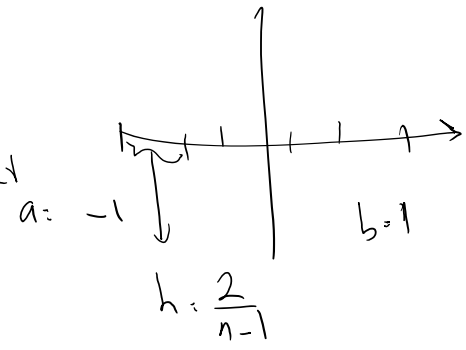


$f(\xi) = \alpha_0 + \alpha_1 \xi + \dots + \alpha_n \xi^n$ \rightarrow order of polynomial

NC $\int_{-1}^1 f(\xi) d\xi = (1 - -1) \left(\sum_{i=1}^n w_i f(\xi_i) \right)$
 equi-distinct



GA $\int_{-1}^1 f(\xi) d\xi = \sum_{i=1}^n w_i f(\xi_i)$

$n \rightarrow 0$
 NC $n = \begin{cases} n-1 & \text{even } n \\ n & \text{odd } n \end{cases}$

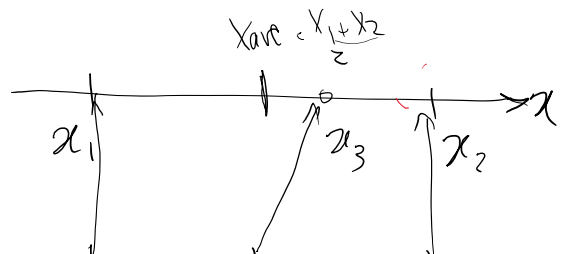
$n \rightarrow 0$
 GC $n = 2n-1 \rightarrow$
 each GP is worth twice NC point

$0 \rightarrow n$
 NC $n = \begin{cases} 0 & 0 \text{ odd} \\ 0+1 & 0 \text{ even} \end{cases} \rightarrow$ often the case

GA $n = \text{ceil} \left(\frac{0+1}{2} \right)$

Application of this:

$\int_{-1}^1 e^{x^3} BDB dx$ $T_{n+1}(f)$

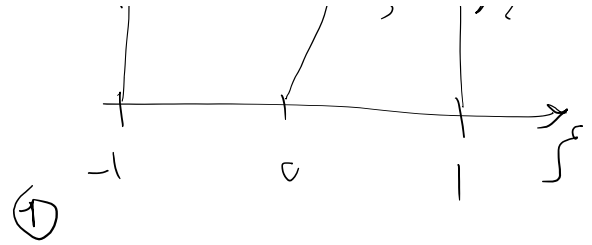


$$k^e = \int_{x_1}^{x_3} B^T D B dx$$

$$= \frac{EA(f)}{J(\xi)}$$

$$\xi = 2\xi(x_{\text{ave}} - x_3) + \frac{L_e}{2}$$

$$I_{12}(\xi) = \begin{bmatrix} \xi - \frac{1}{2} \\ \xi + \frac{1}{2} \\ -2\xi \end{bmatrix} \begin{bmatrix} \xi - \frac{1}{2} & \xi + \frac{1}{2} & -2\xi \end{bmatrix} d\xi$$



- E & A are constant
 homog. prismatic cross sect. AE

- unskewed element $x_3 = x_{\text{ave}} \Rightarrow J_e = \frac{L_e}{2}$

Integrand I is a polynomial

$$I_{12}(\xi) = \frac{EA}{L_e/2} (\xi - \frac{1}{2})(\xi + \frac{1}{2}) \quad \text{2nd order polynomial}$$

Maximum order over all i, j is 2

Order = 2

NC $n_{NC} = 0 + 1 = 3$
 Simpson's rule

GC $n_{GC} = \text{ceil}(\frac{0+1}{2}) = \text{ceil}(\frac{3}{2}) = 2$

Integrating k^e with NC

TABLE 5.5 Newton-Cotes numbers and error estimates

n pts	Number of intervals n-1	Cotes numbers							Upper bound on error R_n as a function of the derivative of F
		C_0^1	C_1^1	C_2^1	C_3^1	C_4^1	C_5^1	C_6^1	
2	1	$\frac{1}{2}$	$\frac{1}{2}$						$10^{-1}(b-a)^3 F'''(\tau)$
3	2	$\frac{1}{6}$	$\frac{4}{6}$	$\frac{1}{6}$					$10^{-3}(b-a)^5 F^{IV}(\tau)$
4	3	$\frac{1}{8}$	$\frac{3}{8}$	$\frac{3}{8}$	$\frac{1}{8}$				$10^{-3}(b-a)^5 F^{IV}(\tau)$
5	4	$\frac{7}{90}$	$\frac{32}{90}$	$\frac{12}{90}$	$\frac{32}{90}$	$\frac{7}{90}$			$10^{-6}(b-a)^7 F^{VI}(\tau)$
6	5	$\frac{19}{288}$	$\frac{75}{288}$	$\frac{50}{288}$	$\frac{50}{288}$	$\frac{75}{288}$	$\frac{19}{288}$		$10^{-6}(b-a)^7 F^{VI}(\tau)$
7	6	$\frac{41}{840}$	$\frac{216}{840}$	$\frac{27}{840}$	$\frac{272}{840}$	$\frac{27}{840}$	$\frac{216}{840}$	$\frac{41}{840}$	$10^{-9}(b-a)^9 F^{VIII}(\tau)$

