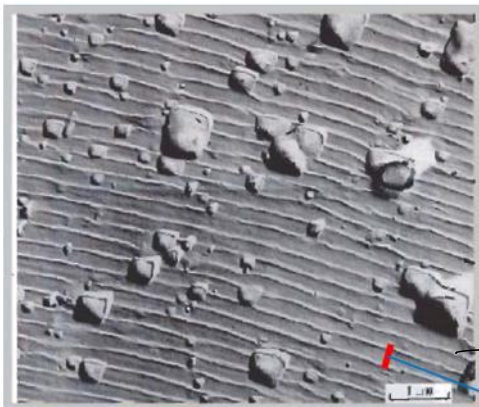
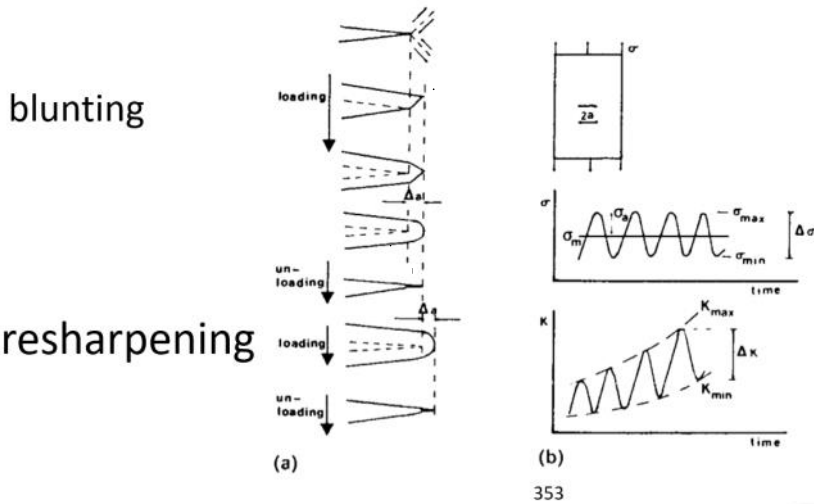


Fatigue fracture is prevalent!

- Deliberately applied load reversals (e.g. rotating systems)
- Vibrations (machine parts)
- Repeated pressurization and depressurization (airplanes)
- Thermal cycling (switching off electronic devices)
- Random forces (ships, vehicles, planes)

(source: Schreurs fracture notes 2012)

Fatigue occurs always and everywhere and is a major source of mechanical failure



Fracture surface of a 2024-T3 aluminum alloy (source S. Suresh MIT)

Incremental crack propagation
 - Each time after sharpening the crack propagates a bit

Fatigue crack growth:

Microcrack formation in **accumulated slip bands** due to repeated loading

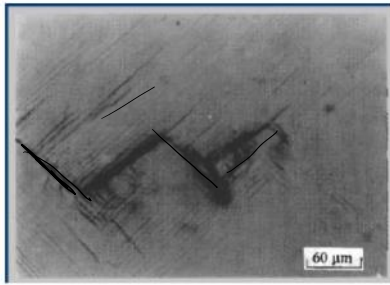
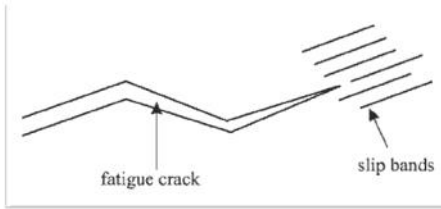
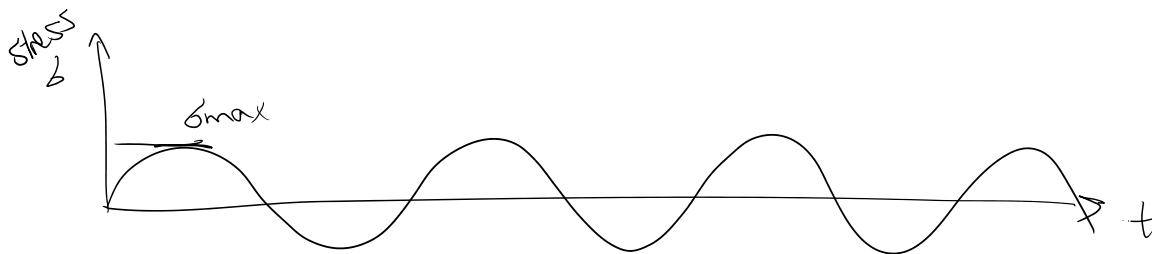
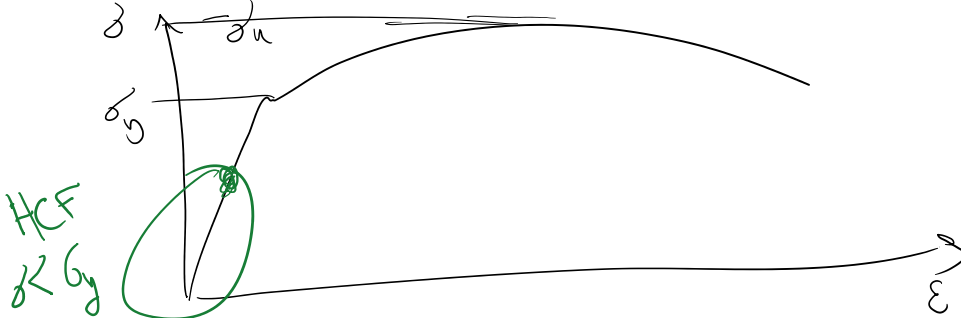


