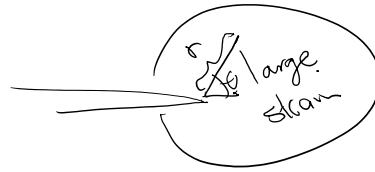


Ramberg-Osgood model

From last time, around the crack tip we had:

$$\epsilon \propto \sigma^n$$



around the crack tip

$$\sigma = \frac{C_1}{r^\alpha}$$

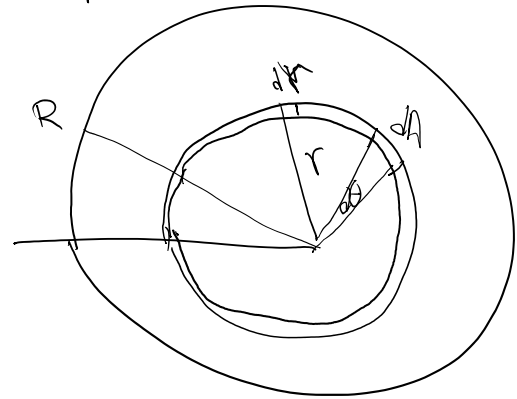
$$\epsilon = \frac{C_2}{r^\beta}$$

strain energy in circle of radius

$$E(R) = \int W(\epsilon) dA$$

$$\propto \int_0^{2\pi} \int_0^R \underbrace{\frac{1}{r^\alpha}}_{\text{stress}} \underbrace{\frac{1}{r^\beta}}_{\text{strain}} \underbrace{dA}_{(r dr d\theta)}$$

$$= \int_0^{2\pi} \int_0^R r^{1-\alpha-\beta} \underbrace{dr d\theta}_{\text{Power}}$$



For the energy to be finite & have certain R dependency we require

$$1 - \alpha - \beta > 0 \quad \alpha + \beta < 1$$

In fact

$$\boxed{\alpha + \beta = 1} \quad (i)$$

$$\begin{cases} \textcircled{2} \quad \epsilon \propto \sigma^n \\ \epsilon = \frac{C_2}{r^\beta}, \quad \sigma = \frac{C_1}{r^\alpha} \end{cases} \implies r^{-\beta} = r^{-n\alpha} \implies \boxed{y = n\alpha} \quad (ii)$$

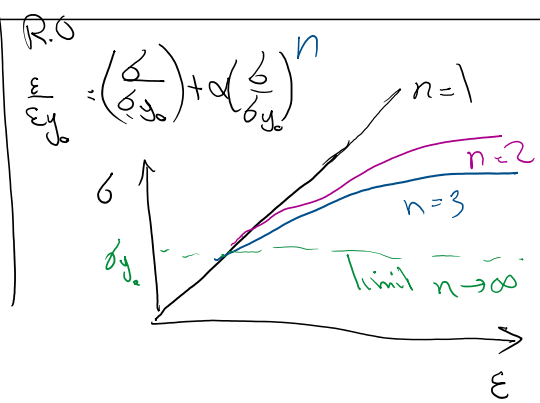
$$\textcircled{i}, \textcircled{ii} \implies \boxed{y = \frac{n}{n+1} \quad \sigma \propto \frac{1}{r}, \quad \epsilon \sim 1}$$

☺☺

①  $y = \frac{1}{n+1}$       $\sigma \propto \frac{1}{r^{\frac{1}{n+1}}}$       $\epsilon \propto \frac{1}{r^{\frac{n}{n+1}}}$   
 $\kappa = \frac{1}{n+1}$

$n=1$ :  
Linear elastic

$\sigma, \epsilon \propto \frac{1}{\sqrt{r}}$  LEFM



LEFM  $\sigma_{ij}^I(r, \theta) = \frac{K_I}{\sqrt{2\pi r}} f_{ij}^I(\theta)$

$n \rightarrow \infty$   $\sigma$  bounded!

