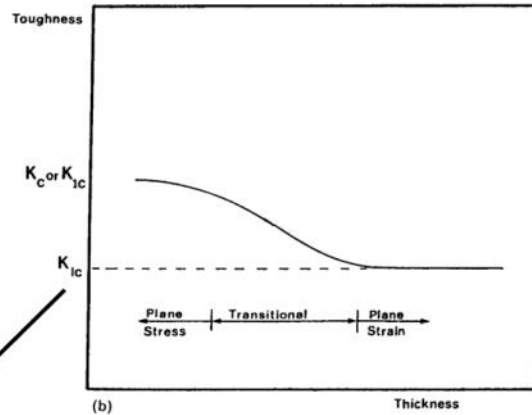
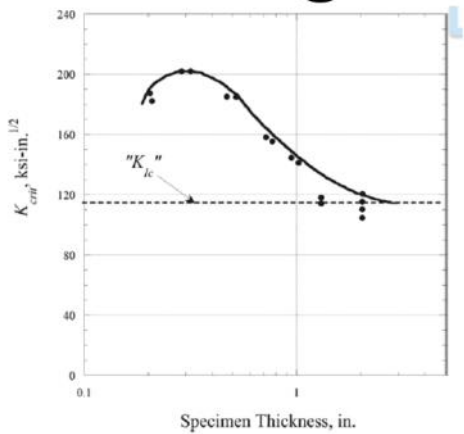


# Toughness vs. thickness



**Plane strain fracture toughness (safe) lowest K**

(Irwin) 
$$K_c = K_{Ic} \left( 1 + \frac{1.4}{B^2} \left[ \frac{K_{Ic}}{\sigma_{ys}} \right]^4 \right)^{1/2}$$

Note that  $\frac{1}{B^2} \left[ \frac{K}{\sigma_{ys}} \right]^4 \propto \left( \frac{r_p}{B} \right)^2$

if  $K = K_{Ic}$  corresponding  $r_p = \left( \frac{K_{Ic}}{\sigma_{ys}} \right)^2$

plane strain critical SF

as  $r_p/B \rightarrow 0$  ( $B \rightarrow \infty$ ; p. strain)  $K_c \rightarrow K_{Ic}$

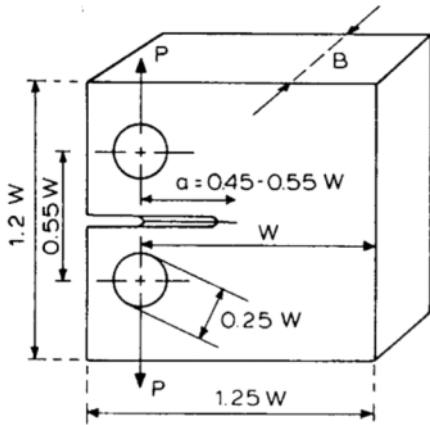
# Fracture toughness tests

- Prediction of failure in real-world applications: need the value of fracture toughness
- Tests on cracked samples: **PLANE STRAIN condition!!!**

Compact Tension Test

$$K_I = \frac{P}{B\sqrt{W}} \frac{\left(2 + \frac{a}{W}\right) \left[0.886 + 4.64 \frac{a}{W} - 13.32 \left(\frac{a}{W}\right)^2 + 14.72 \left(\frac{a}{W}\right)^3 - 5.6 \left(\frac{a}{W}\right)^4\right]}{\left(1 - \frac{a}{W}\right)^{3/2}}$$

$a, B, (W-a) \geq 2.5 \left(\frac{K_{Ic}}{\sigma_{YS}}\right)^2$



ASTM (based on Irwin's model) for plane strain condition:

$$a, B, (W - a) \geq 2.5 \left( \frac{K_{Ic}}{\sigma_Y} \right)^2$$

P. strain toughness

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## 5.3. J Integral (Rice 1958)