Table 7.1 Classification of fatigue damage

Fatigue	Failure cycles N_R	Pertinent stress	Strain ratio $\Delta\epsilon^p / \Delta\epsilon^e$	Energy ratio $\Delta W^p / \Delta W^e$
VHCF Very high cycle fatigue	$> 10^7$	$< \sigma_F$	≈ 0	≈ 0
HCF High cycle fatigue	10^5 to 10^6	$< \sigma_Y$	≈ 0	≈ 0
LCF Low cycle fatigue	10^2 to 10^4	σ_Y to σ_U	1 to 10	1 to 10
VLCF Very low cycle fatigue	1 to 20	$\approx \sigma_U$	10 to 100	10 to 100

Source: Dufailly and Lemaitre (1995)



HCF $\sigma_{max} < \sigma_y$
 $N \quad 10^6 - 10^7$

$N \ll 10^6 - 10^7$
 Almost all in elastic regime
LEFM model can be used
Stress-based : we study the evolution of stress
 Plastic strain & energy ≈ 0

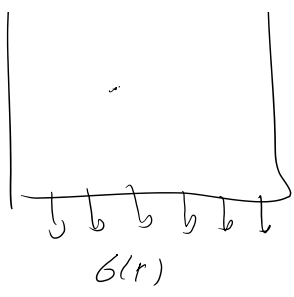
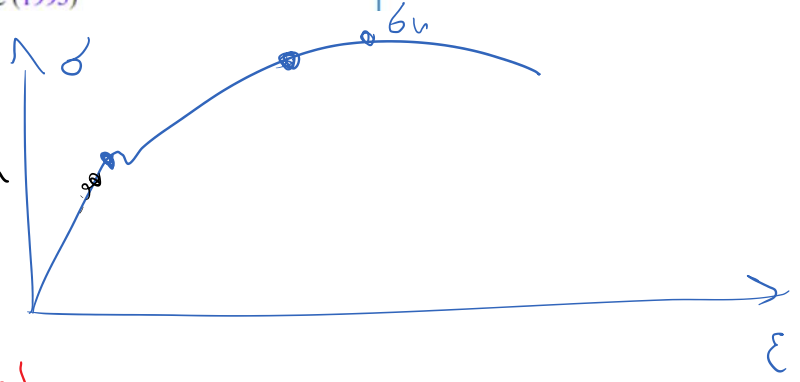


Table 7.1 Classification of fatigue damage

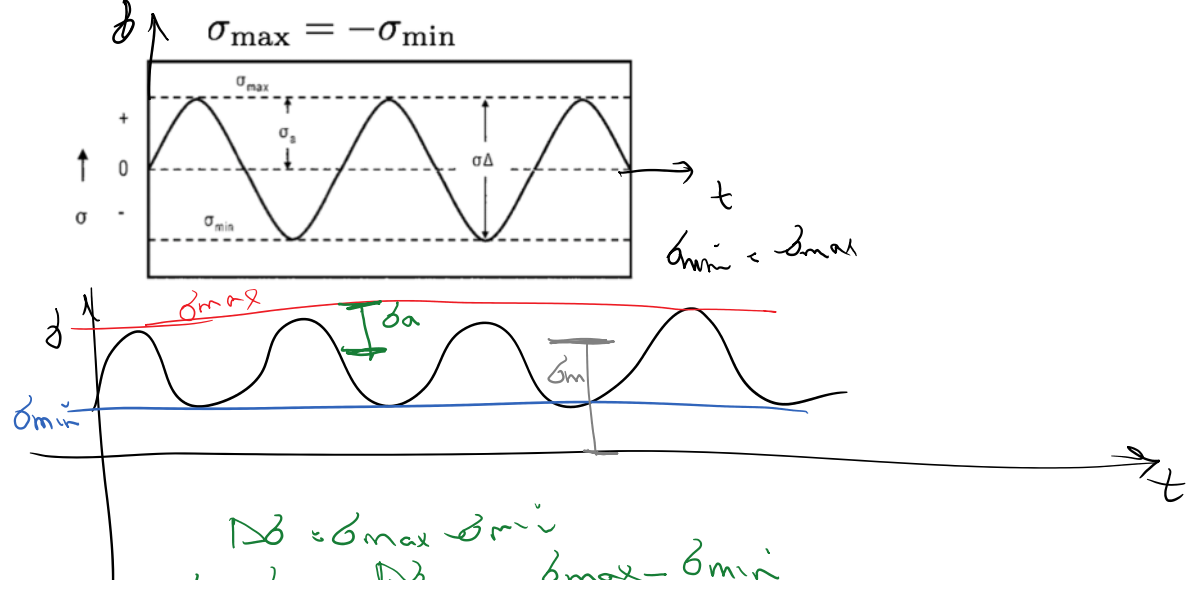
Fatigue	Failure cycles N_R	Pertinent stress	Strain ratio $\Delta \epsilon^p / \Delta \epsilon^e$	Energy ratio $\Delta W^p / \Delta W^e$
Very high cycle fatigue	$> 10^7$	$< \sigma_F$	≈ 0	≈ 0
High cycle fatigue	10^5 to 10^6	$< \sigma_Y$	≈ 0	≈ 0
Low cycle fatigue	10^2 to 10^4	σ_Y to σ_U	1 to 10	1 to 10
Very low cycle fatigue	1 to 20	$\approx \sigma_U$	10 to 100	10 to 100

Source: Dufailly and Lemaitre (1995)

