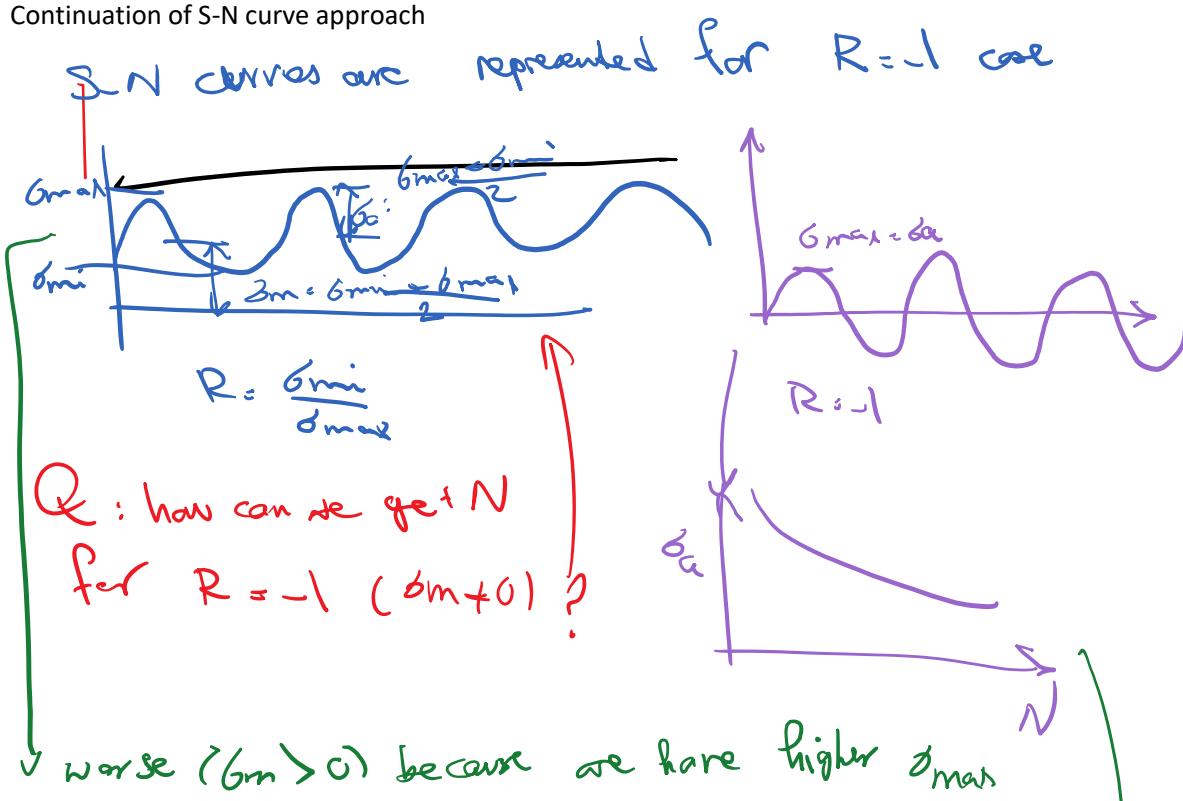
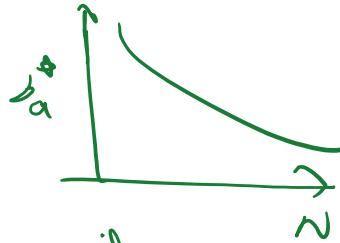


Continuation of S-N curve approach



Idea modify  $\sigma_a \rightarrow \sigma_a^*$  then go back to

$$\sigma_m \rightarrow \sigma_y$$



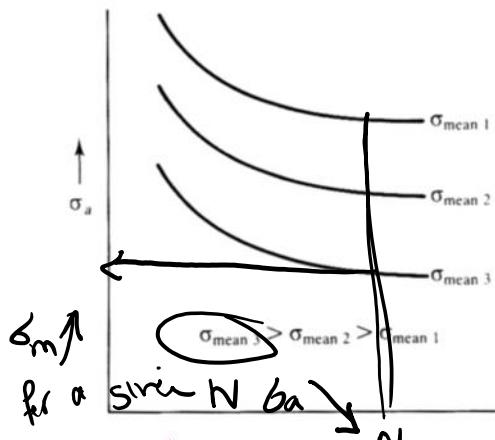
$N \rightarrow \infty$  because material

should fail by yielding

$$\sigma_a \rightarrow \sigma_a^* = \sigma_a \left[ 1 - \left( \frac{\sigma_m}{\sigma_u \text{ or } \sigma_y} \right)^n \right]$$

Ultimate stress

## Effect of mean stress



**Approach 2:**  
Correction-factor formulas

$$\sigma_a = \sigma_{f0} \left[ 1 - \left( \frac{\sigma_m}{\sigma_u} \right)^r \right]$$

where  $\sigma_a$  is the amplitude of allowable stress (alternating stress).

