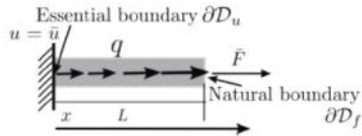


Discrete Galerkin formulation for solid bar



• F_r , F_N , and F_D are given by,

$$F_r = (\phi^T, q)_r = \int_D [\phi]^T q \, dv = \int_0^L \begin{bmatrix} \phi_1 \\ \phi_2 \\ \vdots \\ \phi_{n_f} \end{bmatrix} q \, dx \quad (301a)$$

$$F_N = (\phi^T, \bar{F})_N = \int_{\partial D_f} [\phi]^T \bar{F} \cdot \mathbf{N} \, ds = \left(\begin{bmatrix} \phi_1 \\ \phi_2 \\ \vdots \\ \phi_{n_f} \end{bmatrix} \bar{F} \right)_{x=L} \quad (301b)$$

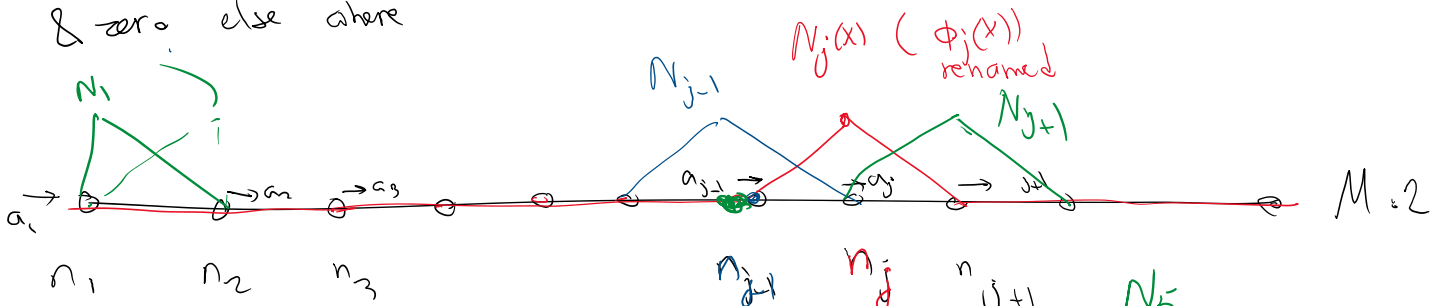
$$F_D = \mathcal{A}(\phi^T, \phi_p) = \int_D \frac{d}{dx} [\phi]^T EA \frac{d}{dx} \phi_p \, dv = \int_0^L \frac{d}{dx} \begin{bmatrix} \phi_1 \\ \phi_2 \\ \vdots \\ \phi_{n_f} \end{bmatrix} EA \frac{d}{dx} \phi_p \, dx \quad (301c)$$

$(\int_m(\Phi))^T D (\int_m \Phi)$

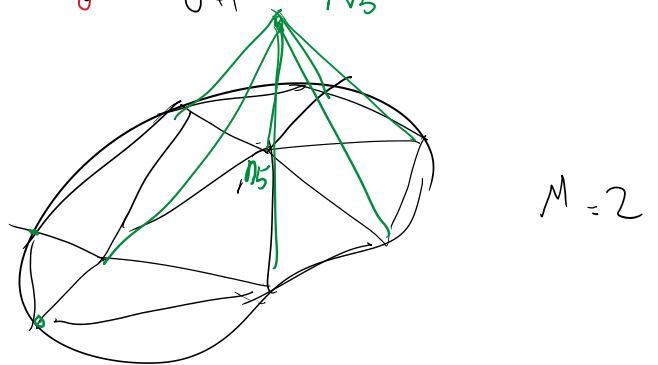
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In finite element method (FEM) we denote basis functions (phis) by N and call them shape functions.