Low cycle fatigue
 — ϵ_p, W^p very high
 — LEFM (K) cannot be used
 — **Strain based**



Baseline fatigue loading:

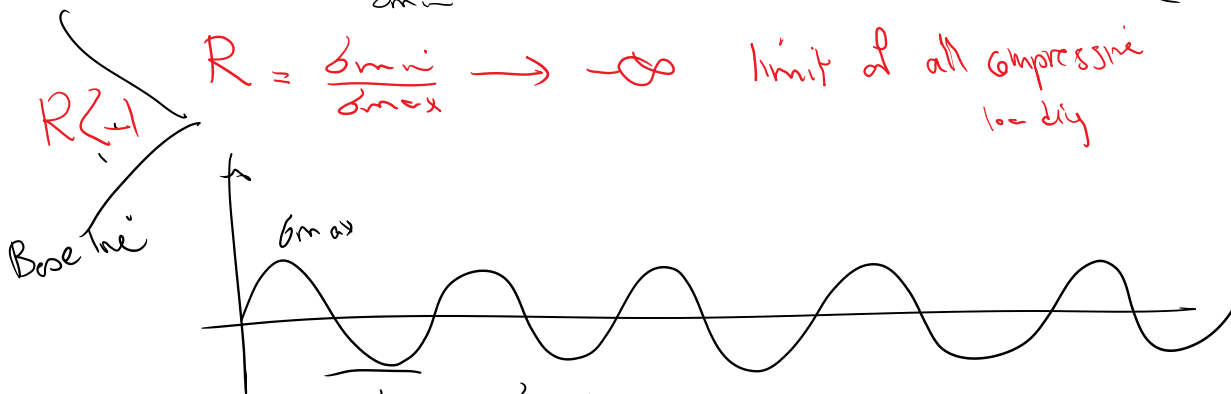
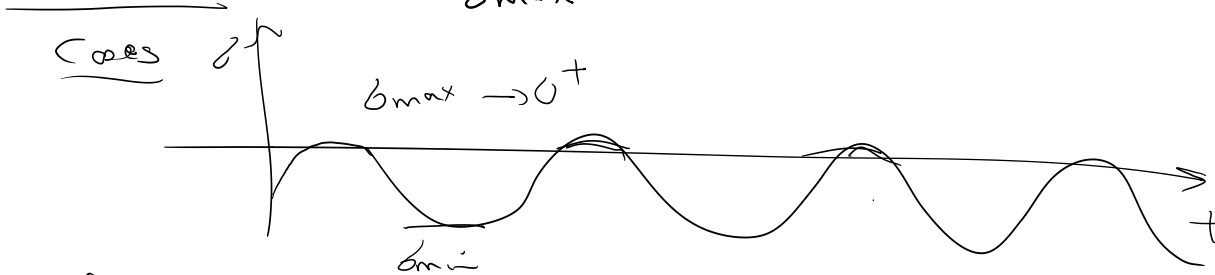


$$\Delta \sigma = \sigma_{max} - \sigma_{min}$$

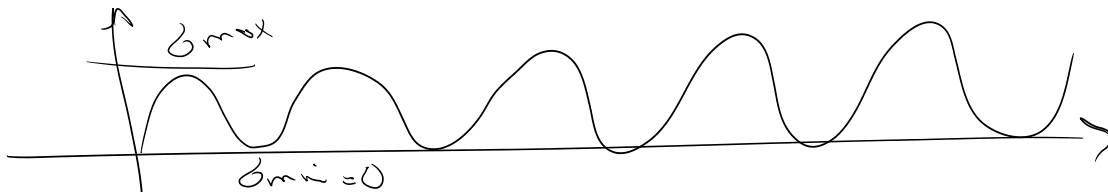
$$\sigma_a = \frac{\Delta \sigma}{2} = \frac{\sigma_{max} - \sigma_{min}}{2}$$

