

In fact both J_1 (J) and J_2 are related to SIFs:

$$J_1 = \int_{\Gamma} \left(w dy - t \frac{\partial u}{\partial x} d\Gamma \right) \quad \text{J1 \& J2: crack advance for } (\theta = 0, 90) \text{ degrees}$$

$$J_2 = \int_{\Gamma} \left(w dx - t \frac{\partial u}{\partial y} d\Gamma \right) \quad \theta = \frac{\pi}{2}$$

Hellen and Blackburn (1975)

$$J = \frac{(1+\nu)(1+\kappa)}{4E} (K_I^2 + K_{II}^2 + 2iK_I K_{II}) \rightarrow$$

$$J_1 = \frac{K_I^2 + K_{II}^2}{E'} \quad J_2 = \frac{-K_I K_{II}}{E'} \rightarrow K_{II} = \frac{-J_2 E'}{K_I}$$

$$K_I^2 + \left(\frac{-J_2 E'}{K_I} \right)^2 = J_1 E'$$

$$K_I^4 - (J_1 E') K_I^2 + (J_2 E')^2 = 0$$

$$z^2 - (J_1 E') z + (J_2 E')^2 = 0 \quad z = K_I^2$$

The solutions will be $K_I^2 = a^2$ or $b^2 \rightarrow$

$$\begin{aligned} K_I &= \pm a & K_{II} &= \mp b \\ K_I &= \pm b & K_{II} &= \mp a \end{aligned}$$

We'll choose the right values and signs based on the loading and direction of loading.

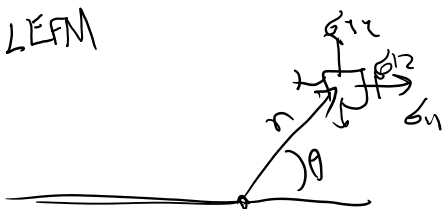
$$\begin{aligned} J_1 &= \frac{K_I^2 + K_{II}^2}{E'} \\ J_2 &= \frac{-2K_I K_{II}}{E'} \end{aligned}$$

$$E' = \begin{cases} E & \text{plane strain} \\ \frac{E}{1-\nu^2} & \text{plane stress} \end{cases}$$

Note that if $K_I = a, K_{II} = b$ is a solution the general solution is:
 $K_I = \pm a, K_{II} = \pm b$ and $K_I = \pm b, K_{II} = \pm a$

Second use of J integral -> local stress distribution

LEFM



$$\sigma_{ij} = \frac{K_I}{\sqrt{2\pi r}} \sum_{\Gamma}^{ij} (\theta) \quad \dots \text{mode II}$$

PFM

$$\sigma_{ij} \propto J r^{-\beta}$$

Do we have an asymptotic solution of this form?

5.3.5 Plastic crack tip fields:
Hutchinson, Rice, and Rosengren (HRR) solution

Ramberg - Osgood model

$$\frac{\epsilon}{\epsilon_{y0}} = \frac{\sigma}{\sigma_{y0}} + \alpha \left(\frac{\sigma}{\sigma_{y0}} \right)^n$$

$n=1$ }
similar to ϵ^e

$\approx \epsilon^p$

Compare with plasticity model

$$\epsilon = \epsilon^e + \epsilon^p$$

$$\frac{\epsilon}{\epsilon_{y0}} = \left(\frac{\sigma}{\sigma_{y0}} \right) + \alpha \left(\frac{\sigma}{\sigma_{y0}} \right)^n$$

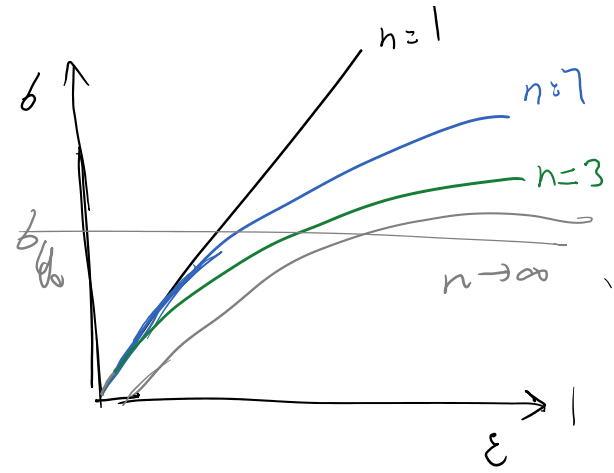
very large around the crack tip

around the crack tip

$$\frac{\epsilon}{\epsilon_{y0}} \approx \alpha \left(\frac{\sigma}{\sigma_{y0}} \right)^n$$