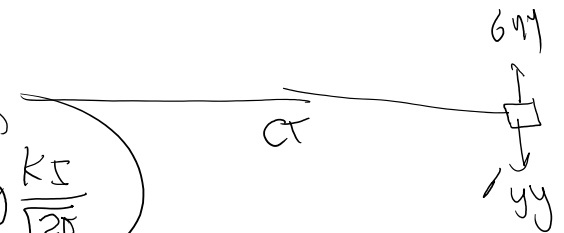
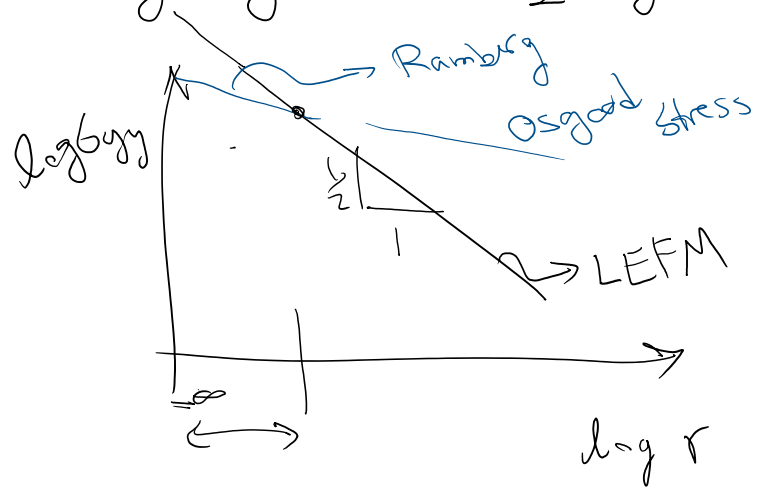
$\epsilon \propto \frac{1}{r}$  all singularity in strain

$1 < n < \infty$   $\epsilon$  is always more singular

LEFM

$\sigma_{yy}(r, \theta=0) = \frac{K_I}{\sqrt{2\pi r}}$

$\log \sigma_{yy} = -\frac{1}{2} \log r + \log \frac{K_I}{\sqrt{2\pi}}$

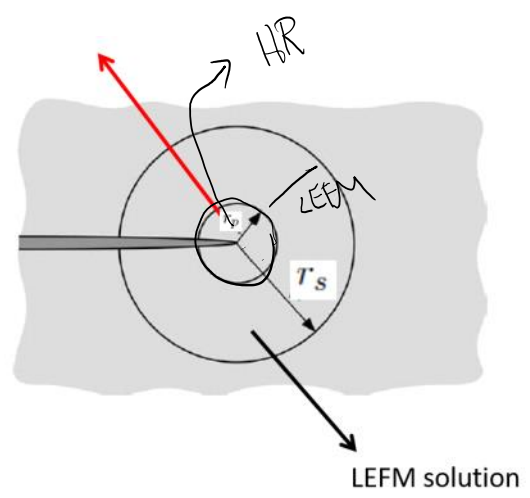
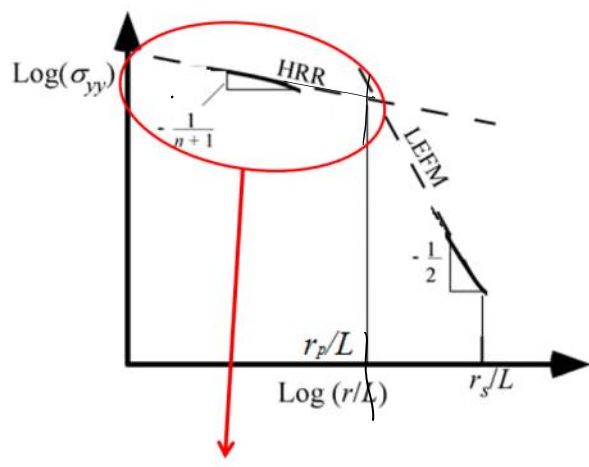