$$\sigma_m = \frac{\sigma_{max} + \sigma_{min}}{2}$$

$$R = \frac{\sigma_{min}}{\sigma_{max}} \quad \text{load ratio}$$

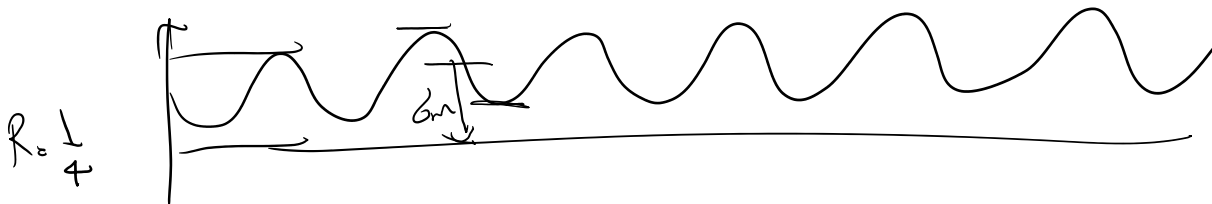


$$R = -1$$



$$R = \frac{\sigma_{min}}{\sigma_{max}} = 0$$

as $\sigma_m \rightarrow R \uparrow$



as $\sigma_m \rightarrow \sigma \quad R \rightarrow 1$

$$\boxed{\text{Re}(-\infty, 1)}$$

$R = -1$ sym base line
 $R = 0$ touching $\sigma = 0$ all tensile

$$\Delta\sigma = \sigma_{\max} - \sigma_{\min}$$

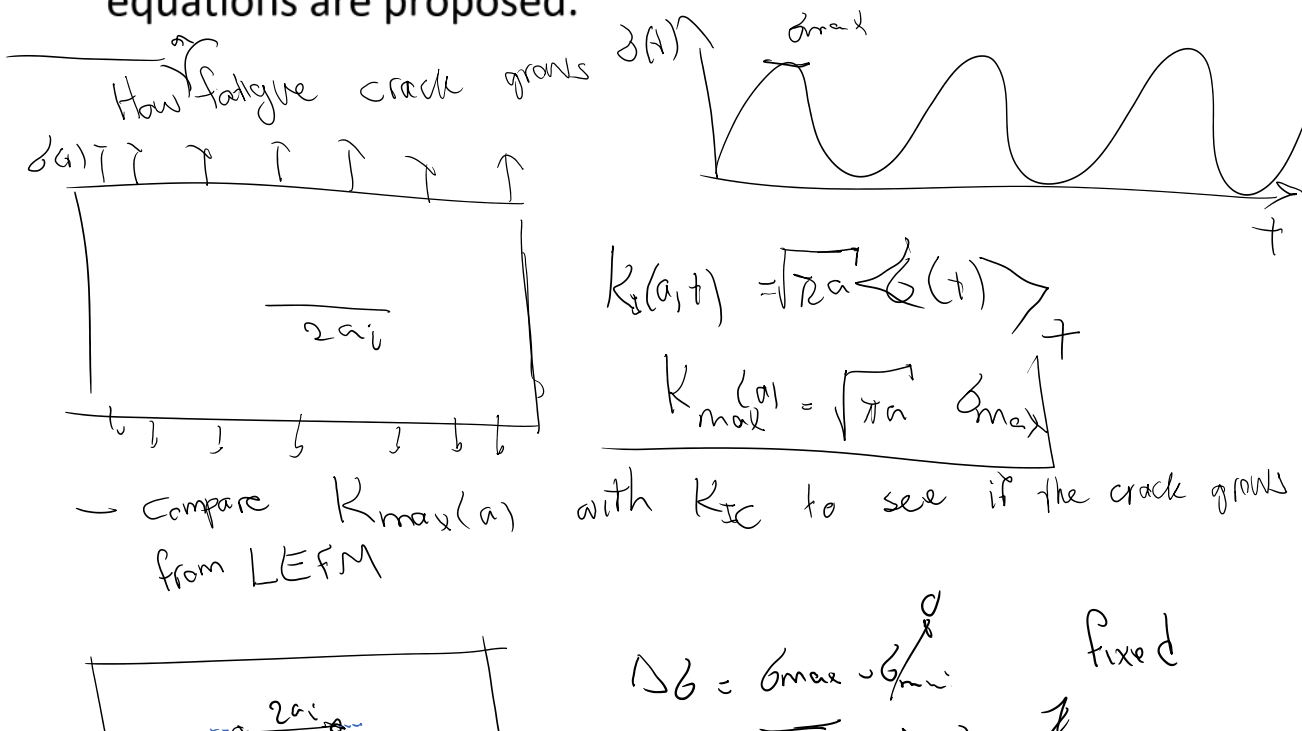
$$\sigma_a = 0.5(\sigma_{\max} - \sigma_{\min})$$

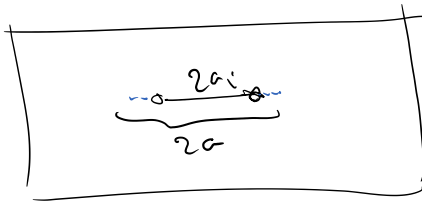
$$\sigma_m = 0.5(\sigma_{\max} + \sigma_{\min})$$

$$R = \frac{\sigma_{\min}}{\sigma_{\max}} \quad \text{load ratio}$$

Cyclic vs. static loadings

- Static: Until K reaches K_c , crack will not grow
- Cyclic: K applied can be well below K_c , crack still grows!!!
- 1961, Paris Erdogan used the theory of LEFM to explain fatigue cracking successfully.
- Methodology: experiments first, then empirical equations are proposed.





$$\Delta\sigma = \sigma_{max} - \sigma_{min}$$

$$\Delta K = \sqrt{\pi a} \Delta\sigma$$

solves

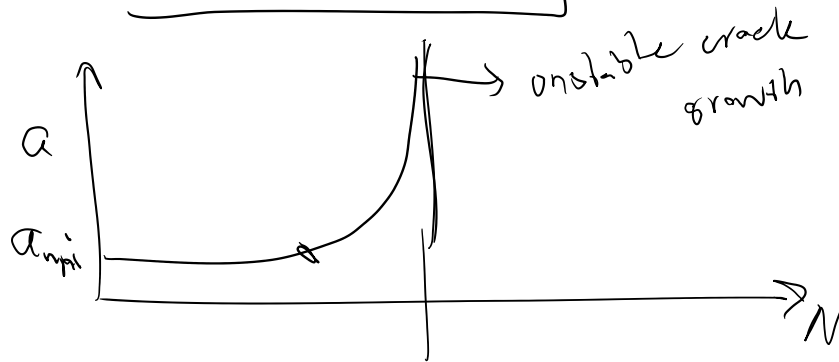
$$K_{max} = \sqrt{\pi a} \sigma_{max}$$

↑ fixed
as a grows

From a_i up to some crack length (a_c) crack grows slowly by fatigue process and a_c it unstably propagates.