$$\boxed{\epsilon \propto \sigma^n} \quad (1)$$

around the crack tip

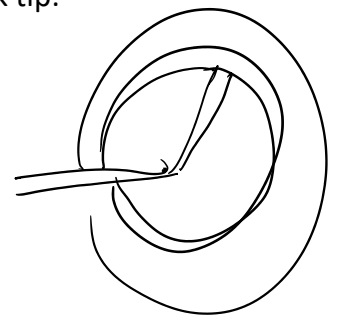


We are looking for the powers of singularity for strain and stress around the crack tip:

$$\textcircled{2} \quad \sigma = \frac{c_1}{r^\alpha} \quad \epsilon = \frac{c_2}{r^\beta}$$

From energy argument

$$\mathcal{E} = \int_0^R \underbrace{\mathcal{W}(\epsilon)}_{\epsilon: \sigma} \underbrace{dA}_{r dr d\theta}$$



$$\int \frac{r}{r^\alpha} \cdot \frac{1}{r^\beta} r dr d\theta$$

$r^{1-\alpha-\beta}$

$$1 - \alpha - \beta \geq 0$$

$$\alpha + \beta \leq 1 \quad \text{in fact } \alpha + \beta = 1$$

$$\textcircled{1}, \textcircled{2} \quad \epsilon \propto \sigma^n$$

$$\rightarrow (n+1)y = 1 \quad y = \frac{1}{1+n}$$

$$\textcircled{1,2} \quad \begin{matrix} \epsilon \propto \sigma^n \\ \downarrow \\ \frac{C_1}{r^{\frac{1}{n}}} \propto \left(\frac{C_1}{r\sigma}\right)^n \end{matrix} \rightarrow \alpha = \eta \gamma \quad \left. \begin{matrix} (n+1) \gamma = 1 \\ \gamma = \frac{1}{1+n} \\ \alpha = \frac{n}{1+n} \end{matrix} \right\}$$

we expect to have the following dependence on r around the crack tip

$$\sigma \propto \frac{1}{r^{\frac{1}{n+1}}} \quad \epsilon \propto \frac{1}{r^{\frac{n}{n+1}}}$$

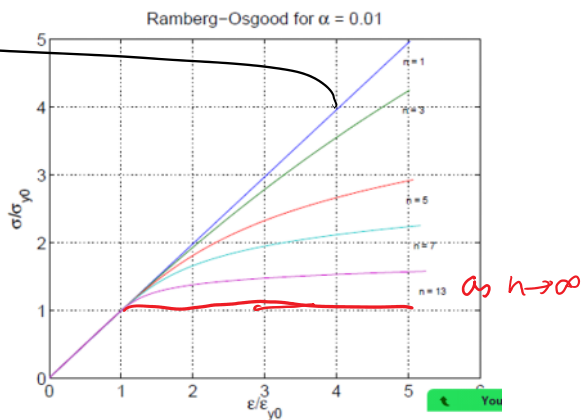
$n=1 \rightarrow$ LEFM (linear elastic)