Ramberg - Osgood

$\sigma_{yy} \propto \frac{1}{r^{\frac{1}{1+n}}}$

$\log \sigma_{yy} = \frac{-1}{1+n} \log r + C$   
constant

0 0 0 1+n 0 0  
 condition



Stress is still singular but with a weaker power of singularity!

$$\sigma \propto \frac{1}{r^{1/(n+1)}} \quad \epsilon \propto \frac{1}{r^{n/(n+1)}}$$

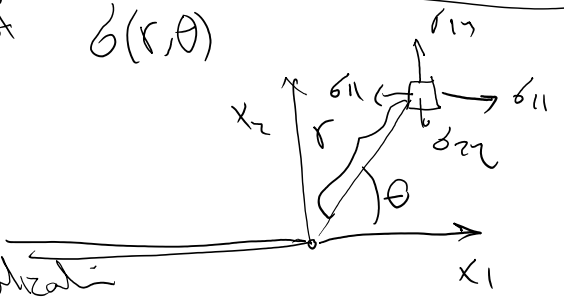
What about the full expression of  $\sigma(r, \theta)$

plays the role of  $K$

$$\sigma_{ij} = \sigma_0 \left( \frac{EJ}{\alpha \sigma_0^2 I_n r} \right)^{1/(n+1)} \bar{\sigma}_{ij}(n, \theta)$$

$$\epsilon_{ij} = \frac{\alpha \sigma_0}{E} \left( \frac{EJ}{\alpha \sigma_0^2 I_n r} \right)^{n/(n+1)} \bar{\epsilon}_{ij}(n, \theta)$$

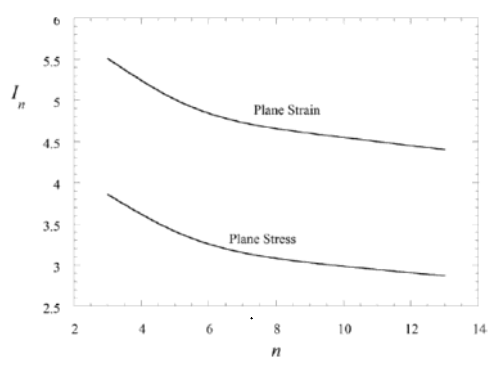
generalization of



LEFM

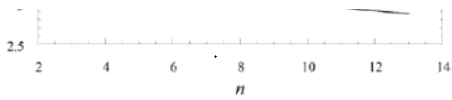
$$\sigma_{ij}^I(r, \theta) = \frac{K_I}{2\pi r^{1/2}} f_{ij}^I(\theta)$$

$$RO \left( \frac{\epsilon}{\epsilon_{y_2}} \right) = \left( \frac{\sigma}{\sigma_0} \right) + \alpha \left( \frac{\sigma}{\sigma_0} \right)^n$$

