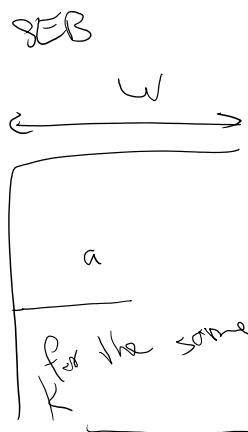
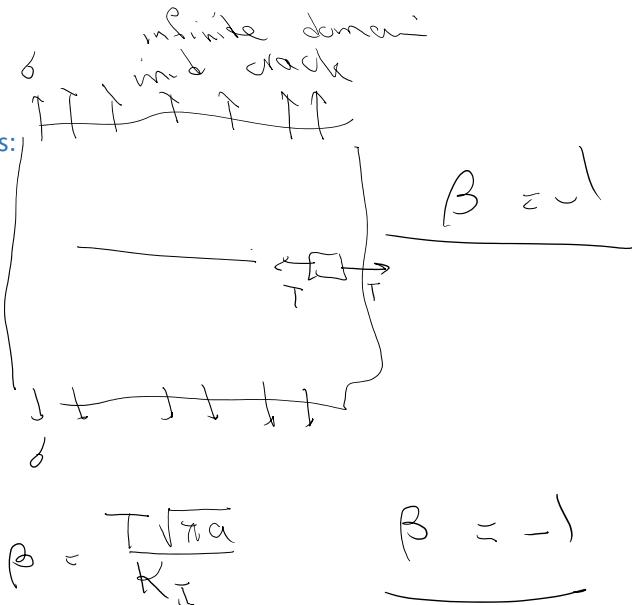
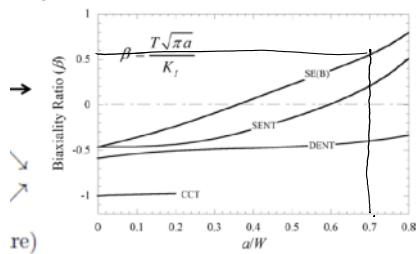


Different crack geometries have different T stresses:



SEB $\beta \nearrow$ $\sigma_{11} \approx \sigma_{22}$ get closer \Rightarrow more difficult to yield

$$\frac{\sigma}{\sigma_0} = 0.7 \Rightarrow \beta = +0.52$$

This was a lower R

T (or β) $\nearrow \Rightarrow$ more biaxial loading

$\Rightarrow R$ (toughness) \searrow

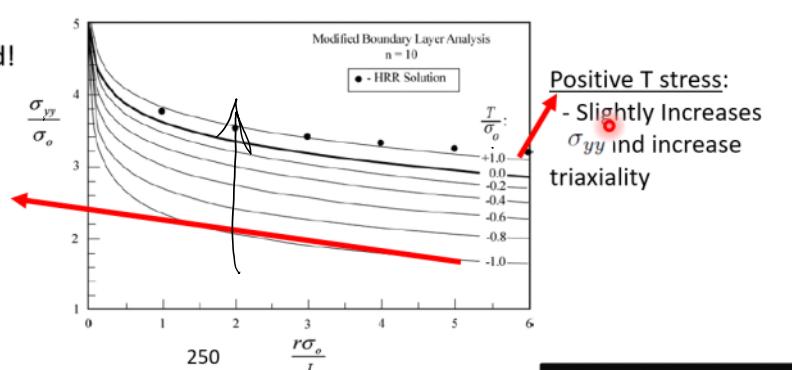
What's the effect of T (beta) in terms of local stress distribution:

Plastic analysis: σ_{yy} is redistributed!