$$\sigma \propto \frac{1}{\sqrt{r}} \quad \epsilon \propto \frac{1}{\sqrt{r}}$$

we had this before

$$\sigma_{ij} = \frac{K_I}{\sqrt{2\pi r}} \Sigma_{ij}^{\epsilon}(\theta)$$

$$\epsilon_{ij} = D_{ijkl} \sigma_{kl} \propto \frac{1}{\sqrt{r}}$$

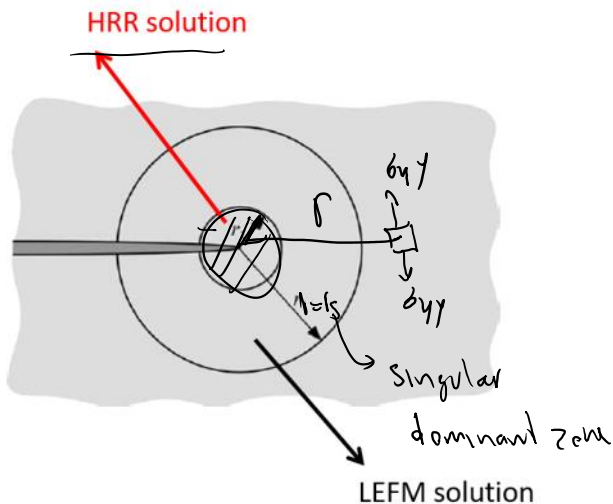
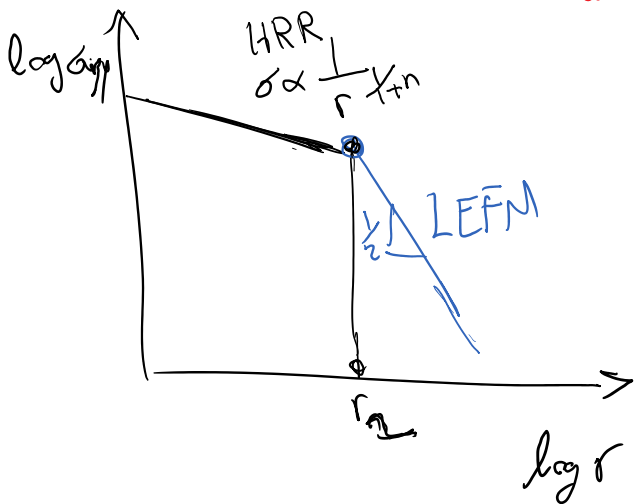


How about the limit as $n \rightarrow \infty$ (approximate an elastic-perfectly plastic model)

$\sigma \rightarrow \frac{1}{r^0} \rightarrow \text{const}$ no longer singular in the limit of elastic perfectly plastic model

σ is almost bounded by σ_{y0} (as if we have yield stress)

$\epsilon \propto \frac{1}{r^{n/(n+1)}} \rightarrow \frac{1}{r}$ all singularity goes to strain



- This model still predicts that stress is singular close to the crack tip, but the power of singularity for stress is lower.
- Only predicts bounded stress for the limit of $n \rightarrow \infty$ which corresponds to elastic-perfectly plastic model.

Final form of HRR solution

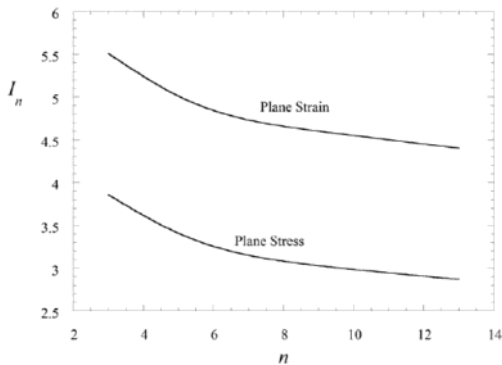
$$\sigma_{ij} = \sigma_0 \left(\frac{EJ}{\alpha \sigma_0^2 I_n r} \right)^{\frac{1}{n+1}} \tilde{\sigma}_{ij}(n, \theta)$$

$$\epsilon_{ij} = \frac{\alpha \sigma_0}{E} \left(\frac{EJ}{\alpha \sigma_0^2 I_n r} \right)^{\frac{n}{n+1}} \tilde{\epsilon}_{ij}(n, \theta)$$