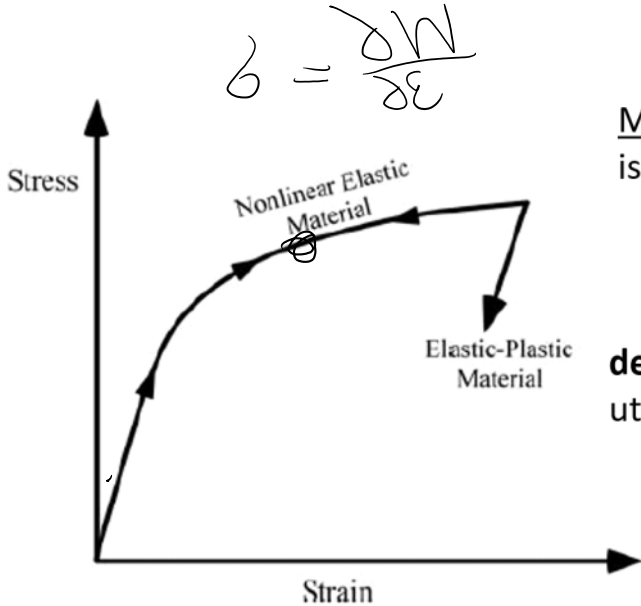
	LEFM (good when SSY holds)	PFM & NLFM
Global perspective "energy"	$G = \frac{-d\Pi}{dA}$ energy release rate crack growth $G = R$	$G = J$ J integral
Local perspective "stress", ...	 $\sigma_{ij}^I = \frac{\sqrt{K_I}}{\sqrt{2\pi r}} f_{ij}^I(\theta)$ pure mode I	$\sigma_{yy} = J^{1/2} r^{-1/2}$
relationship	$k^2 + k_{II}^2$	...

relationship

$$G = \frac{k_I^2 + k_{II}^2}{E'} \text{ in plane}$$

**Idea**

Replace complicated plastic model with nonlinear elasticity (no unloading)



Monotonic loading: an elastic-plastic material is equivalent to a nonlinear elastic material

deformation theory of plasticity can be utilized

For now Linear Elasticity  $\rightarrow$  Nonlinear Elasticity

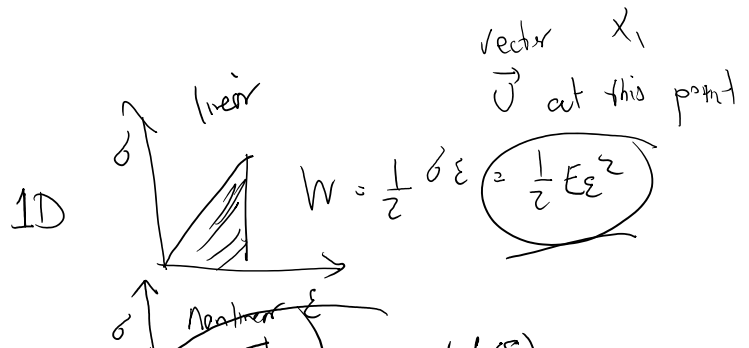
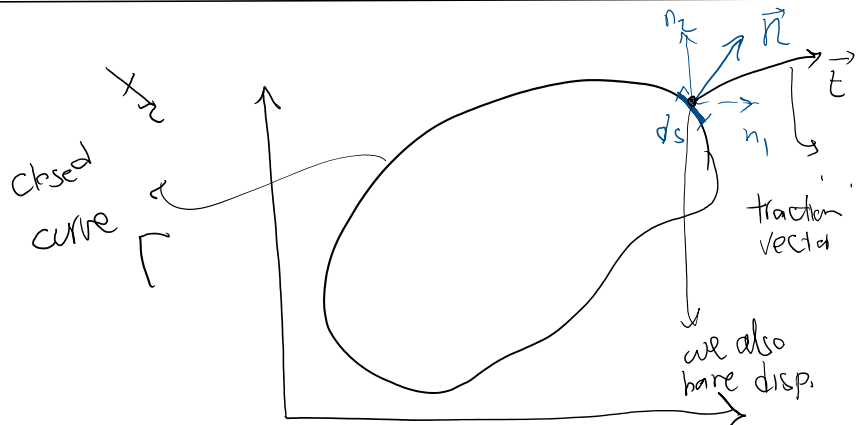
NL elasticity

Eshelby, Cherepanov 1967

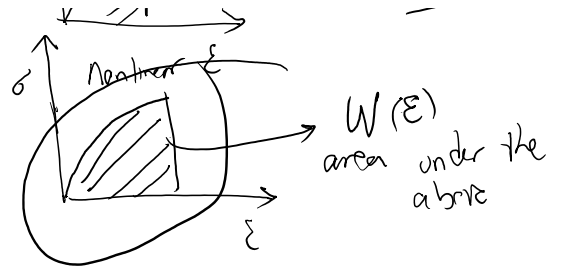
$$J_k = \int (W n_k - t \cdot \frac{\partial u}{\partial x_k}) ds$$