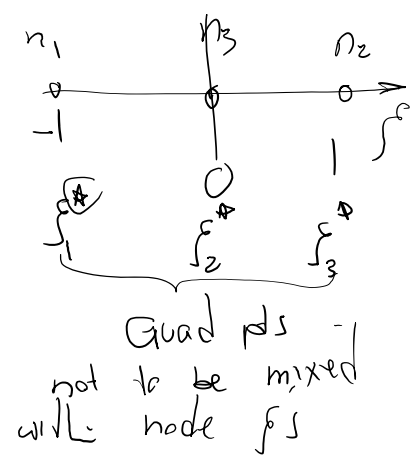
Simpson's rule with 3 pts

← n pts

no. 1 what

7

$$k^e = \int_{a_i^{-1}}^{b_i^1} I_{3 \times 3}(f) df = \underbrace{\begin{pmatrix} b-a \\ 4 & -1 \end{pmatrix}}_{\text{need this in } \Delta AC} \left(\frac{1}{6} I_{3 \times 3}(f_1^*) + \frac{4}{6} I_{3 \times 3}(f_2^*) + \frac{1}{6} I_{3 \times 3}(f_3^*) \right) =$$



$$k^e = \frac{1}{3} \left(I_{3 \times 3}(-1) + 4 I_{3 \times 3}(0) + I_{3 \times 3}(1) \right)$$

$$I_{3 \times 3}(f) = \frac{EA(f)}{J(f)} = 2f(x_{arc} - x_s) + \frac{b_0}{2}$$

$$\begin{bmatrix} f - \frac{1}{2} \\ f + \frac{1}{2} \\ -2f \end{bmatrix} \begin{bmatrix} f - \frac{1}{2} & f + \frac{1}{2} & -2f \end{bmatrix}$$

$$[2 - AC]$$

Same integrals with GQ:

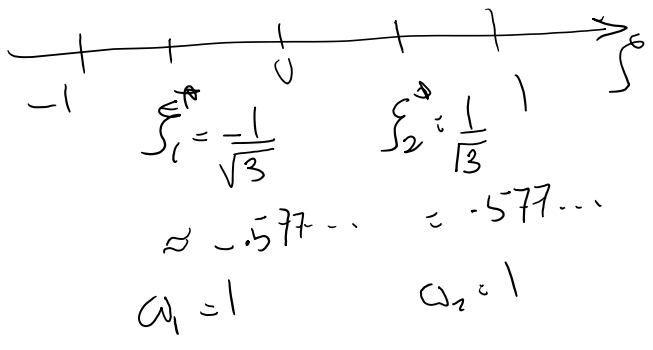


TABLE 5.6 Sampling points and weights in Gauss-Legendre numerical integration (interval -1 to +1)

n	r_i			α_i		
1	0.	(15 zeros)		2.	(15 zeros)	
2	± 0.57735	02691	89626	1.00000	00000	00000
3	± 0.77459	66692	41483	0.55555	55555	55556
	0.00000	00000	00000	0.88888	88888	88889
4	± 0.86113	63115	94053	0.34785	48451	37454
	± 0.33998	10435	84856	0.65214	51548	62546
5	± 0.90617	98459	38664	0.23692	68850	56189
	± 0.53846	93101	05683	0.47862	86704	99366
	0.00000	00000	00000	0.56888	88888	88889
6	± 0.93246	95142	03152	0.17132	44923	79170
	± 0.66120	93864	66265	0.36076	15730	48139
	± 0.23861	91860	83197	0.46791	39345	72691

already incorporated to ω_i in GQ

$$k^e = \int_{-1}^1 I_{3 \times 3}(f) df = \cancel{\begin{pmatrix} 1 & -1 \end{pmatrix}} \left(\omega_1 I_{3 \times 3}(f_1^*) + \omega_2 I_{3 \times 3}(f_2^*) \right)$$

$$\Rightarrow K^e = I_{3 \times 3} \left(\frac{-1}{\sqrt{3}} \right) + I_{3 \times 3} \left(\frac{1}{\sqrt{3}} \right)$$