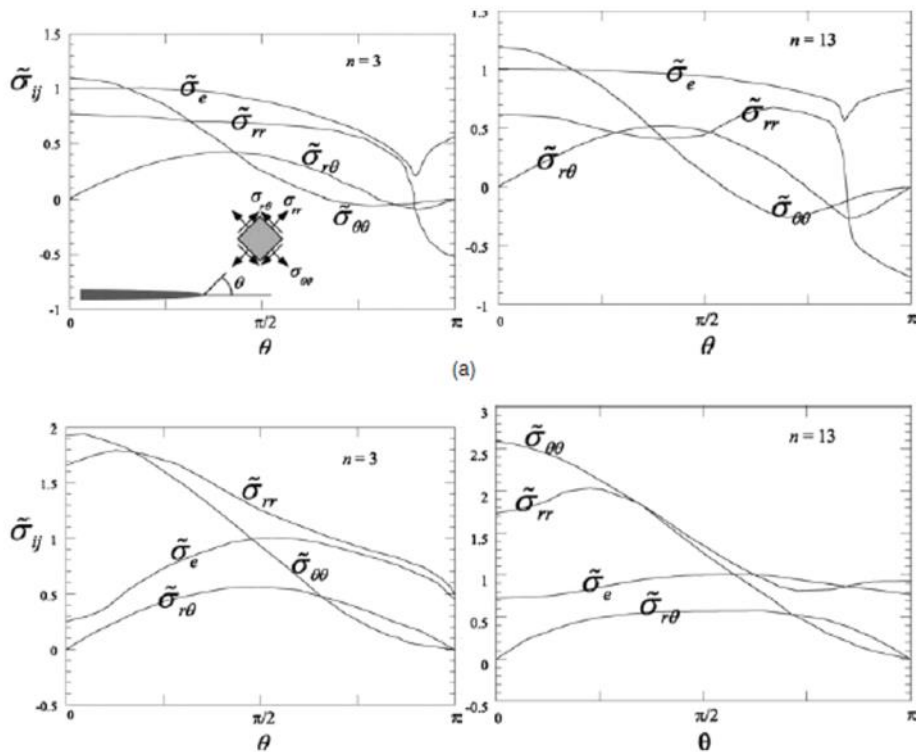
plays the role of K
 expressed in terms of G it has the power of σ_0

$$\tilde{\sigma}_{ij} = \frac{K \sigma}{\sqrt{2n}} \sum_{ij}^I(\theta)$$

terms in green show dependency on power n

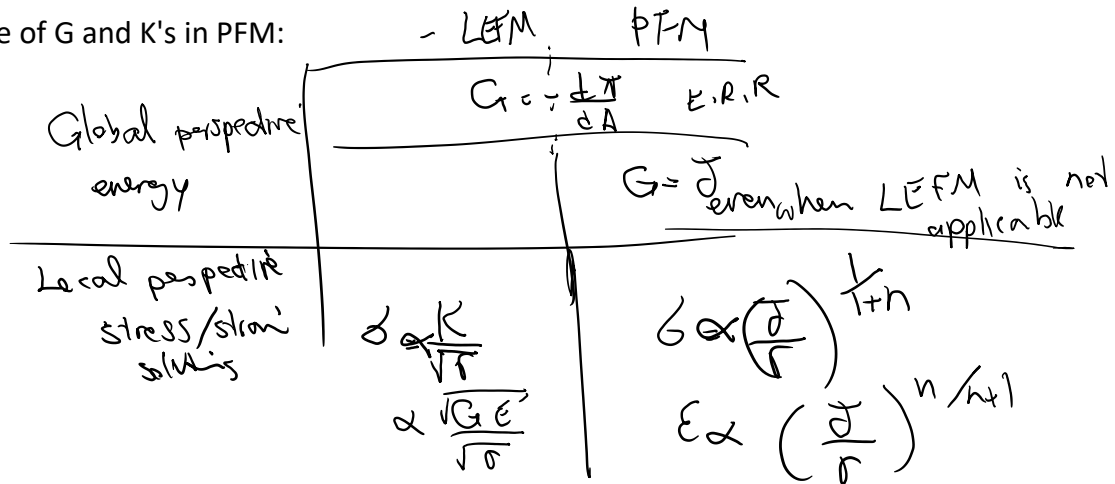


HRR solution: Angular functions



Summary:

J plays the role of G and K's in PFM:



Energy release rate of J integral: Assumptions

1. Homogeneous body

2. Linear or non-linear elastic solid

→ approximate plasticity

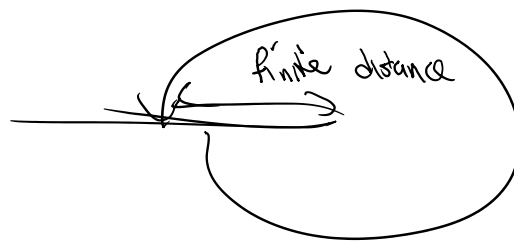
3. No inertia, or body forces; no initial stresses

$$\nabla \cdot \sigma + \rho b = \rho \ddot{u}$$

4. No thermal loading

5. 2-D stress and deformation field

6. Plane stress or plane strain



~~7. Mode I loading~~

8. Stress free crack

We can extend the way we compute the J integral so that none of these constraints become relevant (we will do this when covering the computational methods for computing J)