Kirk, Dodds, Anderson

High negative T stress:

- Decreases σ_{yy}
- Decreases triaxiality



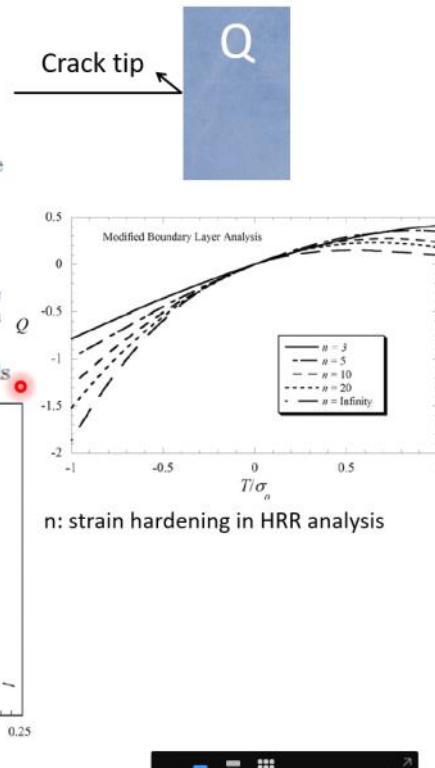
LSY: When a single parameter (G, K, J, CTOD) is not enough? J-Q theory

- ***Q* parameter (J-Q theory)** Valid for **nonlinear** analysis

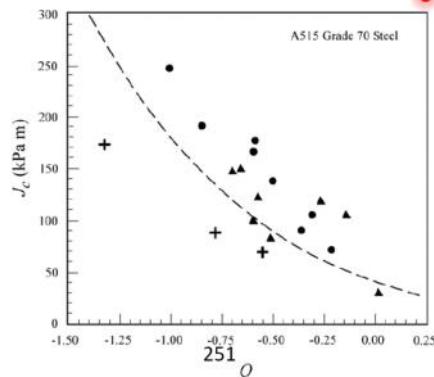
* Added as a **hydrostatic shift** in front of crack to (HRR) stress fields

$$\sigma_{ij} \approx (\sigma_{ij})_{T=0} + Q\sigma_0\delta_{ij} \quad (|\theta| \leq \frac{\pi}{2})$$

* Similar to *T* positive *Q* increases triaxiality and reduces fracture resistance



- **More number of parameters:** With extensive deformation two-parameter models such as *K,T* or *J,Q* eventually break.