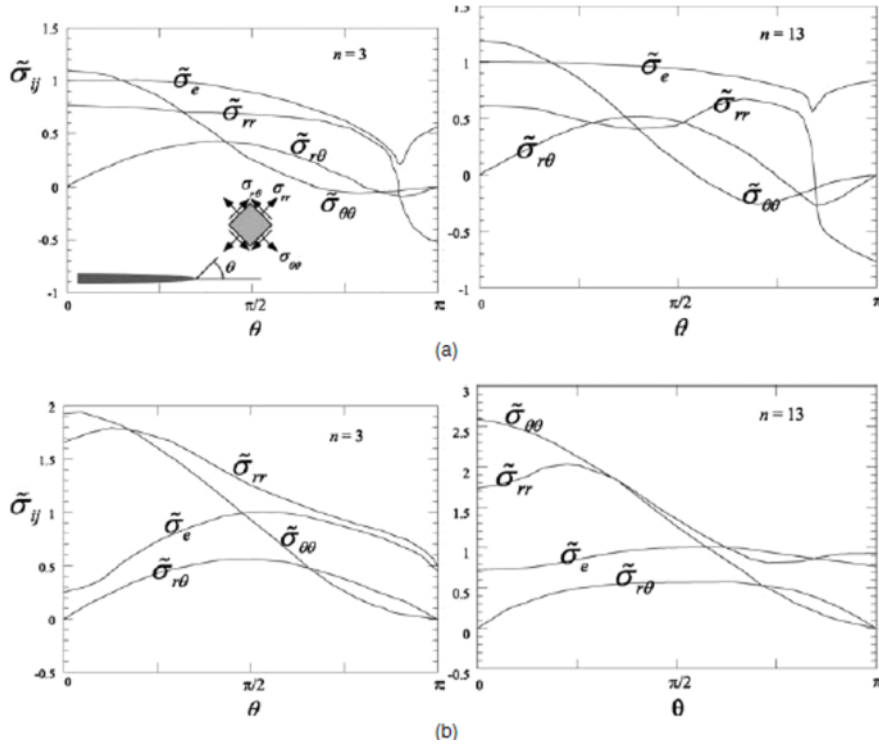


Dim Analysis of

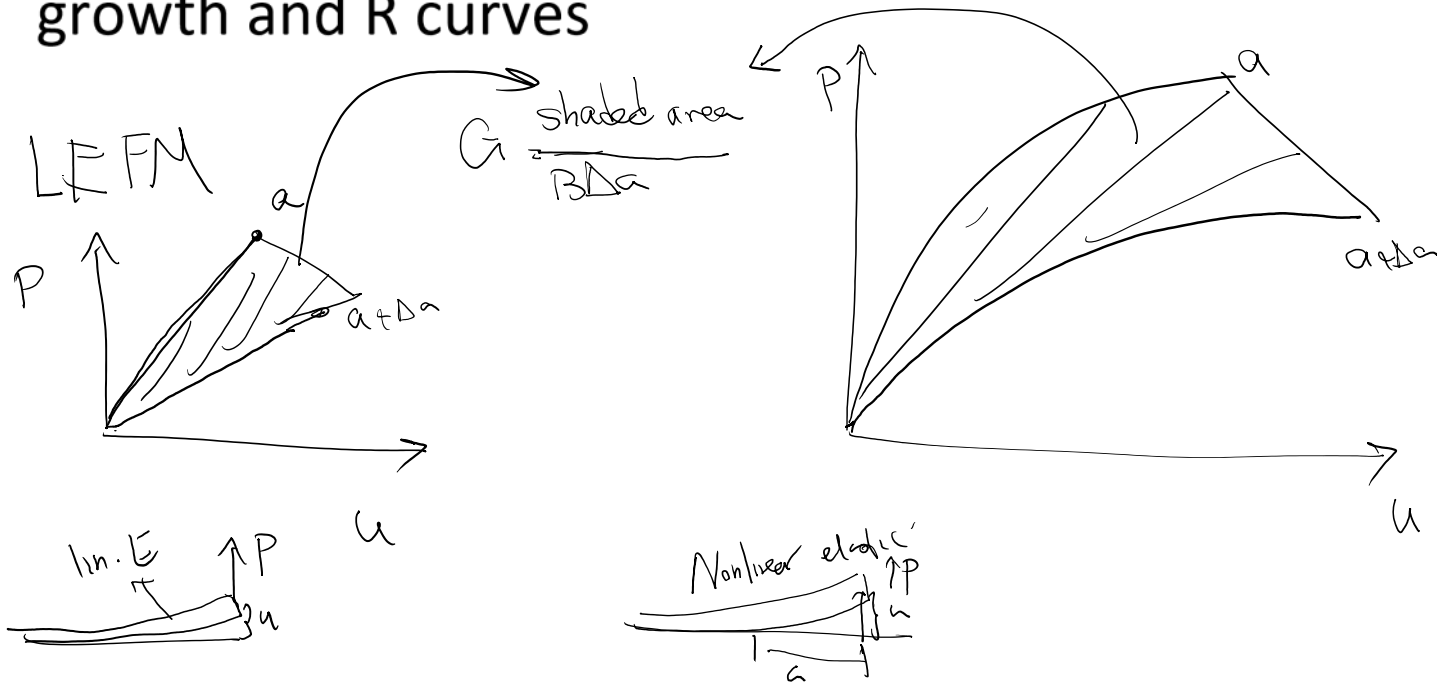
$$\left[ \frac{EJ}{\alpha \sigma_0^2 I_n r} \right] = \frac{[\sigma][\sigma][L]}{[\sigma]^2 [L]} = 1$$