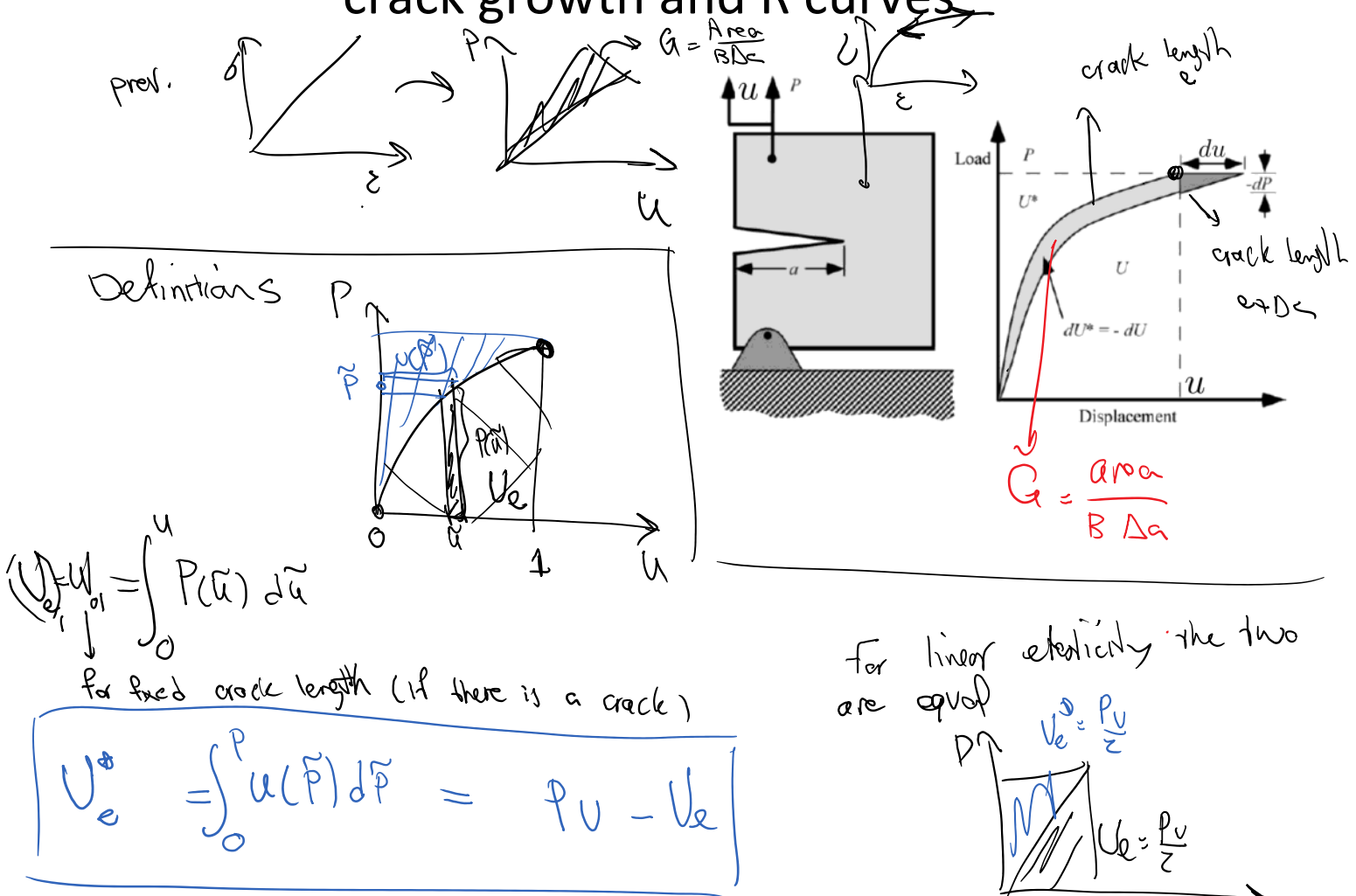
Generalization of J integral

- Dynamic loading
- Surface tractions on crack surfaces
- Body force
- Initial strains (e.g. thermal loading)
- Initial stress from pore pressures

