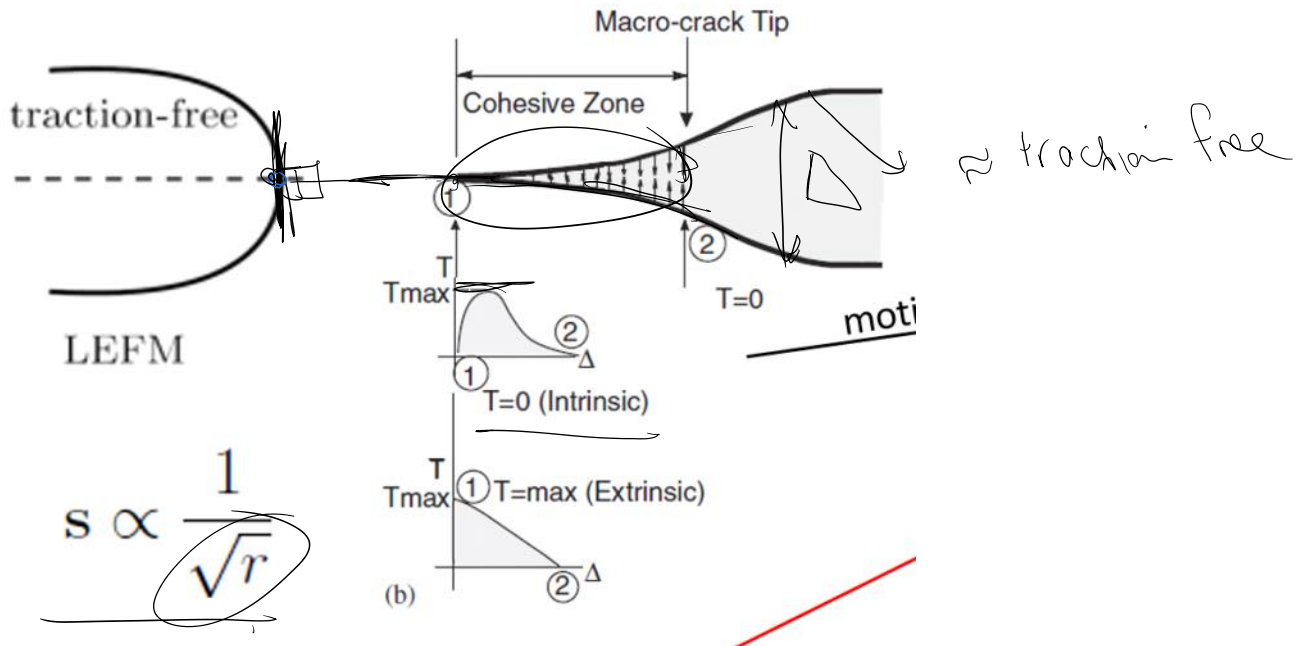
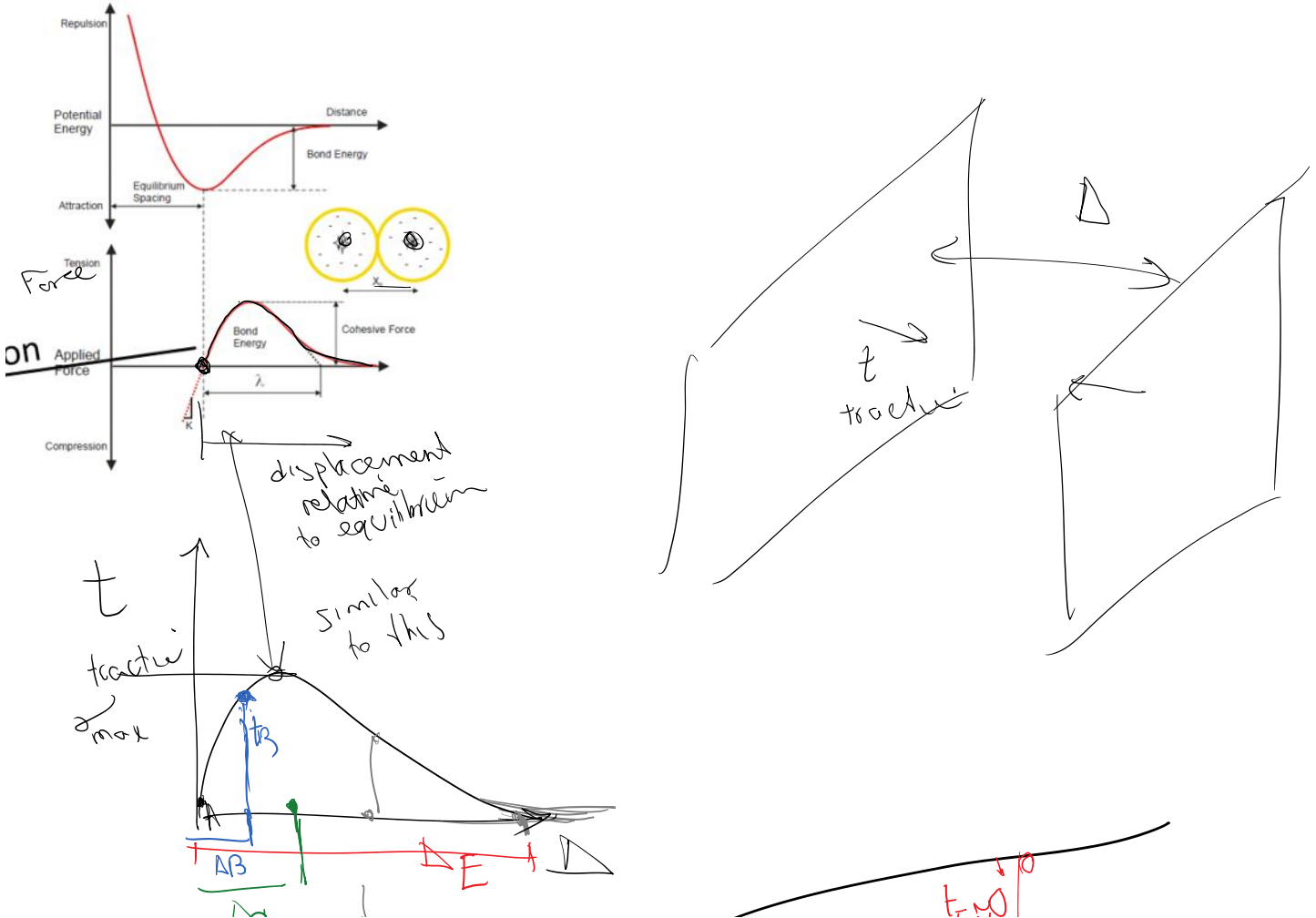