for  $k=1 \& 2$

$W$ : Strain energy density



$W$ : Strain energy density



$$d(\epsilon) = \frac{dW(\epsilon)}{d\epsilon}$$

$k=1$

$$J_1 = \int_{\Gamma} (W n_1 - t_i \frac{\partial u_i}{\partial x_1}) ds$$

$$J_2 = \int_{\Gamma} (W n_2 - t_i \frac{\partial u_i}{\partial x_2}) ds$$

2D/3D linear

$$W = \frac{1}{2} \sigma : \epsilon = \frac{1}{2} \epsilon : C \epsilon$$

in general

$$= \frac{1}{2} \sigma_{ij} \epsilon_{ij} = \frac{1}{2} \sum_{i=1}^d \sum_{j=1}^d \sigma_{ij} \epsilon_{ij}$$

$$\sigma = \frac{\partial W(\epsilon)}{\partial \epsilon}$$

$$\sigma_{ij} = \frac{\partial W(\epsilon)}{\partial \epsilon_{ij}}$$

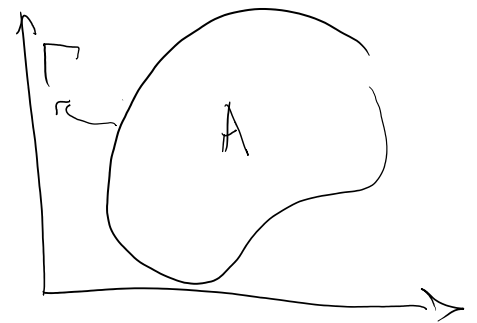
Eshelby and Cherpanov showed that this integral ( $J_1$  and  $J_2$ ) is zero over closed path  $\Gamma$  for smooth solutions of elastostatic problems with **no body force**.

Equation of motion

$$\nabla \cdot \sigma + \rho b = \rho \ddot{u}$$

acceleration

$\nabla \cdot \sigma = 0$  elastostatic  
no body force



1) Proof of why  $J = 0$  over a closed path:

$$J_k = \int_{\Gamma} (W(\epsilon) n_k - t_i \frac{\partial u_i}{\partial x_k}) ds$$

$\vec{t} \cdot \vec{u}$

FYI  $a_i, b_i$

$\vec{T}$ 

$$\vec{T} \cdot \frac{\partial \vec{u}}{\partial x_k}$$

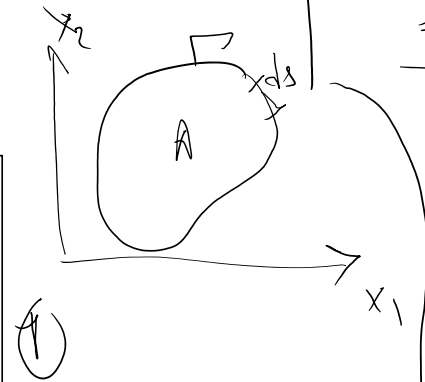
$$t_i = \delta_{ij} n_j \quad \vec{T} = \vec{\sigma} n$$

FYI  $a_i b_i$   
 dimension  $\rightarrow$   
 $= \sum_{i=1}^d a_i b_i$   
 $t_i \frac{\partial u_i}{\partial x_k} = \sum_{i=1}^2 t_i \frac{\partial u_i}{\partial x_k}$

$$J_k = \int_{\Gamma} \left[ W(\epsilon) n_k - \underbrace{(\delta_{ij} n_j)}_{t_i} \frac{\partial u_i}{\partial x_k} \right] ds$$

We'll use divergence theorem

$$J_k = \int_A \left[ \underbrace{\frac{\partial W}{\partial x_k}}_{(1)} - \underbrace{\frac{\partial}{\partial x_j} (\delta_{ij} \frac{\partial u_i}{\partial x_k})}_{(2)} \right] dV$$



$t_i = \delta_{ij} n_j$   
 $= \sum_{j=1}^2 \delta_{ij} n_j$

$\int_{\Gamma} f_i n_i ds =$   
 $\int_A \frac{\partial f_i}{\partial x_i} dV$   
 general form of Dir theorem

$$W_{,k} = \frac{\partial W(\epsilon)}{\partial x_k} = \frac{\partial W(\epsilon)}{\partial \epsilon_{mn}} \frac{\partial \epsilon_{mn}}{\partial x_k}$$

$$\delta_{mn} \left( \frac{\partial}{\partial x_k} \left( \frac{u_{m,n} + u_{n,m}}{2} \right) \right) = \delta_{mn} \left( \frac{u_{,m,nk} + u_{,n,mk}}{2} \right)$$

we know  $\delta_{mn} = \delta_{nm}$  (stress tensor is symmetric)