$$I_{3 \times 3} = \frac{EA(\xi)}{J(\xi)}$$

$$\simeq 2\xi(X_{\text{ave}} - X_0) + \frac{l_0}{2}$$

$$\begin{bmatrix} \xi - \frac{1}{2} \\ \xi + \frac{1}{2} \\ -2\xi \end{bmatrix} \begin{bmatrix} \xi - \frac{1}{2} & \xi + \frac{1}{2} & -2\xi \end{bmatrix}$$

$$|2 - G \bar{Q}|$$

$$n_G = 2$$

better
fewer
quadrature
pts
(50%
here)

$$K^e = \frac{1}{3} \left(I_{3 \times 3}(-1) + 4 I_{3 \times 3}(0) + I_{3 \times 3}(1) \right)$$

$$I_{3 \times 3} = \frac{EA(\xi)}{J(\xi)}$$

$$\simeq 2\xi(X_{\text{ave}} - X_0) + \frac{l_0}{2}$$

$$\begin{bmatrix} \xi - \frac{1}{2} \\ \xi + \frac{1}{2} \\ -2\xi \end{bmatrix} \begin{bmatrix} \xi - \frac{1}{2} & \xi + \frac{1}{2} & -2\xi \end{bmatrix}$$

$$n_K = 3$$

$$|2 - K|$$

Q1 : EA, J constant, would eqns 2 give us the exact

$$K^e = \frac{AE}{L} \begin{bmatrix} \frac{7}{3} & -\frac{8}{3} & \frac{1}{3} \\ \text{sym} & \frac{7}{3} & -\frac{8}{3} \\ & & \frac{16}{3} \end{bmatrix} \begin{matrix} 2 \\ a \end{matrix}$$

Yes: Because the integrand is a polynomial of

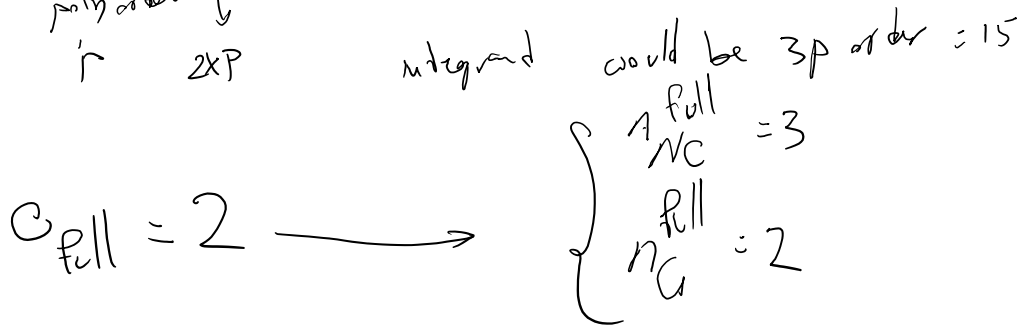
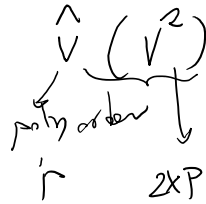
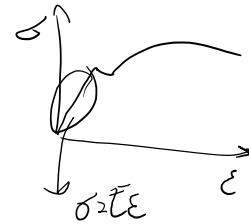
order = 2 & two schemes ($n_K = 3$ or $n_G = 2$)

both integrate polynomial order = 2 exactly

Full Integration Order:

Number of quad points that integrate the element EXACTLY when:

- Element is homogeneous E
- Constant section (1D and 2D) A
- Not-skewed geometry ($J = \text{constant}$) J
- For nonlinear problem like solid mechanics we assume the material is linear in determining number of quad points.



The integration will be exact if the actual integrand is a polynomial of n_{full} (in the problem shown A, E, J are all constant)

What if the element was:

- Skewed J not constant
- Inhomogeneous E not constant
- Non-prismatic A not constant

$$K^e = \int_{-1}^1 T(\xi) d\xi$$

$$I(\xi) = \frac{EA(\xi)}{J(\xi)}$$

$$\approx 2\xi(x_{arc} - x_s) + \frac{le}{2}$$

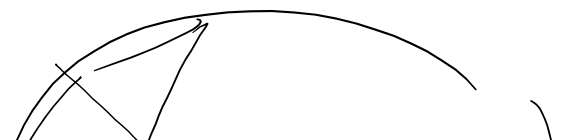
$$\begin{bmatrix} \xi - \frac{1}{2} \\ \xi + \frac{1}{2} \\ -2\xi \end{bmatrix} \begin{bmatrix} \xi - \frac{1}{2} & \xi + \frac{1}{2} & -2\xi \end{bmatrix}$$

We almost always still use the full integration scheme!