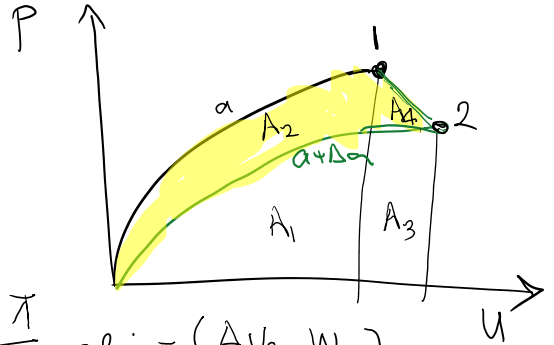
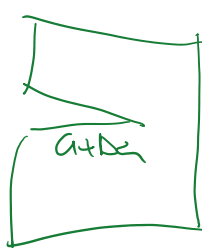
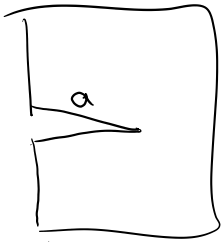
cf. Saouma 13.11 & 13.12 for details

Linear -> nonlinear **elasticity**, extending the P-delta system analysis

5.3.4. Energy Release Rate, crack growth and R curves



$$\left| \begin{array}{c} \circ \\ \cup \\ \hline \end{array} \right| \quad \left| \begin{array}{c} \cup \\ - \cup \\ \hline \end{array} \right|$$



want to show

$$G = \lim_{\Delta a \rightarrow 0} \frac{\text{shaded area}}{\Delta a \cdot B}$$

$$G = -\frac{dW}{dA} = -\frac{dW}{B da} = \lim_{\Delta a \rightarrow 0} -\frac{(A_3 - A_2) - (A_3 + A_4)}{B \Delta a}$$

$$\Delta u_e = u_e(a + \Delta a) - u_e(a) = (A_1 + A_3) - (A_1 + A_2) = A_3 - A_2$$

$$u_e(a + \Delta a) = A_1 + A_3$$

$$u_e(a) = A_1 + A_2$$

$$W_{12} = \int_0^2 P du = A_3 + A_4$$

$$G = \lim_{\Delta a \rightarrow 0} -\frac{(A_3 - A_2) - (A_3 + A_4)}{B \Delta a}$$

$$\rightarrow \lim_{\Delta a \rightarrow 0} \frac{A_2 + A_4}{B \Delta a}$$

linear & nonlinear elasticity

$$G = \lim_{\Delta a \rightarrow 0} \frac{\text{shaded area}}{B \Delta a}$$

Do we have equations similar to

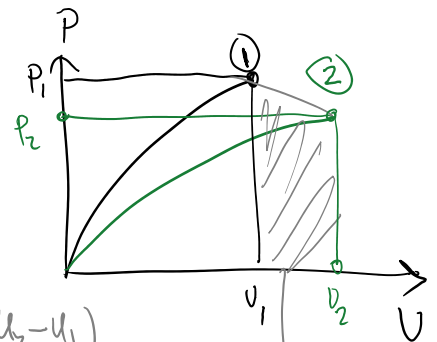
$$G = \frac{P^2}{2B} \frac{dC}{da} = -\frac{U^2}{2B} \frac{dK}{da}$$

Compliance $C = \frac{U}{P}$ stiffness $K = \frac{P}{U}$

Recall

$$G = \lim_{\Delta a \rightarrow 0} -\frac{\Delta u_e}{B \Delta a} = \lim_{\Delta a \rightarrow 0} -\frac{(u_e(a + \Delta a) - u_e(a))}{B \Delta a}$$

$$= \frac{1}{B} \lim_{\Delta a \rightarrow 0} -\frac{(u_e(a + \Delta a) - u_e(a))}{\Delta a} + \lim_{\Delta a \rightarrow 0} \frac{(P_1 + P_2)}{2} \frac{(u_2 - u_1)}{\Delta a}$$



$$W_{12} \approx \frac{(P_1 + P_2)}{2} (u_2 - u_1)$$

$$= \frac{1}{2} (P_1 + P_2) (u_2 - u_1)$$

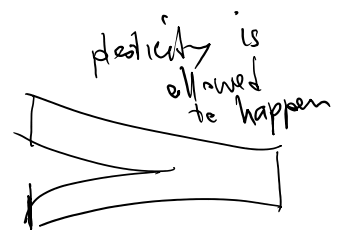
$$G = \lim_{\Delta a \rightarrow 0} \frac{-1}{B} \frac{dU_e(a)}{da} + \frac{1}{B} \lim_{\Delta a \rightarrow 0} \left(\frac{P(a) + P(a+\Delta a)}{2} \right) \left(\frac{U(a+\Delta a) - U(a)}{\Delta a} \right)$$