$$K_{max}(a_c) = \sqrt{\pi a_c} \sigma_{max} = K_{Ic}$$

$$a_c = \frac{1}{\pi} \left(\frac{K_{Ic}}{\sigma_{max}} \right)^2$$



There are two approaches for modeling fatigue

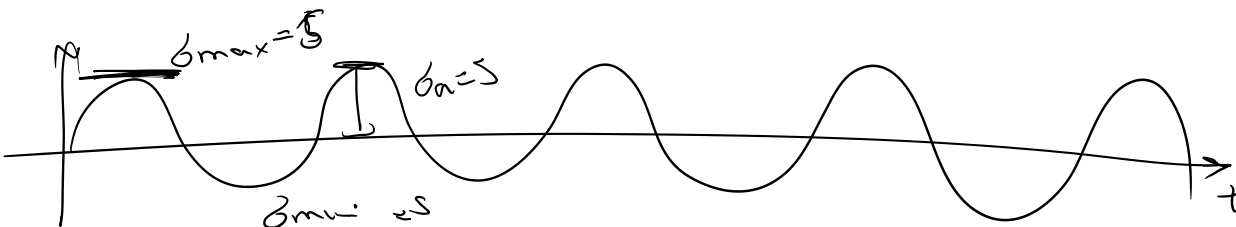
Older approach:

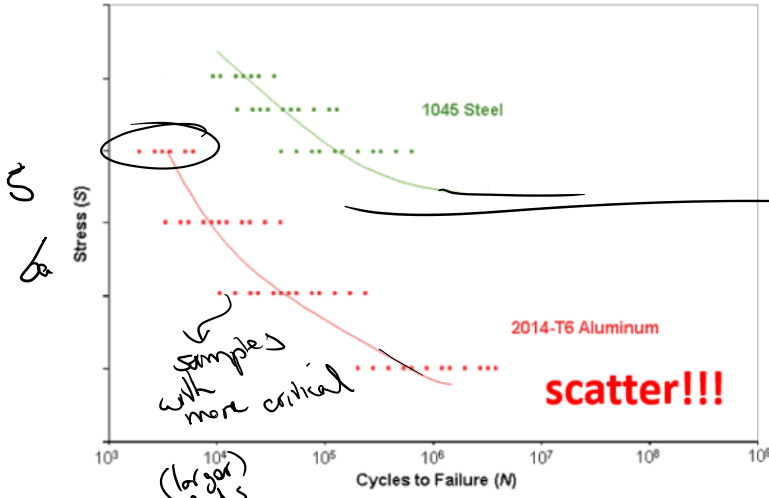
- Stress-based
- Does not incorporate initial material flaws

Newer approach:

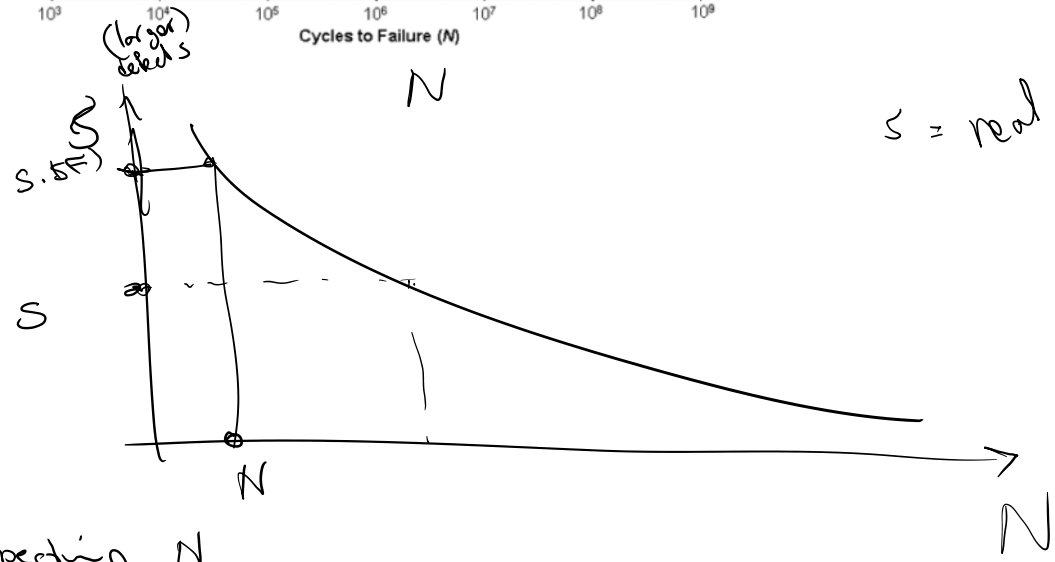
- K-based (SIF-based)
- We start with some initial crack length