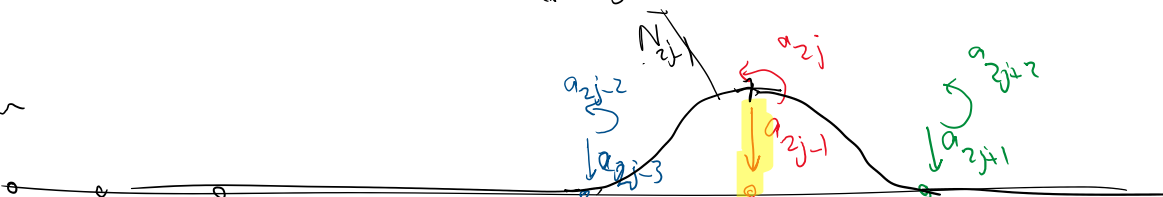
Shape functions take the value of 1 @ one degree of freedom (dof) & zero else where



2D heat conduction:

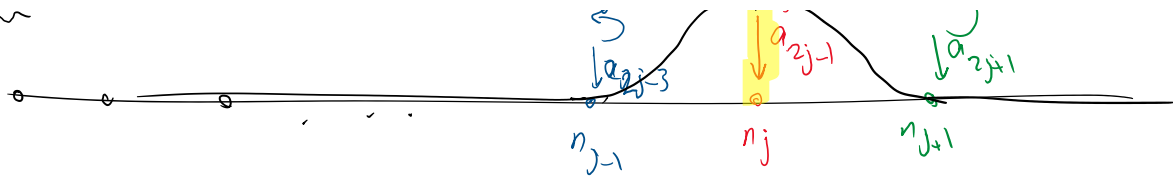


beam problem



$(E I y^{(4)}) + q = 0$
M=4

problem

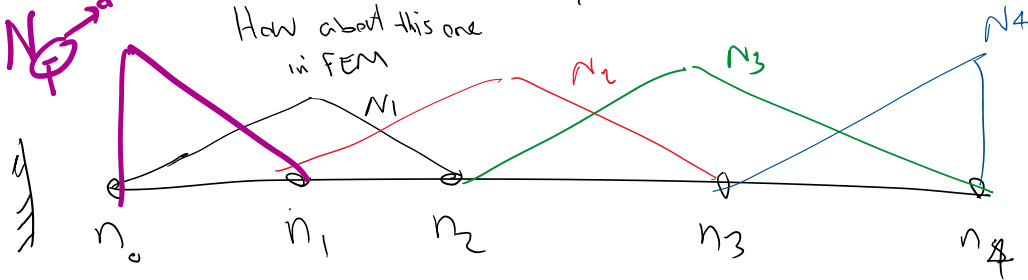


$$E I y'' + q = 0$$

$$M = 4$$

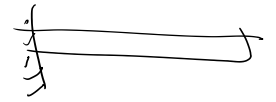
$$u^h(x) = \phi_p(x) + \sum_{i=1}^n a_i \Phi_i(x)$$

prescribed dof #1



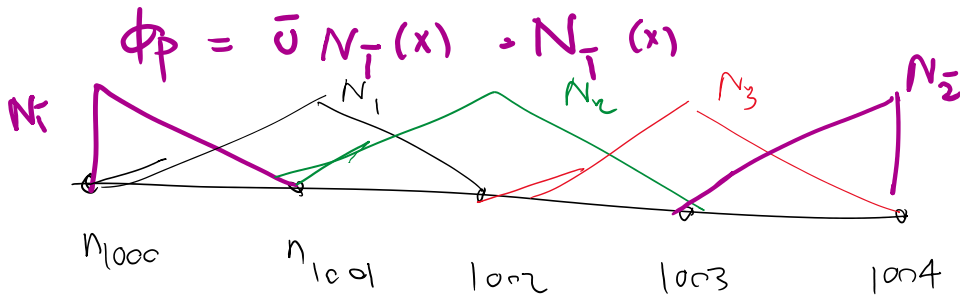
$$n_f = \# \text{ free dofs} = 4$$

$$n_p = \# \text{ prescribed dofs} = 1$$



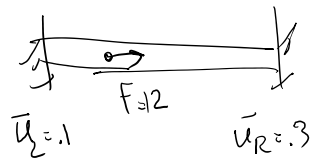
$$u: \bar{u} = 1$$

we used $\phi_p = \bar{u} = 1$ before how about now?



$$n_f = 3$$

$$n_p = 2$$



$N(i)$ or N_i^- : i refers to dof #

$$u^h(x) = \phi_p(x) + \sum_{i=1}^{n_f} a_i N_i(x) = \underbrace{\left(\sum_{i=1}^{n_f} a_i^- N_i^-(x) \right)}_{\phi_p} + \sum_{i=1}^{n_f} a_i N_i(x)$$