# HRR solution: Angular functions

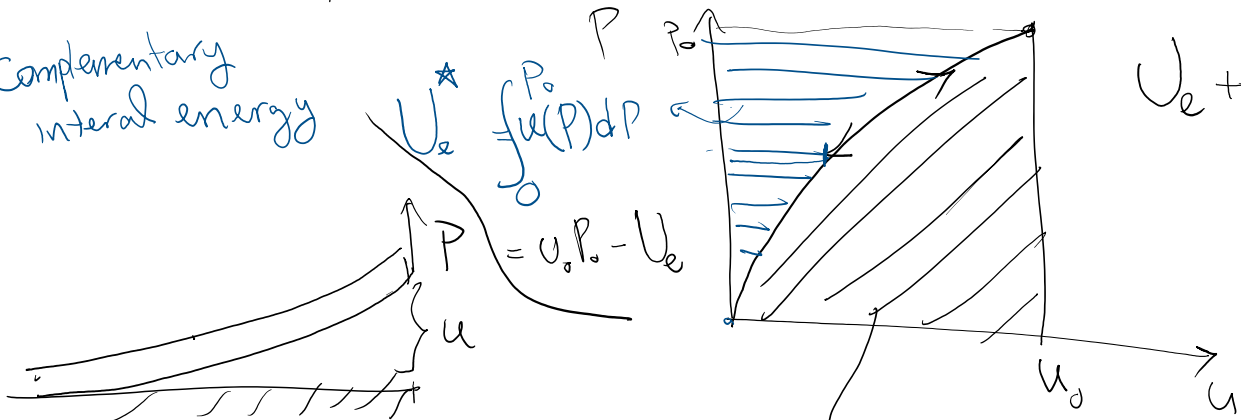


## 5.3. 4. Energy Release Rate, crack growth and R curves



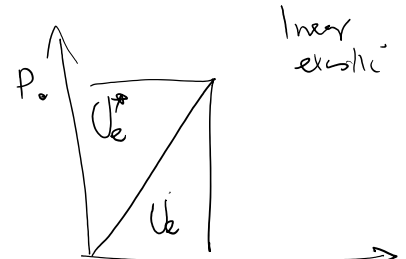
Nonlinear elasticity

Complementary internal energy



We're going to use  $U_e$  &  $U_e^*$  to express  $G$

Internal energy  $U_e = \int_{u=0}^{u_0} P(u) du$

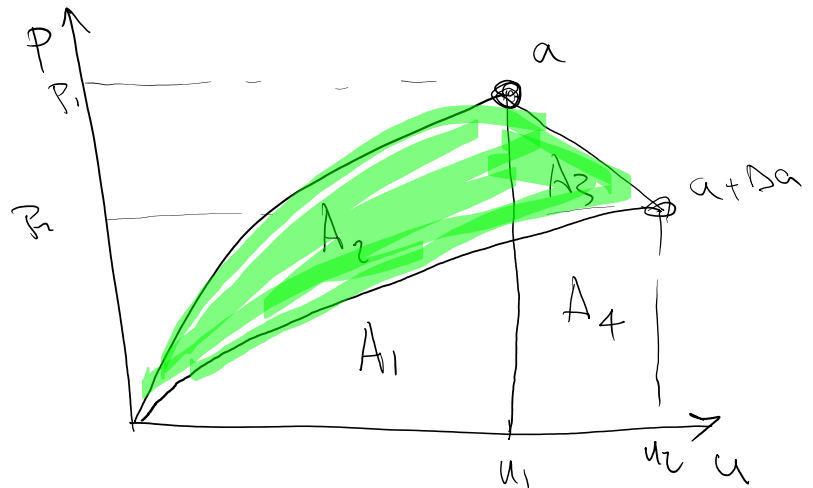


$$U_e = U_e^* = \frac{1}{2} P_0 u_0$$

Why  $G = \frac{\text{shaded area}}{B \Delta a}$

$$G = - \frac{\Delta \Pi}{\Delta A} = - \frac{\Delta (U_e - W_{ext})}{\Delta B a}$$

$\lim_{\Delta A \rightarrow 0}$



$$G = \frac{1}{B} \frac{1}{\Delta a} (-U_{e2} + U_{e1} - W_{ext_{1 \rightarrow 2}})$$

$$\begin{array}{l} U_{e1} = A_1 + A_2 \\ U_{e2} = A_1 + A_4 \end{array} \quad \left| \quad W_{ext_{1 \rightarrow 2}} = \int_{u_1}^{u_2} P du = A_3 + A_4$$



shaded area

$$U_2 = A_1 + A_4 \quad | \quad U_1 = A_2 + A_3$$

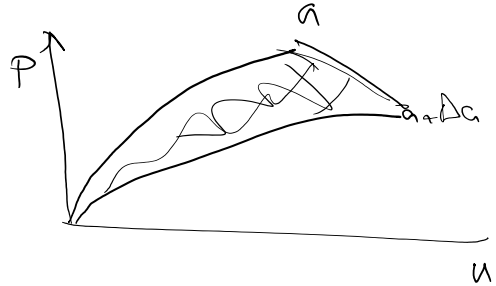