6.1 Fracture mechanics in Finite Element Methods (FEM)

6.1.1. Introduction to Finite Element method

6.1.2. Singular stress finite elements

6.1.3. Extraction of *K* (SIF), *G*

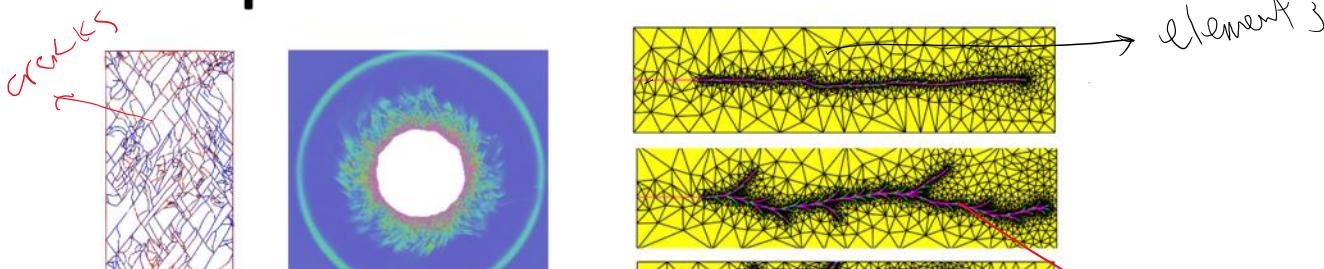
6.1.4. *J* integral

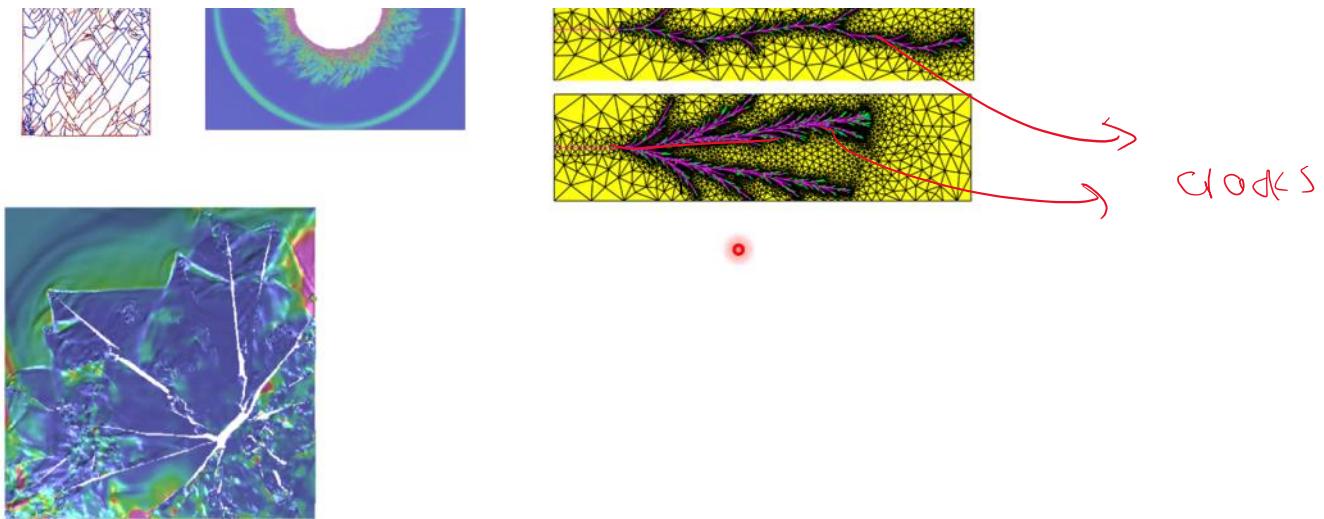
6.1.5. Finite Element mesh design for fracture mechanics

6.1.6. Computational crack growth

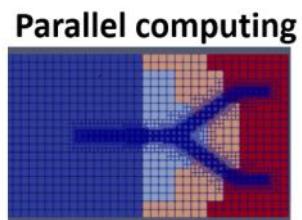
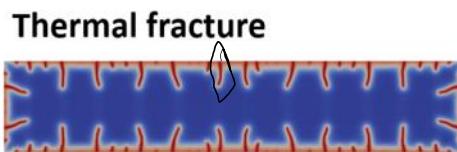
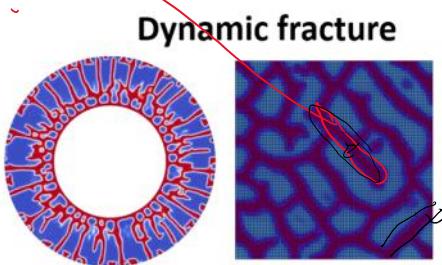
6.1.7. Extended Finite Element Method (XFEM)

Sharp interface models

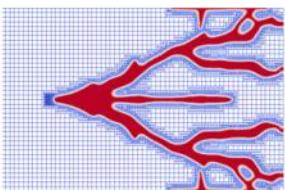




Bulk models

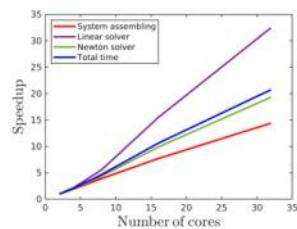


Mesh adaptivity



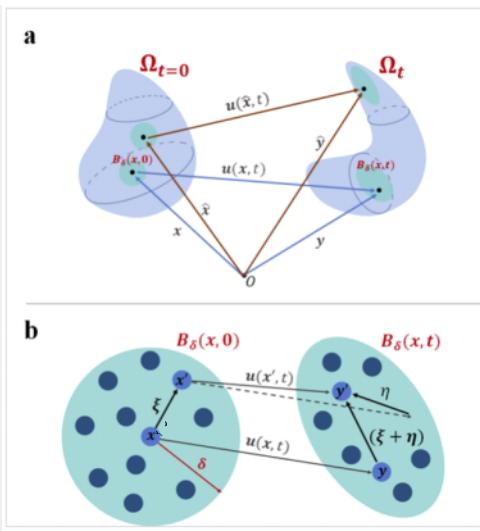
Crack has thickness in these models

Credit: Giang Huynh

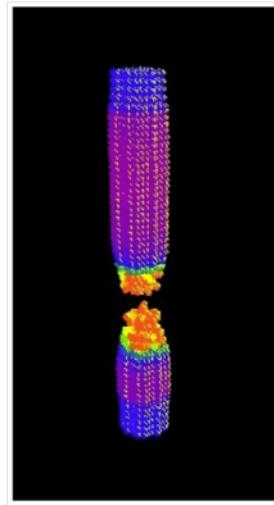


- Continuum (bulk) damage model
- Phase field

Particle models



(a) Kinematics of material body Ω_t within peridynamic theory. (b) Representation of peridynamic horizon of x .