uses ↓
S-N or S-N-P plots



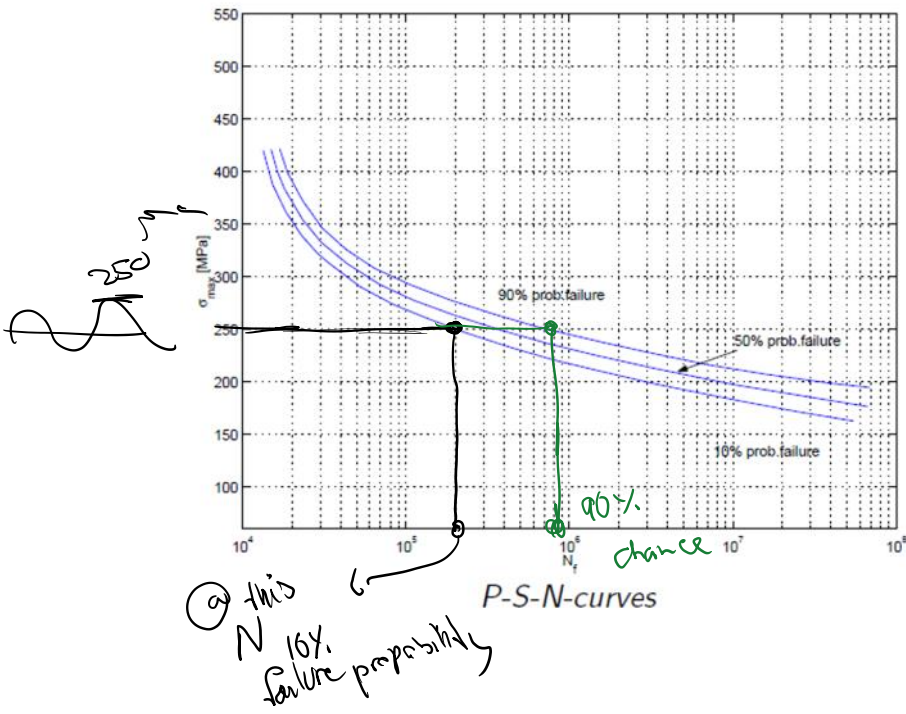


Some like steel have a threshold



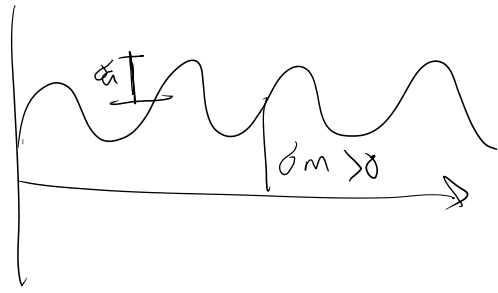
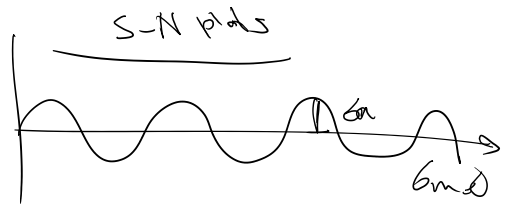
Inspection interval = $\frac{N}{SF_2}$

S-N-P curve: scatter effects

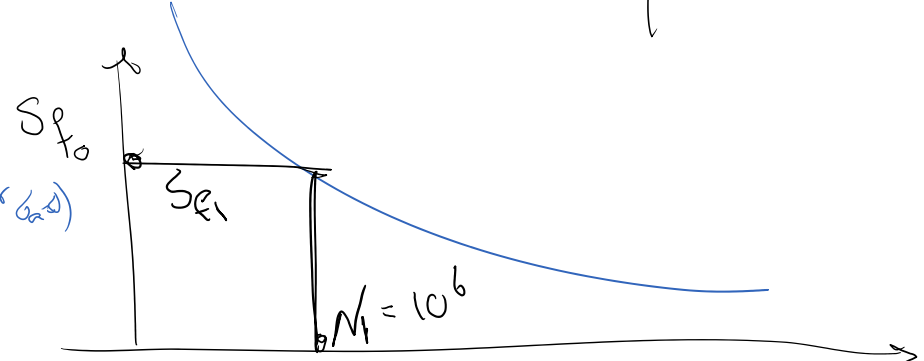


S-N (and S-N-P) plots are calibrated for $R = -1$.

How to use them when $R \neq -1$



get S_{f_0}
 $= (S \text{ for } \sigma_a)$

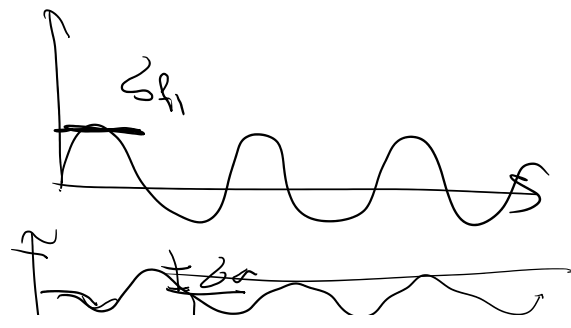


$$\sigma_a = S_{f_0} \left[1 - \left(\frac{\sigma_m}{\sigma_a} \right)^r \right]$$