$$G = \frac{1}{B \Delta a} \left( -(A_1 + A_4) + (A_1 + A_2) + (A_3 + A_4) \right) = \frac{A_2 + A_3}{B \Delta a}$$

shaded area

②

Similar to linear elastic case we have

$$G = \frac{\text{shaded area}}{B \Delta a}$$



Can we have a relation like  $G = \frac{P^2}{2B} \frac{dC}{de} = \frac{-u^2}{2B} \frac{dk}{da}$

linear elastics

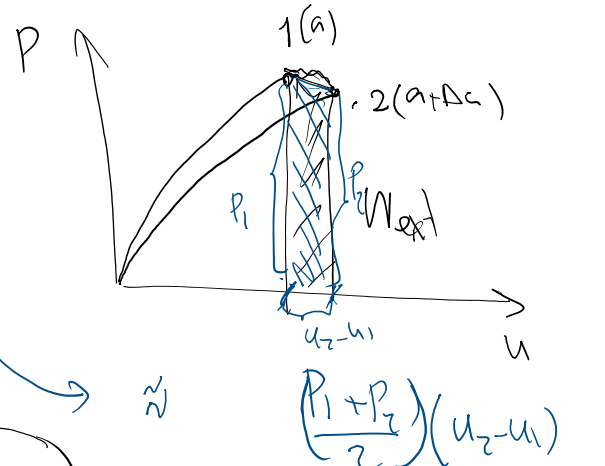
but for nonlinear elasticity

$$G = \lim_{\Delta a \rightarrow 0} \frac{1}{B \Delta a} (\Delta U_e + W_{ext})$$

$$= \frac{1}{B} \left\{ - \lim_{\Delta a \rightarrow 0} \frac{\Delta U_e}{\Delta a} + \lim_{\Delta a \rightarrow 0} \left( \int_{u_1}^{u_2} P du \right) \right\}$$

$$= \frac{1}{B} \left\{ - \lim_{\Delta a \rightarrow 0} \frac{\Delta U_e}{\Delta a} + \lim_{\Delta a \rightarrow 0} \left( \frac{P_1 + P_2}{2} \right) \frac{(u_2 - u_1)}{\Delta a} \right\}$$

$$= \frac{1}{B} \left\{ - \frac{dU_e(a)}{da} + P \frac{du}{da} \right\}$$



Differential formula  
Nonlinear elasticity

$$G = \frac{1}{B} \left\{ -\frac{dU_e}{da} + P \frac{du}{da} \right\} \quad (3)$$

