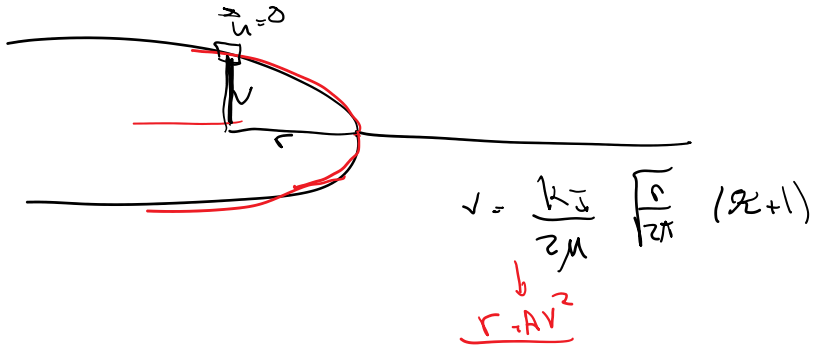
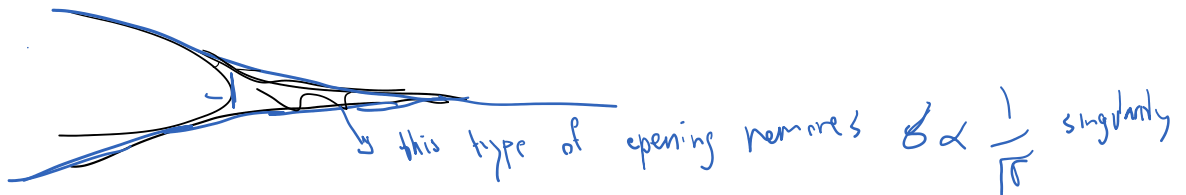


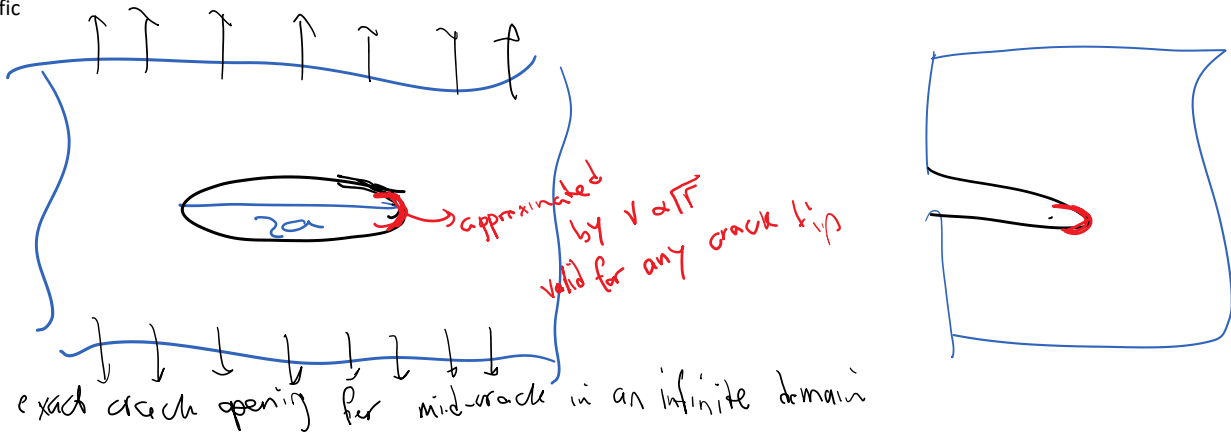
From last time, we were discussing the displacement profile



The real crack opening is like:



We want to emphasize the this parabolic opening (LEFM) theory, is the asymptotic opening and the actual crack opening is problem specific