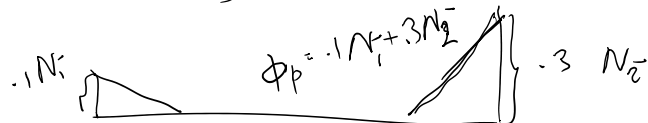
same as

$$u^h(x) = \begin{bmatrix} N_1(x) & N_2(x) \end{bmatrix} \begin{bmatrix} a_1^- \\ a_2^- \end{bmatrix} + \begin{bmatrix} N_1(x) & N_2(x) & N_3(x) \end{bmatrix} \begin{bmatrix} a_1 \\ a_2 \\ a_3 \end{bmatrix}$$

this example

$$a_1^- = 0.1$$

$$a_2^- = 3$$



in terms of nodal we write

In terms of nodal we write

$$u^h = \sum_{i=1}^{n_p} a_i^- N_i^-(x) + \sum_{i=1}^n a_i N_i(x)$$

$$= \bar{N} \bar{a} + N a$$

$$\bar{N} = [N_1^- \dots N_{n_p}^-] \quad \bar{a} = \begin{bmatrix} a_1^- \\ \vdots \\ a_{n_p}^- \end{bmatrix}$$

$$N = [N_1 \dots N_{n_f}] \quad a = \begin{bmatrix} a_1 \\ \vdots \\ a_{n_f} \end{bmatrix}$$

$n_f = \#$ free dofs

$n_p = \#$ prescribed dofs

another notati:

$$u^h = \sum_{i=1}^{n_p} a_i^P N_i^P(x) + \sum_{i=1}^{n_f} a_i^F N_i^F(x)$$

$$\Phi_P = N^P a^P + N^F a^F$$

$$N^P = [N_1^P \dots N_{n_p}^P] \quad a^P = \begin{bmatrix} a_1^P \\ \vdots \\ a_{n_p}^P \end{bmatrix}$$

$$N^F = [N_1^F \dots N_{n_f}^F] \quad a^F = \begin{bmatrix} a_1^F \\ \vdots \\ a_{n_f}^F \end{bmatrix}$$

often a^P & N^P are simply written as a & N

2D example

Heat conduction:

each node has 1 dof

(Temperature)

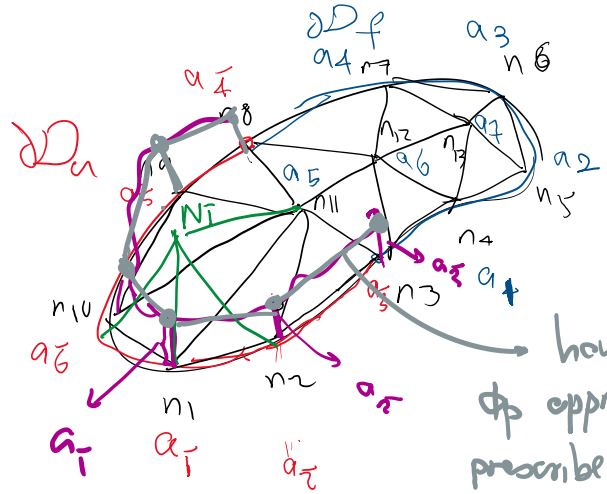
$$n = 8$$

$$n_f = 7$$

$$n_p = 6$$



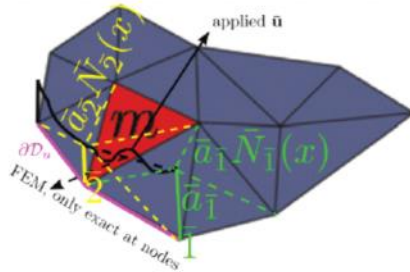
(x, y)



$$\Phi_P = a_1^- N_1^-(x) + a_2^- N_2^-(x) + \dots + a_6^- N_6^-(x)$$

$$\Phi_P(n_2) = 0 \quad a_2^- \downarrow 1 \quad \dots \quad 0 \quad = \quad 0_2^-$$