What if the material is linear



$$U_e = \frac{1}{2} P u = \frac{1}{2} k u^2$$

u plug into (3)

$$G = \frac{1}{B} \left\{ -\frac{d \left( \frac{1}{2} k u^2 \right)}{da} + P \frac{du}{da} \right\} =$$

$$\frac{1}{B} \left\{ -\frac{1}{2} \frac{dk}{da} u^2 + \frac{-k}{2} \times 2u \frac{du}{da} + P \frac{du}{da} \right\} =$$

$$\frac{-u^2}{2B} \frac{dk}{da} + \frac{1}{B} \left( -P \frac{du}{da} + P \frac{du}{da} \right) = \frac{-u^2}{2B} \frac{dk}{da}$$

we recovered the simpler linear relation we had before

Special cases of nonlinear elasticity G formula:

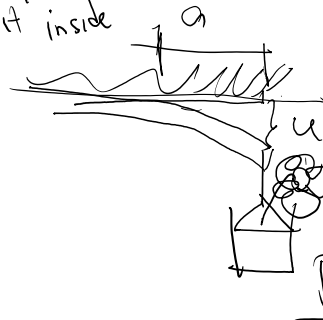
Case 1: dead load

$$G = \frac{1}{B} \left\{ -\frac{dU_e}{da} + P \frac{du}{da} \right\} \quad (3)$$

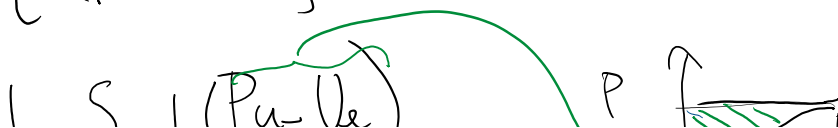
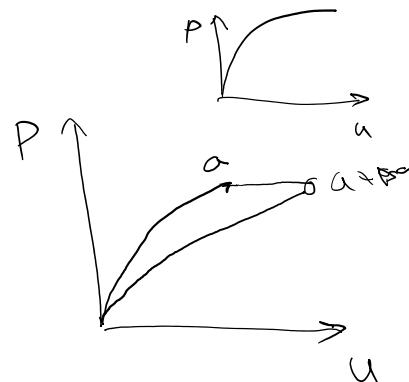
since P is fixed we can move it inside  $\frac{d}{da}$

$$G = \frac{1}{B} \left\{ -\frac{dU_e}{da} + P \frac{du}{da} \right\}$$

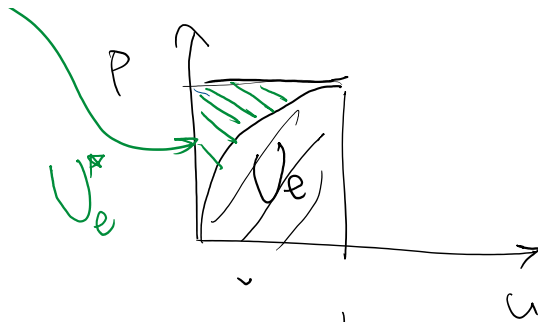
$$G = \frac{1}{B} \left\{ -\frac{dU_e}{da} + \frac{d(Pu)}{da} \right\}$$



