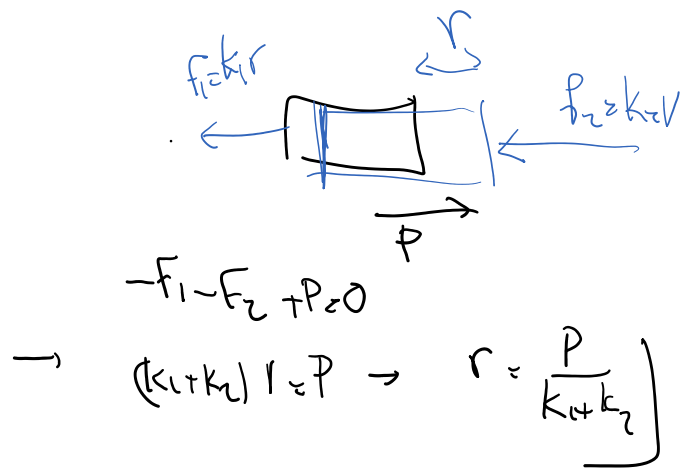
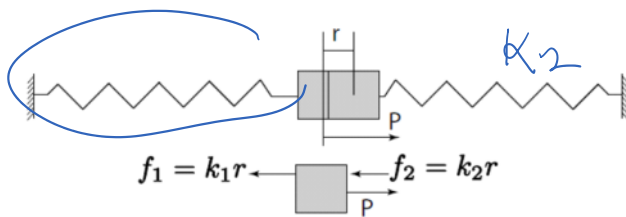
1. Balance law

- Why start with a balance law?
  - They are the actual physics laws.
  - They contain more *information* than their corresponding PDEs.
  - Larger solution space than the PDEs.
- Can we directly start the FE formulation from a PDE?
  - Yes, FE formulation starts from a differential equation.
  - A PDE may not be derived from a balance law.



Balance of mass, force (linear momentum), energy, ...

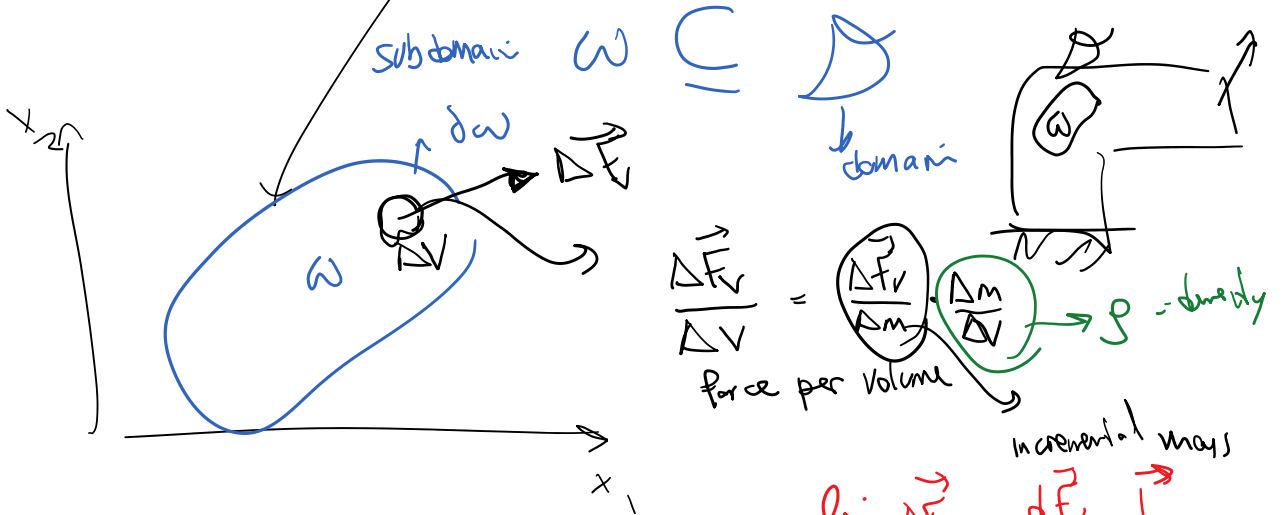
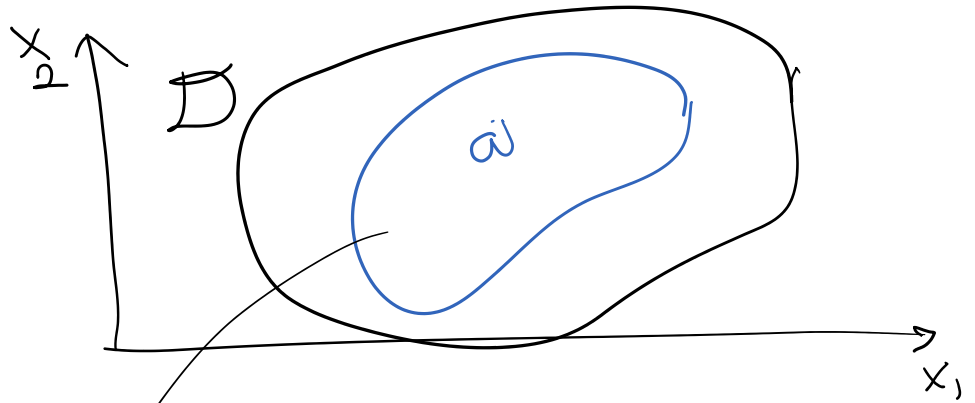
Example of balance of force in discrete setting:



Continuum:

Balance of forces

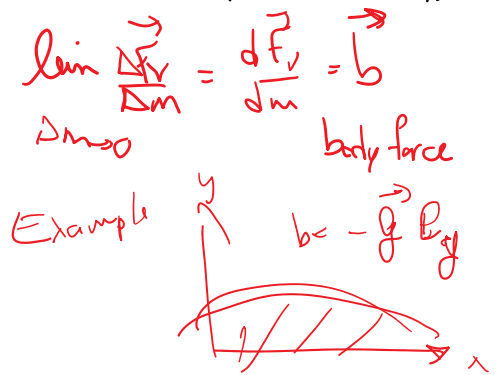
$$\sum F = 0$$



Types of forces: 1. Volumetric force

$$\vec{F}_V = \sum \Delta \vec{F}_V = \sum \frac{\Delta \vec{F}_V}{\Delta V} \Delta V$$

$$= \int_{\omega} (\rho \vec{b}) dV$$

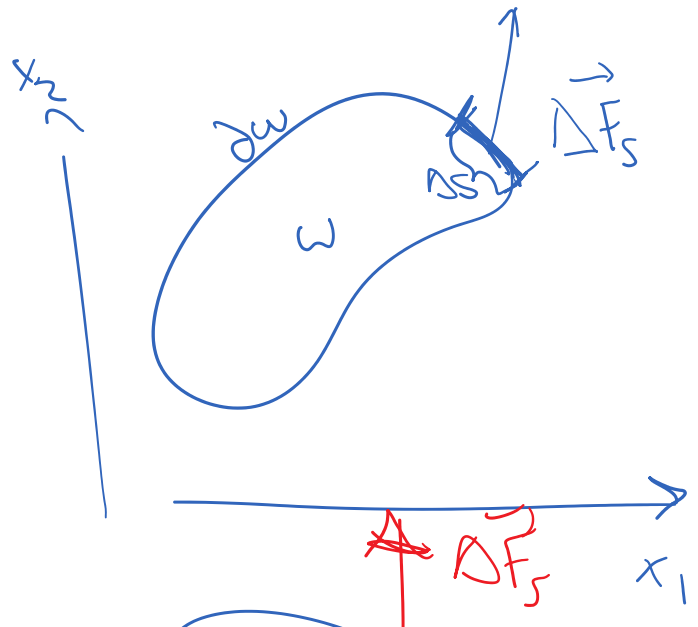


Surface force

$$\vec{F}_S = \sum \Delta \vec{F}_S$$

$$= \sum \frac{\Delta \vec{F}_S}{\Delta S} \Delta S$$

let  $\Delta S \rightarrow 0$



~~area~~  $\Delta S \rightarrow 0$

$$= \int \left[ \lim_{\Delta S \rightarrow 0} \left( \frac{\Delta F_s}{\Delta S} \right) \right] dS$$



intensity of force  
"force per area"  $\lim_{\Delta S \rightarrow 0} \frac{\Delta F_s}{\Delta S} = \vec{t}$

$$\lim_{\Delta S \rightarrow 0} \frac{\Delta F_s}{\Delta S} = \vec{t}$$

MPa Pa  
psi

$$\vec{F}_s = \int \vec{t} dS$$

$$\vec{t} = \sigma \cdot \vec{n}$$

stress tensor

$$\sigma = \begin{pmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & - & \\ & & \sigma_{33} \end{pmatrix}$$

stress tensor

$$t_s = \begin{pmatrix} t_1 \\ t_2 \\ t_3 \end{pmatrix}$$

$$n_s = \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}$$