$$\Rightarrow W_{,k} = \delta_{mn} u_{m,nk} \quad \text{term 1}$$

$$(2) \quad \frac{\partial}{\partial x_j} (\delta_{ij} \frac{\partial u_i}{\partial x_k}) = \underbrace{\frac{\partial \delta_{ij}}{\partial x_j}}_{\leftarrow + \rightarrow} \frac{\partial u_i}{\partial x_k} + \delta_{ij} \frac{\partial^2 u_i}{\partial x_j \partial x_k}$$

$$\delta x_j \delta x_k \quad \delta x_j \delta x_k \quad \delta x_j \delta x_k$$

$$(\nabla \cdot \sigma)_i = 0$$

equation of motion ( $\ddot{u}$  &  $b=0$ )

$$\frac{\partial}{\partial x_j} \left( \delta_{ij} \frac{\partial u_i}{\partial x_k} \right) = \delta_{ij} \frac{\partial^2 u_i}{\partial x_j \partial x_k} \quad \text{term 2}$$

plug terms 1 & 2 in equation (1)

$$J_k = \int_A \left[ \underbrace{\frac{\partial W}{\partial x_k}}_{(1)} - \underbrace{\frac{\partial}{\partial x_j} \left( \delta_{ij} \frac{\partial u_i}{\partial x_k} \right)}_{(2)} \right] dV \quad (1)$$

$$J_k = \int_A \left( \sigma_{mn} u_{m,nk} - \delta_{ij} u_{i,jk} \right) dV$$

$\downarrow \downarrow$   
 $m \quad n$

$$= \int_A \left( \sigma_{mn} u_{m,nk} - \sigma_{mn} u_{m,nk} \right) dV = 0$$

$$J_k = 0 \quad \text{☺}$$

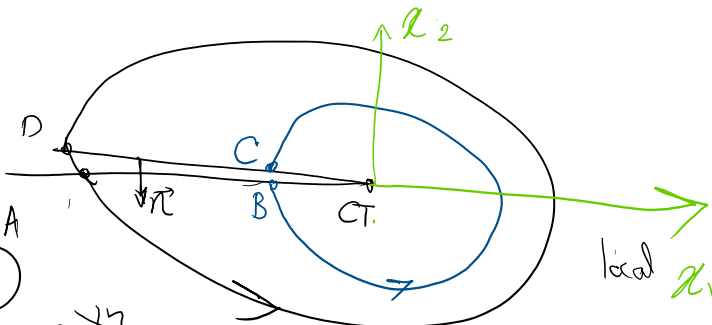
Recap: Eshelby and Cherpanov showed that  $J_k = 0$  over closed curves for C1 smooth solutions

Second part (2):


We'll show that J integral is equal over any closed curve around the crack tip

$$J_{AD} = \int_A^D (u_{i,j} \sigma_{ij}) ds$$

$$J_{AD} = J_{BC} \neq \text{○}$$



value does not depend on path

we're doing this for  $J_1$  

$$J_{ADCBA} = 0 = J_{AD} + J_{DC} + J_{CB} + J_{BA}$$

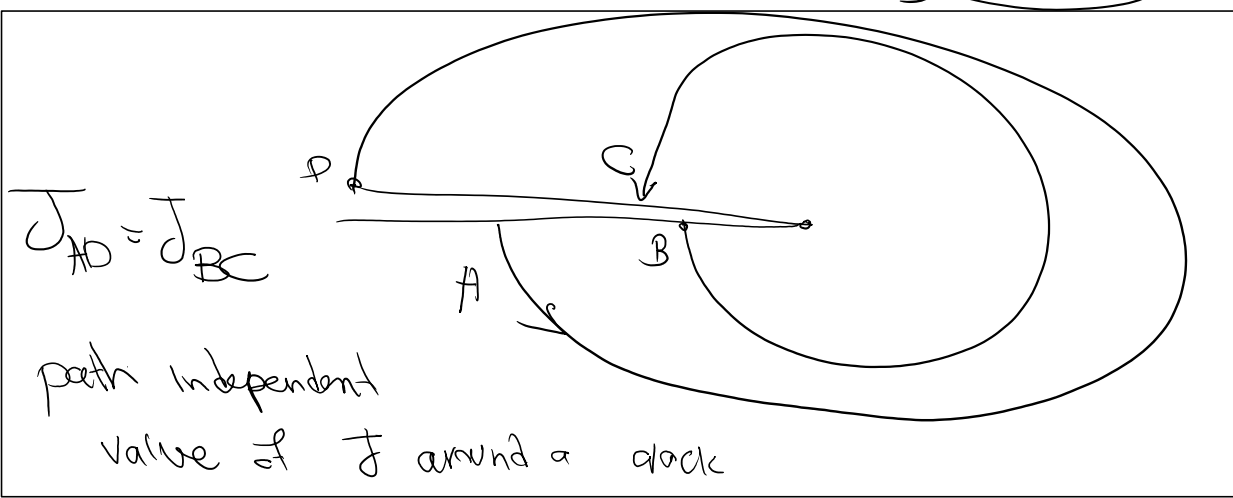
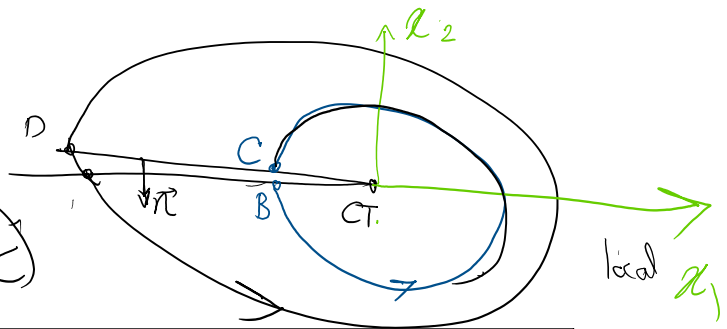
$J_{DC} = \int_C^D (W/h_1 - \vec{t} \cdot \frac{\partial \vec{u}}{\partial x_1}) ds = 0$  if we have TRACTION FREE CRACK SURFACE