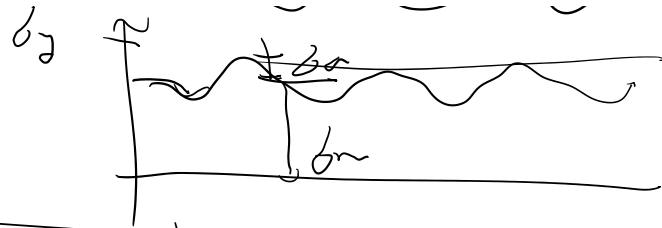
\downarrow or σ_y

$\sigma_m \rightarrow \sigma_y$, $\sigma_a \rightarrow 0$

σ_y , σ_p

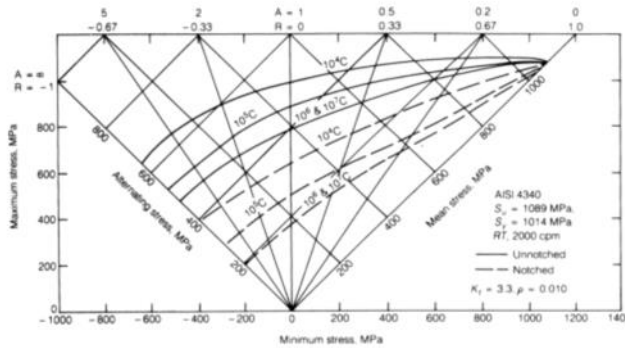
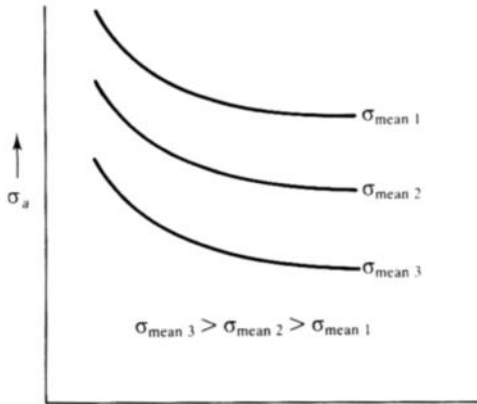


$$\sigma_a = -2 \sigma_{f1}$$



So as $\sigma_m \rightarrow \sigma_y$ $\sigma_a \rightarrow 0$ and we tend to
 σ_{bu}
 steady state yielding failure

Effect of mean stress



Approach 1:
Master diagram

$$R = \frac{\sigma_{min}}{\sigma_{max}}$$

$$A = \frac{\sigma_a}{\sigma_m} = \frac{1-R}{1+R}$$

Approach 2:

Correction-factor formulas

$$\sigma_a = \sigma_{f0} \left[1 - \left(\frac{\sigma_m}{\sigma_u} \right)^r \right]$$

where σ_a is the amplitude of allowable stress (alternating stress).

σ_{f0} is the stress at fatigue fracture when the material under zero mean stress cyclic loading

σ_m is the mean stress of the actual loading.

σ_u is the tensile strength of the material.

$r = 1$ is called Goodman line which is close to the results of notched specimens.

$r = 2$ is the Gerber parabola which better represents ductile metals.

Other correction factor

Gerber (1874) $\frac{\sigma_a^*}{\sigma_u} = 1 - \left(\frac{\sigma_m}{\sigma_u} \right)^2$

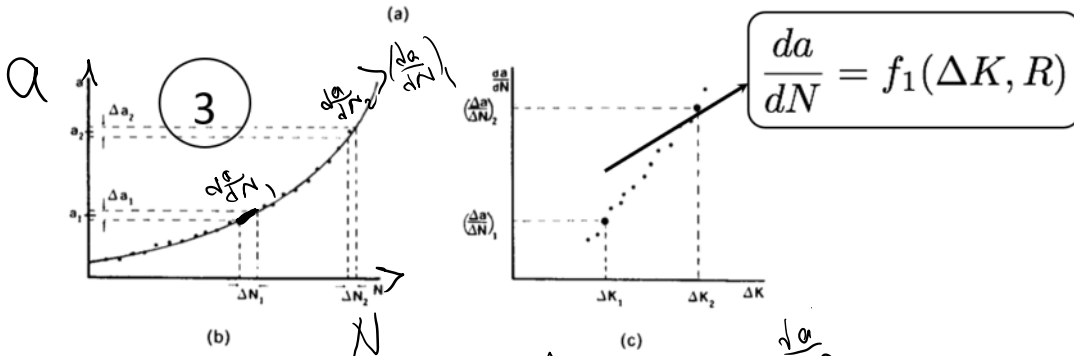
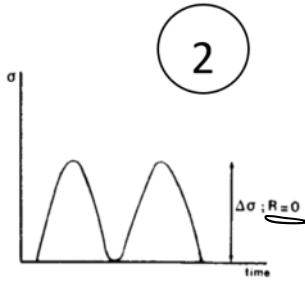
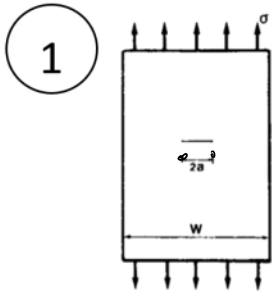
Goodman (1899) $\frac{\sigma_a^*}{\sigma_u} = 1 - \frac{\sigma_m}{\sigma_u}$

Soderberg (1939) $\frac{\sigma_a^*}{\sigma_u} = 1 - \frac{\sigma_m}{\sigma_{y0}}$

New Approach

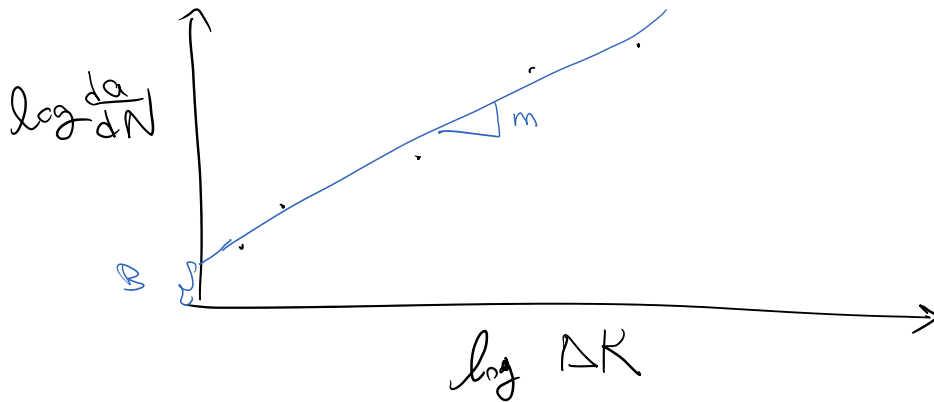
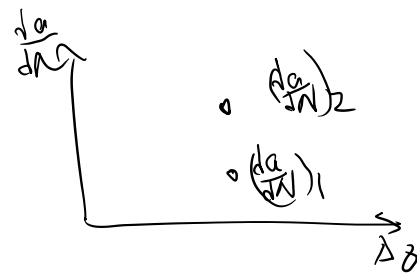
Crack growth data