$\sigma_{f0}$  is the stress at fatigue fracture when the material under zero mean stress cycled loading

$\sigma_m$  is the mean stress of the actual loading.

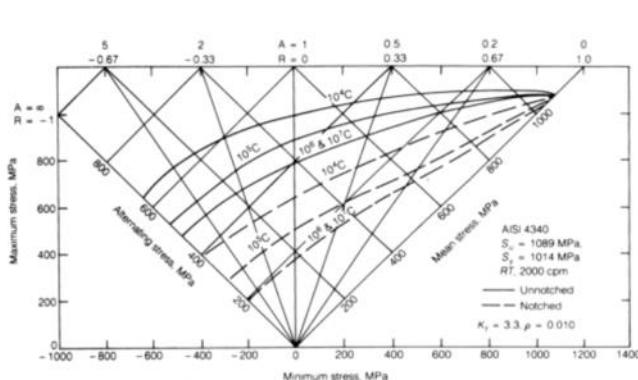
$\sigma_u$  is the tensile strength of the material.

$r = 1$  is called Goodman line which is close to the results of notched specimens.

$r = 2$  is the Gerber parabola which better represents ductile metals.

see

**Approach 1:**  
Master diagram



$$R = \frac{\sigma_{min}}{\sigma_{max}}$$

$$A = \frac{\sigma_a}{\sigma_m} = \frac{1-R}{1+R}$$

Other correction factor

Gerber (1874)

$$\frac{\sigma_a^*}{\sigma_a} = 1 - \left( \frac{\sigma_m}{\sigma_u} \right)^2$$

Goodman (1899)

$$\frac{\sigma_a^*}{\sigma_a} = 1 - \frac{\sigma_m}{\sigma_u}$$

Soderberg (1939)

$$\frac{\sigma_a^*}{\sigma_a} = 1 - \frac{\sigma_m}{\sigma_{y0}}$$

allowed  
stress variation  
goes to zero

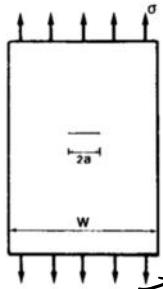
$\sigma_m \rightarrow \sigma_u$  or  $\sigma_{y0}$   
 $\sigma_a^* \rightarrow 0$

New approach

Paris law

$$K = \sigma \sqrt{\pi a}$$

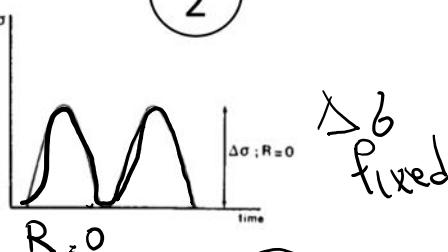
1



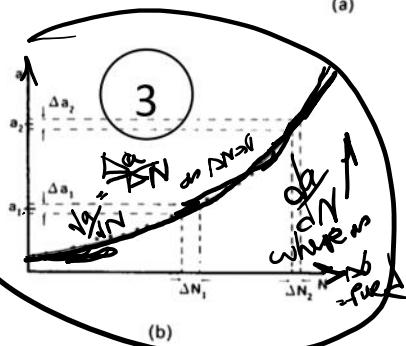
assume  
a  
very  
big domain  
 $K = \sigma \sqrt{\pi a}$

(a)

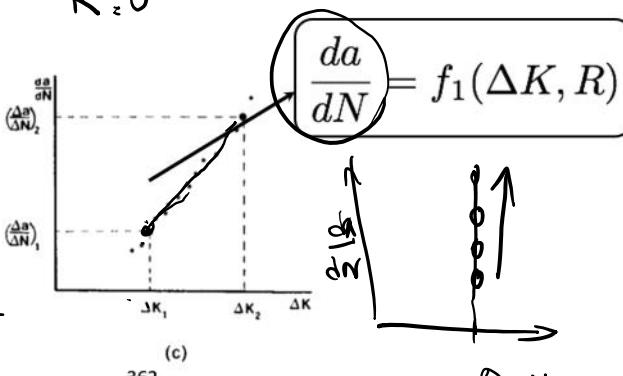
2



$\Delta \sigma$  fixed



(b)