Computer model of the necking \square of an aluminum rod under tension. Colors indicate temperature increase due to plastic heating. Calculation performed with the Emu computer code using peridynamic state-based framework.

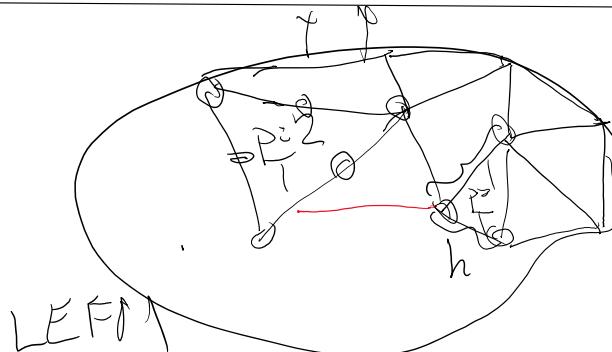
<https://en.wikipedia.org/wiki/Peridynamics>

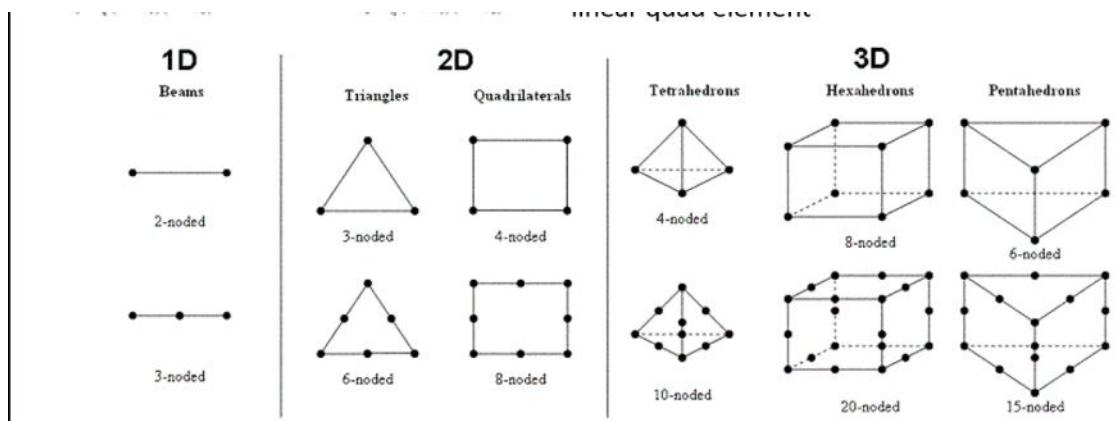


267

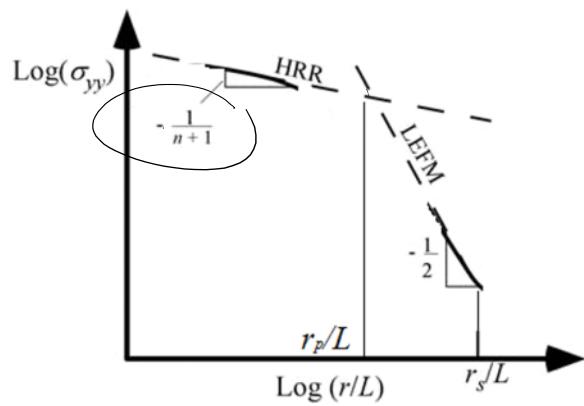
* Peridynamics

h smaller
relevant
fracture length scale
 $\left\{ \begin{array}{l} r_s \\ r_p \end{array} \right.$



