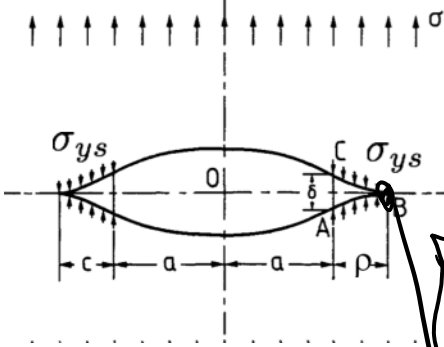


Continued from the last time:

superposition principle

$$K_I = K_I^\sigma + K_I^{\sigma_{ys}}$$



$$K_I^\sigma = \sigma \sqrt{\pi(a + \rho)}$$

$$K_I^{\sigma_{ys}} = +2\sigma_{ys} \sqrt{\frac{a + \rho}{\pi}} \cos^{-1} \left( \frac{a}{a + \rho} \right)$$

other requiring  $K @ a + \rho \text{ local} = 0$

$$K = K_I^\sigma + K_I^{\sigma_{ys}} = 0 \rightarrow$$

$$2\sigma_{ys} \sqrt{\frac{a + \rho}{\pi}} \cos^{-1} \left( \frac{a}{a + \rho} \right) = \sigma \sqrt{\pi(a + \rho)}$$

$$\cos \left( \frac{\pi \sigma}{2\sigma_{ys}} \right) = \frac{a}{a + \rho} \quad \rho = ?$$

$$\frac{\sigma}{\sigma_{ys}} \ll 1 \quad \text{SSY}$$

$$\cos x = 1 - \frac{x^2}{2} \quad \text{for small } x = \frac{\pi \sigma}{2 \sigma_{ys}}$$

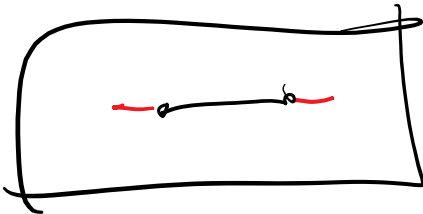
We obtain:

Strip yield model

$$\rho = \frac{\pi^2 \sigma^2 a}{8 \sigma_{ys}^2} = \frac{\pi}{8} \left( \frac{K_I}{\sigma_{ys}} \right)^2$$

312

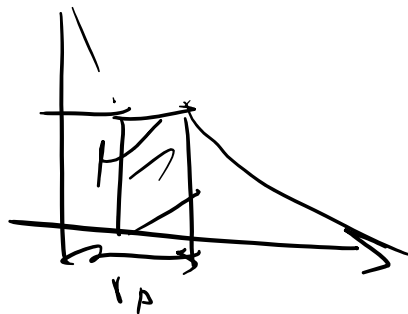
with 1D stress redistribution



Irvin

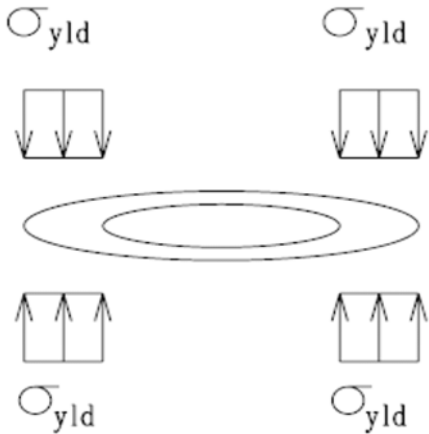
$$r_p = \frac{1}{\pi} \left( \frac{K_I}{\sigma_{ys}} \right)^2$$

318



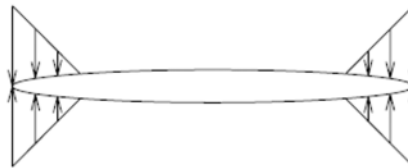
# 3. Strip Yield Model: Dugdale vs Barenblatt model

Dugdale: Uniform stress



More appropriate for polymers

Barenblatt: Linear stress



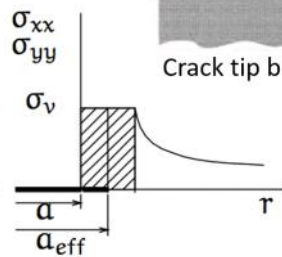
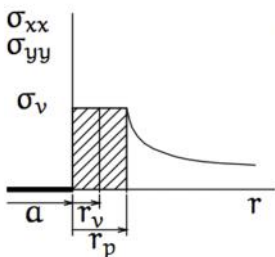
More appropriate for metals

182

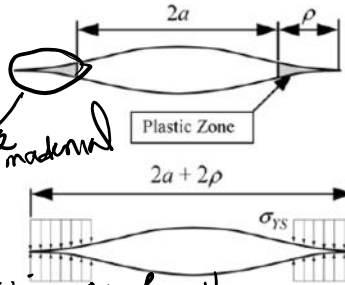
These are predecessors to Traction-Separation-Relations (TSRs) or cohesive models that we will later discuss.