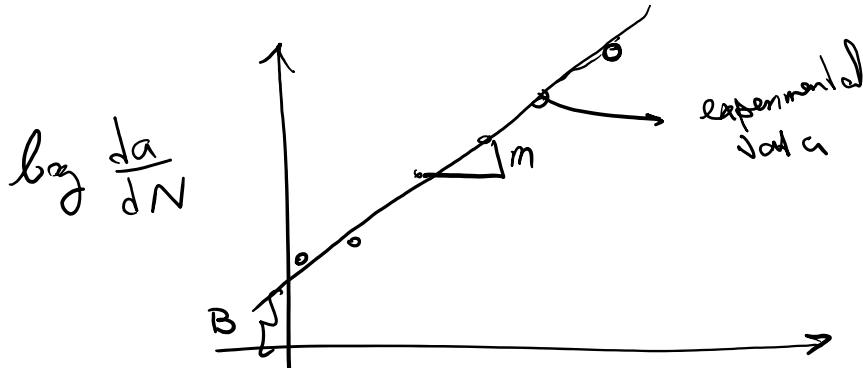
362

$\Delta \sigma'$

$$K = \sigma \sqrt{\pi a} \quad \underline{\Delta K = \Delta \sigma \sqrt{\pi a}} \quad \text{as } a \uparrow \\ \Delta K \text{ increases}$$

$\frac{da}{dN}$  is it only a function of  $\Delta \sigma$ ?

$$\frac{da}{dN} = f(\Delta \sigma) \quad \text{fixed}$$



$\log \frac{da}{dN}$

$$\log \frac{da}{dN} \approx B + m \log \Delta K$$

$$\frac{da}{dN} = e^{B + m \log \Delta K} = (e^B) (e^{m \log \Delta K})$$

$$\frac{da}{dN} = e^{B + m \log \Delta K} = \underbrace{\left(e^B\right)}_C \left(e^{\log(\Delta K)}\right)^m$$

"fixed"

$\Delta K^m$

$$\frac{da}{dN} = C \Delta K^m$$

Paris law

We must be careful about units

$$\frac{da}{dN} \cdot \frac{1}{L} = L$$

$\Delta K = [G] / \sqrt{E}$   
units: MPa-fm<sup>0.5</sup>

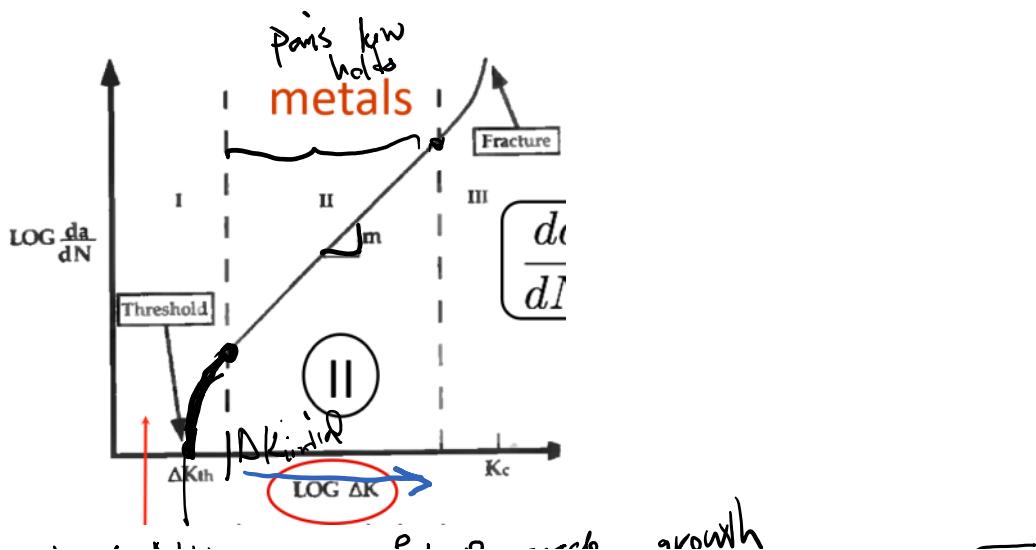
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↓