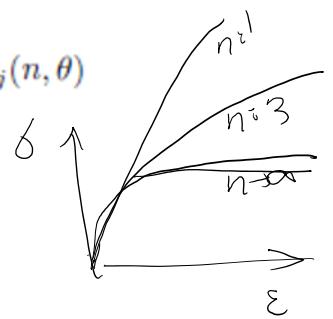


L \bar{E} FM $\varepsilon, \delta \propto r^{-\frac{1}{n+1}}$



$$\sigma_{ij} = \sigma_0 \left(\frac{EJ}{\alpha \sigma_0^2 I_n r} \right)^{\frac{1}{n+1}} \tilde{\sigma}_{ij}(n, \theta)$$

$$\epsilon_{ij} = \frac{\alpha \sigma_0}{E} \left(\frac{EJ}{\alpha \sigma_0^2 I_n r} \right)^{\frac{n}{n+1}} \tilde{\epsilon}_{ij}(n, \theta)$$

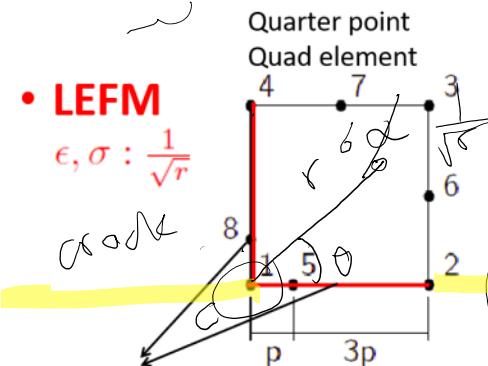


- NLFM (PFM): For HRR solution stress $\frac{1}{r^{\frac{1}{n+1}}}$ and strain $\frac{1}{r^{\frac{n}{n+1}}}$ are still singular \Rightarrow
 - for elastic-perfectly plastic ($n \rightarrow \infty$) stress is bounded and strain is $\frac{1}{r}$ singular

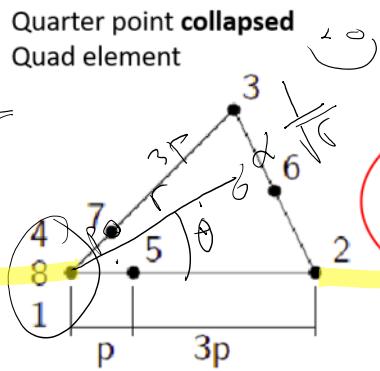
Isoparametric singular elements

• LEFM

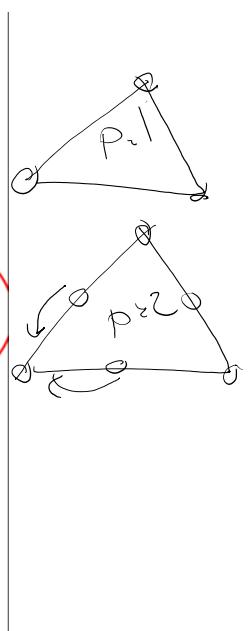
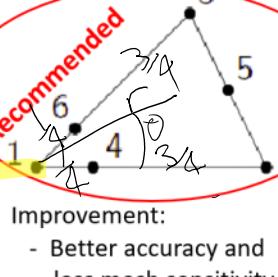
$$\epsilon, \sigma : \frac{1}{\sqrt{r}}$$



Quarter point collapsed Quad element



Quarter point Tri element

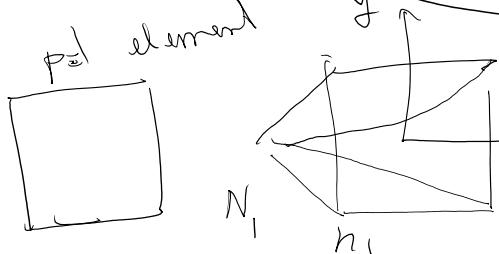


Improvement:
- $\frac{1}{\sqrt{r}}$ from inside all element

Problem

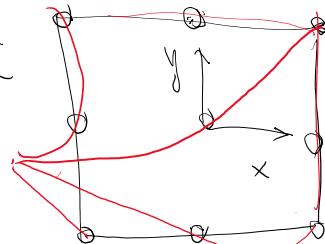
Solution inaccuracy and sensitivity
when opposite edge 3-6-2 is curved