B. Essential Boundary Conditions



$$\begin{aligned}
 n_p &:= \text{number of (prescribed) dof on } \partial D_u \\
 (\cdot) &= \text{decoration for prescribed dofs} \\
 \{\bar{1}, \dots, \bar{n}_p\} &= \text{global dofs on } \partial D_u \\
 \bar{a} = [\bar{a}_{\bar{1}}, \dots, \bar{a}_{\bar{n}_p}]^T &= \text{vector of prescribed values for these dofs} \\
 \bar{N} = [\bar{N}_{\bar{1}}, \dots, \bar{N}_{\bar{n}_p}] &= \text{(row) vector of shape functions for these dofs} \\
 \bar{B} = [\bar{B}_{\bar{1}}, \dots, \bar{B}_{\bar{n}_p}] &= \text{(row) vector of "displacement to strain" for these dofs}
 \end{aligned} \tag{308}$$

- ϕ_p is formed by,

$$\phi_p = \bar{N}\bar{a} = \sum_{i=1}^{n_p} \bar{a}_{\bar{i}} \bar{N}_{\bar{i}} \Rightarrow \frac{d\phi_p}{dx} = \bar{B}\bar{a} = \sum_{i=1}^{n_p} \bar{a}_{\bar{i}} \bar{B}_{\bar{i}} \tag{309}$$

- While this construction guarantees satisfaction of essential boundary conditions at nodes $\bar{1}, \dots, \bar{n}_p$, as shown in the figure the intermediate values may not match \bar{u} .

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Now that we have established how the solution is interpolated, we want to form K, FD, Fr, FN (and a new force Fn)

1) K matrix

$$K = \int_D \begin{bmatrix} \phi_1 \\ \vdots \\ \phi_n \end{bmatrix} D \begin{bmatrix} \phi_1 & \dots & \phi_n \end{bmatrix} dV$$

\downarrow
 material
 section
 property

$n = \# \text{ unknowns}$

in FEM language

$$\phi_1 = N_1 \quad \dots$$

$$\phi_{n_f} = N_{n_f}$$

$$K = \int_D \begin{bmatrix} N_1 \\ N_2 \\ \vdots \\ N_{n_f} \end{bmatrix} D \underbrace{\begin{bmatrix} N_1 & \dots & N_{n_f} \end{bmatrix}}_{B^T \text{ (or } B)} dV$$

$$K = K^{pp} = \int (\mathbf{B}^p)^t \mathbf{D} \mathbf{B}^p dV$$

$\mathbf{B}^p = \mathbf{L}_m \mathbf{N}^p$ \mathbf{B} "displacement to strain map"

(2)

What about \mathbf{F}_D Dirichlet BC force vector

Recall

$$\mathbf{F}_D = \int_D \mathbf{L}_m \begin{bmatrix} \phi_1 \\ \vdots \\ \phi_n \end{bmatrix} \mathbf{D} \mathbf{L}_m^p \phi_p dV$$

compare this with

$$K = \int_D \mathbf{L}_m \begin{bmatrix} \phi_1 \\ \vdots \\ \phi_n \end{bmatrix} \mathbf{D} \mathbf{L}_m^p (\phi_1, \dots, \phi_n) dV$$

in FEM language

$$\mathbf{F}_D = \int_D \mathbf{L}_m \begin{bmatrix} N_1 \\ \vdots \\ N_{np} \end{bmatrix} \mathbf{D} \mathbf{L}_m^p \left(\sum_{i=1}^{np} a_i^p N_i^p \right) dV$$

$$= \int_D \mathbf{L}_m \begin{bmatrix} N_1 \\ \vdots \\ N_{np} \end{bmatrix} \mathbf{D} \mathbf{L}_m^p \left([N_1^p \dots N_{np}^p] \begin{bmatrix} a_1^p \\ \vdots \\ a_{np}^p \end{bmatrix} \right) dV$$