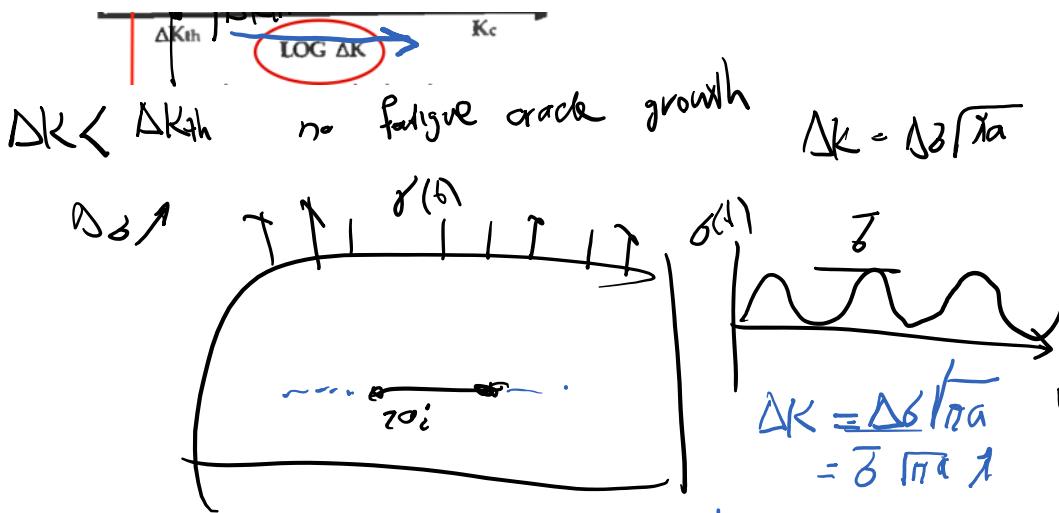
$C$  is not dimensionless

& its value changes from one system  
to another.

$m$  is fixed for a material

In fact, Paris law ONLY holds for stage II fatigue crack propagation as shown below





$$K_{max} = K_t$$

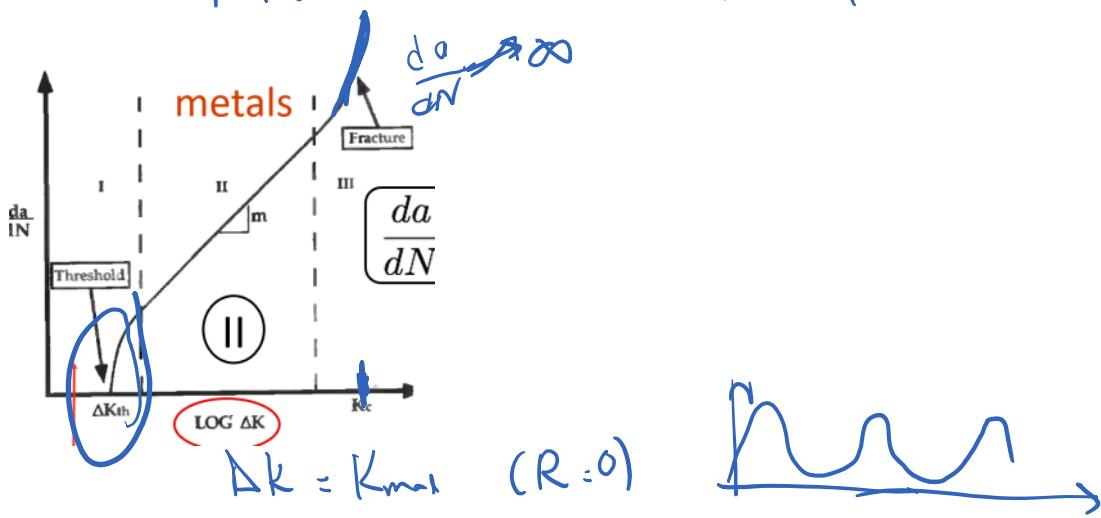
fixed  $a^*$

crack propagates right away

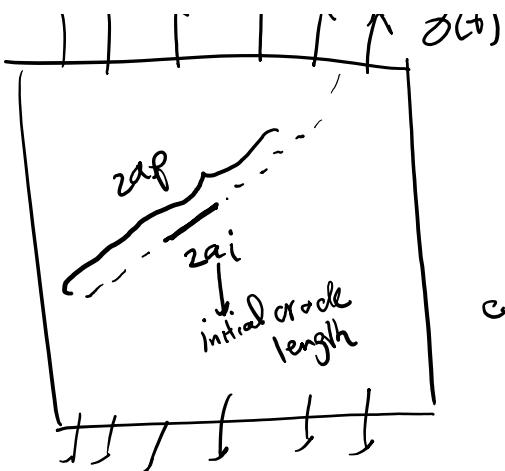
$$\sigma_{max} \sqrt{a \alpha_f} = K_{Ic} \rightarrow \alpha_f \text{ when } a \rightarrow a_f$$