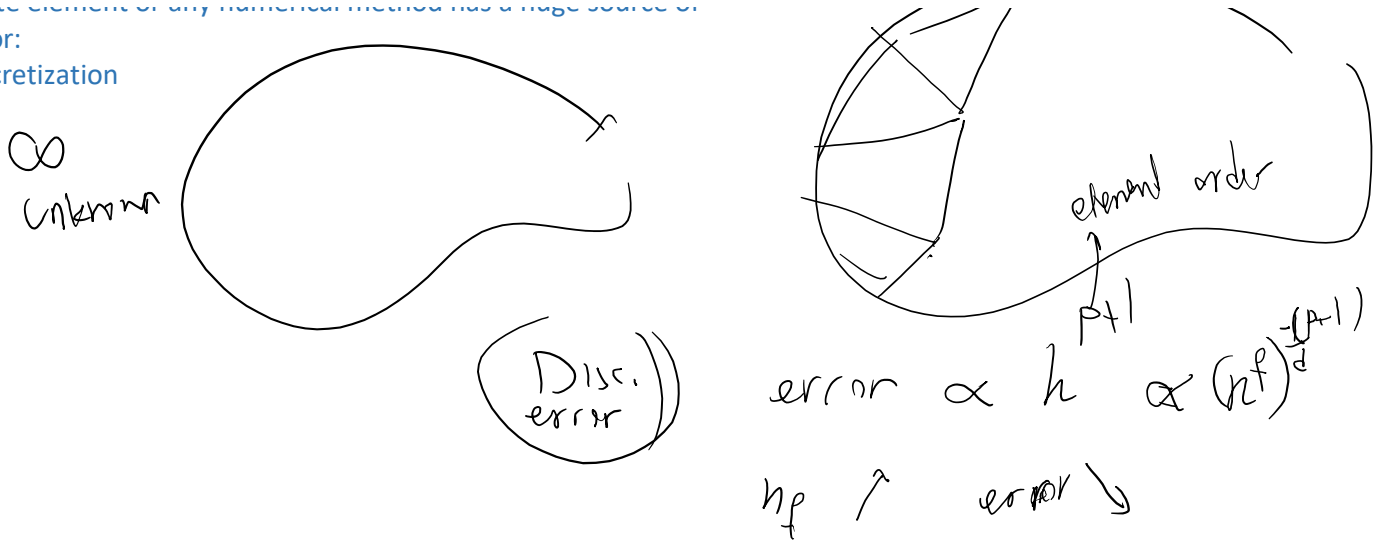
Reason:
Finite element or any numerical method has a huge source of error:
Discretization



n_G dof unknowns



Finite element of any numerical method has a huge source of error:
Discretization



Any error we introduce whose error is similar or goes to zero faster than disc error is OK.