pel element



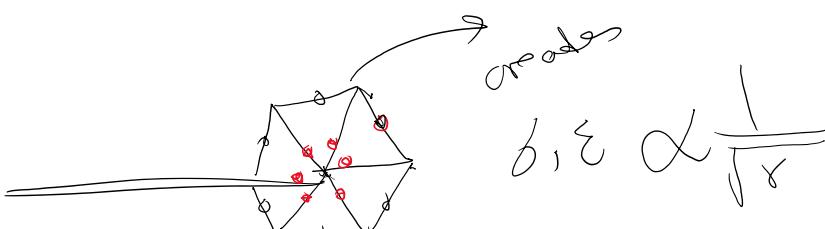
solution is
a linear combinati

x
 y
 xy



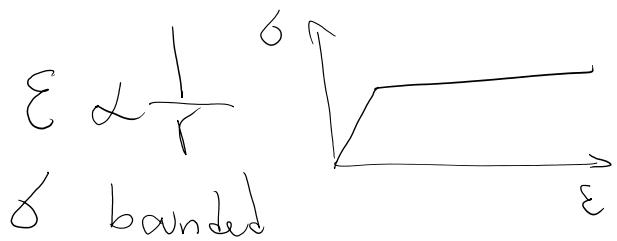
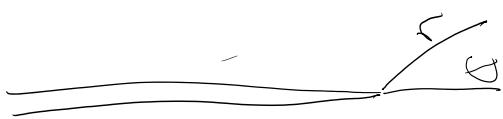
1
 x
 y
 x^2
 xy
 y^2
 x^2y
 xy^2

LEFM



around the crack tip

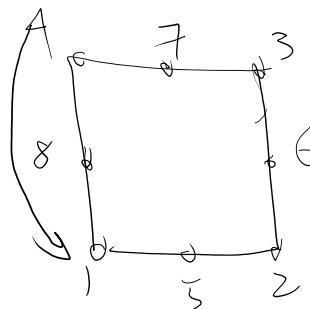
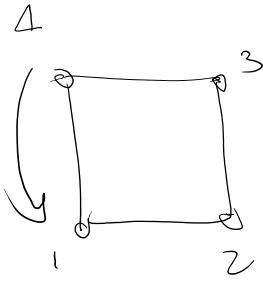
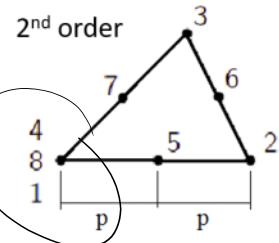
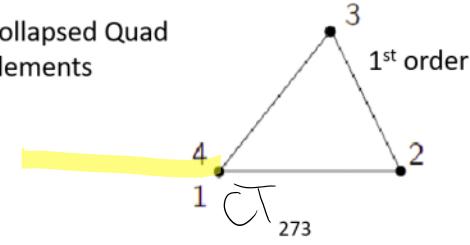
Elastic Perfectly plastic