Cohesive models continued:

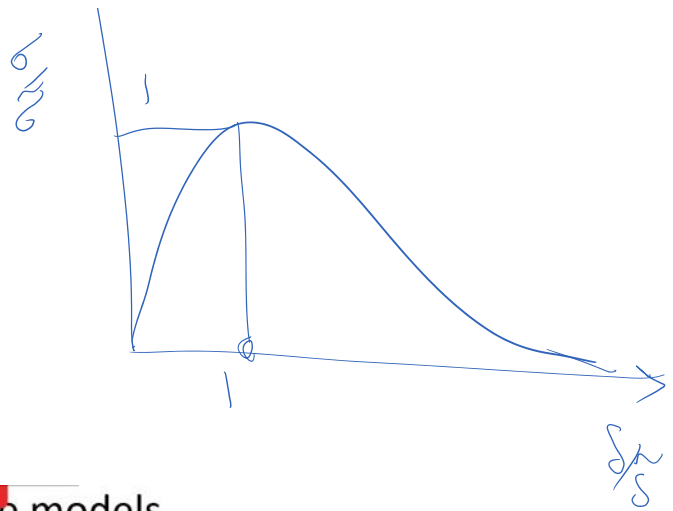


Cohesive models eliminate the singularity of the stress and strain fields around the crack tip



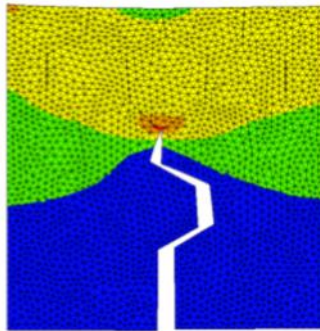
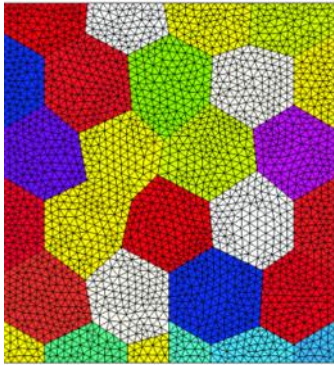
$$\begin{cases} -\tilde{\sigma}, \tilde{\delta} \\ -\tilde{\sigma}, \tilde{\delta} \\ -\tilde{\sigma}, \tilde{\delta} \end{cases}$$

non-dimensional expression $\sigma = \tilde{\sigma} f(\frac{\delta}{\tilde{\delta}})$