$(K \rightarrow K_{Ic})$

crack propagates in an unstable fashion



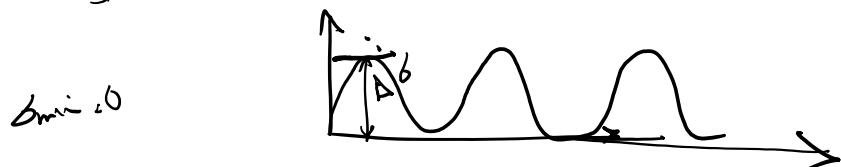
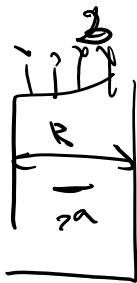
$$K = \sigma \sqrt{2a} \gamma(a)$$



$$K = \sigma \sqrt{\pi a} Y(a)$$

geometry needed  
sober

$$\text{c) } Y(a) = \sec\left(\frac{\pi a}{L}\right) = \frac{1}{\cos\left(\frac{\pi a}{L}\right)}$$



$$\Delta K = \Delta \sigma \sqrt{\pi a} Y(a)$$

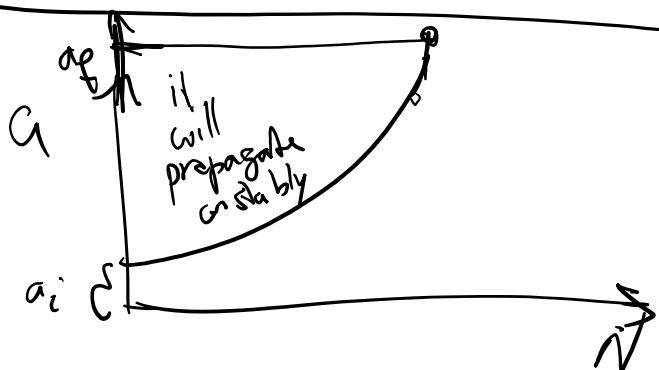
$$\frac{da}{dN} = C \Delta K^m$$

Paris law

This is an ODE E that should be integrated in time

$$\frac{da}{dN} = C \Delta \sigma^m a^{\frac{m}{2}} a^{\frac{m}{2}} Y(a) \quad \text{eq ①}$$

$$a(N=0) = a_i \quad a_i < a \leq a_f$$



$a_f$  is obtained from instantaneous crack propagation criterion

$$K = \sigma \sqrt{\pi a} Y(a)$$

$$K_{\text{max}} = G_m \sqrt{\pi a} Y(a) \quad \text{Graph: } \begin{array}{c} \delta_{\text{max}} \\ \downarrow \\ \text{Graph of } \delta(t) \end{array}$$

$$= K_{IC} \implies$$

Solve  $a_f$  from  $Y(a_f) \sqrt{\pi a_f} = \frac{K_{IC}}{G_{\text{max}}}$

Summary

$$\frac{da}{dN} = C(\Delta k)^m \quad (\star)$$

$$\Delta k = \sqrt{\pi a} Y(a)$$

$$IC, a = a_i \quad \textcircled{a} \quad N = 0$$

Final condition is  $a = a_f$  :  $\delta_{\text{max}} \sqrt{\pi a_f} Y(a_f) : KE$

Integrate  $(\star)$  from  $a = a_i$  to  $a = a_f$

to get  $N_f$  (the life of this structure)

What about the case where  $Y(a) \approx 1$  is acceptable?



$$\frac{dN}{da} = C(\Delta k)^m = C \left( \pi^{\frac{k}{2}} \Delta \delta^{\frac{k}{2}} a^{\frac{k}{2}} \right)^m$$

$$\Delta k = \sqrt{\pi a} \Delta \delta P^{\frac{1}{2}}$$

$$\rightarrow \frac{dN}{da} = C \pi \Delta b^{\frac{m}{2}} a^{\frac{m}{2}}$$

$A$

$\frac{da}{dN} = A a^{\frac{m}{2}}, A = C \pi \Delta b^{\frac{m}{2}}$

(3)