## Effective crack length

Irwin



Dugdale



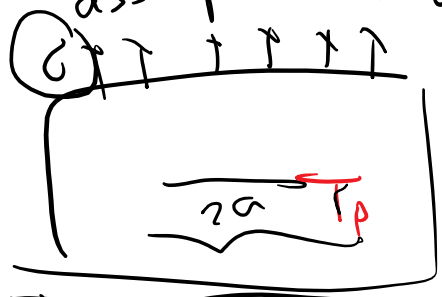
*Since material yielding here, it is as if the effective crack length is longer than 2a*

$$a \rightarrow a_{eff} = a + \rho \rightarrow \text{Fracture Process Zone (FPZ)}$$

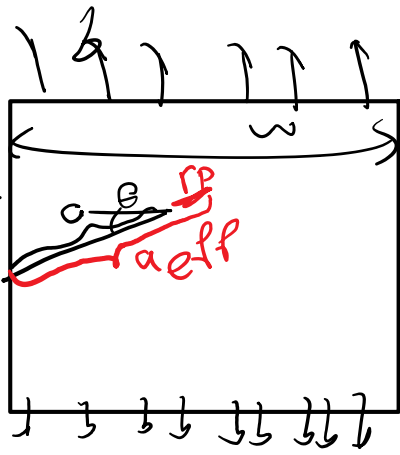
the applicability of LEFM theory is increased

size

This means we can apply the theory for slightly higher loads than we could have had with SSY assumption & still get acceptable results



$\sigma \uparrow \rightarrow r_p \uparrow$



$$K_{eff} = f(a_{eff}, W, \theta, \dots) \sqrt{2a_{eff}} \sigma$$

$$a_{eff} = a + r_p = a + \frac{1}{2\pi} \left( \frac{K_{eff}}{\sigma_{ys}} \right)^2$$

2 nonlinear equations, 2 unknowns

Iterations needed in general

Idea: (1) use  $a_{eff}$ ?  $f(a, W, \theta) \approx f(a)$   $a_{eff} = a \rightarrow K_{eff}$   
 (2)  $K_{eff} \rightarrow a_{eff}$

Infinite domain

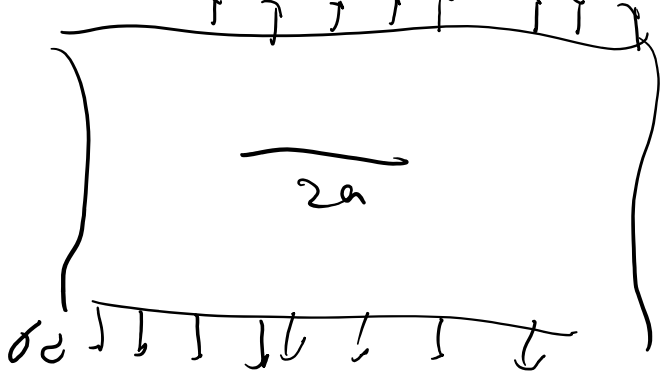
Mid-crack case

$$K_{eff} = f \sqrt{2a_{eff}} \sigma$$

$$a_{eff} = a + \frac{1}{2\pi} \left( \frac{K_{eff}}{\sigma_{ys}} \right)^2$$

1st order approximation

$$K_{eff} = \sqrt{\pi} \left( a + \frac{1}{2\pi} \left( \frac{K_{eff}}{\sigma_{ys}} \right)^2 \right) \sigma$$