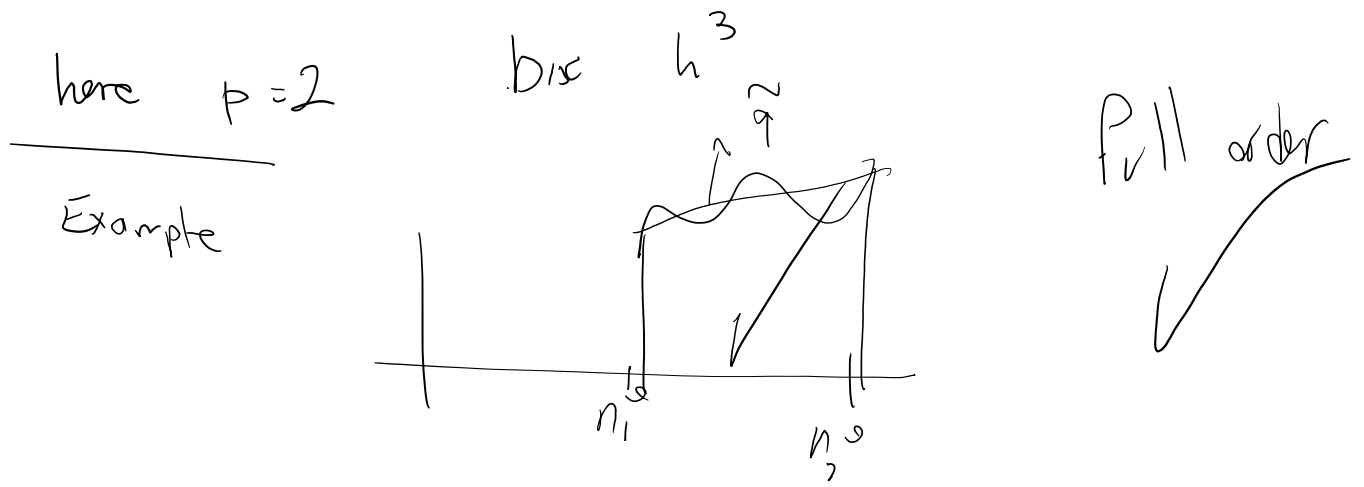


TABLE 5.5 Newton-Cotes numbers and error estimates

Number of intervals n	C_0	C_1	C_2	C_3	C_4	C_5	C_6	Upper bound on error R_n , as a function of the derivative of F
1	$\frac{1}{2}$	$\frac{1}{2}$						$10^{-1}(b-a)^3 F''(r)$
2	$\frac{1}{6}$	$\frac{4}{6}$	$\frac{1}{6}$					$10^{-3}(b-a)^5 F^{IV}(r)$
3	$\frac{1}{8}$	$\frac{3}{8}$	$\frac{3}{8}$	$\frac{1}{8}$				$10^{-3}(b-a)^5 F^{IV}(r)$
4	$\frac{7}{90}$	$\frac{32}{90}$	$\frac{12}{90}$	$\frac{32}{90}$	$\frac{7}{90}$			$10^{-6}(b-a)^7 F^{VI}(r)$
5	$\frac{19}{288}$	$\frac{75}{288}$	$\frac{50}{288}$	$\frac{50}{288}$	$\frac{75}{288}$	$\frac{19}{288}$		$10^{-6}(b-a)^7 F^{VI}(r)$
6	$\frac{41}{840}$	$\frac{216}{840}$	$\frac{27}{840}$	$\frac{272}{840}$	$\frac{27}{840}$	$\frac{216}{840}$	$\frac{41}{840}$	$10^{-9}(b-a)^9 F^{VIII}(r)$

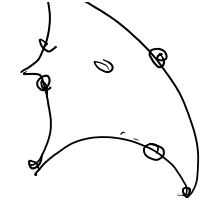
Disc h^3

highly nonlinear



We recommend that full numerical integration⁹ always be used for a displacement-based or mixed finite element formulation, where we define "full" numerical integration as

We recommend that *full numerical integration*⁹ always be used for a displacement-based or mixed finite element formulation, where we define "full" numerical integration as the order that gives the exact matrices (i.e., the analytically integrated values) when the elements are geometrically undistorted. Table 5.9 lists this order for elements used in two-dimensional analyses.

nonlinear
& skewed 
use more than full order

with

J non constant

is there any any
NC or G scheme
that integrates this exactly

$$I(\xi) = \frac{EA(\xi)}{2 \times 3} J(\xi)$$

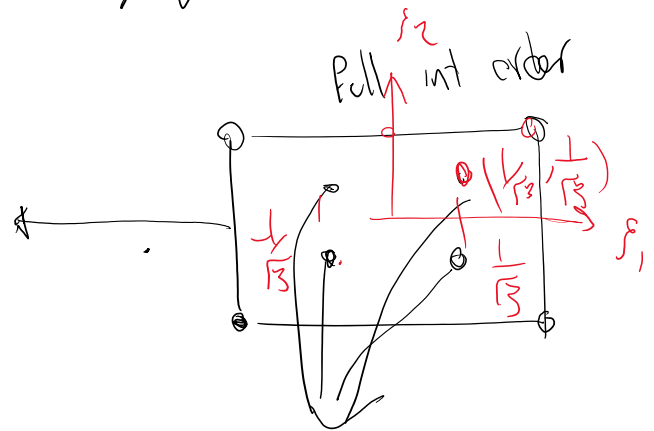
$$\approx 2\xi(X_{arc} - X_0) + \frac{le}{2}$$

$$\begin{bmatrix} \xi - \frac{1}{2} \\ \xi + \frac{1}{2} \\ -2\xi \end{bmatrix} \begin{bmatrix} \xi - \frac{1}{2} & \xi + \frac{1}{2} & -2\xi \end{bmatrix}$$

NO

Q: Can we even use a fewer number of quad points?