- **Elastic-perfectly plastic**

$$\epsilon : \frac{1}{r}$$

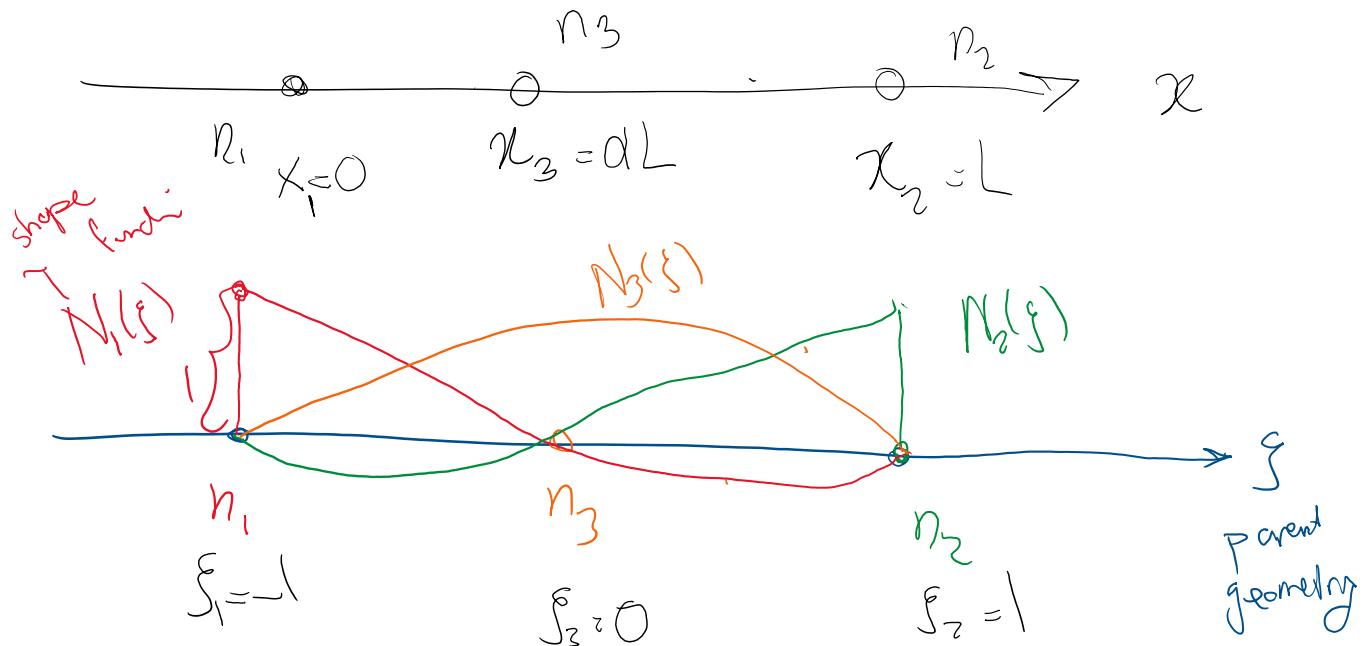
Collapsed Quad elements



Motivation:

Why 1/4, 3/4 positions generate 1/sqrt(r) singularity?

$$\alpha =$$



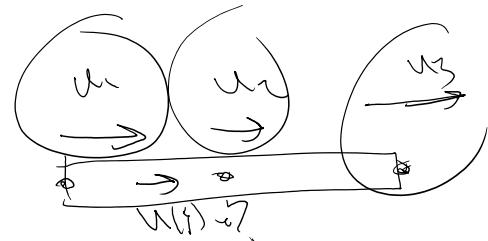
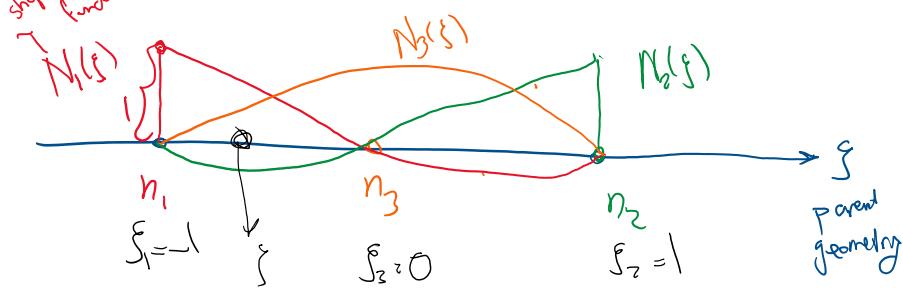
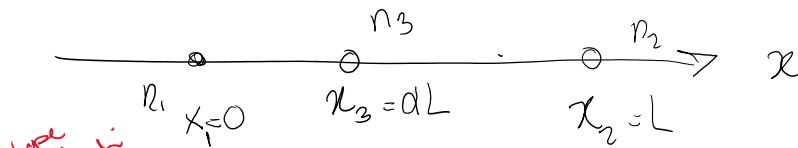
$$N_1(\xi) = \frac{(\xi - \xi_1)(\xi - \xi_3)}{(\xi_1 - \xi_2)(\xi_1 - \xi_3)} = \frac{\xi(\xi-1)}{2}$$