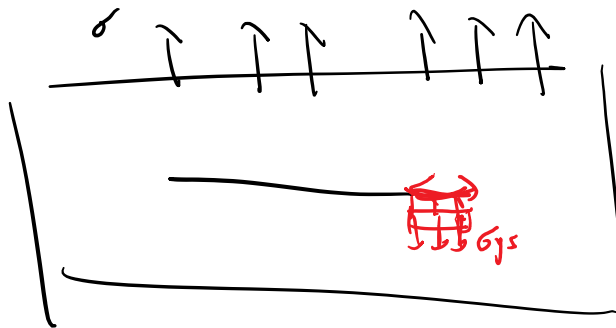
$$K_{eff} = \sqrt{\pi} \left( a + \frac{1}{2\pi} \left( \frac{K_{eff}}{\sigma_{ys}} \right)^2 \right) \sigma$$

$$K_{eff}^2 = \pi a \sigma^2 + \frac{1}{2} K_{eff}^2 \left( \frac{\sigma}{\sigma_{ys}} \right)^2$$

$$K_{eff} = \frac{\sqrt{\pi a} \sigma}{\sqrt{1 - \frac{1}{2} \left( \frac{\sigma}{\sigma_{ys}} \right)^2}}$$

← without self correction

$\frac{\sigma}{\sigma_{ys}} \ll 1$   
 we can ignore  
 the term  
 $K_{eff} = \sqrt{\pi a} \sigma$



$\left( \frac{\sigma}{\sigma_{ys}} \right)^2 \ll 1$   
 LEFM :-

$\left( \frac{\sigma}{\sigma_{ys}} \right)^2 \lesssim 1$   
 sigma getting close to sigma<sub>ys</sub>  
 self idea extends the applicability of the theory

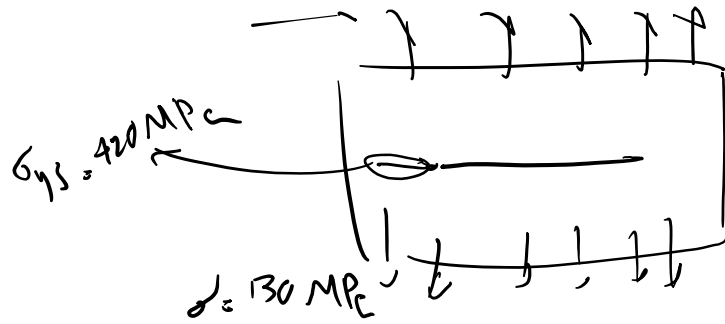
$\frac{\sigma}{\sigma_{ys}} \gtrsim 1$   
 PFM  
 other theories

Example:

Consider a large central cracked plate subjected to a uniform stress of 130 MPa. The fracture toughness  $K_c = 50 \text{ MPa}\sqrt{\text{m}}$ , the yield strength  $\sigma_{ys} = 420 \text{ MPa}$ .

- (a) What is the maximum allowable crack length? <sup>with self correction</sup>
- (b) What is the maximum crack length if plastic correction is taken into account. Plane stress and Irwin's correction.





$$\frac{\sigma}{\sigma_{y_s}} = \frac{130}{420} \approx 0.3 \rightarrow 0.3 \text{ rel. large}$$

use the correction

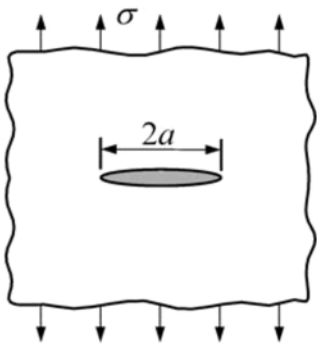
(a)  
Part

$$K_{Ic} = \frac{\sigma \sqrt{\pi a}}{130} \rightarrow a = 94.2 \text{ mm}$$

(Part b)

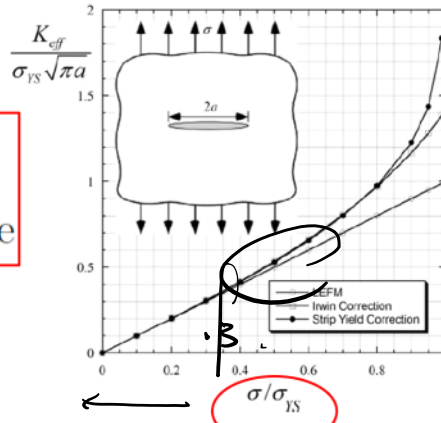
$$K_{Ic} = 470 = \frac{\sigma \sqrt{\pi a}}{\sqrt{1 - \frac{1}{2} \left(\frac{\sigma}{\sigma_{y_s}}\right)^2}} = \frac{130 \sqrt{\pi a}}{\sqrt{1 - \left(\frac{130}{420}\right)^2}}$$

$$a = \dots \text{ m} \rightarrow a_{\text{eff}} = 89.7 \text{ mm}$$



$$K_{eff} = \frac{\sigma \sqrt{\pi a}}{\sqrt{1 - \frac{1}{2} \left( \frac{\sigma}{\sigma_{YS}} \right)^2}}$$