2D $P=1$ element



Gauss pts

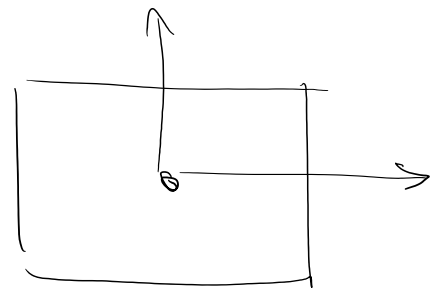
Reduced integral

is an explicit scheme

1 pt

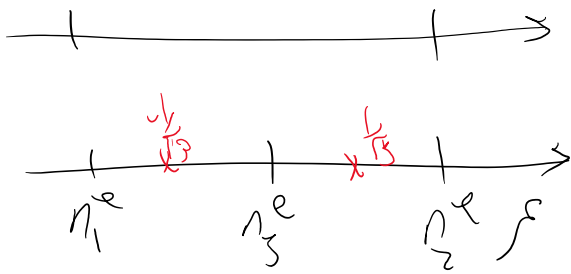
1/4

Comp time



$$k^e = \int_{\Omega} B^T D B \, d\Omega$$

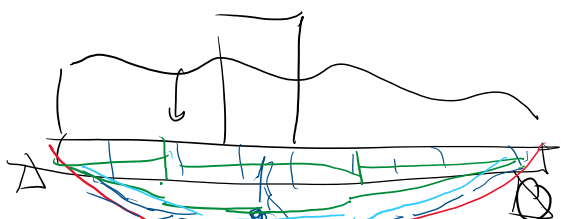
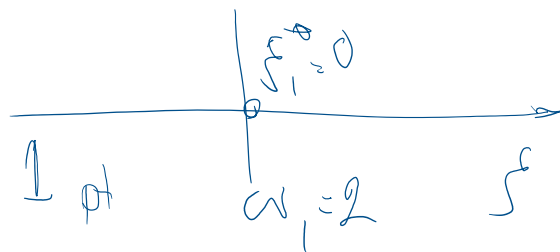
Cost
Reason



2nd motivation

FEM is stiff

can we integrate it with 1 pt

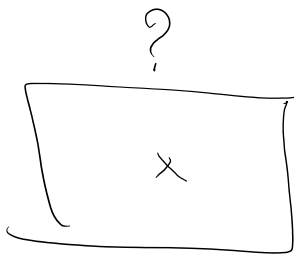
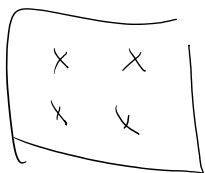