P is fixed



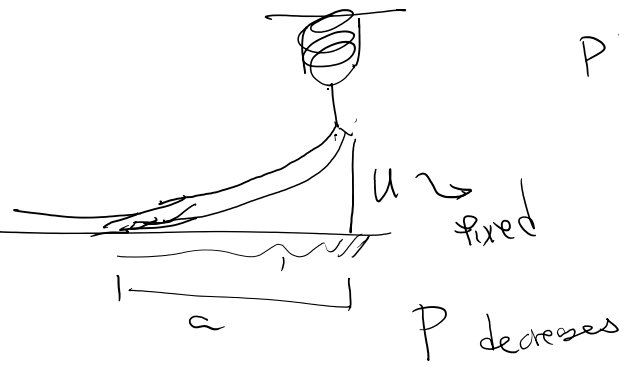
$$= \frac{1}{B} \left\{ \frac{d(Pu)}{da} \right\}$$



Dead load

$$G = \frac{1}{B} \frac{dU_e^*}{da}$$

$$G = \frac{1}{B} \left( \frac{dU_e}{da} + P \frac{da}{da} \right)$$



Fixed grip

$$G = -\frac{1}{B} \frac{dU_e}{da}$$

Summary

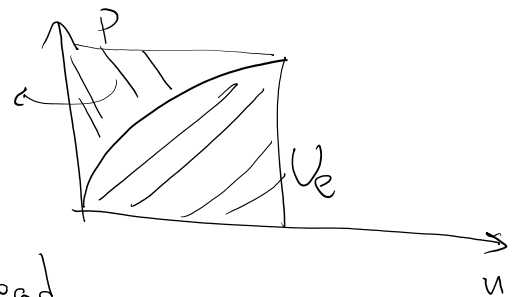
$$G = \frac{1}{B} \left( -\frac{dU_e}{da} + P \frac{da}{da} \right) \text{ (general) } U_e^*$$

$$= \frac{1}{B} \frac{dU_e^*}{da}$$

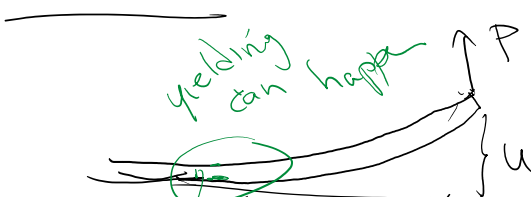
dead (fixed) load

$$= -\frac{1}{B} \frac{dU_e}{da}$$

fixed grip



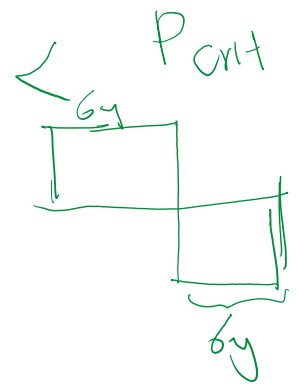
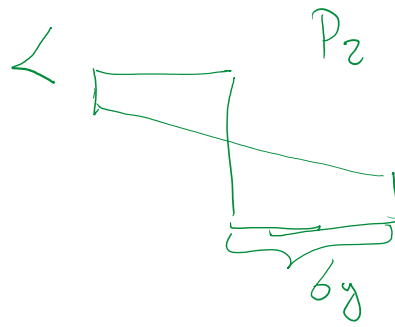
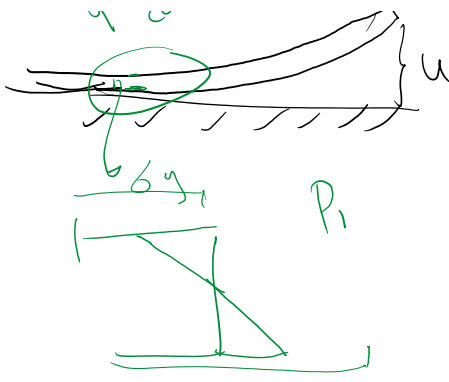
⊕



dead load

$$G = \frac{1}{B} \frac{dU_e^*}{da}$$





$$u = \overline{B} \overline{d} a$$