$$\frac{da}{a^{\frac{m}{2}}} = A dN : a^{-\frac{m}{2}} da = dN \quad m > 2$$

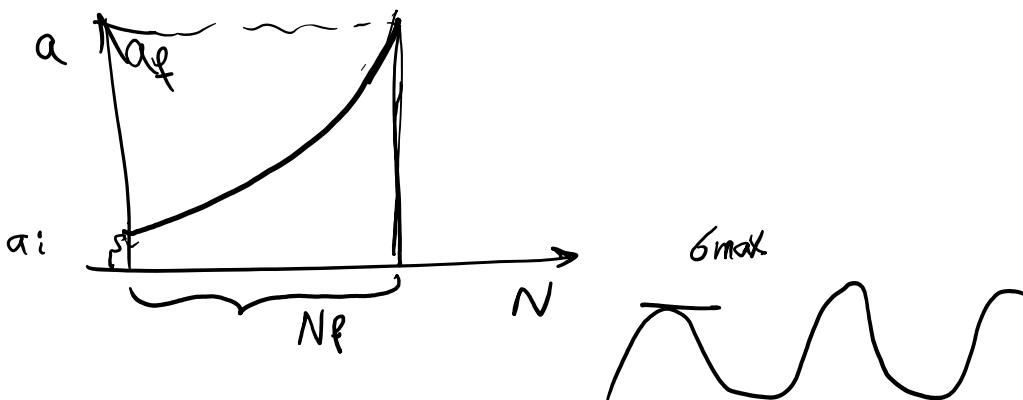
$$\rightarrow \text{integrate} \quad \frac{1}{1-\frac{m}{2}} a^{1-\frac{m}{2}} \Big|_{a_i}^a = N \Big|_{N_0}^N$$

OK  
typical  
of all materials

$$\frac{1}{\frac{m}{2}-1} - \frac{1}{a^{\frac{m}{2}-1}}$$

$$-\frac{1}{\frac{m}{2}-1} \frac{1}{a^{\frac{m}{2}-1}} + \frac{1}{\frac{m}{2}-1} \frac{1}{a_i^{\frac{m}{2}-1}} = N(a)$$

$$N(a) = \frac{1}{C \pi \Delta b^{\frac{m}{2}}} \left( \frac{1}{a_i^{\frac{m}{2}-1}} - \frac{1}{a^{\frac{m}{2}-1}} \right) \quad (4)$$



$$a_f = ?$$

$$K_{max} = \sigma_{max} \sqrt{2 a_p} = K_{SC}$$

$$\rightarrow \boxed{a_f < \frac{1}{k} \left( \frac{k_{cc}}{\delta_{mat}} \right)^{\frac{1}{m}}} \quad (5)$$

Ans →

$$N_f = \frac{1}{\left( \frac{m-1}{2} \right) C Y^{\frac{m}{2}} \Delta \sigma^{\frac{m}{2}}} \left( \frac{1}{a_i^{\frac{m-1}{2}}} - \frac{1}{a_f^{\frac{m-1}{2}}} \right) \quad (5)$$

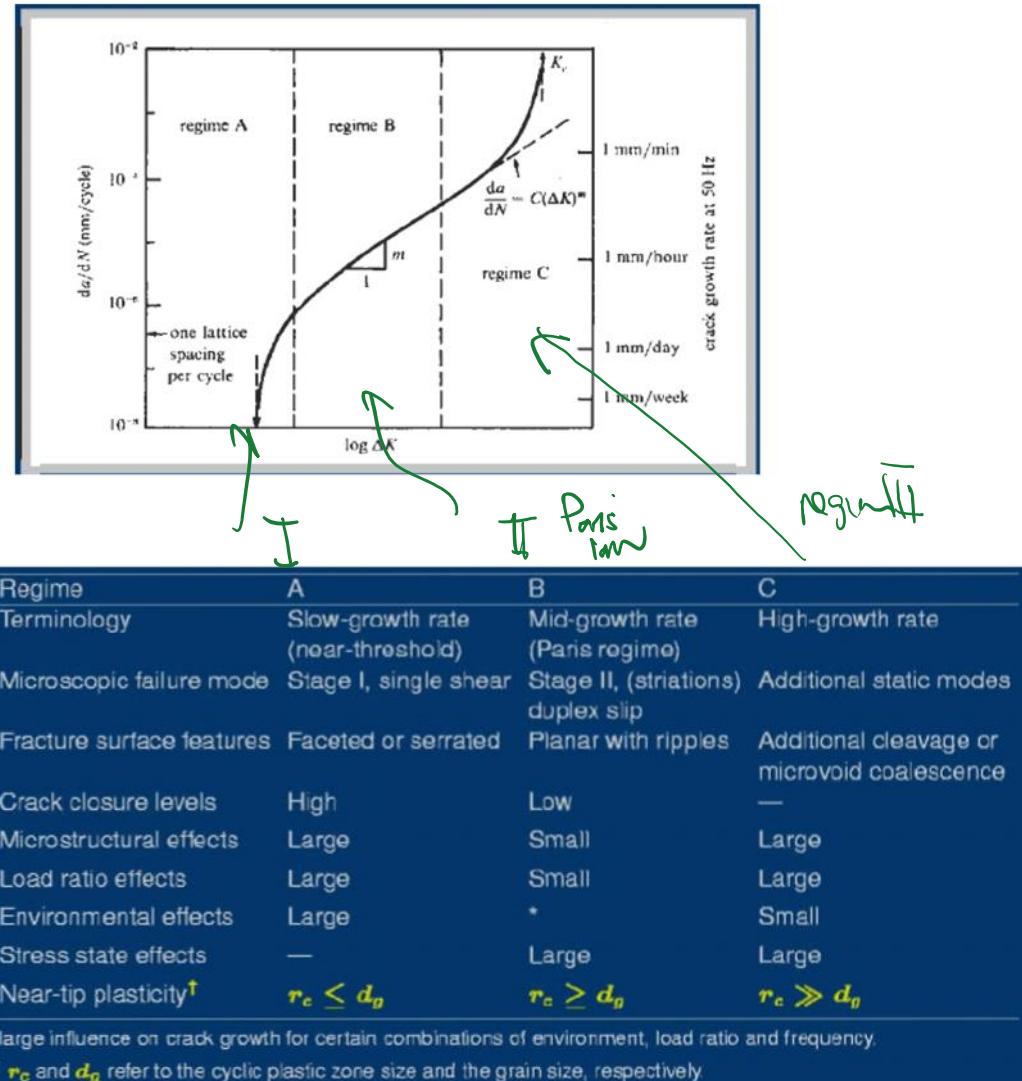
For  $m > 2$ :