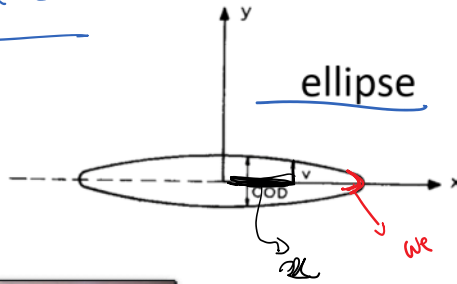
$$y = 0, -a \leq x \leq a$$

$$2\mu v = \frac{\kappa + 1}{2} \text{Im} \tilde{Z} - y \text{Re} Z \longrightarrow v = \frac{\kappa + 1}{4\mu} \text{Im} \tilde{Z}$$

$$Z(z) = \frac{\sigma x}{\sqrt{x^2 - a^2}} \longrightarrow \tilde{Z}(z) = \sigma \sqrt{x^2 - a^2}$$

$$-a \leq x \leq a \quad i = \sqrt{-1} \longrightarrow \tilde{Z}(z) = i(\sigma \sqrt{a^2 - x^2})$$

$$v = \frac{\kappa + 1}{4\mu} \sigma \sqrt{a^2 - x^2} \quad \left( \frac{v}{A} \right)^2 + x^2 = a^2$$



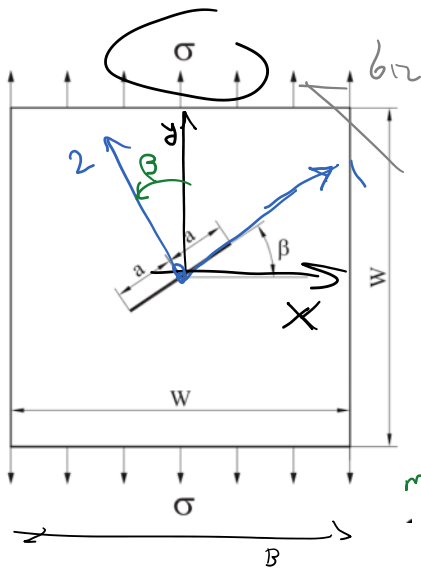
Crack Opening Displacement

$$\text{COD} = 2v = \frac{\kappa + 1}{2\mu} \sigma \sqrt{a^2 - x^2}$$

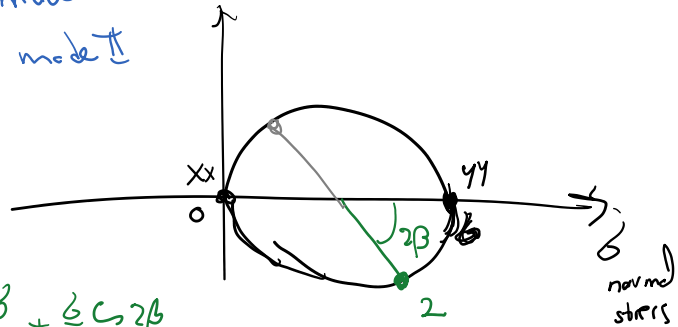
$\hookrightarrow G = \frac{E}{2(1+\nu)}$

Calculation of stress intensity factors

$a \ll W, B \rightarrow$  we can use infinite domain SFs



$\sigma_{22} \rightarrow$  mode I  
 $\sigma_{12} \rightarrow$  mode II



$$\sigma_{22} = \frac{\sigma}{2} + \frac{\sigma}{2} \cos 2\beta$$

$$\sigma_{11} = \frac{\sigma}{2} - \frac{\sigma}{2} \cos 2\beta$$

$$\sigma_{12} = \frac{\sigma}{2} (\sin 2\beta) = \sigma \sin \beta \cos \beta$$

$$K_I = \sqrt{\pi a} \quad (\text{far field normal stress}) = \sqrt{\pi a} \cdot \sigma_{22} = \sqrt{\pi a} \sigma \cos^2 \beta$$

$$K_{II} = \sqrt{\pi a} \quad (\text{shear}) = \sqrt{\pi a} \sigma_{12} = \sqrt{\pi a} \sigma \sin \beta \cos \beta$$

max @  $\beta = 45^\circ$

- A key assumption is that the infinite domain SIF formulas are used. If the crack is long enough, we need to take the domain size into effect.