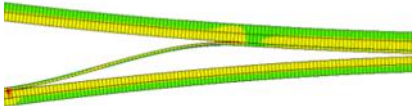


Sample applications of cohesive models

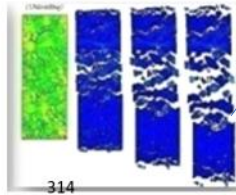
fracture of polycrystalline material



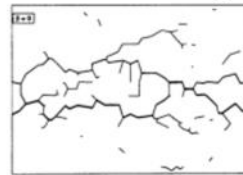
delamination of composites



fragmentation



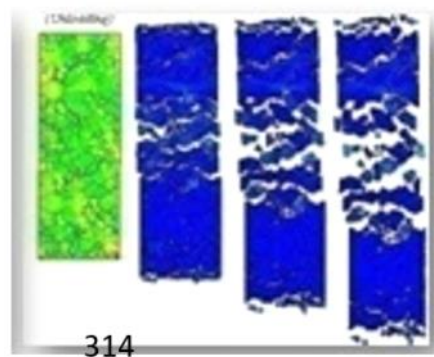
Microcracking and branching



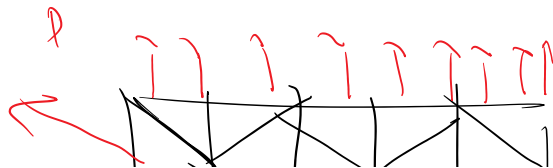
Extrinsic

vs

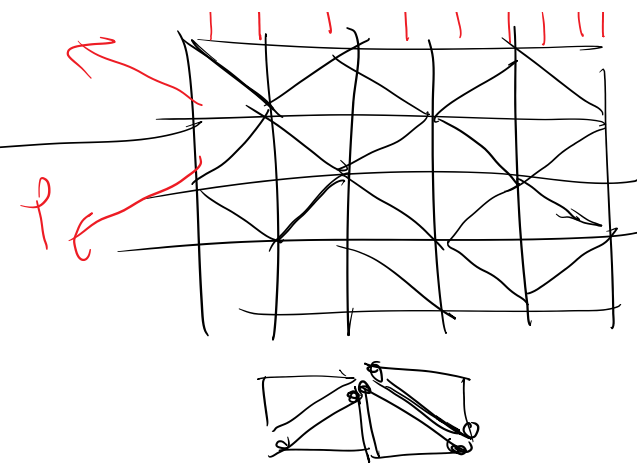
intrinsic models



number of elements

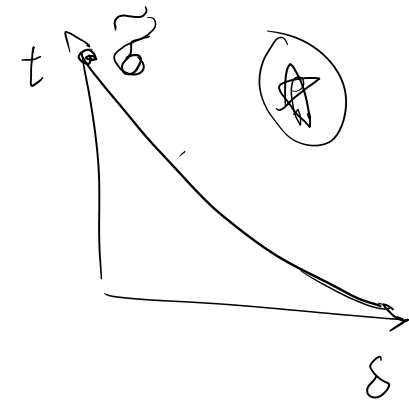
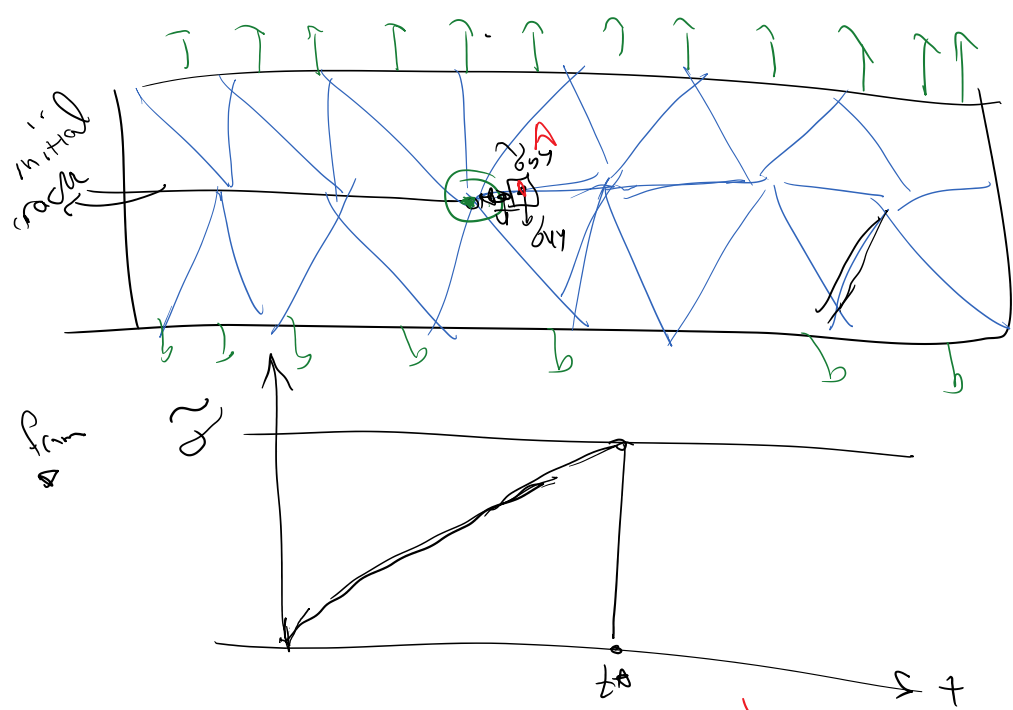


all