$$\begin{cases} N_1(\xi = -1) = 1 \\ N_1(\xi = 1) = 0 \\ N_1(\xi = 0) = 0 \end{cases}$$

$$N_2(\xi) = \frac{(\xi - \xi_1)(\xi - \xi_3)}{(\xi_2 - \xi_1)(\xi_2 - \xi_3)} = \frac{\xi(\xi+1)}{2}$$

$$N_2(\xi) = \frac{(\xi - \xi_1)(\xi - \xi_3)}{(\xi_2 - \xi_1)(\xi_2 - \xi_3)} = \frac{\xi^2 - \xi_1 \xi_3}{\xi_2^2 - \xi_1 \xi_3}$$

$$N_3(\xi) = \frac{(\xi - \xi_1)(\xi - \xi_2)}{(\xi_3 - \xi_1)(\xi_3 - \xi_2)} \rightarrow \text{Is } \xi \text{?}$$



$$u(\xi) = u_1 N_1(\xi) + u_2 N_2(\xi) + u_3 N_3(\xi)$$

↓ displacement

$$\epsilon = \frac{du(\xi)}{dx} = \frac{du(\xi)}{d\xi} \frac{d\xi}{dx}$$

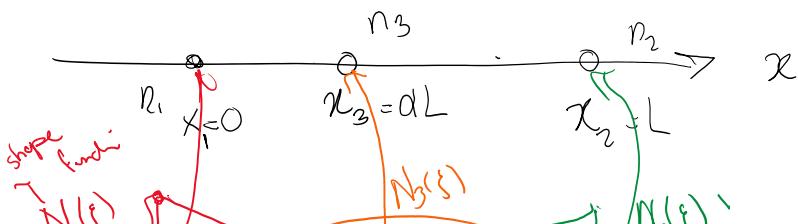
↓ strain

$$\epsilon(\xi = -1) = \infty$$

$$\boxed{\epsilon = \frac{du(\xi)}{d\xi}}$$

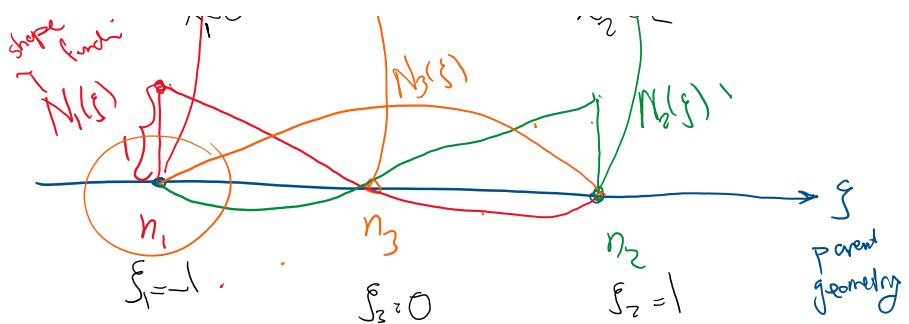
$\frac{dX(\xi)}{d\xi}$

We need to write
 $X(\xi)$



$$\xi = \xi_1 = -1 \Rightarrow x \cdot x_1 = 0$$

$$\xi - \xi_1 = 1 \Rightarrow x - x_1 = L$$



$$\xi = \xi_2 = 0 \Rightarrow x = x_2 = l$$

$$\xi = \xi_3 = 1 \Rightarrow x = x_3 = aL$$

$$x(\xi) = x_1 N_1(\xi) + x_2 N_2(\xi) + x_3 N_3(\xi)$$

check this

$$x(\xi = \xi_1) = x_1 \underbrace{N_1(\xi_1)}_{\neq 0} + x_2 N_2(\xi_1) + x_3 N_3(\xi_1)$$

$$= x_1 \checkmark$$

easily verify $x(\xi = \xi_2) = x_2$

$$x(\xi = \xi_3) = x_3$$

$$v_1 \rightarrow u_1 \rightarrow u_2 \rightarrow u_3$$

$$u(\xi) = u_1 N_1(\xi) + u_2 N_2(\xi) + u_3 N_3(\xi)$$

$$x(\xi) = x_1 N_1(\xi) + x_2 N_2(\xi) + x_3 N_3(\xi)$$