As  $\frac{\sigma}{\sigma_{ys}}$  increases  $\Rightarrow$   
LEFM becomes less accurate



$\left( \frac{\sigma}{\sigma_{ys}} \right) \approx .1$

Relating this to SSY:

crack tip

K-dominant region

Inelastic region

$r_p = \frac{1}{2} \left( \frac{K}{\sigma_{ys}} \right)^2$

LEFM leading term is dominant

$r \rightarrow \infty$   
 $\sigma \rightarrow 0$

$\sigma_{yy} \rightarrow \sigma$   
 $a_2 \rightarrow a$

$\sigma(r, \theta=0) = \frac{K}{\sqrt{2\pi r}} = \sigma$

$r_p = \frac{1}{2\pi} \left( \frac{K}{\sigma} \right)^2$

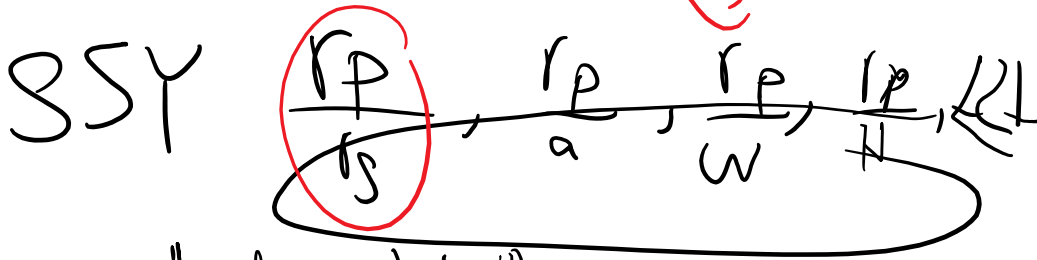
$r_s = ?$

$$r_s = \frac{1}{2\pi} \left( \frac{1}{\sigma} \right)$$

$$r_p = \frac{1}{2\pi} \left( \frac{K}{\sigma_{ys}} \right)^2$$

$$\left( \frac{r_p}{r_s} \right) = \left( \frac{\sigma}{\sigma_{ys}} \right)^2$$

$$\frac{r_p}{r_s} \ll 1$$



all relevant lengths

LEFM provides accurate global response & local = outside FPZ

$$\frac{r_p}{r_s} \ll 1 \quad \frac{b}{\sigma_{ys}} \ll .3$$

## Plastic yield criteria

von-Mises criterion

$$\sigma_v = \sqrt{3J_2}$$

$$= \sqrt{\frac{(\sigma_{11} - \sigma_{22})^2 + (\sigma_{22} - \sigma_{33})^2 + (\sigma_{33} - \sigma_{11})^2 + 6(\sigma_{12}^2 + \sigma_{23}^2 + \sigma_{31}^2)}{2}}$$

$$= \sqrt{\frac{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_1 - \sigma_3)^2}{2}}$$

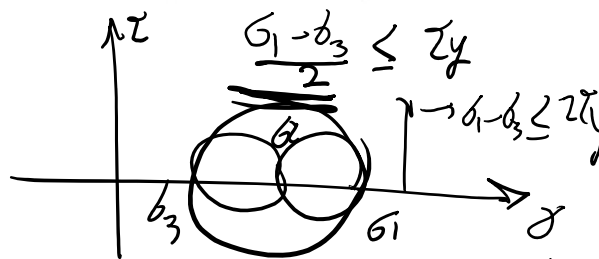
$$= \sqrt{\frac{3}{2} s_{ij} s_{ij}} \quad s \text{ is stress deviator tensor}$$