 mesh elements
 have
 cohesive
 elements

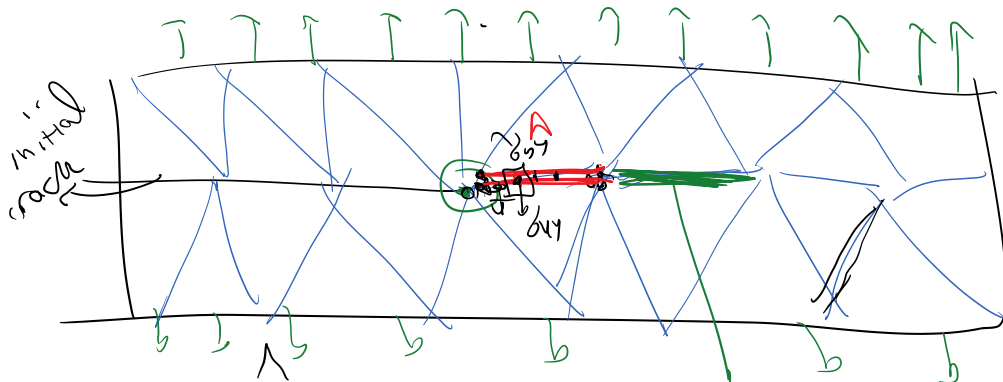


Artificial compliance -> as the mesh gets finer the structure gets more and more compliant

Extrinsic model

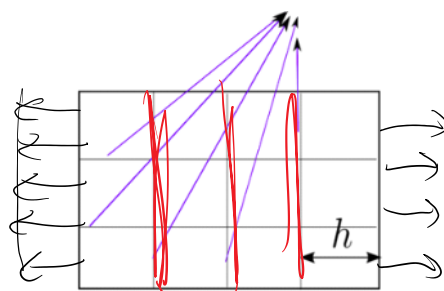
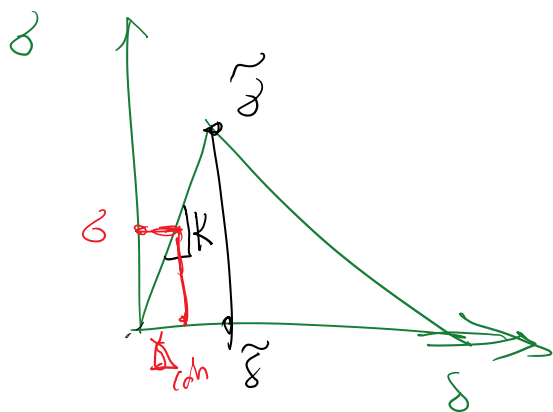


We insert a cohesive model at point A or the whole element edge



at a border time that edge becomes cohesive

We can do similar fragmentation problems with extrinsic models, but handling the mesh connectivity and interactive insertion of cohesive surfaces is more difficult.