## Cylindrical pressure vessel with an inclined through-thickness crack

closed-ends

$\frac{R}{t} \geq 10$  thin-walled pressure

$\sigma_z = \frac{(\pi R^2) p}{\pi R t} = \frac{R}{2t} p$

$\sigma_z = \frac{1}{2} \frac{R}{t} p$

Area =  $2 t dz$   
 tensile force  $G_\theta (2 t dz)$

Compressive force =  $(2 R dz) p$   
 area

$G_\theta (2 t dz)$

$G_\theta = \frac{R}{t} p = 2 \sigma_z$

We want to calculate SIFs for a short crack at angle  $\beta$  on the pressure vessel.

## Cylindrical pressure vessel with an inclined through-thickness crack

Equilibrium

$\sigma_\theta = 2 \sigma_z = \frac{p R}{t}$

This is why an overcooked hotdog usually cracks along the longitudinal direction first (i.e. its skin fails from hoop stress, generated by internal steam pressure).

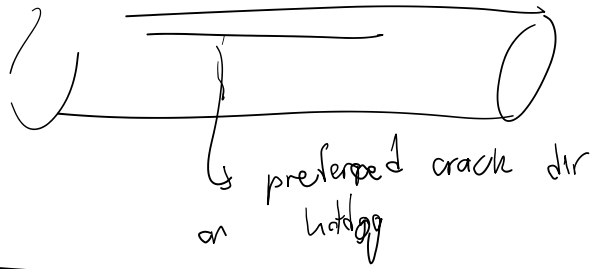
$G_{22} = G_{22} \cos^2 \beta + G_{\theta\theta} \sin^2 \beta$   
 $G_{12} = -G_{22} \sin \beta \cos \beta + G_{\theta\theta} \sin \beta \cos \beta$

$\rightarrow G_{22} = \frac{p R}{2t} (1 + \sin^2 \beta)$   
 $G_{12} = \frac{p R}{2t} \sin \beta \cos \beta$

$K_I = G_{22} \sqrt{\pi a} \rightarrow \text{max @ } \beta = \frac{\pi}{2}$

$$K_I = \sigma_{22} \sqrt{\pi a} \rightarrow \text{max } \textcircled{a} \quad \beta = \frac{11}{2}$$

$$K_{II} = |\sigma_{12}| \sqrt{\pi a}$$



How to calculate SIF:

- SIF handbook
- Computational methods (e.g. FEM)
- Experimental methods (e.g. photo sensitive material)

## Computation of SIFs

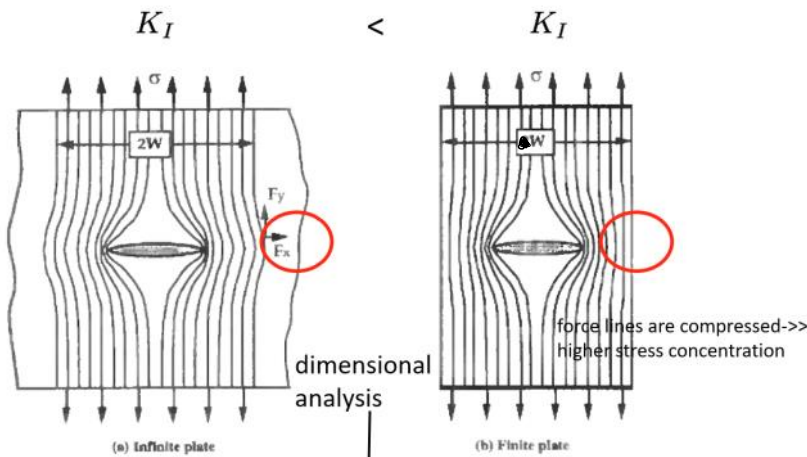
- Analytical methods (limitation: simple geometry)
  - superposition methods
  - weight/Green functions
- Numerical methods (FEM, BEM, XFEM)
  - numerical solutions -> data fit -> **SIF handbooks**
- Experimental methods
  - photoelasticity

### SIF for finite size samples

Exact (closed-form) solution for SIFs: simple crack geometries in an **infinite** plate.