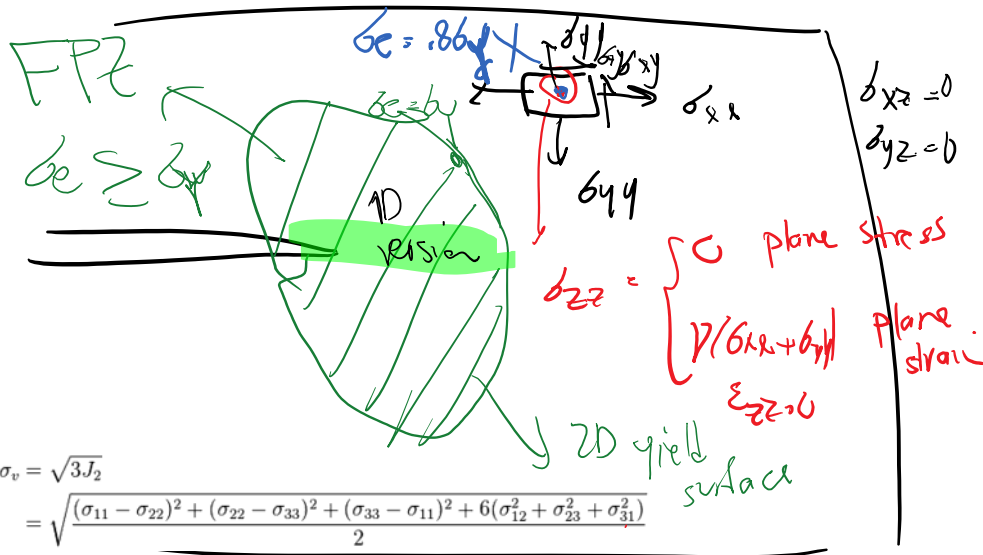
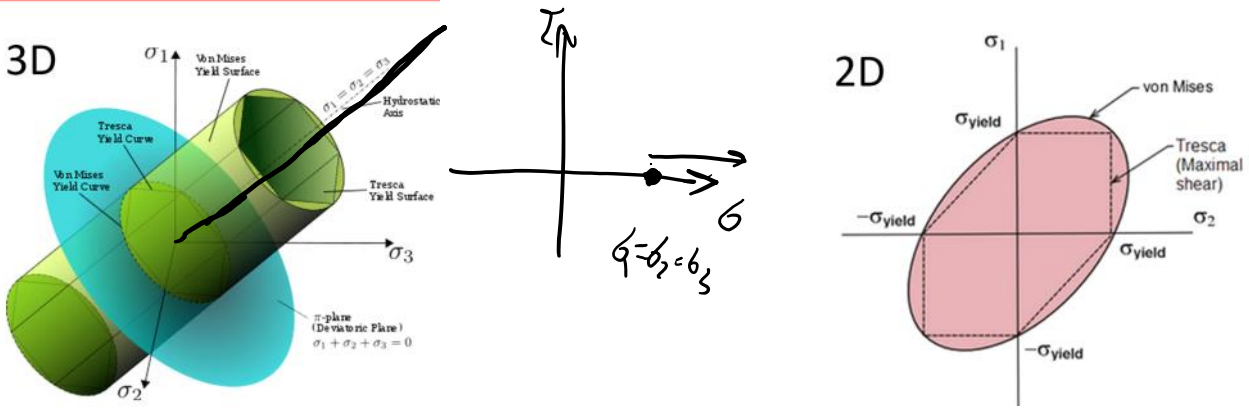
$$\sigma^{dev} = \sigma - \frac{1}{3} (\text{tr } \sigma) \mathbf{I}$$

Tresca criterion

Maximum shear stress

$$\sigma_{tresca} = \sigma_1 - \sigma_3 > \sigma_{max}$$





von-Mises criterion  
 $\sigma_e = \sigma_{ys}$

Principal stresses:

Mode I, principal stresses

$$\begin{cases} \sigma_1 = \frac{K_I}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left( 1 + \sin \frac{\theta}{2} \right) \\ \sigma_2 = \frac{K_I}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left( 1 - \sin \frac{\theta}{2} \right) \\ \sigma_3 = \begin{cases} 0 & \text{plane stress} \\ \frac{2\nu K_I}{\sqrt{2\pi r}} \cos \frac{\theta}{2} & \text{plane strain } \nu(b_1 + b_2) \end{cases} \end{cases}$$



$$r_y(\theta) = \frac{1}{4\pi} \left( \frac{K_I}{\sigma_{ys}} \right)^2 \left[ 1 + \cos \theta + \frac{3}{2} \sin^2 \theta \right] \quad \text{plane stress}$$

$$r_y(\theta) = \frac{1}{4\pi} \left( \frac{K_I}{\sigma_{ys}} \right)^2 \left[ (1 - 2\mu)^2 (1 + \cos \theta) + \frac{3}{2} \sin^2 \theta \right] \quad \text{plane strain}$$