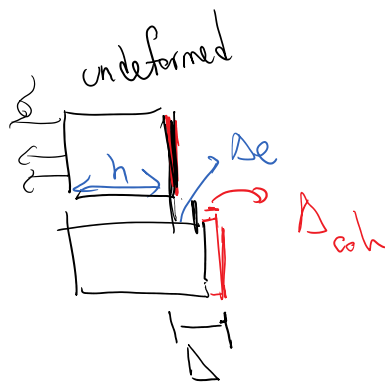


cohesive models

Bulk part

$$\Delta_e = \epsilon h = \frac{\sigma}{E} h$$

$$\Delta_{coh} = \frac{\sigma}{K}$$



$$\Delta = \frac{\sigma}{E} h + \frac{\sigma}{K} = \sigma \left(\frac{h}{E} + \frac{1}{K} \right)$$

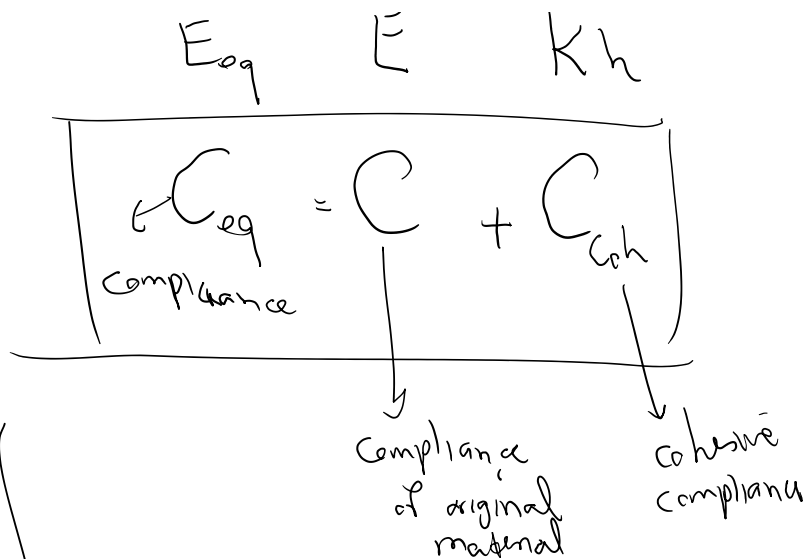
for a material with equivalent stiffness E_{eq}

$$\Delta = \frac{\sigma}{E_{eq}} h$$

$$\frac{1}{E_{eq}} = \frac{1}{E} + \frac{1}{K}$$

$$\sigma (\ddot{E} + \dot{K}) \sim E_{eq}$$

$$E_{eq} = \frac{1}{\frac{1}{E} + \frac{1}{Kh}}$$



$$E_{eq} = E \cdot \frac{1}{1 + \frac{E}{Kh}}$$

Kh vs E

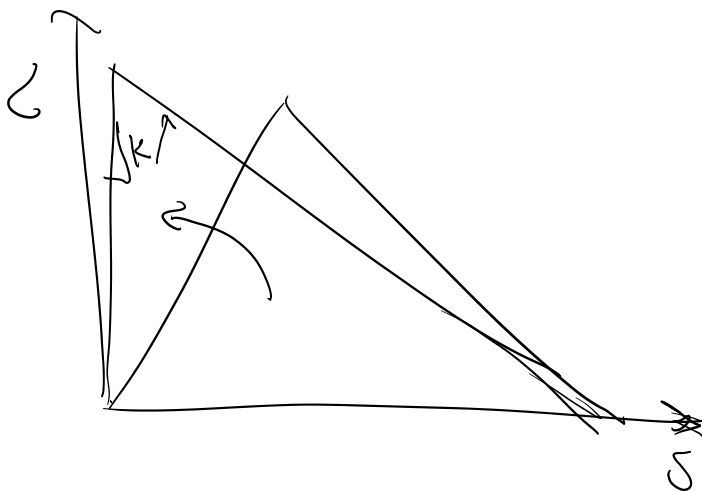
$$\frac{Kh}{E} \rightarrow 0$$

$$E_{eq} \rightarrow 0 \quad (C_{eq} \rightarrow \infty)$$

As $h \rightarrow 0$ (better FEM solution), $\frac{Kh}{E} \rightarrow 0 \rightarrow$ Compliance \uparrow