real deflection

more stiff response

with reduce order the response becomes more compliant

more compliant
Reason 2

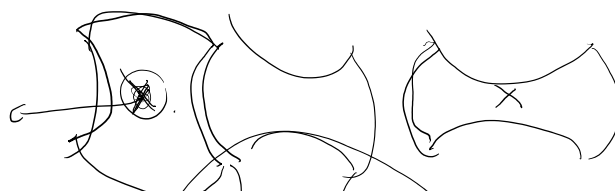


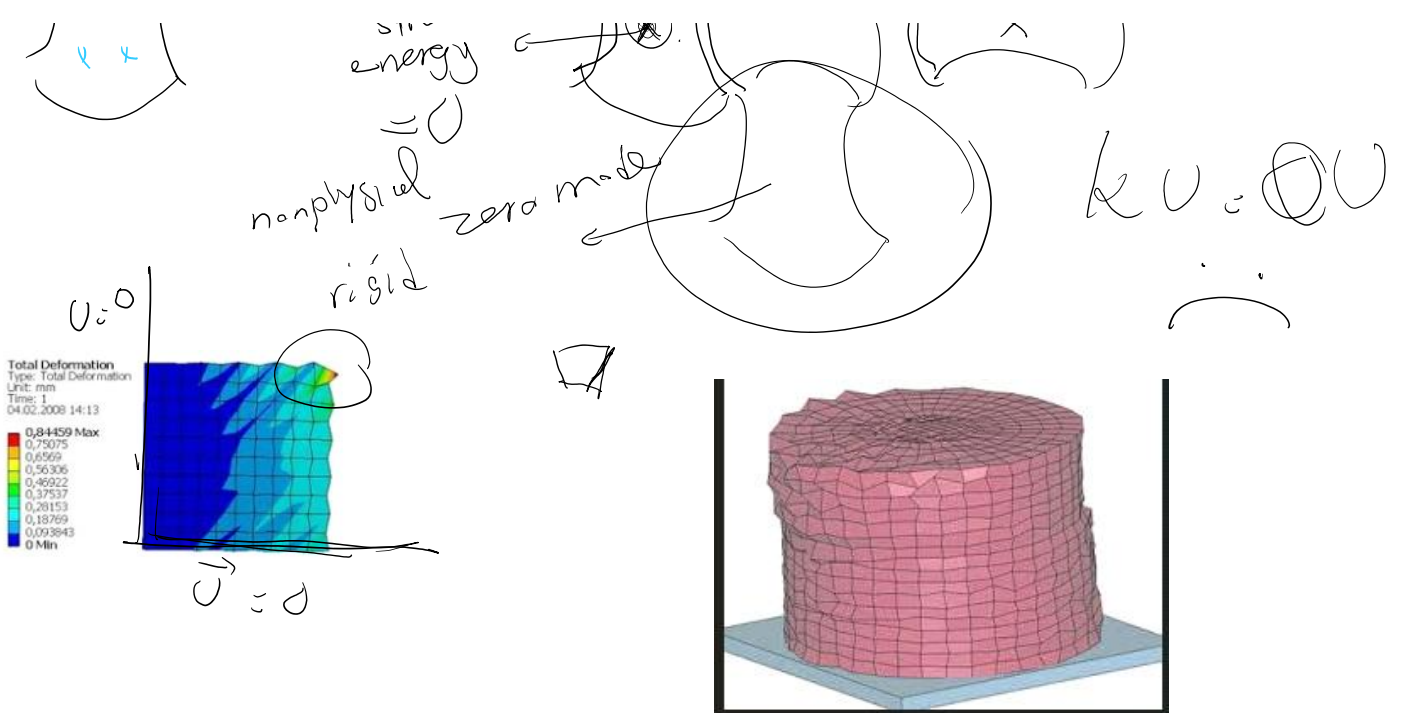
internal energy $\rightarrow 0$



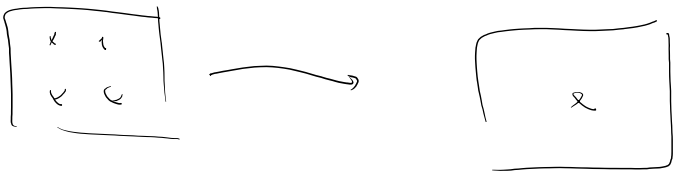
we create some nonphysical zero energy modes

strain energy





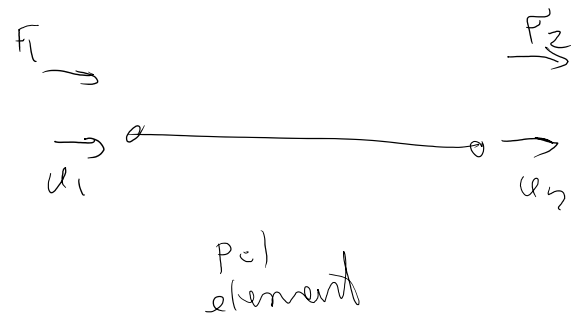
In general, we don't want to do reduced order integration (anything lower than full integration order)
 However, there are advanced techniques that afford going to reduced order integration.



if we do reduced order integration, how can we know it's not too much & we don't introduce nonphysical zero modes

$$\begin{bmatrix} F_1 \\ F_2 \end{bmatrix} = \underbrace{\frac{k}{AE}}_{k^e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

eigenvalues of k^e



eigenvalues λ


$$k U = \lambda U \iff \det(k - \lambda I) = 0$$

\downarrow eigen vector \downarrow eigenvalue

$$\det \begin{bmatrix} k - \lambda & -k \\ -k & k - \lambda \end{bmatrix} = 0 \quad (k - \lambda)^2 - k^2 = 0 \quad (k = \frac{AE}{L})$$

$$\implies k - \lambda = \pm k \quad \rightarrow \begin{cases} \lambda = 0 \\ \lambda = 2k \end{cases}$$

$\lambda = 2k$

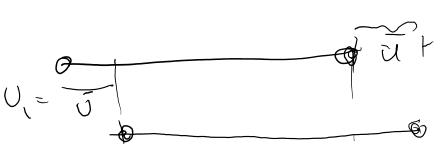
$$\begin{bmatrix} k & -k \\ -k & k \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = 2k \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} \implies u_2 = -u_1$$


$\lambda = 0$

$$\begin{bmatrix} k & -k \\ -k & k \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = 0 \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$\begin{bmatrix} k(u_1 - u_2) \\ k(-u_1 + u_2) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \rightarrow u_1 = u_2$$

$\Delta u = u_2 - u_1 = \bar{u} - \bar{u} = 0$



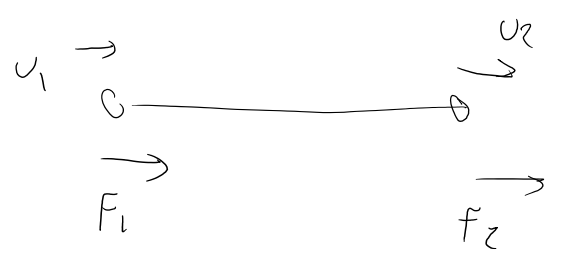
Rigid body motion

$$E = \frac{D U}{L_e} = 0$$

Zero energy mode

① Zero energy mode
Rigid motion $u(x) = \bar{u}$

$$\begin{pmatrix} k \\ k \\ 2k \end{pmatrix}$$



$$\text{rank}(K) = \underbrace{\dim(K)}_{\text{size of matrix}} - \underbrace{\dim(\text{null}(K))}_{\# \text{ of zero eigenvalues}} = 2$$

>> K = [1 -1; -1 1]