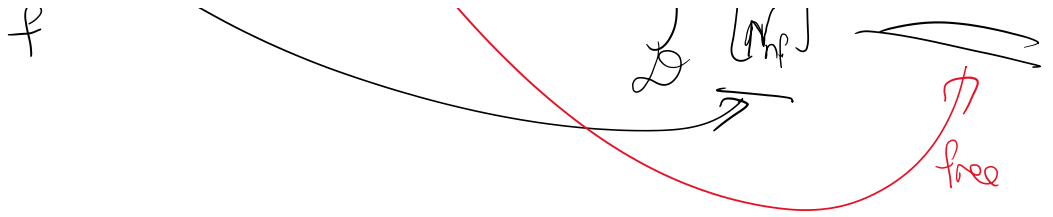
$$\mathbf{F}_D = \left(\int_D \mathbf{L}_m \begin{bmatrix} N_1 \\ \vdots \\ N_{np} \end{bmatrix} \mathbf{D} \mathbf{L}_m^p \left([N_1^p \dots N_{np}^p] \right) \right) \begin{bmatrix} a_1^p \\ \vdots \\ a_{np}^p \end{bmatrix}$$

prescribed

recall

$$K^{pp} = K = \int_D \mathbf{L}_m \begin{bmatrix} N_1 \\ \vdots \\ N_{np} \end{bmatrix} \mathbf{D} \mathbf{L}_m^p [N_1^p \dots N_{np}^p] dV$$

both
f



Summary of K & F_D

$$K = K^{fp} = \int_{\mathcal{D}} (B)^{ft} \underbrace{D B^f}_{\text{a.k.a } B} dV$$

$$B^f = L_m(N_1^f \dots N_{n_f}^f)$$

$$B^p = L_m(N_1^p \dots N_{n_p}^p)$$

$$F_D = K^{fp} a^p$$

$$K^{fp} = \int_{\mathcal{D}} (B^p)^t D B^p dV$$

(2)

Numerical example

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Bar problem

$$\int_{\mathcal{D}} \underbrace{w'}_{L_m(w)} \underbrace{EA}_{D} u' \dots$$

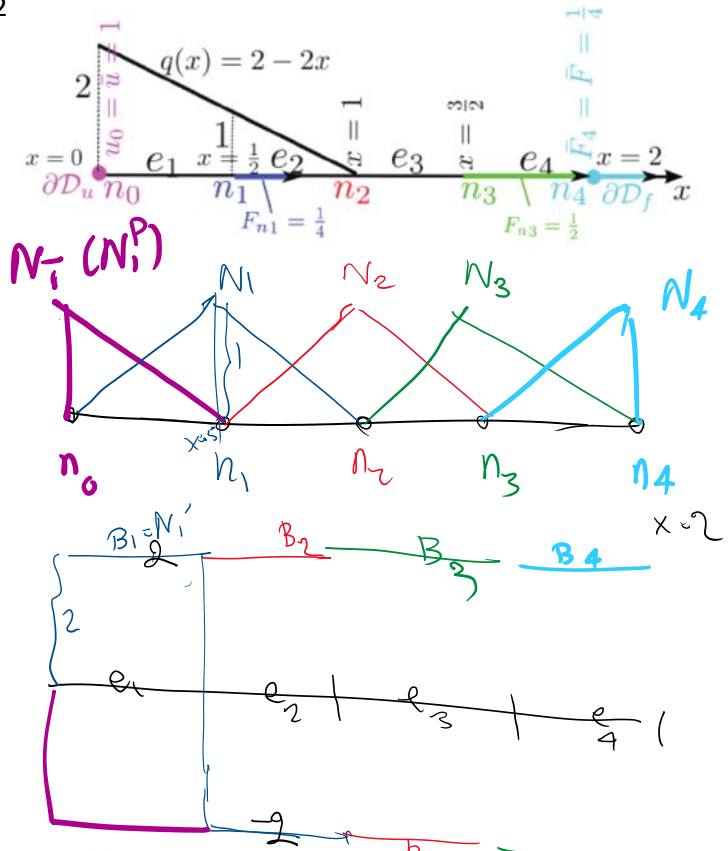
$$L_m(w) \rightarrow \text{dim} = ()'$$

$$n_f = 4$$

$$n_p = 1 \quad a_p (= \bar{a}) = [1]$$

$$a = (a^p) = \begin{pmatrix} a_1 \\ a_2 \\ a_3 \\ a_4 \end{pmatrix} \Rightarrow \text{free}$$

r2

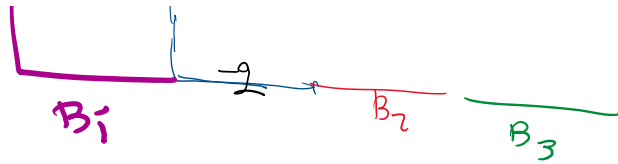


$$K = \int_0^2 B_f^t \frac{D}{EA} B_f dx$$

$$B_f = \text{dim} [N_1 \ N_2 \ N_3 \ N_4]$$

$$= \frac{1}{dx} [N_1 \ N_2 \ N_3 \ N_4]$$

$$= [N_1' \ N_2' \ N_3' \ N_4']$$



$$K = \int_0^2 \begin{bmatrix} B_1 \\ B_2 \\ B_3 \\ B_4 \end{bmatrix} \frac{1}{EA} \begin{bmatrix} B_1 & B_2 & B_3 & B_4 \end{bmatrix} dx$$