What if for a given h

choose $\frac{Kh}{E}$ large enough, eg 10, 100



Unfortunately, we cannot use arbitrarily high K (penalties) in general, as the system matrices become difficult to solve (bad

conditioning)

- Artificial compliance becomes important if cohesive surfaces are added between all elements for intrinsic models to find crack propagation path.
- The artificial compliance is computed as,

$$\Delta = \Delta_e + \Delta_c, \quad \Delta_e = \text{elastic displacement}, \Delta_c = \text{cohesive separation} \Rightarrow$$

$$\frac{\sigma}{E_{\text{eff}}} h = \frac{\sigma}{E} h + \frac{\sigma}{K} \Rightarrow$$

$$\frac{1}{E_{\text{eff}}} = \frac{1}{E} + \frac{1}{Kh} \Rightarrow$$

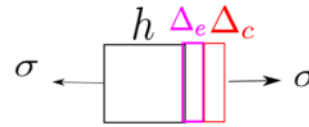
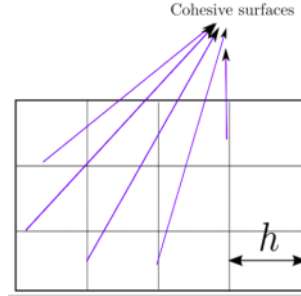
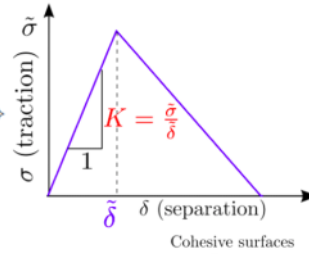
Artificial compliance is,

$$C_c = \frac{1}{Kh} = \frac{\delta}{\sigma h} = \frac{1}{E_c}, \text{ where}$$

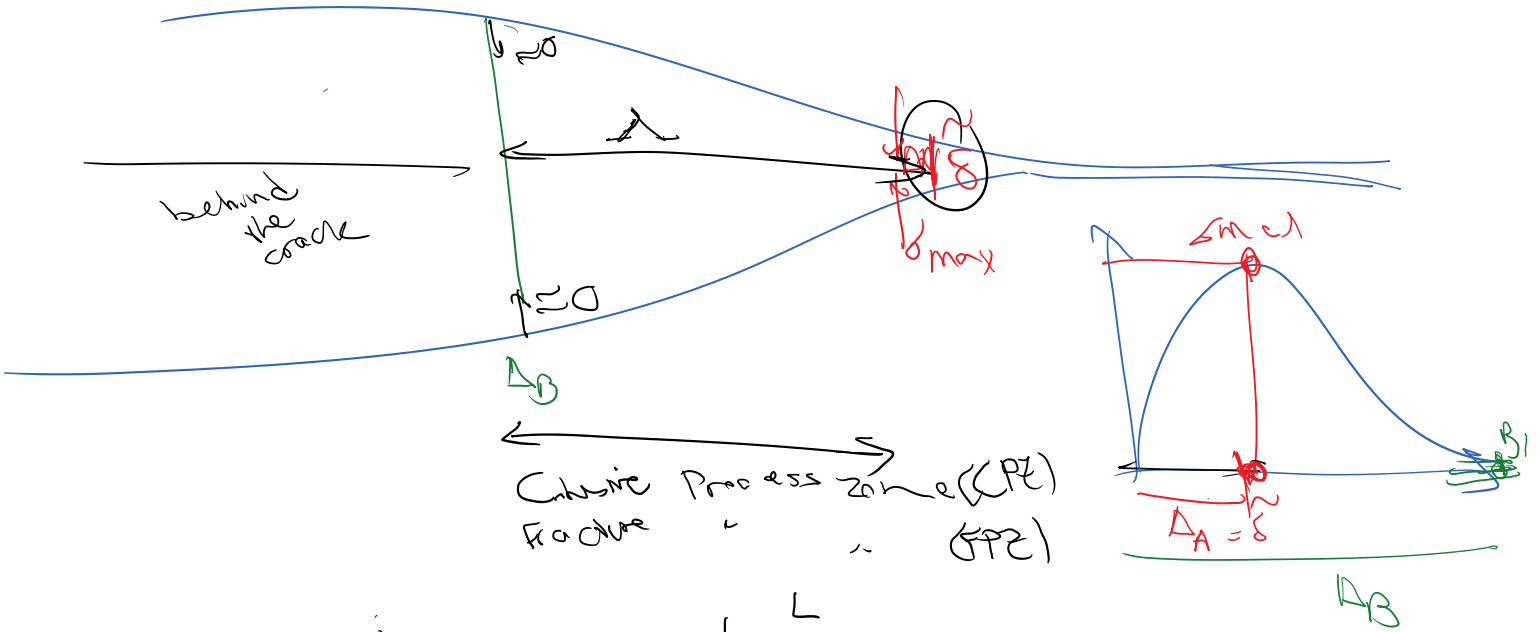
$$E_c = Kh = \frac{\sigma h}{\delta}, \text{ and effective elastic modulus is}$$