$$N_f = \frac{2}{(m-2) C Y^m (\Delta \sigma)^m \pi^{m/2}} \left[ \frac{1}{(a_0)^{(m-2)/2}} - \frac{1}{(a_f)^{(m-2)/2}} \right]$$

For  $m = 2$ :

$$N_f = \frac{1}{C Y^2 (\Delta \sigma)^2 \pi} \ln \frac{a_f}{a_0}$$

(source Course presentation S. Suresh MIT)



(source Course presentation S. Suresh MIT)

not depends on load ratio R

$$\frac{da}{dN} = C(\Delta K)^m, \quad \Delta K = K_{\max} - K_{\min}$$

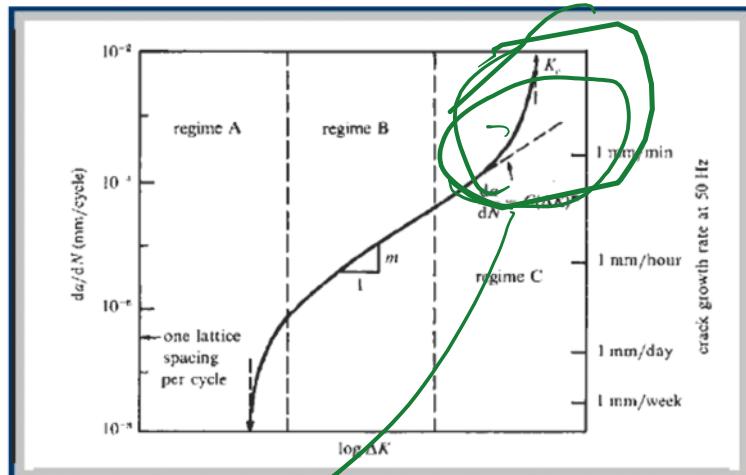
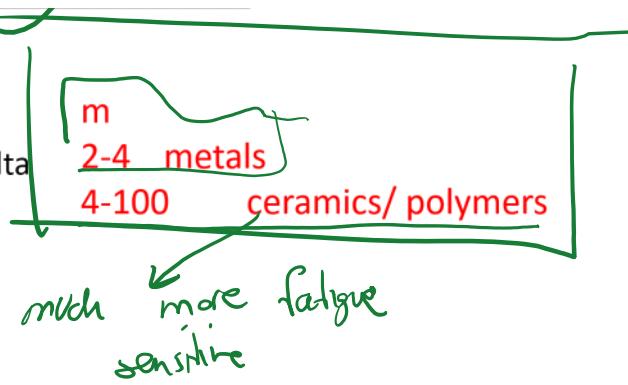
Table 1: Numerical parameters in the Paris equation.

alloy	$m$	$A$
Steel	3	$10^{-11}$
Aluminum	3	$10^{-12}$
Nickel	3.3	$4 \times 10^{-12}$
Titanium	5	$10^{-11}$

$C, m$

$C, m$

are material properties that must be determined experimentally from a log(delta K)-log(da/dN) plot.