Cracks in finite plate: influence of external boundaries cannot be neglected -> generally, no exact solution

## SIF for finite size samples



geometry/correction factor [-]

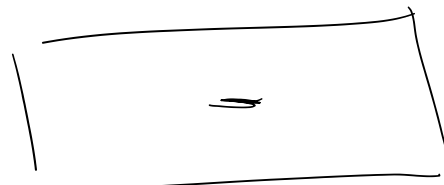
$$K_I = f(a/W) \sigma \sqrt{\pi a} \quad a \ll W : f(a/W) \approx 1$$

For this mid-crack example we have an analytical solution:

$$f(a, W) = \sqrt{\frac{1}{\cos \frac{\pi a}{W}}} = \sqrt{\sec \frac{\pi a}{W}}$$

$$a \ll W \rightarrow f(a, W) \rightarrow 1$$

$$\dots \rightarrow \frac{\pi W}{2} \dots$$

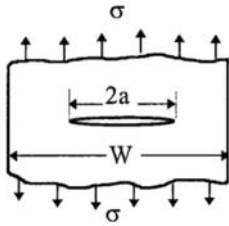


$a \ll W \rightarrow f(a, W) \rightarrow 1$

$a \rightarrow W \rightarrow C \frac{\pi a}{W} = C \frac{\pi \frac{W}{2}}{W} \rightarrow 0$   
 $f(a, W) \& K_I \rightarrow \infty$



2. Centre crack in a strip of finite width



$K_I = \sqrt{\sec \frac{\pi a}{W}} \sigma \sqrt{\pi a}$

secant function

$\sec \theta = \frac{1}{\cos \theta}$

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Typical convention in providing SIF correction factors

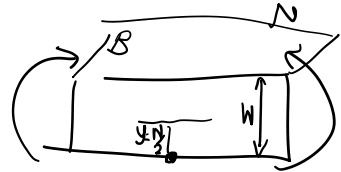
Ex



$\sigma_{max}$  w/o crack

$\sigma_{max} = \frac{My}{EI}$

for  $y = \frac{W}{2}$

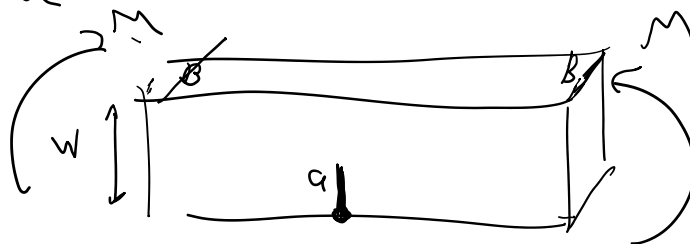


$\sigma_{max} = \frac{M \frac{W}{2}}{E \frac{1}{12} B W^3} = \frac{6M}{B W^2}$

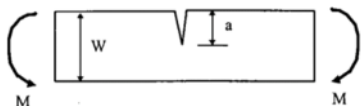
$K_I = f\left(\frac{a}{W}\right) \sigma_{max} \sqrt{\pi a}$

Correction factor depending on geometry

Maximum tensile (mode I) stress for this problem



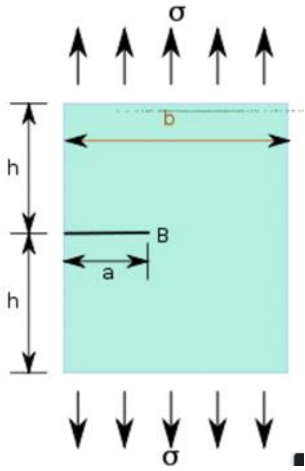
5. Edge crack in a beam of width B subjected to bending



$K_I = f\left(\frac{a}{W}\right) \sigma \sqrt{\pi a}$  where  $\sigma = \frac{6M}{B W^2}$

$a/W$	$f(a/W)$
0.1	1.044
0.2	1.055

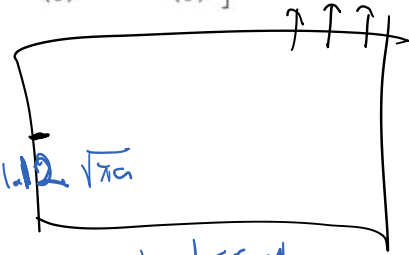
0.3	1.125
0.4	1.257
0.5	1.500
0.6	1.915



$$h/b \geq 1 \text{ and } a/b \leq 0.6.$$

$$K_I = \sigma \sqrt{\pi a} \left[ 1.12 - 0.23 \left( \frac{a}{b} \right) + 10.6 \left( \frac{a}{b} \right)^2 - 21.7 \left( \frac{a}{b} \right)^3 + 30.4 \left( \frac{a}{b} \right)^4 \right]$$