$$\frac{1}{E_{\text{eff}}} = \frac{1}{E} + \frac{1}{E_c} \Rightarrow E_{\text{eff}} = \frac{EE_c}{E + E_c}$$

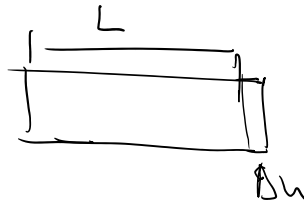
- That is the smaller element spacing h or softer the initial slope K of TSR the higher artificial compliance (higher errors)
- While extrinsic cohesive models do not have the same problem, adaptive insertion of cohesive surfaces is more challenging for them.



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$$\epsilon = \frac{\Delta u}{L}$$



$$\Delta u = \epsilon L$$

change of length

$$u = \epsilon L$$

Maximum of displacement

c, h, p

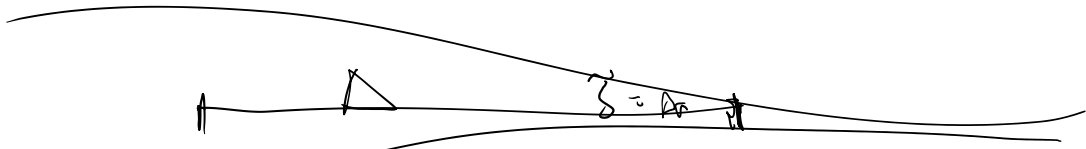
$$\frac{\text{Magnitude of displacement}}{\text{Magnitude of length scales}} = \text{Scale of strain}$$

$$\epsilon = \frac{\delta}{E} \quad \text{for fracture } \delta \rightsquigarrow \delta \text{ strength}$$

$$\boxed{\tilde{\epsilon} = \frac{\tilde{\delta}}{E}} \quad \text{strain scale}$$

$$\tilde{u} = \tilde{\epsilon} L$$

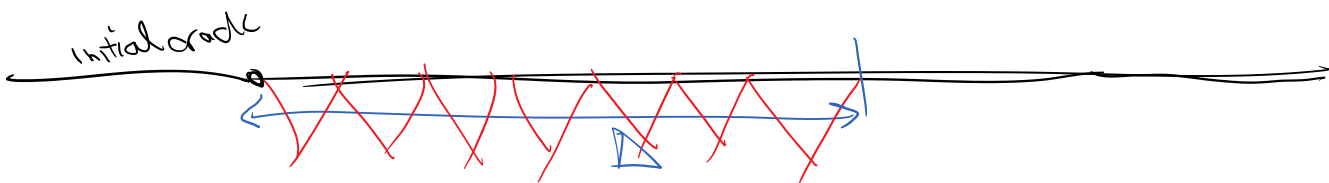
$$\boxed{\tilde{u} = \frac{\tilde{\delta}}{E} L}$$



$$\tilde{\delta} \propto \frac{\tilde{\delta}}{E} \Delta$$

$$\boxed{\Delta \propto \tilde{\delta} \frac{E}{\sigma}}$$

Finite Element



recommendation 6-10 elements across Δ

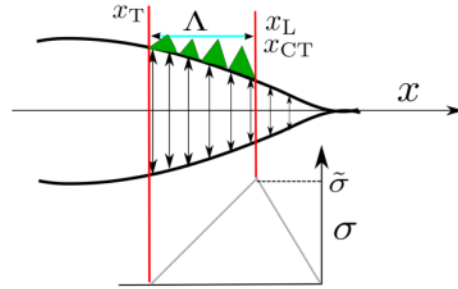
Why process zone size is important?