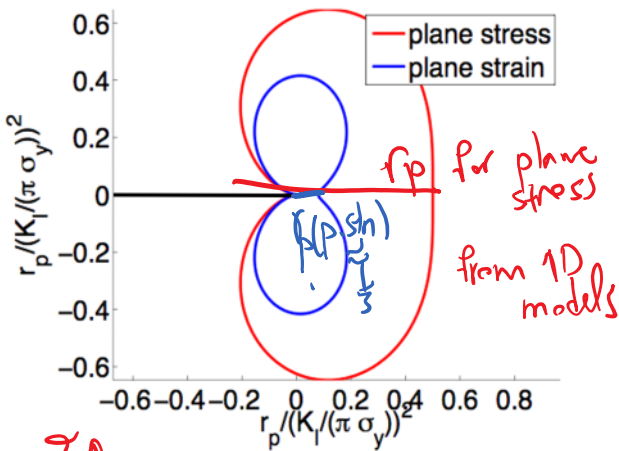
*we had this from 1D models*

*angle-dependent terms*

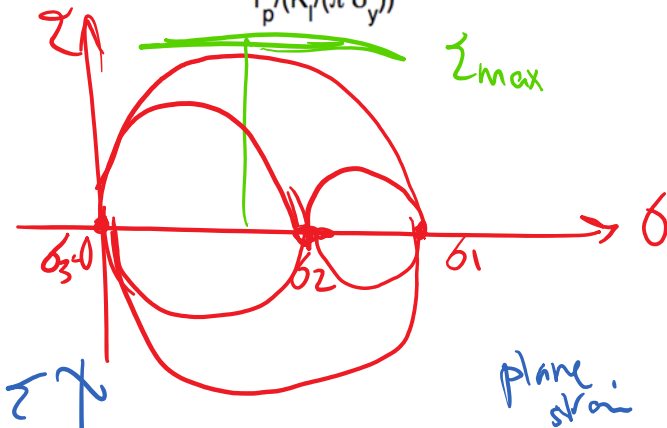
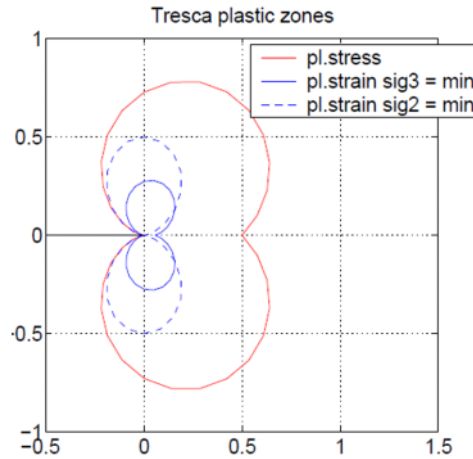
### von-Mises criterion

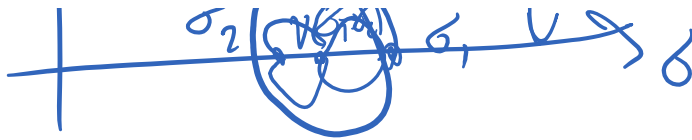
$$r_y(\theta) = \frac{1}{4\pi} \left( \frac{K_I}{\sigma_{ys}} \right)^2 \left[ 1 + \cos \theta + \frac{3}{2} \sin^2 \theta \right]$$

plane stress

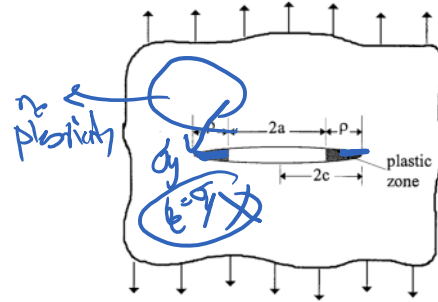
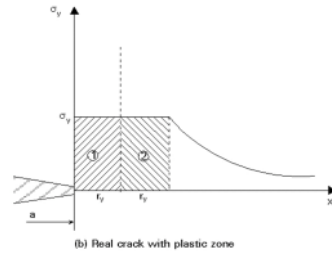
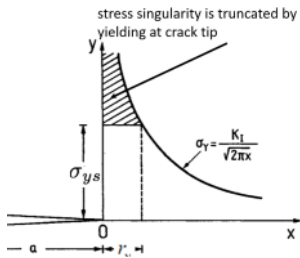