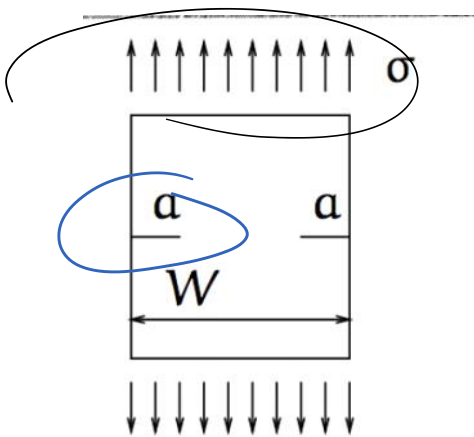
What about  $a \ll b$



$$K_{II} = 1.12 \sigma \sqrt{\pi a}$$

for a very short crack on the edge

12% higher than a very short mid crack



$$K_{I \pm} \quad \frac{a}{W} \rightarrow 0$$

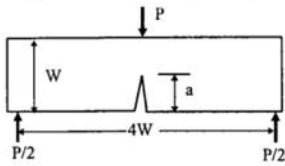
$$1.12 \sigma \sqrt{\pi a}$$

$$K_I = \sigma \sqrt{a} \left[ 1.12 \sqrt{\pi} + 0.76 \frac{a}{W} - 8.48 \left( \frac{a}{W} \right)^2 + 27.36 \left( \frac{a}{W} \right)^3 \right]$$

$$\approx 1.12 \sigma \sqrt{\pi a}$$



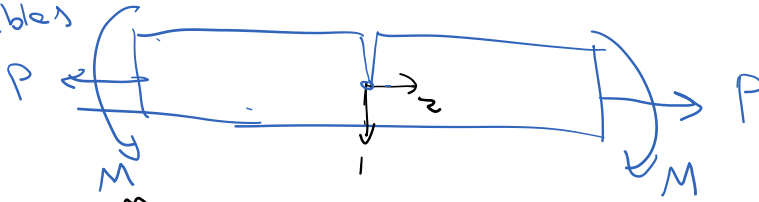
9. Single-edge notch bend (SENB), thickness  $B = W/2$



$$K_I = Y \frac{4P\sqrt{\pi}}{B\sqrt{W}}$$

$$Y = 1.63 \left(\frac{a}{W}\right)^{1/2} - 2.6 \left(\frac{a}{W}\right)^{3/2} + 12.3 \left(\frac{a}{W}\right)^{5/2} - 21.3 \left(\frac{a}{W}\right)^{7/2} + 21.9 \left(\frac{a}{W}\right)^{9/2}$$

Use of tables



$$\sigma_{ij} = \Delta_{ij}^P + \sigma_{ij}^M$$

$$= K_I^P \frac{1}{\sqrt{2\pi r}} f_{ij}^I(\theta) + K_I^M \frac{1}{\sqrt{2\pi r}} f_{ij}^I(\theta)$$

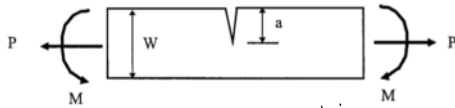
$$= \frac{(K_I^P + K_I^M)}{\sqrt{2\pi r}} f_{ij}^I(\theta)$$

$$K_I = K_I^P + K_I^M$$

Determine the stress intensity factor for an edge cracked plate subjected to a combined tension and bending.