- Importance of process zone size A

- Static estimate:

$$A = \varsigma \pi \frac{\mu}{1-\nu} \frac{\bar{\phi}}{\bar{\sigma}^2} \propto \bar{L}$$

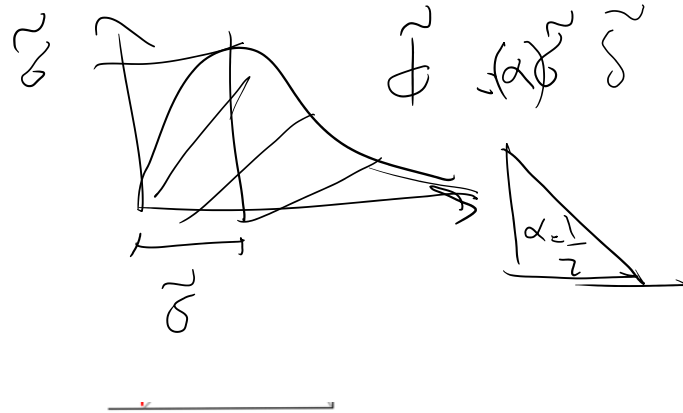
$$\varsigma = \begin{cases} \frac{1}{4} & \text{Dugdale model} \\ \frac{9}{16} & \text{Potential-based TSRs} \end{cases}$$



Handwritten notes:

$$\propto E \frac{\phi}{\sigma^2} \propto \frac{\mu}{\sigma^2}$$

$$\propto \left(\frac{\mu}{\sigma^2} \right) \propto \frac{\mu}{\sigma^2}$$

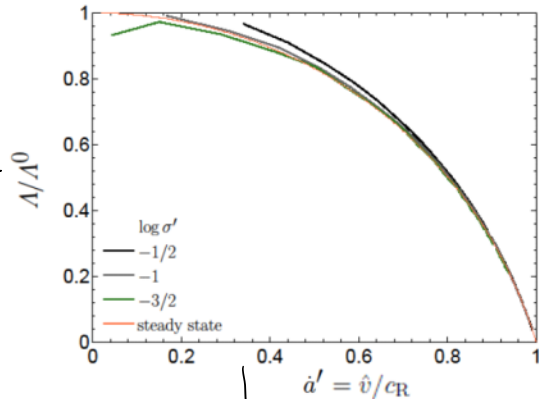
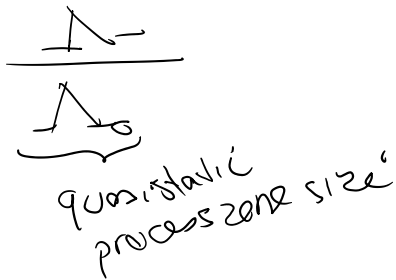


- Minimum number of elements in process zone size:
There should be at least 4-10 elements along the PZ

- Dynamic estimate: PZS decreases as crack speed \hat{v} approaches Rayleigh wave speed c_R

$$A(\hat{v}) = \frac{A}{A(\hat{v})}, \quad A(\hat{v}) \rightarrow 0 \text{ as } \hat{v} \rightarrow c_R \Rightarrow$$

Smaller elements are needed in PZT as crack accelerates!



normalized crack speed

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$\tilde{\phi} = \tilde{\sigma} \tilde{\delta}$ Energy 2 out of the three are independent
 $\tilde{p} = \rho \tilde{v} = \frac{\tilde{\sigma}}{c_d}$ Linear momentum
 $\tilde{E} = \frac{\tilde{v}}{c_d} = \frac{\tilde{\sigma}}{\rho c_d^2} \propto \frac{\tilde{\sigma}}{\|C\|}$ Strain
 $\tilde{v} = \frac{\tilde{\delta}}{\tilde{\tau}} = \frac{\tilde{\sigma}}{\rho c_d}$ Velocity
 $\tilde{L} = c_d \tilde{\tau} = \frac{\rho c_d^2 \tilde{\delta}}{\tilde{\sigma}} \propto \Lambda^0, \tilde{\delta} \frac{\Lambda}{\rho c_d}$ Length **Process zone size Λ**
 $\tilde{\tau} = \frac{\rho c_d \tilde{\delta}}{\tilde{\sigma}}$ Time **Influences time step for time marching methods**