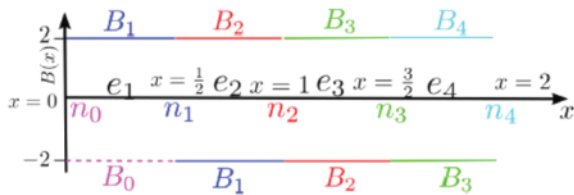
$$K_{11} = \int_0^2 B_1 B_1 dx = \int_{e_1} B_1 B_1 dx + \int_{e_2} B_1 B_1 dx =$$

$$\int_0^1 (2x)(2x) dx + \int_{\frac{1}{2}}^1 (-2)(-2) dx = \frac{4}{2} + \frac{4}{2} = 4$$

$$K_{12} = \int_0^2 B_1 B_2 dx = \int_{e_1} B_1 B_2 dx$$

$$= \int_{\frac{1}{2}}^1 (-2)(2x) dx = -2$$

Bar Example: Step 1: Stiffness matrix



$$K = \int_0^2 \begin{bmatrix} B_1 \\ B_2 \\ B_3 \\ B_4 \end{bmatrix} EA \begin{bmatrix} B_1 & B_2 & B_3 & B_4 \end{bmatrix} dx = \begin{bmatrix} \int_0^2 B_1 B_1 dx & \int_0^2 B_1 B_2 dx & \int_0^2 B_1 B_3 dx & \int_0^2 B_1 B_4 dx \\ \text{sym.} & \int_0^2 B_2 B_2 dx & \int_0^2 B_2 B_3 dx & \int_0^2 B_2 B_4 dx \\ \int_{e_1} B_1 B_1 dx + \int_{e_2} B_1 B_1 dx & \int_{e_2} B_1 B_2 dx & 0 & 0 \\ \int_{e_2} B_2 B_2 dx + \int_{e_3} B_2 B_2 dx & \int_{e_3} B_2 B_3 dx & \int_{e_3} B_2 B_4 dx & \int_{e_4} B_2 B_4 dx \\ \int_{e_3} B_3 B_3 dx + \int_{e_4} B_3 B_3 dx & \int_{e_4} B_3 B_4 dx & \int_{e_4} B_4 B_4 dx & \int_{e_4} B_4 B_4 dx \end{bmatrix}$$

$$= \begin{bmatrix} \frac{1}{2} \cdot (2) \cdot (2) + \frac{1}{2} \cdot (-2) \cdot (-2) & \frac{1}{2} \cdot (-2) \cdot (2) & 0 & 0 \\ \text{sym.} & \frac{1}{2} \cdot (2) \cdot (2) + \frac{1}{2} \cdot (-2) \cdot (-2) & \frac{1}{2} \cdot (-2) \cdot (2) & 0 \\ \frac{1}{2} \cdot (2) \cdot (2) + \frac{1}{2} \cdot (-2) \cdot (-2) & \frac{1}{2} \cdot (-2) \cdot (2) & \frac{1}{2} \cdot (2) \cdot (2) + \frac{1}{2} \cdot (-2) \cdot (-2) & \frac{1}{2} \cdot (-2) \cdot (2) \\ \frac{1}{2} \cdot (2) \cdot (2) + \frac{1}{2} \cdot (-2) \cdot (-2) & \frac{1}{2} \cdot (-2) \cdot (2) & \frac{1}{2} \cdot (2) \cdot (2) & \frac{1}{2} \cdot (2) \cdot (2) \end{bmatrix} \Rightarrow$$

$$K = \begin{bmatrix} 4 & -2 & 0 & 0 \\ \text{sym.} & 4 & -2 & 0 \\ & & 4 & -2 \\ & & & 2 \end{bmatrix} \quad (316)$$