### Tresca criterion





Remember 1D analyses

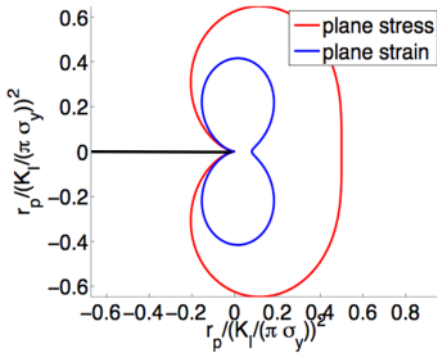


2D

von-Mises criterion

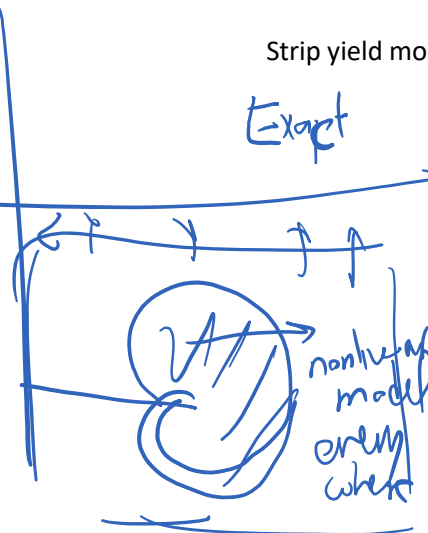
$$r_y(\theta) = \frac{1}{4\pi} \left( \frac{K_I}{\sigma_{ys}} \right)^2 \left[ 1 + \cos \theta + \frac{3}{2} \sin^2 \theta \right]$$

plane stress



Strip yield model

Exact



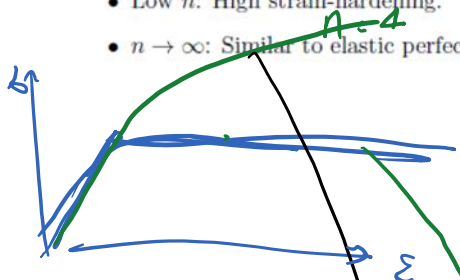
Dodds, 1991, FEM solutions

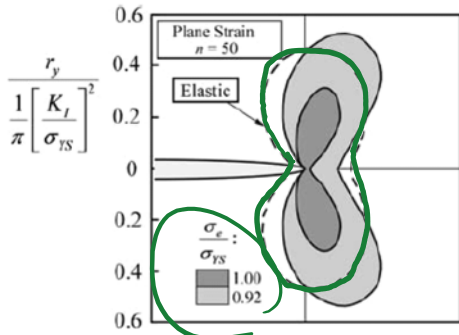
Ramberg-Osgood material model

$$\frac{\epsilon}{\epsilon_o} = \frac{\sigma}{\sigma_o} + \alpha \left( \frac{\sigma}{\sigma_o} \right)^n$$

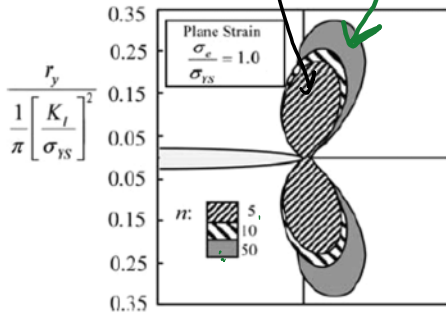
$n \rightarrow \infty$

- Low  $n$ : High strain-hardening.
- $n \rightarrow \infty$ : Similar to elastic perfectly plastic.





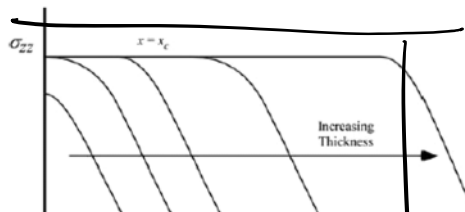
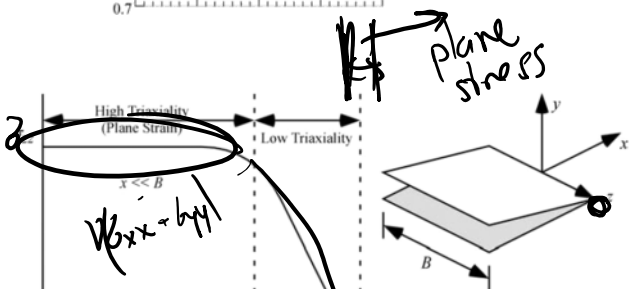
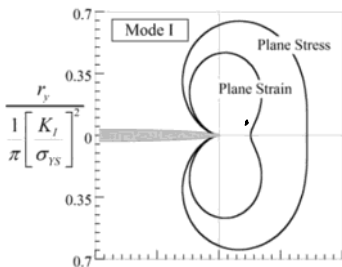
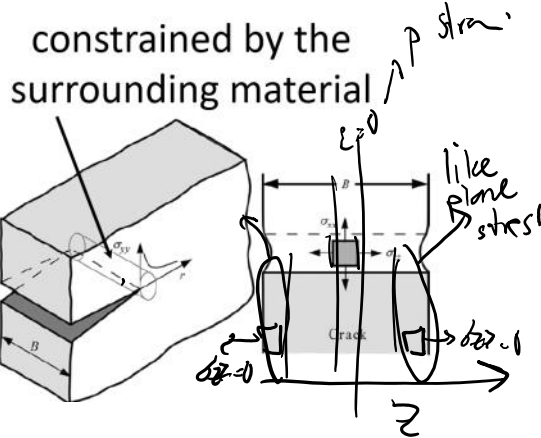
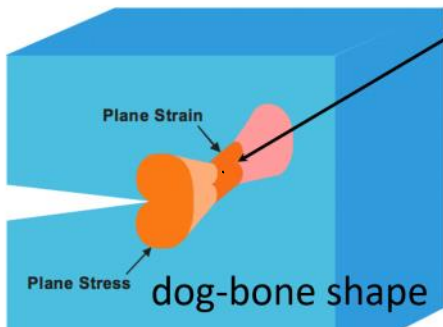
Effect of definition of yield (some level of ambiguity)