$$K = \sigma \sqrt{\pi a}$$



Hypothesis $\frac{da}{dN}$ depends on $\Delta \sigma$

$\frac{da}{dN}$ should be characterized with something other than $\Delta \sigma$



$$\log \frac{da}{dN} = B + m \log \Delta K$$

$$e \left(\frac{da}{dN} \right) = e \left(e^B \cdot e^{m \log \Delta K} \right)$$

$$\frac{da}{dN} = e^B \cdot e^{m \log \Delta K} = \frac{e^B}{c} (\Delta K)^m$$

$da \propto \Delta K^m$ Paris-Erdogan

$$\frac{da}{dN} = C \Delta K^m \quad \text{Paris - Erdogan relation}$$

$$K = \sigma \sqrt{\pi a} Y$$

$$\Delta K = \Delta \sigma \sqrt{\pi c}$$

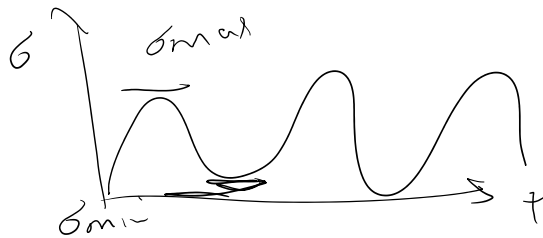
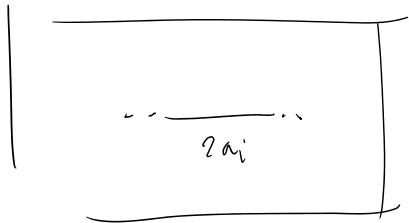
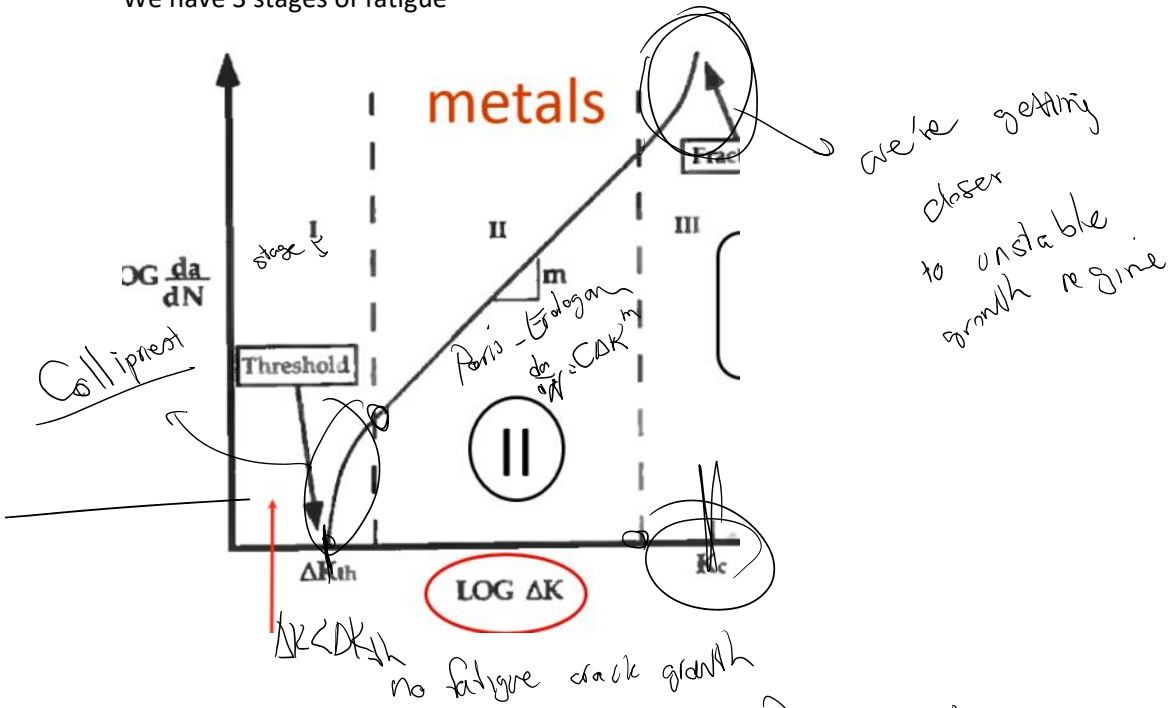
$$\left[\frac{da}{dN} \right] = L \quad \text{length} \quad \left\{ \rightarrow C \text{ is dimensional} \right.$$

$$\left[\Delta K \right]^m \rightarrow \left[\sigma \sqrt{L} \right]^m$$

We need to clearly mention what units are used to calculate C

* m is constant for a material

We have 3 stages of fatigue



$$K_{max} = \sigma_{max} \sqrt{\pi a}$$

$$K_{\max} = \sigma_{\max} \sqrt{\pi a}$$

stage 3 ;

$$K_{\max} = \sigma_{\max} \sqrt{\pi c} = K_{\sigma}$$

$$\sigma_c^2 = \frac{1}{\pi} \left(\frac{K_c}{\sigma_{\max}} \right)^2$$