Similarly  $J_{BA} = 0$

why these two are zero

$$J_{AD} + J_{CB} = 0$$

$$\Rightarrow J_{AD} = -J_{CB} = (-J_{BC})$$



Last step, shown by Rice,

$$J = G \quad (J_{AD} = J_{BC} = G)$$

Why  $J = G$ ?

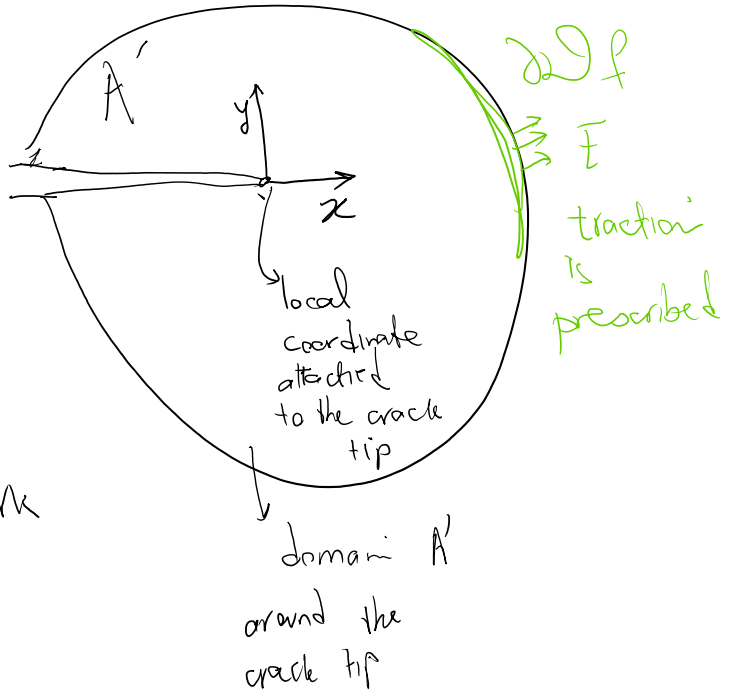
<https://rezaabedi.com/wp-content/uploads/Courses/FractureMechanics/J=G.pdf>

Why  $J = G$ ?

<https://rezaabedi.com/wp-content/uploads/Courses/FractureMechanics/J=G.pdf>

$$G = - \frac{d\Pi}{dA} = - \frac{d\Pi}{B da}$$

crack surface
crack length



$$\Pi = \underbrace{U_e}_{\text{internal energy}} - \underbrace{W_{\text{ext}}}_{\text{external work}}$$

$$U_e = \int_{A'} W(\epsilon) dV$$

strain energy density

$$W_{\text{ext}} = \int_{A'} u \cdot p / b \, dV + \int_{\partial\Omega_f} \bar{u} \bar{t} \, ds$$

$$\Rightarrow G = \frac{-1}{B} \frac{d}{da} \left( \underbrace{\Pi}_{U_e - W_{\text{ext}}} \right) = \frac{-1}{B} \frac{d}{da} \left( \int_{A'} W(\epsilon) dV - \int_{\partial\Omega_f} \bar{u} \bar{t} \, ds \right)$$