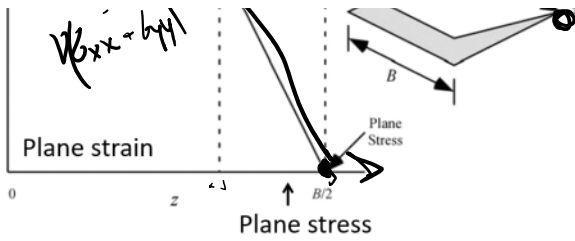


Effect of strain-hardening: Higher hardening (lower n) => smaller zone

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## Plane stress/plane strain

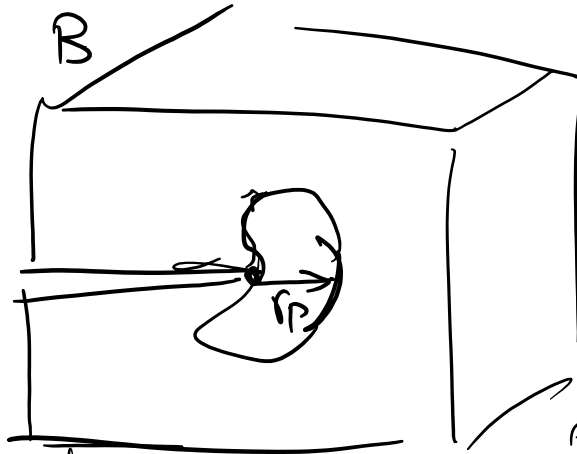




As the thickness increases more through the thickness behaves as plane strain

# Fracture Mechanics

$$r_p \propto \left( \frac{K_{Ic}}{\sigma_y} \right)^2$$

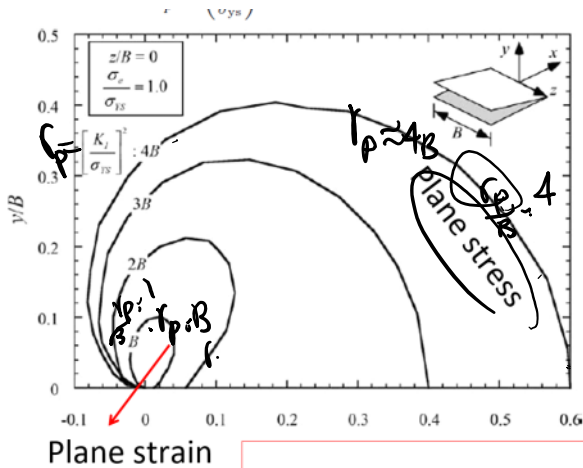


$\frac{r_p}{B}$	very small	P. strain
$\frac{r_p}{B}$	large	P. stress

$r_p$  fixed

B changed and look @

FPZ size a shape change



$$\frac{\left( \frac{K}{\sigma_{ys}} \right)^2}{B} \propto \frac{r_p}{B} : \begin{cases} \text{low (high } B) & \text{plane strain} \\ \text{high (low } B) & \text{plane stress} \end{cases}$$

Change of plastic loci to plane stress mode as "relative B decreases". Nakamura & Park, ASME 1988

For plane strain condition we must have  $B > \left( \frac{K}{\sigma_{ys}} \right)^2 \propto r_p$