$W_{1/2} \approx \left(\frac{P_1 + P_2}{2} \right) (U_2 - U_1)$
 $f'(x) = \lim_{\Delta x \rightarrow 0} \frac{f(x+\Delta x) - f(x)}{\Delta x}$

$$G = -\frac{1}{B} \frac{dU_e}{da} + \frac{P}{B} \frac{dU}{da}$$

General formula for G

$G = \frac{1}{B} \left(-\frac{dU_e}{da} + P \frac{dU}{da} \right)$
 $= \frac{1}{B} \frac{dU_e}{da}$ dead load
 $= \frac{-1}{B} \frac{dU_e}{da}$ fixed grip



(*)

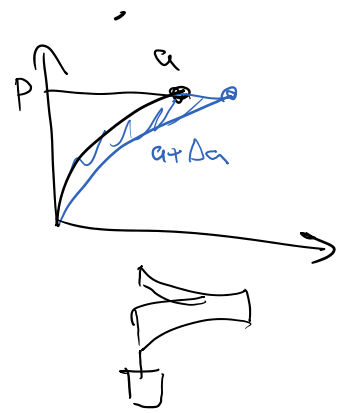
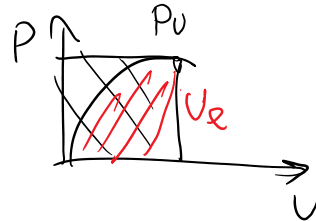
Special case 1: dead load (P is fixed)

$$G = \frac{1}{B} \left(-\frac{dU_e}{da} + P \frac{dU}{da} \right)$$

take P inside $\frac{d}{da}$ because it's fixed

$$= \frac{1}{B} \left(-\frac{dU_e}{da} + \frac{d(PU)}{da} \right) = \frac{1}{B} \frac{d(PU - U_e)}{da}$$

$$= \frac{1}{B} \frac{dU_e}{da}$$
 dead load

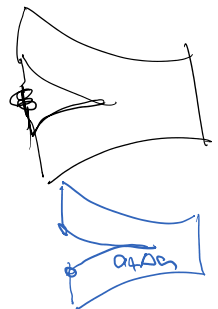
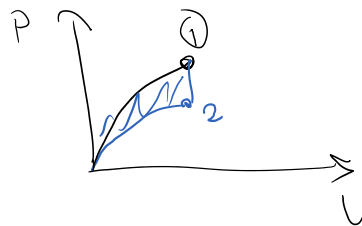


Special case 2: fixed grip

$$G = \frac{1}{B} \left(-\frac{dU_e}{da} + P \frac{dU}{da} \right)$$

U is fixed $\rightarrow \frac{dU}{da} = 0$

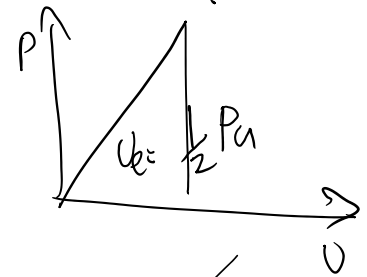
$$G = -\frac{1}{B} \frac{dU_e}{da}$$



Does \otimes match linear results for a linear material?

$$G = \frac{1}{B} \left(-\frac{dU_e}{da} + P \frac{du}{da} \right) \quad \rightarrow$$

$$U_e = \frac{1}{2} P u = \frac{1}{2} \left(\overset{P}{k} u \right) u = \frac{1}{2} k u^2$$



$$G = \frac{1}{B} \left(-\frac{d \frac{1}{2} k u^2}{da} + P \frac{du}{da} \right) = \frac{1}{B} \left(-\frac{1}{2} \frac{dk}{da} \cdot u^2 - \frac{k u du}{P da} + P \frac{du}{da} \right)$$

$$= \frac{1}{2B} u^2 \frac{dk}{da}$$

matches our linear formula

$$\therefore \frac{1}{2B} \frac{P^2}{a} \frac{dC}{da}$$