Improvements to Paris law to also model  
region II & accurate representation for  
transition from stage II to III

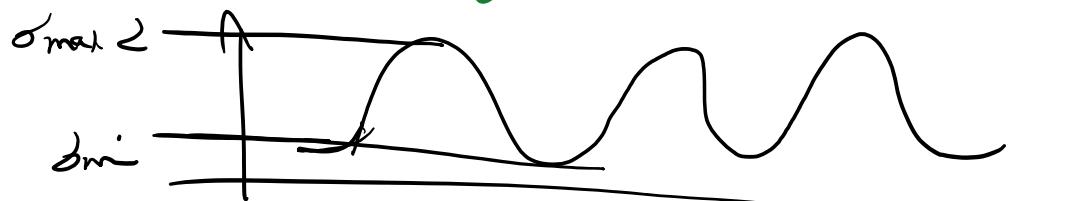
### Forman's model (stage II-III)

$$\frac{da}{dN} = \frac{C(\Delta K)^n}{(1-R)K_c - \Delta K}$$

### Paris' model

$$\frac{da}{dN} = C(\Delta K)^m$$

corr ch.



$$R = \frac{\sigma_{min}}{\sigma_{max}}$$

$$K_{min} = \sigma_{min} \sqrt{\frac{\pi a}{E}}$$

$$K_{max} = \sigma_{max} \sqrt{\frac{\pi a}{E}}$$

$$\rightarrow R = \frac{K_{min}}{K_{max}}$$

$$\text{denominator} = (1 - R) K_{IC} - \Delta K =$$

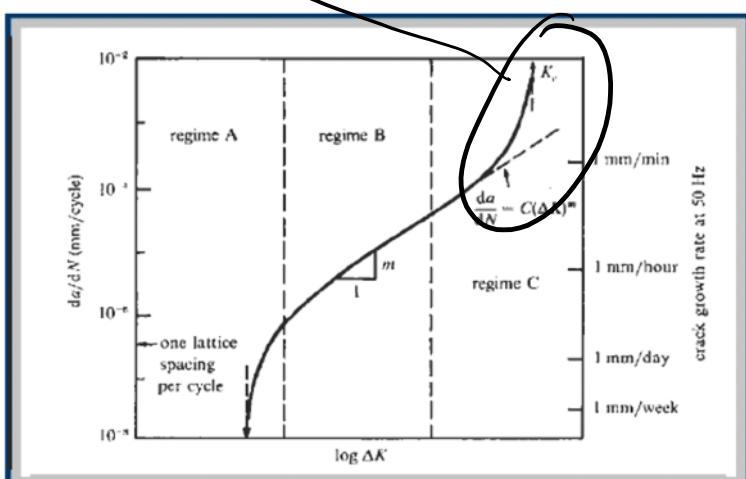
$$(1 - \frac{K_{min}}{K_{max}}) K_{IC} - (K_{max} - K_{min})$$

$$= \frac{(K_{max} - K_{min})}{\Delta K} \left( \frac{K_{IC}}{K_{max}} - 1 \right)$$

$$\textcircled{6} \quad \boxed{\frac{da}{dN} = \frac{C \Delta k^m}{\Delta K \left( \frac{K_{IC}}{K_{max}} - 1 \right)}} \quad \text{Forman's model}$$

$$K_{max} \nearrow K_{max} = Y_{max} \sqrt{P_{fa}} \text{ because } \alpha \uparrow$$

$$K_{max} \nearrow K_{IC} \Rightarrow \frac{da}{dN} \rightarrow \infty$$



This smoothens the transition of  $\frac{da}{dN} \sim C \Delta k^m$  from zone II (Paris law)

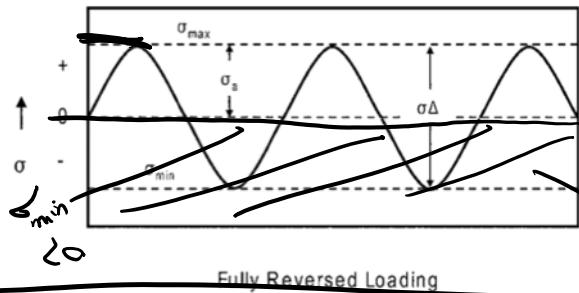
to  $\approx \text{II}$  (Fast crack growth)

Tension-compression loading

$$R = \frac{\sigma_{\min}}{\sigma_{\max}} < 0$$

$$K_{\max} = \sigma_{\max} \sqrt{\pi a}$$

$$K_{\min} = \begin{cases} 0 & ? \\ \sigma_{\min} \sqrt{\pi a} & ? \end{cases}$$



example

$$R \approx -1$$

not doing anything for fatigue

$$\Delta K = (\sigma_{\max} - \max(\sigma_{\min}/\sigma)) \sqrt{\pi a}$$

correction