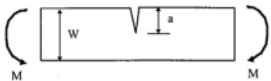
$$a/W = 0.2$$

$B$  thickness



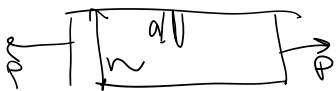
$$K_{II} = K_{II}^{\text{bending}} + K_{II}^P \quad , \quad K_{II}^{\text{bending}} = 1.055 \frac{6M}{BW^2} \sqrt{\pi a}$$

5. Edge crack in a beam of width  $B$  subjected to bending



$$K_I = f\left(\frac{a}{W}\right) \sigma \sqrt{\pi a} \quad \text{where } \sigma = \frac{6M}{BW^2}$$

$a/W$	$f(a/W)$
0.1	1.044
0.2	1.055
0.3	1.125



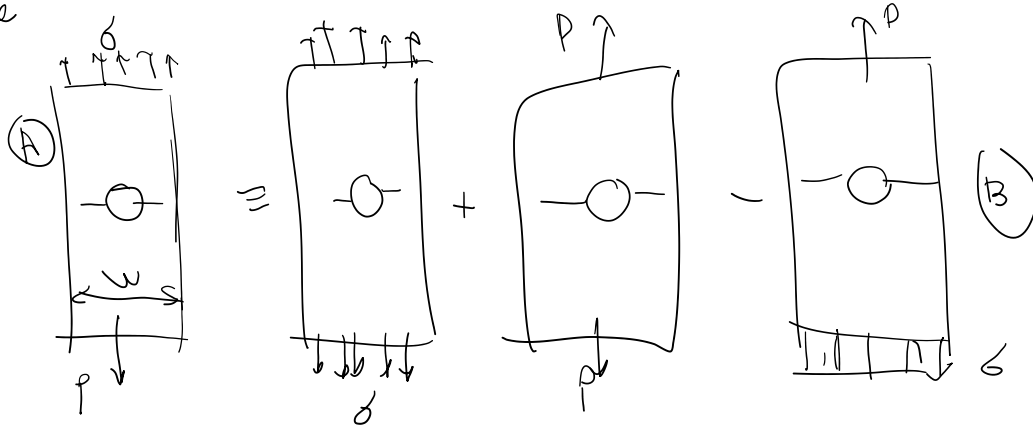
look for table

$$(1.12) \left(\frac{P}{WB}\right) \sqrt{\pi a}$$

Actual calculation for  $\frac{a}{W} = 0.2$  is higher than 1.12  
 i.e. (A)

higher than 1.12  
 Use (A)

Example



P.  $\delta V$

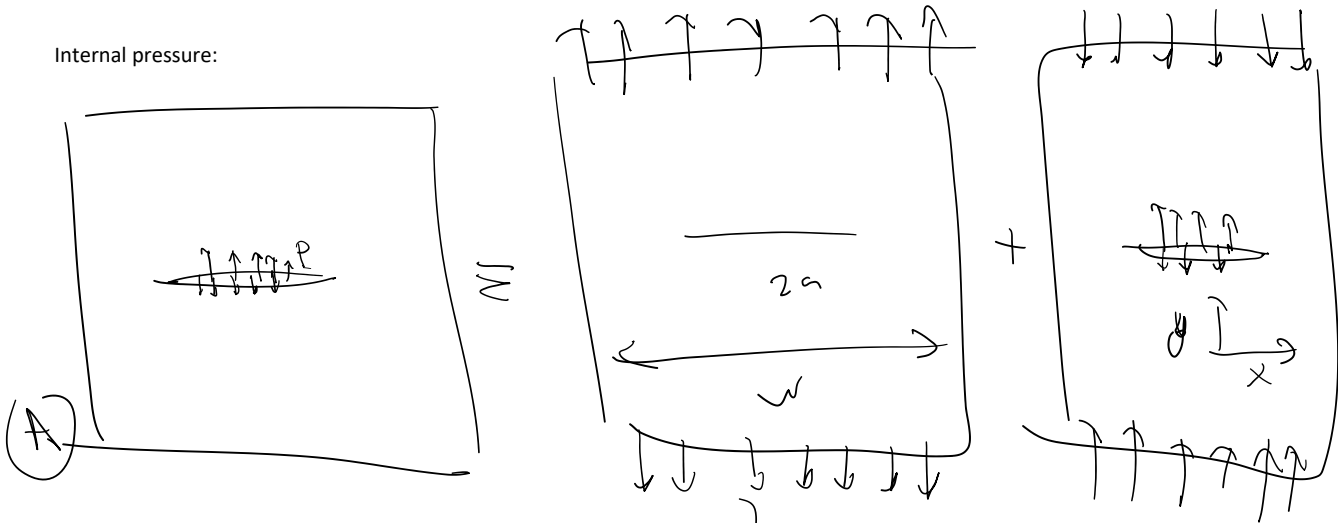
$$K_I^A = K_I^\sigma + K_I^P - K_I^B$$

circle is centered in the domain

$$K_I^B = K_I^A$$

$$\rightarrow K_I^A \geq \frac{k_I^\sigma + k_I^P}{2}$$

Internal pressure:



$$K_I^A =$$

$$K_I^B = \sqrt{\frac{I}{G \frac{\pi a^2}{2}}} \sqrt{\pi a}$$

$$\sigma_{yy} = -P$$

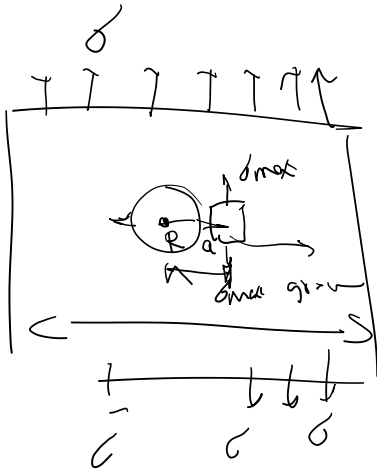
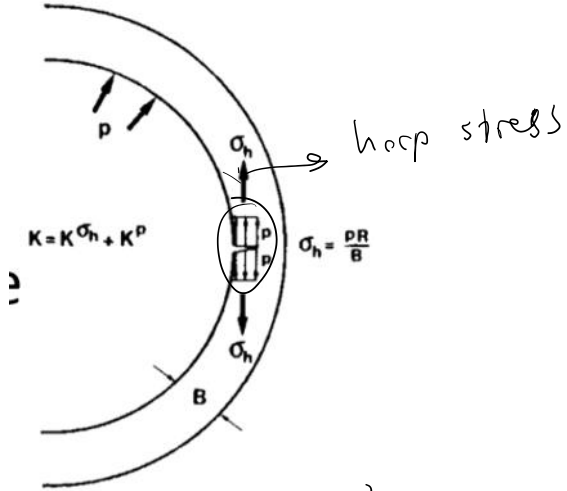
$$k_I = 0$$



Example of internal pressure

$a \ll t$

$K_I = (1.12) \left(\frac{PR}{B}\right) \sqrt{\pi a}$   
 ↓  
 correction

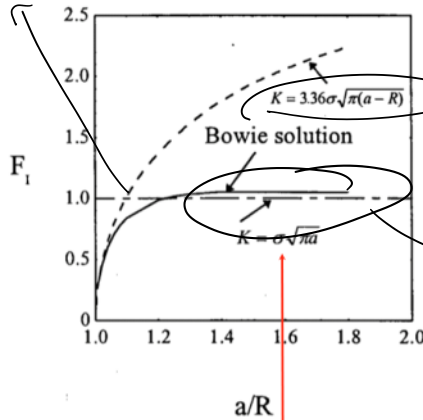
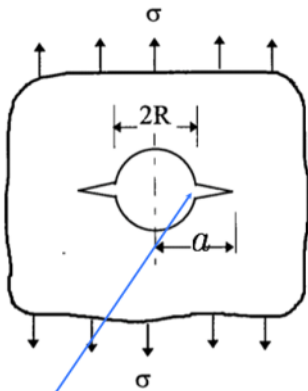


$K_I = (1.12) \sqrt{\pi a} \sigma_{max} \approx 3.36 \sqrt{\pi a} \sigma$   
 ↓  
 correction factor  
 $a \ll R, W$

$a \gg R$  but still  $a \ll W$

$\sqrt{\pi(R+a)} \sigma \approx (\sqrt{\pi a}) \sigma$   
 $R \ll a$

very good approx  $a \ll R$



other solution

$R \ll a$

**3σ** edge crack

hole as a part of the crack

$K_I = 1.12(3\sigma)\sqrt{\pi(a-R)} = 3.36\sigma\sqrt{\pi a} \sqrt{1 - \frac{R}{a}} = 3.36\sqrt{1 - \frac{1}{a/R}} \sigma \sqrt{\pi a}$