K =

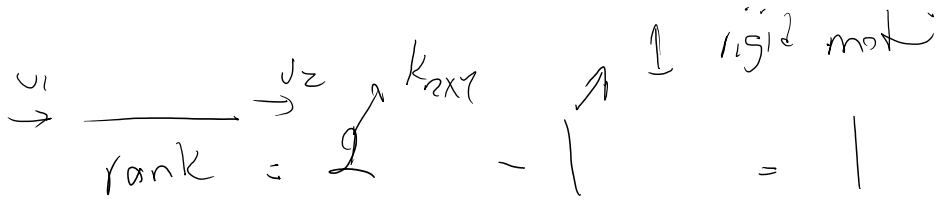
$$\begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

>> rank(K)

ans =

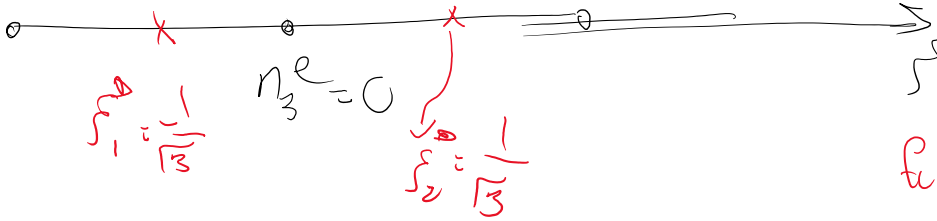
1

Physically for a bar problem we must have 1 zero mode (rigid mode)



$$n_1^e = -1$$

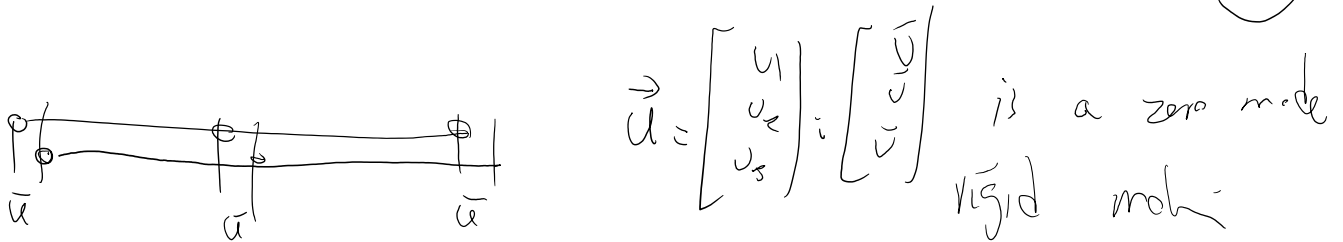
$$n_2^e = 1$$



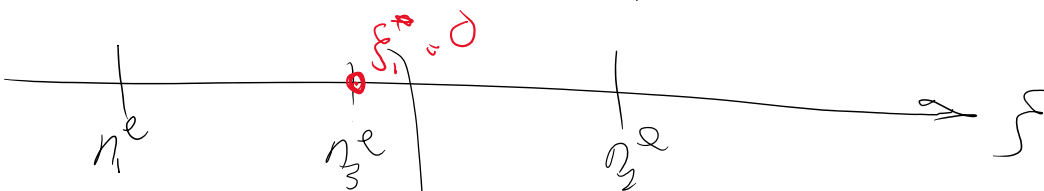
$K_{3 \times 3}$

$$\text{rank}(K) = \dim(K) - \# \text{ zero modes} = 3 - 1 = 2$$

Full int order gives rank = 2
😊



Reduced order integrals

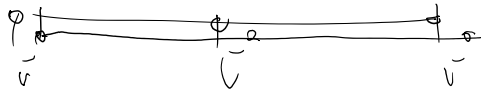


$$\begin{array}{ccc}
 n_1^e & n_3^e & \frac{1}{2}e \\
 & \downarrow & \rightarrow \int \\
 k=2 & I_{3 \times 3}(\vec{0}) &
 \end{array}$$

if we calculate the rank of this k
 we would get 1 instead of 2

Reason we have 2 (instead of 1) zero modes

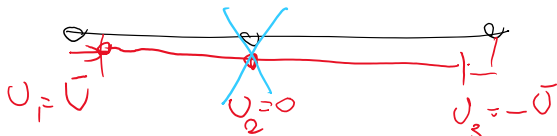
1)



$$U = \vec{u} \begin{pmatrix} 1 \\ 1 \\ 1 \end{pmatrix} \quad \text{rigid body}$$

$$F = kU = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \quad \checkmark \quad \text{smiley face}$$

2)



$$U = \vec{u} \begin{pmatrix} 1 \\ 0 \\ -1 \end{pmatrix}$$

$$F = kU = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \quad \text{frowny face}$$

nonphysical

Any coord tran

$$\det J \geq 0$$



... a zero measure set

